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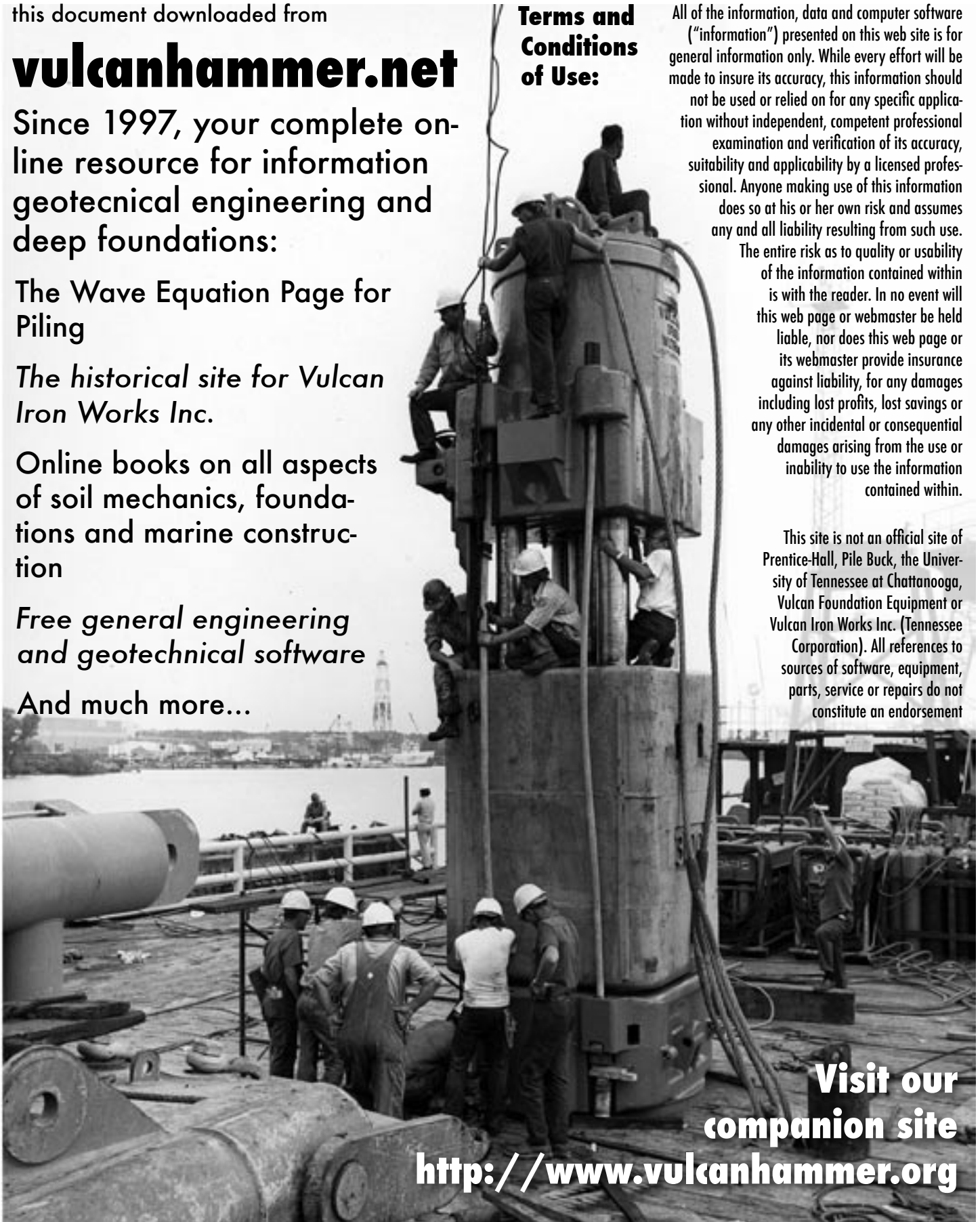
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ELECTRONIC COMPUTER PROGRAM ABSTRACT			
TITLE OF PROGRAM WESWEAP -- Wave Equation Analysis for Piles		PROGRAM NO. 741-F3-R0010	
PREPARING AGENCY U. S. Army Engineer Waterways Experiment Station, Geotechnical Laboratory, P. O. Box 631, Vicksburg, MS 39180			
AUTHOR(S) G. G. Goble and Frank Rausche WES Contact: Hugh M. Taylor, Jr.		DATE PROGRAM COMPLETED July 1976	STATUS OF PROGRAM
		PHASE INIT	STAGE OP
A. PURPOSE OF PROGRAM The program performs wave equation analysis of piles driven by a single blow of any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesels and refined mechanical hammer models. The program can be used to predict impact stresses in piles during driving and to estimate static soil resistance on piles at the time of driving.			
B. PROGRAM SPECIFICATIONS The program development was aimed at providing a simple input and both a flexible and extensive output that include automatic plotting capabilities. The computer language is FORTRAN IV.			
C. METHODS The pile and driving systems are represented by a series of discrete masses and springs. The soil is modeled by a spring and a dashpot attached to each mass. The soil resistance so represented are linear elastic plastic. The elastic resistances are linearly proportional to the element velocity for the velocity. By using Newton's Second Law, accelerations and displacements are calculated and the computation proceeds to the next time increment.			
D. EQUIPMENT DETAILS			
E. INPUT-OUTPUT A short input and long or complete input forms are available. Common hammer property data are stored in a file. Input data is reprinted, options of printed and plotted parameters are available, and time plots are optional.			
F. ADDITIONAL REMARKS Manuals by the Federal Highway Administration that describe this program and its use are: Vol. I, Background, Report No. FHWA-IP-76-14.1; Vol. II, Users Manual, Report No. FHWA-IP-76-14.2, Vol. III, Program Documentation, Report No. FHWA-IP-76-14.3; and Vol. IV, Narrative Presentation, Report No. FHWA-IP-76-14.4.			

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WAVE EQUATION ANALYSIS OF PILE DRIVING

WEAP PROGRAM

Volume I - Background



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U.S. DEPARTMENT OF TRANSPORTATION
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16. Abstract A computer program was written and tested that performs a realistic Wave Equation Analysis of Piles driven by any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesel and refined mechanical hammer models. The program development was aimed at providing a simple input and both a flexible and extensive output that includes automatic plotting capabilities. Pile Driving Hammer data were prepared and stored in a file for most of the commonly encountered models. The computer language is FORTRAN IV. The program was extensively tested against measured pile top force and velocity data and against measured diesel combustion pressure and stroke. This volume is the first in a series. The others in the series are:																					
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PREFACE

During the past 15 years wave equation computer programs have enjoyed a gradual but continual increase in use for the analysis of pile driving. The motivation for the preparation of the WEAP program (Wave Equation Analysis of Piles) came from problems which were experienced by the New York Department of Transportation when they attempted to implement routine wave equation analyses into their pile driving practice. They used a program prepared by the Texas Transportation Institute. In spite of the fact that this program is probably the most widely used wave equation program in the United States, serious difficulties were encountered in that unrealistic stresses were sometimes obtained for piles driven by diesel hammers.

The authors of this report have performed extensive research studies on pile driving emphasizing the measurement of force and acceleration during driving. These measurements involving piles driven by all types of hammers have been made for several states including New York. In order to take advantage of these measurements the Federal Highway Administration contracted with the authors to prepare a wave equation program which would accurately model the diesel hammer. Several years have passed since the TTI program was developed, so it could be expected that other general improvements could be introduced into the program for all types of hammers. Finally the large

volume of available measurements of force and acceleration at the pile top were used to test the program performance. No currently available program has been subjected to such a demanding and thorough testing.

This report is presented in four volumes. The first presents a general discussion of the use of the wave equation and how this particular program models the hammer-pile-soil system. Emphasis is placed on a discussion of the operation of diesel hammers and how that operation is modeled by WEAP. The second volume provides a description of program input and output and can serve as a user's manual for the program. It is strongly recommended that all users read Volume I prior to the User's Manual so that they will understand the assumptions contained in the program and how it is intended that it be used. The third volume was prepared to aid the computer operator during the initial stages of program and data file loading. It also contains a flow chart which may be of interest to those users who want to study the program in greater detail. The fourth volume contains the three parts of a lecture which is also available in the form of a tape/slide show. The contents of this narrative report deal with background, models and applications of the Wave Equation.

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New York Department of Transportation, EDP Bureau

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WAVE EQUATION ANALYSIS FOR PILES
RESEARCH REPORT

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CHAPTER 1

INTRODUCTION

The purpose of this report is to describe the development, modeling and basic ideas behind a computer program called WEAP (Wave Equation Analysis of Piles) which was developed under contract with the Federal Highway Administration. This study was conducted to serve the following purposes:

- (a) To produce a program for analyzing a pile driven by a diesel hammer using a thorough model of both the thermo-dynamic and mechanical hammer operation.
- (b) To improve and refine existing techniques for wave analysis of piles driven by air-steam hammers.
- (c) To study the performance of the program by comparing computed values of pile top force and velocity with those measured previously by the Case project (1)[†]
- (d) To provide a program that requires minimal effort for the preparation of input data for "typical" cases and puts the volume of output information in the control of the user.

The wave equation concept is not new. Smith (2,3) first proposed the use of this discrete method for modeling the hammer-pile-soil system. Among the researchers that have further contributed to the advancement of the art are Forehand and Reese (4) and Samson et al (5). Probably the largest research efforts were made by the Texas Transportation Institute of Texas A&M University (6,7)* (TTI). The program versions that they created were pub-

* Only two of the large number of their reports are referenced here.

[†] Numbers in paranthesis pertain to references listed at end of text.

lished and made generally available in the United States.

When the TTI program was written, air-steam hammer operation was of primary interest and concern. Therefore, it is not surprising that results were much less satisfactory when diesel hammer systems were analyzed. Particularly the prediction of driving stresses for these cases has been found to be unsatisfactory by many program users.

Prior to the development of the WEAP program the performance of wave equation programs was tested either by analysis of simple cases where closed form solutions giving force-time relationships were available, or for real cases where the predicted capacity was compared so that measured in a static load test. Since only a few cases with oversimplified hammer systems can be solved in closed form, the primary testing was against load test results. Such a comparison involves only one parameter in a domain where more than twenty are unknown and must be found. To make matters worse, the system is non-linear and simplifying assumptions, interpolations and extrapolations do not always hold. Other problems such as time - dependent characteristics of the soil, numerical defects of the discrete model and inadequate soil modeling can also yield erroneous results. While large volumes of measurements have been published by the Case Research Project, apparently this data was not used to any substantial degree in checking the performance of other programs. One of the tasks of the project reported here was to subject the program to extensive testing by comparing measured and calculated force and velocity records at the pile top for a wide variety of hammer and pile types.

As indicated above, the principal goal of this work was to improve analysis capabilities for diesel hammers. The WEAP program differs from the TTI program for diesel hammers in that, first, WEAP includes the determination of the gas pressure in the combustion chamber using a thermodynamic analysis rather than a constant, specified pressure and, secondly, the hammer stroke is calculated in the dynamic analysis rather than being specified in advance. Diesel hammers can operate at a wide variety of strokes that cannot be estimated in advance by intuitive means. Moreover, the hammer's effectiveness is strongly dependent on the stroke.

In Chapter 2 of this report the basic use of the wave equation is discussed. Real hammer performance is discussed in Chapter 3 and the model for hammer operation is described with emphasis placed on diesel hammers. The soil model is also described and an alternate approach to that used by Smith is presented. Some further elaboration on diesel hammer operation is contained in Appendix A.

The information necessary for the preparation of the program input data is described in Chapter 4 (and also in the User's Manual). Chapter 5 gives a general description of the program organization and flow. The extensive study of program performance which compares calculated and measured values of force and velocity for 17 different test piles is reported in Chapter 6. Chapter 7 gives some conclusions and recommendations.

CHAPTER 2

BASIC OPERATION AND USE OF THE WAVE EQUATION

For over 100 years foundation engineers have used dynamic formulas to estimate pile bearing capacity (or the inverse, to estimate the required blow count for a specified capacity). The use of these formulas has been severely criticized since all of them have been proven grossly inaccurate and unreliable. Their use persists in spite of the criticism because of their simplicity and the lack of something better. Also, some dynamic means of capacity prediction will continue to be required since pile design loads have tended to increase making static load tests increasingly difficult to perform.

In order to place the wave equation in context it is appropriate to review the use of a dynamic formula. Consider a typical example

$$R = \frac{2 Wh}{S+C} \quad (2.1)$$

where R is the design load, W is the ram weight, h is the ram stroke, S is the permanent set of the pile per hammer blow and C is a term which represents the energy losses and carries the same units as set. Contained in the constant of equation 2.1 is a theoretical factor of safety of 6 plus the quantities necessary to make the units correct. The product Wh usually is used to represent the rated hammer energy. This formula can be represented by the curve shown in Figure 2.1. The design pile capacity is given as a function of blow count and is known as a bearing graph. A number of applications can be visualized. When a particular blow count is observed and the rated hammer

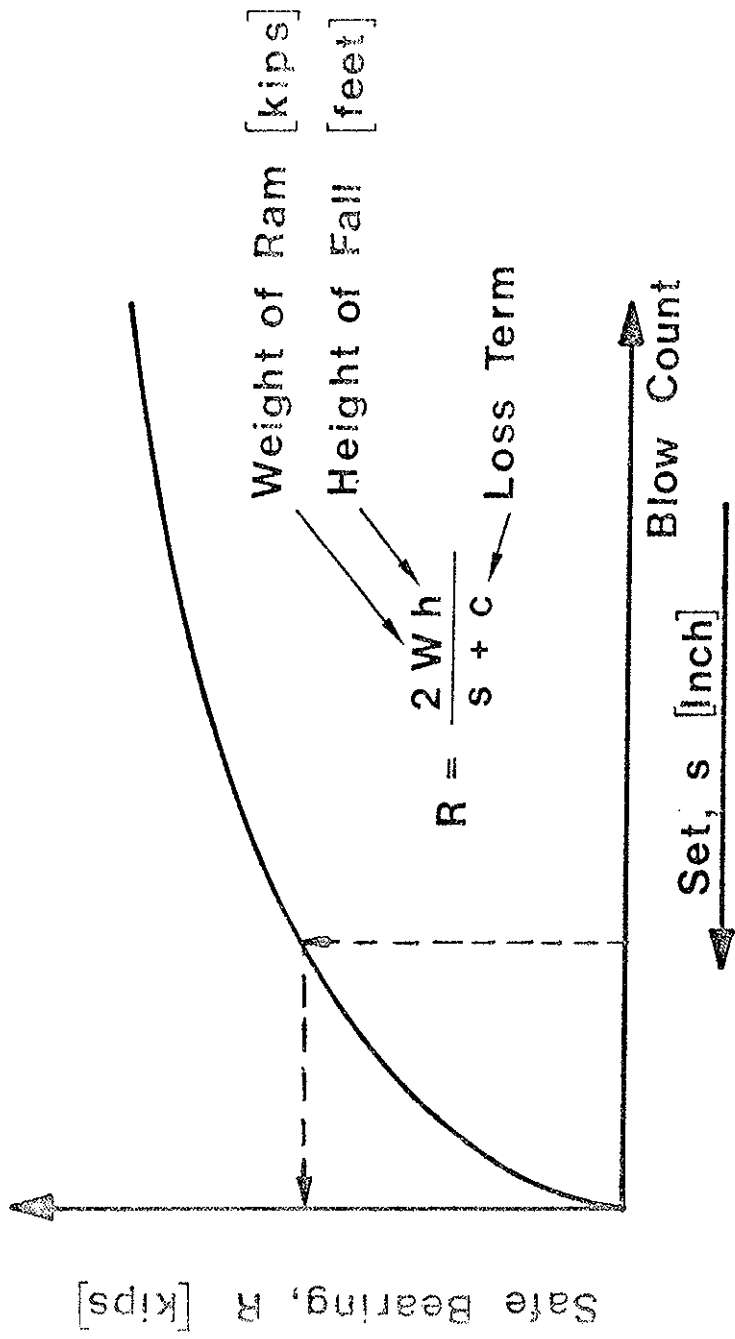


FIGURE 2-1: A BEARING GRAPH DERIVED FROM THE ENGINEERING NEWS FORMULA

energy is known, then with an estimate of loss the pile capacity can be determined as shown in Figure 2.1. Of course, the reverse path is also possible. The blow count required for a specified capacity can be determined.

The other common problem is the selection of equipment. For a particular job the capacity is usually specified. Then for a given driving system (rated hammer energy and estimated losses) the blow count can be determined. If this blow count is not judged to be satisfactory, the equipment can be changed.

The shortcomings of the dynamic formulas can be placed in three categories. In general, these three categories are good for evaluation of all dynamic procedures for capacity determination:

1. The dynamic formulas do a poor job of representing the driving system. Though the rated energy is the most important hammer parameter, it is the only one included. Some dynamic formulas include other parameters such as ram weight but they have not proven to be more accurate. An attempt is made to consider the driving system with the loss term, C , but this approach is greatly oversimplified. Since only a rated performance is included the dynamic formula does not attempt to deal with poor equipment performance.
2. All effects of pile flexibility are neglected since in its derivation the dynamic formula assumes a rigid pile.
3. The soil resistance is assumed to be constant. This kind of soil model is certainly far too simple to even approximately represent a real soil.

In view of the above inadequacies, it is not surprising that an unexpected blow count is observed when driving begins on a job. The questions that

commonly arise are: Is the bearing capacity sufficient? Would further driving produce pile damage? Is the hammer performing properly? Is the driving system of the correct size?

However, the observation of blow count is a very convenient way to gage the quality of the pile installation. Therefore, the wave equation approach was developed. The one dimensional wave equation can be derived by applying Newton's Second Law to a rod element of infinitesimal length. It is written

$$\rho \frac{\partial^2 u}{\partial t^2} = E \frac{\partial^2 u}{\partial x^2} \quad (2.2)$$

where ρ is the material mass density, E is the modulus of elasticity of the material and u is the axial displacement of a point on the rod at location x and time t. Thus, $\partial^2 u / \partial t^2$ is the acceleration and $\partial^2 u / \partial x^2$ is the strain gradient at x and t.

Using this continuous form of the wave equation for pile analysis is usually not practical for the real boundary conditions which must be handled. However, a similar equation can be derived if elements of finite length, ΔL , are chosen having mass, $m = \rho A \Delta L$ and spring stiffness, $K = EA / \Delta L$. Here A is the pile cross sectional area. Newton's Second Law leads to

$$ma = K (\Delta u_t - \Delta u_b) \quad (2.3)$$

where a is the acceleration of the mass and Δu_t and Δu_b are the compression of the springs at the top and bottom, respectively, of the mass

under consideration.

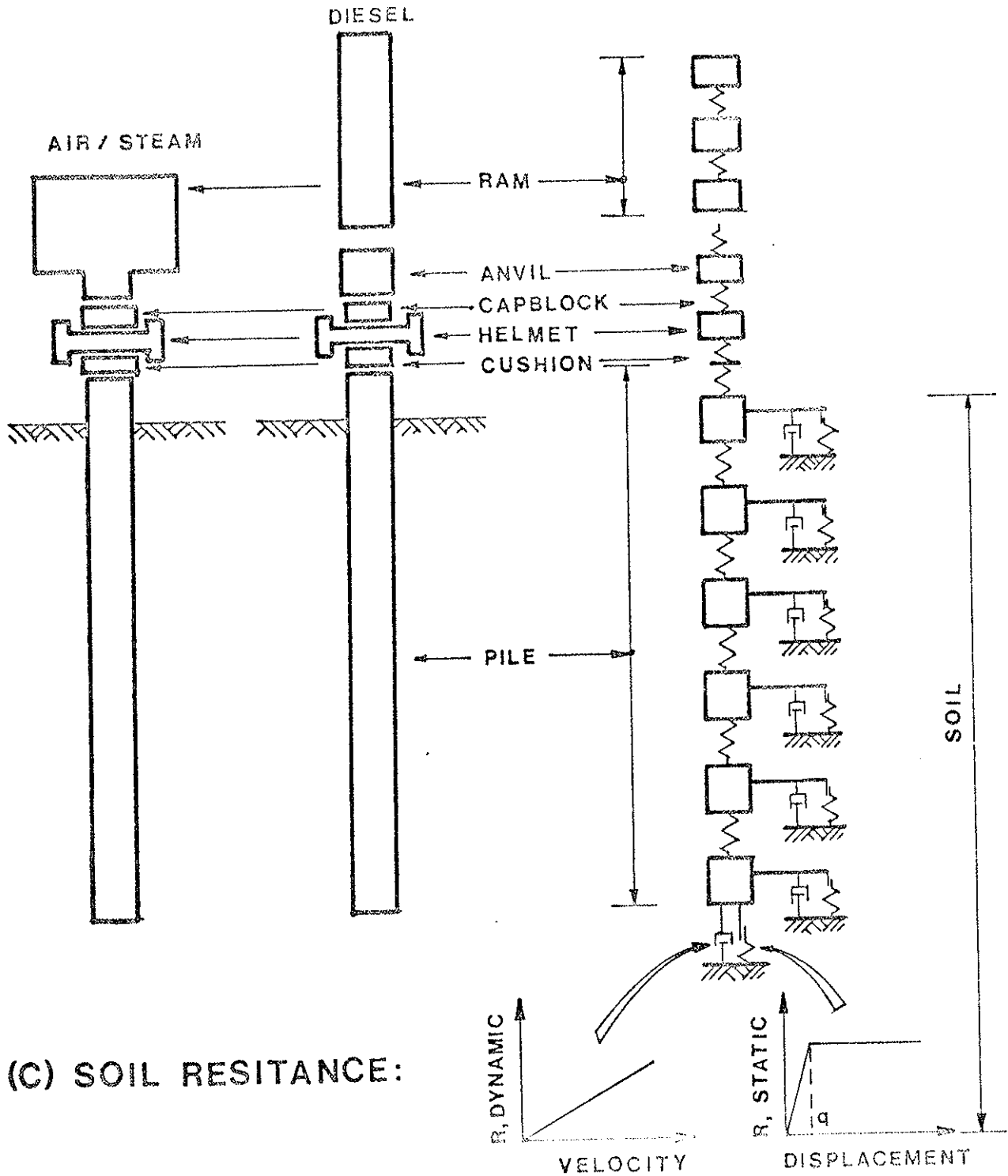
The Wave Equation (the term Wave Equation is the name that has been attached to computer programs for discrete dynamic pile analysis. In the remainder of this report, this usage will be adopted) makes use of the concept of equation 2.3 by representing the pile and driving system (Figure 2.2a) by a series of masses and springs as shown in Figure 2.2b. The soil is modeled by a spring (R, Static) and a dashpot (R, Dynamic) attached to each mass. The soil resistances so represented are shown in Figure 2.2c and are linear elastic plastic for the spring where the maximum force, R_u , is reached at a displacement q , called the quake, and linearly proportional to the element velocity for the dashpot (commonly known as the damping force).

The analysis proceeds by giving the ram an initial velocity. At each element the displacement can be calculated for a small time increment with element velocities determined from the previous time increment. With these displacements and velocities the forces acting on each mass can be determined. They arise from the pile spring deformations, from the soil spring deformation and from the dashpot force. Using Newton's Second Law in the form represented in Equation 2.3 the mass accelerations can be calculated and by integration also the velocity. The computation then proceeds to the next time increment.

In application a set of soil forces R_u and damping forces are assigned at each element. Then the ram is given it's rated impact velocity and the dynamic computation outlined above is continued through successive time

(A) ACTUAL SYSTEM

(B) MODEL



(C) SOIL RESISTANCE:

FIGURE 2-2: (A) THE SYSTEM TO BE ANALYZED;
(B) THE WAVE EQUATION MODEL AND
(C) THE COMPONENTS OF THE SOIL RESISTANCE MODEL

increments until all soil forces are less than R_u . The total permanent displacement will have then been calculated and a point on the bearing graph is known. The capacity value is known as R_{ut} and is equal to the sum of the R_u values at each element. The blow count is obtained from the calculated permanent set. In this procedure the permanent set (or blow count) is determined for a set of assigned resistances. However, the bearing graph is plotted, by tradition, with the blow count as the independent variable. A variety of R_{ut} values can be used to calculate the total shape of the bearing graph.

In addition to the bearing graph the wave equation also gives stresses in the pile and they can also be shown as a function of blow count.

In practice, the wave equation bearing graph can be used in a manner quite similar to the dynamic formula bearing graph. In addition, driving stresses can be rationally limited. While the shape of the two curves are quite similar the differences are substantial. A particular wave equation bearing graph is associated with a single driving system pile type, soil profile and a particular pile penetration. If any one of the above items are changed, the bearing graph changes.

The above description summarizes very briefly the operation of traditional wave equation programs such as the TTI program. The system model will be described in greater detail in Chapter 3. The operation of the WEAP program will also be described emphasizing those aspects which are different.

The stroke of a diesel hammer should not be specified as an input quantity as was done in the above description. In the WEAP program the mathematical model is constructed like that shown in Figure 2.2 except that a combustion chamber force is introduced between the ram and the anvil. The program operation begins by dropping the ram from some initial preassigned height. The ram velocity at the exhaust ports can be calculated directly from the free fall distance. When the exhaust ports are closed by the ram it continues to fall against the confined gas in the combustion chamber. In this stage the gas pressure and ram velocity are calculated incrementally. The gas pressure can be determined from the gas law since the volume is known as the ram falls. When impact occurs, and the velocity at impact has been calculated, a dynamic analysis of the general type described above is performed. Shortly after impact ignition occurs in the combustion chamber and the pressure and temperature are given an appropriate increase. At some stage in the calculation separation occurs between the ram and the anvil. The computation now continues until the exhaust port is passed at a known velocity. From this velocity the rebound stroke can be calculated. If the initial stroke is not the same as the rebound the computation is repeated using the rebound stroke as the initial stroke in the next cycle. Convergence usually occurs in two or three cycles.

A bearing graph similar to that previously described is obtained except that stroke and pile stresses are also included. An example is shown in Figure 6 of the User's Manual.

A variation of this concept is necessary for hammers which do not have a definite fuel setting and for which the combustion pressure is not known a priori. In such cases, the stroke is usually kept as desired by adjusting the fuel amount. (In easy driving the stroke is probably always limited by the maximum combustion energy available). To model this process the stroke has to be specified and maximum combustion pressure is adjusted until the rebound stroke equals the specified one.

Now consider the three problems with the dynamic formula as they relate to the wave equation:

1. The driving system can be represented with considerable realism. The various dynamic parameters used to describe the system must be available. Of course, the wave equation cannot be expected to recognize a poorly performing hammer.
2. The pile is accurately represented.
3. The soil model is a substantial improvement over that used in the dynamic formula. However, it is still extremely simple and crude. Even for this simple model it is very difficult to obtain soil constants. Therefore, greater complexity hardly seems justified.

The use of either dynamic formula or the wave equation requires the accurate determination of blow count. Particular care must be used if the driving resistance is changing rapidly. Often when time dependent strength changes occur, it is desirable to restrike the pile. Here the blow count at the very beginning of restrike must be determined since it can be expected to change with driving.

CHAPTER 3

MATHEMATICAL MODELS

3.1 Introduction

In this chapter the construction and operation of pile driving hammers will be discussed. After the hammer operation has been described, the mathematical model which has been developed to represent it will be presented. Since the most important contribution of this report is probably the diesel hammer portion, it will be presented first.

The model used to represent the other parts of the driving system, the pile and the soil will also be described in considerable detail.

In general, the variables used to describe the system will be the same as those used in the program. While this approach is somewhat unwieldy--- here it has substantial advantages for those readers who need to become deeply involved with the program.

3.2 Hammer

3.2.1 Working Principle of the Open End Diesel Hammer

The Open End Diesel hammer (OED) operates on a two stroke diesel cycle. The hammer is started by raising the ram with a lifting mechanism. At the upper end of its travel the lifting mechanism is tripped, the ram is released and descends by gravity. At the time the ram bottom passes the exhaust ports a certain volume of air, V_{IN} , is trapped and is compressed (Figure 3-1a). Usually before the time of exhaust port closure a certain amount of fuel is squirted into the cylinder. Some hammers inject an atomized

OPEN END DIESEL HAMMER

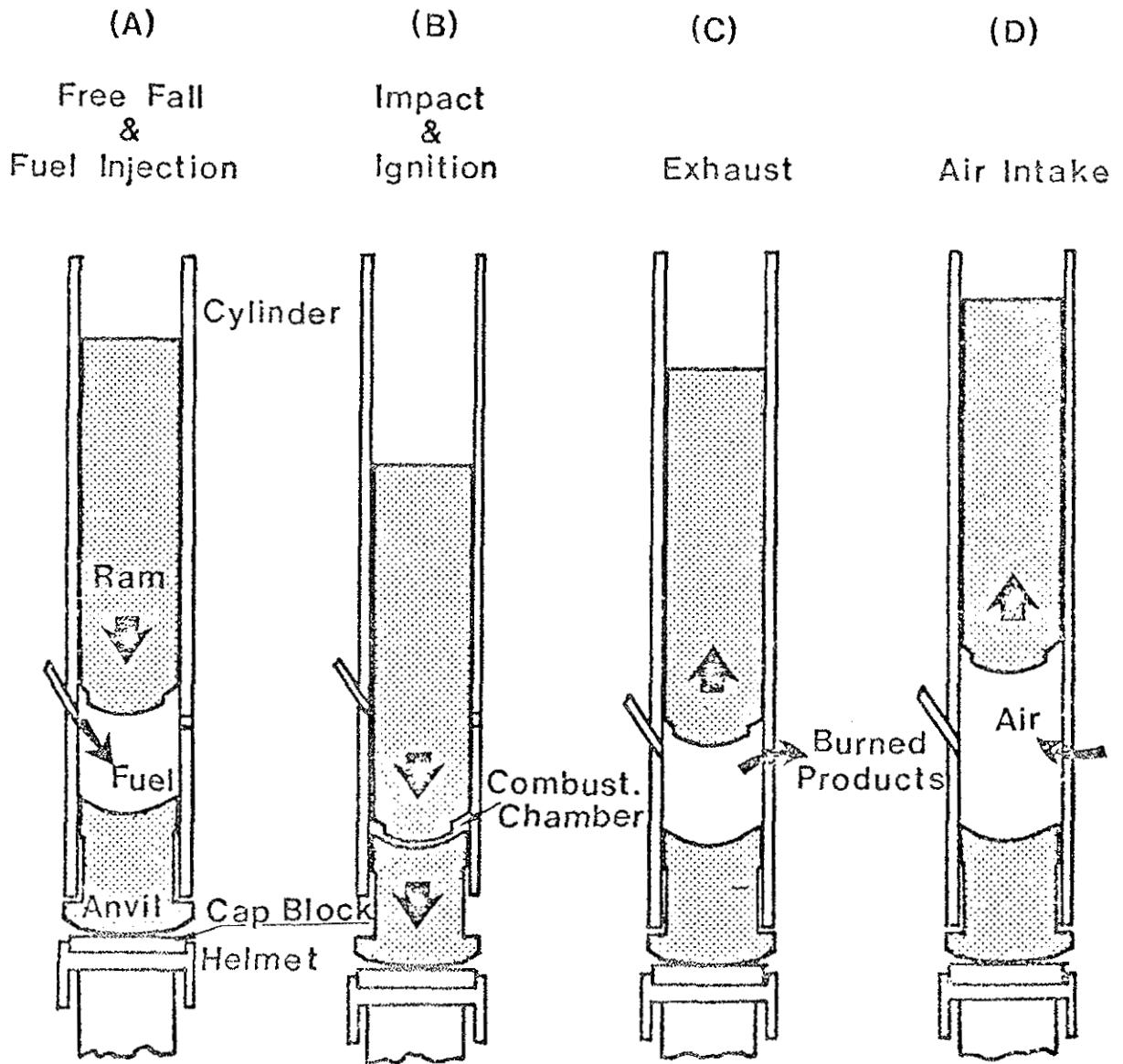


FIGURE 3-1: WORKING PRINCIPLE OF THE OPEN END DIESEL HAMMER

fuel later in the cycle when the combustion chamber pressure is higher.

When the ram impacts against the anvil the air is compressed to a final volume (VFIN). The fuel is splattered by the impact into this final volume (if fuel atomization was not used) and combustion starts at some time after impact (combustion delay). This delay is due to the time that is required for the fuel to mix with the (hot) air and to ignite. More volatile fuels might have a shorter combustion delay than heavier ones. Combustion occurring before impact is called preignition and can be caused by the wrong fuel type or an overheated hammer. For Open End Diesel Hammers preignition is usually considered to be undesirable.

During impact, anvil, capblock and pile top are rapidly driven downward (Figure 3-1b) leaving the cylinder with no support. Thus, it starts to descend by gravity.

Pile rebound and combustion pressure push the ram upwards. When the exhaust ports are cleared some of the combustion products are exhausted leaving in the cylinder a volume of burned gases at ambient pressure that is equal to VIN (Figure 3-1c). As the ram continues upward fresh air, which is drawn in through the exhaust ports, mixes with the remaining burned gases (Figure 3-1d).

Depending on the reaction of the pile and the energy provided by combustion the ram will rise to some height (stroke). It then descends again by gravity to start a new cycle.

3.2.2 The Mechanical Model of the Open End Diesel Hammer

In order to properly model the mechanics of the OED hammer the following characteristic properties must be considered in addition to those described above:

- (a) The ram is relatively long and flexible.
- (b) Metal to metal impact occurs between ram and impact block.
- (c) Energy losses occur on all interfaces of hammer components which transmit the impact.

The capblock, helmet and, if present, the cushion will be considered as a part of the hammer. The helmet is usually a rather heavy steel form that adapts to the pile top. The capblock is cushioning material between anvil and helmet while a cushion is sometimes inserted between helmet and pile top.

Figure 3-2 shows both an actual pile driving hammer (a) and its model (b). The ram was divided into M elements to account for its flexibility. Anvil and cap were represented by one mass each. The spring stiffness above the anvil was determined by the lowest ram element stiffness combined with that of the anvil. Thus, if $HM(I)$ denotes the mass of the I-th hammer segment the following relations hold:

$$HM(I) = \frac{W_R}{M g} \quad \text{for } I \leq M \quad (3-1)$$

where W_R is the weight of the ram (assumed to be uniform) and g is 32.2 ft/s^2 .

OPEN END DIESEL HAMMER

(A)
Schematic

(B)
Model

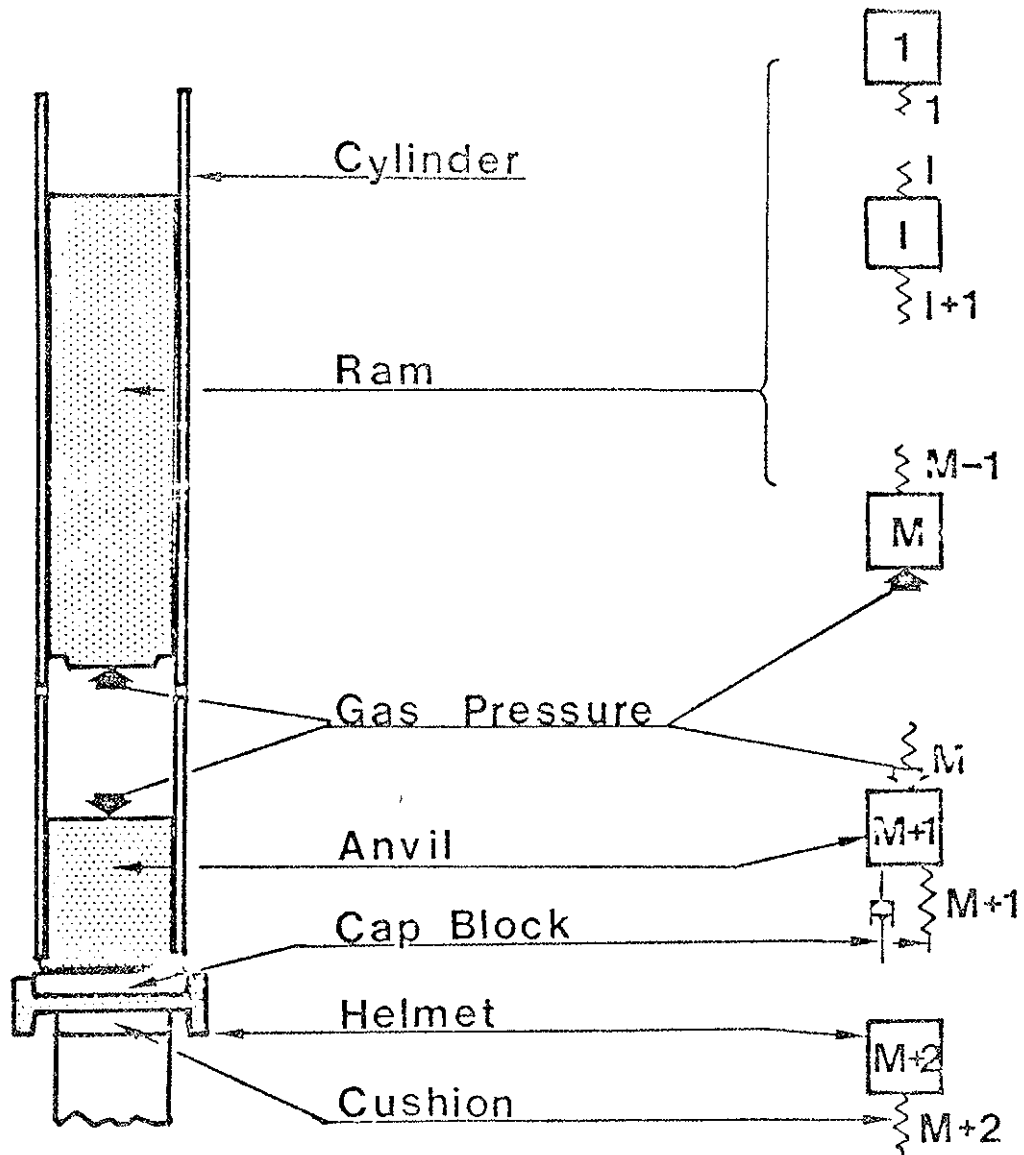


FIGURE 3-2: (A) SCHEMATIC AND (B) MODEL OF OPEN END DIESEL HAMMER

Similarly one obtains

$$HM(M+1) = \frac{W_A}{g} \quad (3-2)$$

and

$$HM(M+2) = \frac{W_c}{g} \quad (3-3)$$

with W_A and W_c being the weight of the anvil and capblock, respectively. Thus, the hammer model always consists of $M+2$ elements. M depends on the length of the ram. Initial studies showed that ram segments of two to three feet in length yield sufficient accuracy.

The springs connecting the hammer masses have the following stiffnesses (if the ram is uniform):

$$STH(I) = \frac{A_R E}{L_R} M \quad I \leq M \quad (3-4)$$

with A_R , L_R and E being the cross sectional area, length and elastic modulus of the ram, respectively. Furthermore,

$$STH(M) = \frac{\left(\frac{A_R M}{L_R}\right) \left(\frac{A}{L_A}\right)}{\frac{A_R M}{L_R} + \frac{A_A}{L_A}} E \quad (3-5)$$

with A_A and L_A being cross sectional area and length of the anvil, respectively. Note that $STH(M)$ is the spring against which the ram impacts.

Since energy losses are usually associated with such an impact, the

unloading stiffness of the anvil spring is:

$$\overline{STH(M)} = \frac{STH(M)}{EANV^2} \quad (3-6)$$

where EANV is the coefficient of restitution of the anvil. Similarly, if ECAP is the coefficient of restitution of the capblock the unloading stiffness of the capblock spring is:

$$\overline{STH(M+1)} = \frac{STH(M+1)}{ECAP^2} \quad (3-6a)$$

STH(M+1) depends solely on the cushion properties in the capblock. If there is a cushion at the pile top with ECUS as a coefficient of restitution, then:

$$\overline{STC} = \frac{STC}{ECUS^2} \quad (3-6b)$$

This stiffness must be combined with that of the pile top element for which

$$\overline{STP(1)} = \frac{STP(1)}{ETOP^2} \quad (3-6c)$$

(the definition of STP(1) will be given below). In this way a combined pile top and cushion stiffness is obtained:

$$STH(M+2) = \frac{(STC)(STP(1))}{STC + STP(1)} \quad (3-7)$$

For unloading the corresponding expressions can be found when the values of Equations (3-6b) and (3-6c) are used.

As a deviation from the usual approach the loading stiffness, say STH was always calculated as $(EA/L)(e^2)$, e being the coefficient of restitution. Then the unloading slope, \overline{STH} , becomes EA/L . In other words, if $k = EA/L$ is the stiffness of a segment or component as determined in a compression test, then k is used as the unloading stiffness in the dynamic application. The effect is a lower stiffness during loading. It is felt that this approach is justified in light of local plastification during the rapid compression which is not present during the expansion phase of the material. The program uses this approach for all springs where coefficient of restitution is less than 1.

Special consideration was given to the fact that the slope of a stress-strain curve of cushioning material usually is gradually increasing and does not - at zero stress - start with its maximum value. Similarly, the force deformation curve of two colliding bodies such as the ram impacting against the anvil cannot show an ideal elastic behavior.

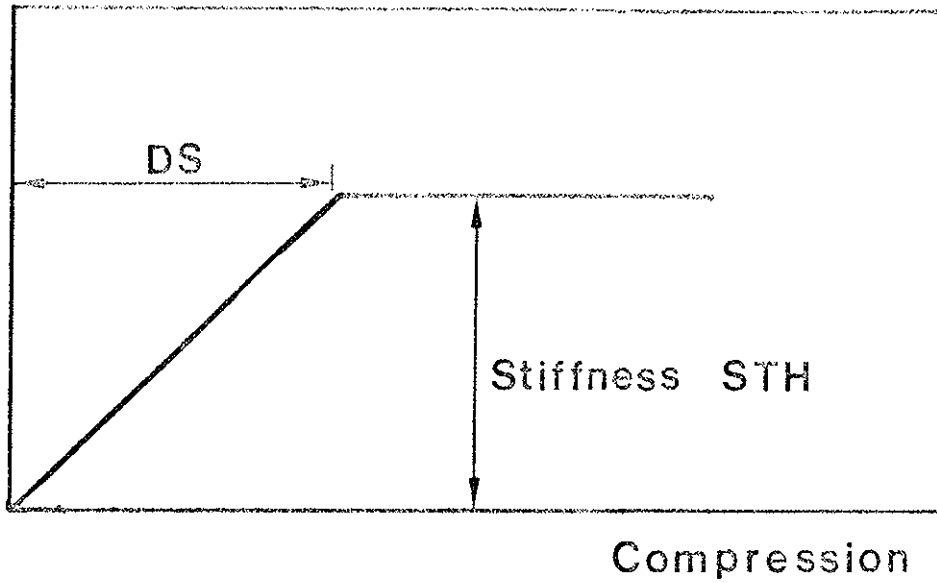
It is neither possible nor necessary to provide a quantitatively exact model. Qualitatively, though, the curves can be rounded and the result can be judged by comparison with measurements.

For this reason three displacement values, DS , were assumed for anvil, cap and cushion. For a deformation less than or equal to DS the stiffness, ST , was assumed to be linearly increasing with the deformation (Figure 3-3a).

Thus, the modified stiffness is:

$$ST = STH(I) \frac{DS(I-1) - DS(I)}{DS}, \quad I = M, M+1, M+2 \quad (3-8)$$

(A)



(B)

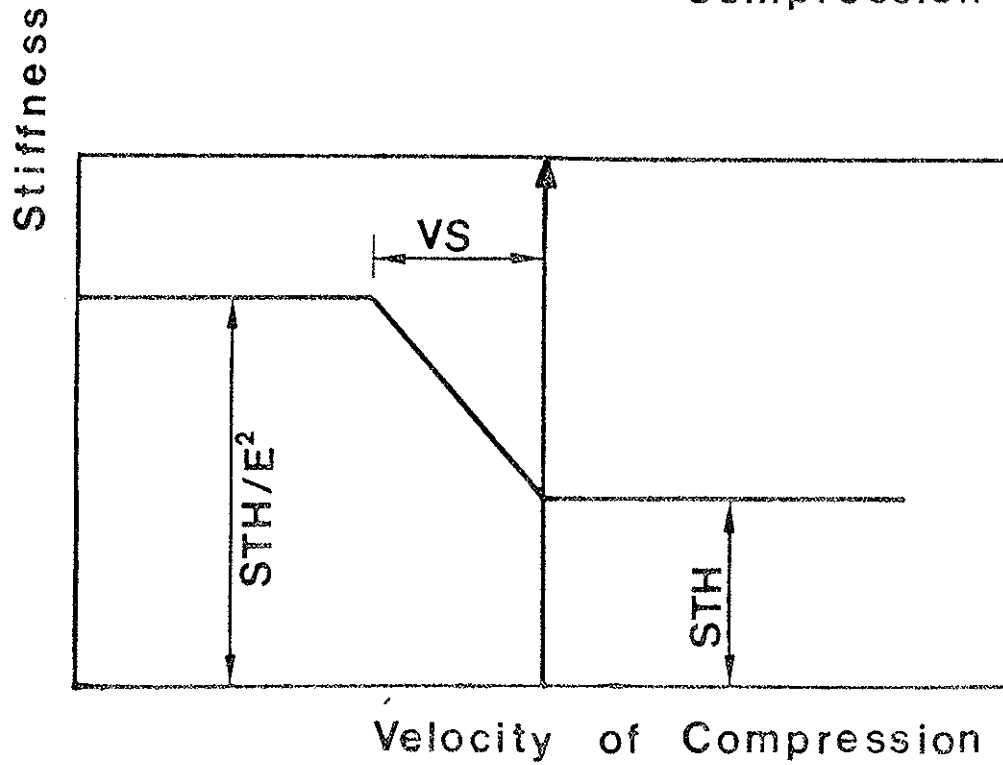


FIGURE 3-3: (A) STIFFNESS VS. COMPRESSION RELATION (B) STIFFNESS VS. COMPRESSION VELOCITY RELATION; BOTH FOR DRIVING SYSTEM COMPONENTS

which is valid for $DS > DNH(I-1) - DNH(I)$, with $DNH(I)$ being the displacement of a hammer element, I , at a certain time. Modifications of this formula have to be made where the deformation becomes small during unloading. A load deformation curve obtained by going through several cycles of loading, unloading and separation is shown in Figure 3-4.

Another point of concern was the change of slope of the load deformation curve when the velocity changes sign. This change was programmed such that the stiffness of hammer springs M , $M+1$ and $M+2$ was changing linearly from the low to the high value between 0 and a negative compression velocity, VS (see Figure 3-3b), which was set at -0.5 ft/sec (0.15 m/sec).

In addition to the bilinear spring a dashpot was added to model the cap-block. The dashpot constant was set at 2% of the ram's critical damping value ($2\sqrt{STH(1)(HM(1))}$). This damper is present more to improve the discrete model's accuracy than for physical reasons. It was found that agreement with measured data improved when using such a dashpot. An additional dashpot was not used with the cushion since the pile top already contains one in its model (see 3.3).

3.2.3 The Thermodynamic Model of the OED Hammer

The thermodynamic model is divided into four parts: (a) compression, (b) combustion delay, (c) ignition and (d) expansion. These four phases are illustrated in Figure 3-5.

Compression begins when the ram has fallen to the level of the exhaust ports. The ram velocity is then:

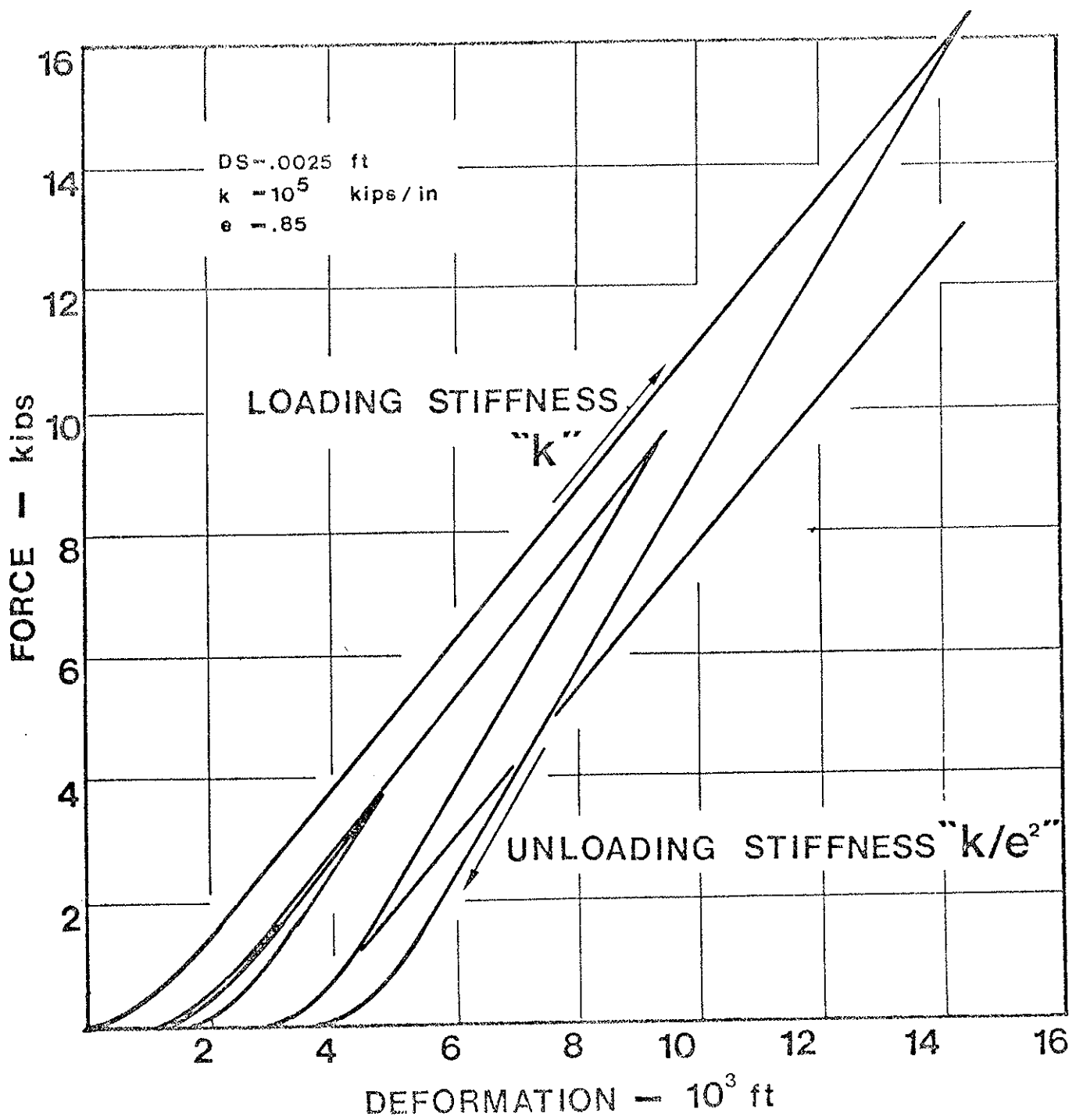


FIGURE 3-4: EXAMPLE OF FORCE VS. DEFORMATION RELATION FOR COMPONENTS OF DRIVING SYSTEM

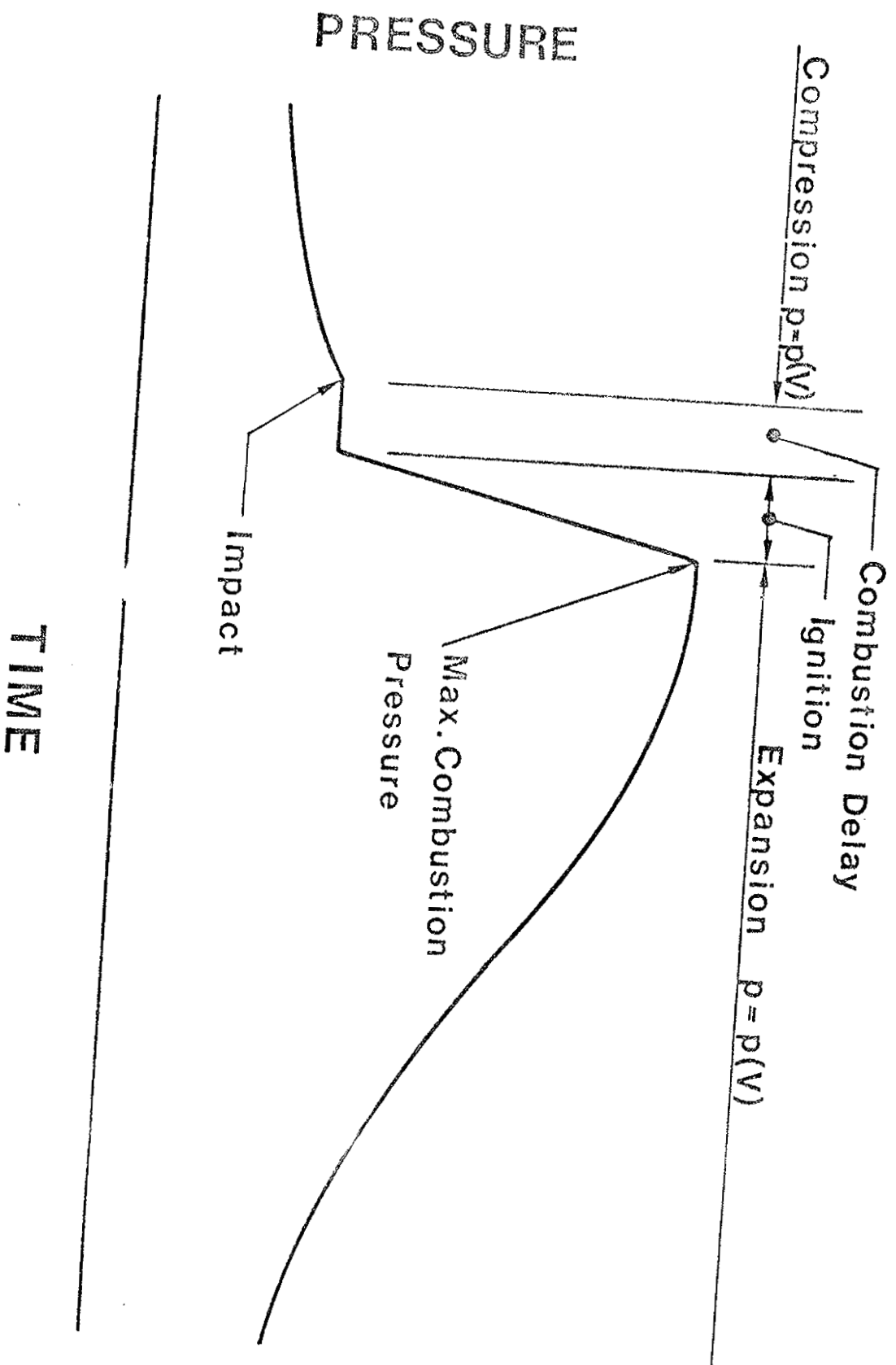


FIGURE 3-1: THE FOUR PHASES OF THE THERMODYNAMIC MODEL

$$V_{R,E} = \sqrt{2g(\text{STROKE} - \text{DEPIB})} \quad (3-9)$$

where STROKE is the ram fall height and DEPIB is the distance between the exhaust ports and the impact block (or anvil). The initial volume of air is

$$V_{IN} = \text{DEPIB} (\text{ARAM}) + \text{VCHAM} \quad (3-10)$$

where ARAM is the cross sectional area of the cylinder and VCHAM is the remaining volume at impact. If during compression the position of the ram is DPOS (measured from the anvil) then the pressure below the ram is

$$P = \left(\frac{V_{IN}}{\text{DPOS} (\text{ARAM})} \right)^{\text{EXPP}} (\text{PATM}) \quad (3-11)$$

Here, EXPP is a parameter depending on the specific heats of the gas in the cylinder and PATM is the atmospheric pressure (14.7 psi or 1.01 bar). For adiabatic compression of air EXPP is 1.4. Since the process is not completely adiabatic EXPP should be chosen less than 1.4. Results of actual measurements were used to determine the correct value (see Appendix A).

With regard to combustion either analytical, experimental or combined approaches can be chosen. The truly analytical approach can only be as good as the available information on many hammer parameters. Actually, there are limits as to the accuracy of the analytical approach since effects of cooling, scavenging, fuel atomization and others can only be estimated. It should be added that efforts were actually made to predict combustion pressures. Appendix A contains a sample calculation and a discussion of such results.

The truly experimental approach can only be as good as the measurements are. In particular it is important to obtain representative data. On the other hand, limitations on the number of independent variables considered will always have to be imposed. For the present program the most important limitations on the independent variables are:

- (a) Normal ambient temperature (approximately 68°F (20°C)).
- (b) Normal atmospheric pressure (14.7 psi (1.01 bar)).
- (c) Normal hammer performance and condition (compression, temperature, lubrication, fuel injection).
- (d) Normal fuel type.

A hammer tested under these conditions, i.e. measurements of pressure stroke and pile top force and acceleration, provides sufficient information. The parameters to be extracted would be time of ignition and magnitude of pressure. Effects of variable stroke on the pressure behavior should be studied. The records should be obtained under normal conditions.

Since it is too difficult to obtain pressure-volume data for most diesel hammers the pressure must be computed in the program after ignition using an appropriate model. An expansion modeled according to Equation (3-11) was found to be sufficiently accurate. This model can be considered a combined experimental and analytical approach.

The validity of the model of Figure 3-5 is shown in Figure 3-6. The solid curve in Figure 3-6 is a pressure record* obtained on a DDMAG D-12 hammer under "normal" conditions. The record exhibits first a gradual increase

*Courtesy of the Foundation Equipment Corporation, Maysesstown, Ohio.

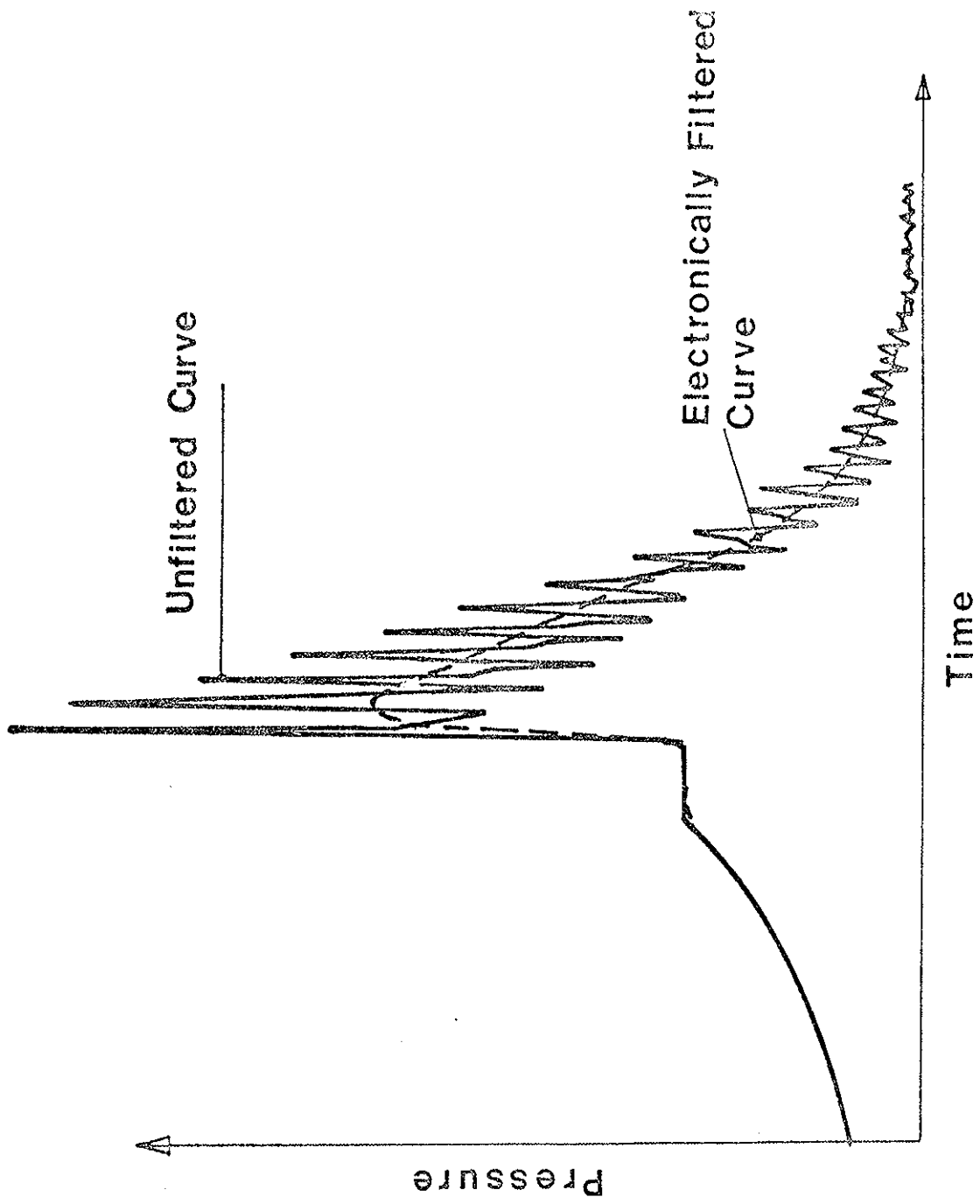


FIGURE 3-6: MEASURED COMBUSTION CHAMBER PRESSURE

during precompression; then it stays constant for a short time period which is the time between impact and combustion and is called the combustion delay; next, it displays a very rapid increase (ignition) and very high magnitude, high frequency variations which gradually decay.

While the timing information on this record is accurate, it is very hard to determine the average or effective maximum pressure. For this reason a low pass, 1 kHz filter was employed which produced the dashed curve of Figure 3-6. The rise of pressure is slower in the filtered than in the unfiltered curve, the maximum pressure value, however, can easily be determined.

Further conclusions for the pressure record are:

- (a) The high frequency components of the record are caused by pressure waves in the chamber and are of little significance to the hammer behavior. This phenomenon has also been reported in measurements made on internal combustion engines.
- (b) The expansion process can only be judged in connection with actual pressure-volume data. It was found by simulation and comparison with measured data that the records represented Otto rather than Diesel cycles and that the expansion model can be rather simple.

In summary, the following values are used in the model:

- (a) The expansion coefficient, EXPP, for the compression phase was set at 1.35.
- (b) The duration of the combustion delay can vary but was taken at 0.002 seconds.
- (c) The maximum combustion pressure depends on measurements.
- (d) The duration of the ignition phase was set to 0.5 milliseconds.

- (e) The expansion coefficient, $EXPP$, for the expansion phase was set to 1.30.

Certain modifications of this model are necessary when a hammer using atomized fuel injection is considered. In this case the combustion delay has to be computed from the start of the fuel injection and the duration of the ignition should be increased to cover the duration of the injection.

For cases where manufacturer's data were not available, both a delay and duration of 10 milliseconds was used with the restriction that ignition occurs at least at impact. It should be mentioned that accurate timing information for atomized injection may be very important.

3.2.4 Working Principle of the Closed End Diesel Hammer

The closed end hammer works very much like the open ended one. In principle the main change consists of a closed cylinder top. Figure 3-7 shows two of these hammer types. When the ram moves upward, air is being compressed at the top of the ram which causes a shorter stroke and, therefore, higher blow rate.

The bounce chamber has ports such that atmospheric pressure exists as long as the ram top is below these ports. As the ram moves toward the cylinder top it creates a pressure which increases until it is just in balance with the weight of the cylinder itself. Further compression is not possible and if the ram still has an upwards velocity uplift of the cylinder will result. This uplifting cannot be tolerated as it can lead both to an unstable driving condition and to the destruction of the hammer. For this reason the fuel amount and, hence maximum combustion chamber pressure, has

CLOSED END DIESEL HAMMERS

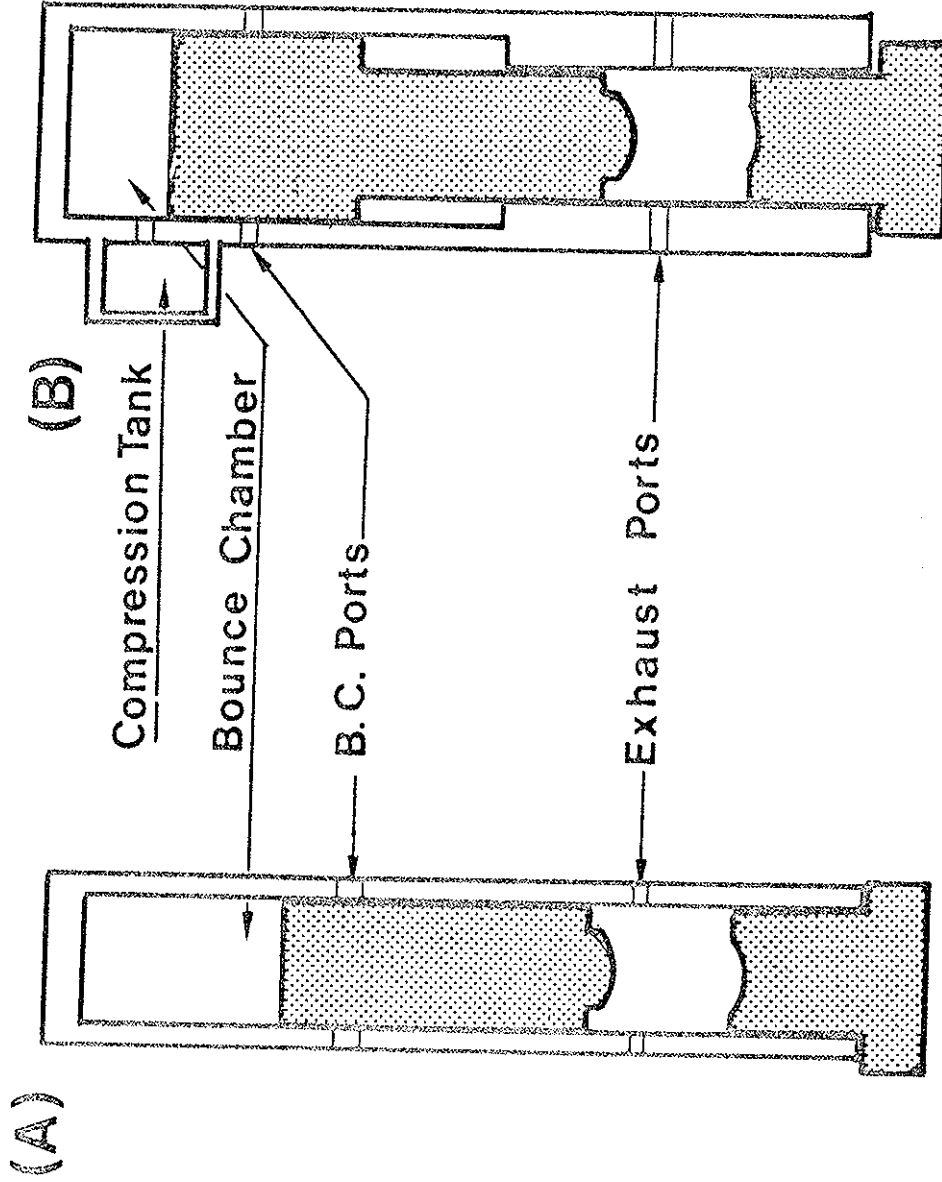


FIGURE 3-7: SCHEMATICS OF CLOSED END DIESEL HAMMERS
(A) UNIFORM (B) NON-UNIFORM RAM

to be reduced such that there is only a very slight lift off or none at all.

Uplift occurs only when the soil resistance forces are sufficiently high. For low resistance forces the stroke will be less than the one for which uplift is imminent.

Another feature of a few closed end hammers (Linkbelt) is an improved scavenging system. This design uses both intake and exhaust ports and an air tank which provides a horizontal air flow through the cylinder when the ram moves downward. It can be expected that the relative amount of unscavenged combustion products present during combustion is smaller than in other hammers.

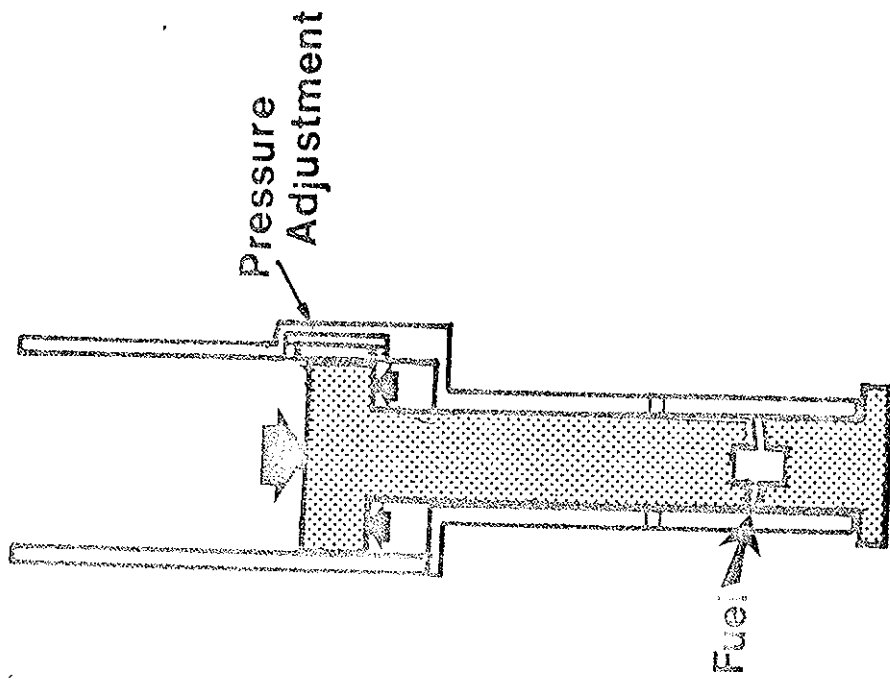
Another closed end hammer type (BSP) should be discussed. This hammer type employs a vacuum chamber below its ram to increase its operating speed. Two phases of this hammer's operation are shown in Figure 3-8. As can be seen, the hammer is not really closed at the top (although a protective cover is usually present). However, the stroke is limited by the vacuum action under the upper portion of the ram such that it does not become visible during operation. In addition to being faster than an open end hammer this type has the advantage of not lifting off during operation since the vacuum force is always less than the cylinder weight.

3.2.4.1 Hammers with Uniform Rams

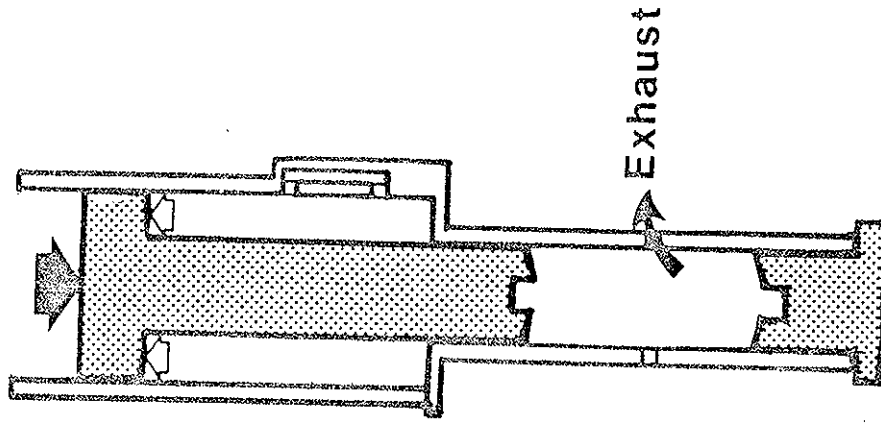
The design of these hammers is not very different from the open end hammers. The principal difference is that the ram on its upward travel

VACUUM CHAMBER - DIESEL HAMMER

Fuel Injection
and Impact



Stroking



Atmospheric
Vacuum

FIGURE 3-8: SCHEMATIC OF A VACUUM CHAMBER DIESEL HAMMER

closes the bounce chamber exhaust ports and, since the cylinder is closed at the top, compresses the air above the ram (Figure 3-7a). Since the bounce chamber is filled with air, compression occurs adiabatically to a chamber pressure:

$$PB = PATM \left(\frac{DEPBB}{DPOS} \right)^{1.4} \quad (3-12)$$

where DEPBB is the distance from exhaust ports and DPOS is the distance from the ram top to the bounce chamber top. Of course, for DPOS being greater than DEPBB

$$PB = PATM. \quad (3-13)$$

There exists a value of DPOS at which the pressure becomes so large that the cylinder starts to lift up. If the weight of the hammer excluding ram and driving system is given by RWH, then this limiting pressure is given by

$$PLIM = \frac{RWH}{ARAM} + PATM. \quad (3-14)$$

From Equation (3-12) one finds the maximum stroke is

$$STRMAX = DBCIB + DEPBB \left(1 - \left(\frac{PATM}{PLIM} \right)^{\frac{1}{1.4}} \right). \quad (3-15)$$

DBCIB is the distance that the ram travels upward before it closes the bounce chamber ports.

3.2.4.2 Hammers with nonuniform rams and compression tanks

Sometimes the top of the ram has a larger cross section (ART) than the

bottom in order to reduce the precompression force for better hammer performance in easy driving. Also, a compression tank may be attached to the bounce chamber. In this way the bounce chamber volume is relatively large until the ram closes the ports of this tank. The remaining volume in the cylinder is called a safety volume. As the ram progresses into it the pressures increase very rapidly and hammer lift off will soon occur.

Because of the existence of a pressure tank of volume VCT, Equation (3-12) has to be revised. The bounce chamber pressure is then:

$$PB = PATM \left(\frac{(DEPBB)(ART) + VCT}{(DPOS)(ART)} \right)^{1.4} \quad (3-16)$$

which is valid for $DPOS \geq DSF$. (DSF is the distance from the pressure tank port to the bounce chamber top).

The maximum pressure in the compression tank, PCT, is given by substituting DSF for DPOS in Equation (3-16). Defining

$$VBIN = (DEPBB)(ART) \quad (3-17a)$$

and

$$VSF = (DSF)(ART) \quad (3-17b)$$

one obtains

$$PCT = PATM \left(\frac{VBIN + VCT}{VSF} \right)^{1.4} \quad (3-18)$$

and for $DPOS < DSF$ one can now find

$$PB = PCT \left(\frac{DSF}{DPOS} \right)^{1.4} \quad (3-19)$$

or

$$\text{STRMAX} = \text{DBIB} + \text{DEFBS-DSF} \left(\frac{\text{PCT}}{\text{PLIM}} \right)^{1/1.4} \quad (3-20a)$$

If the maximum stroke occurs for $\text{DPOS} > \text{DSF}$ then

$$\text{STRMAX} = \text{DBCIB} + \frac{\text{VBIN+VCT}}{\text{ART}} \left(\frac{\text{PATM}}{\text{PLIM}} \right)^{1/1.4} \quad (3-20b)$$

3.2.5 The Vacuum Chamber Hammer

This hammer type utilizes a vacuum rather than compressed air as for other closed end hammers to reduce its stroke and, therefore, increase its blow rate. The essential components and phases of operation are shown in Figure 3-8. Since the vacuum force is limited (maximum: atmospheric pressure, PATM , times the difference in area between ram top and bottom, DELA) uplift cannot occur as long as the cylinder weight exceeds this value. The hammer can therefore be treated as an open ended hammer type as long as the vacuum force is properly considered.

If the distance of the ram above the anvil is DBC , and the pressure in the vacuum chamber starts to decrease when the ram is at a distance DSTART and if $\text{DIN} = \text{VIN}/\text{DELA}$ (with VIN = the volume in the chamber at $\text{DBC} = \text{DSTART}$) then the net force, F , on the ram in downward direction is

$$\text{F} = \text{PATM} (\text{DELA}) \left(1 - \left(\frac{\text{DIN}}{\text{DBC} - \text{DSTART} + \text{DIN}} \right)^{1.4} \right). \quad (3-21)$$

Of course, if DBC is less than DSTART (anvil has moved downward) then no vacuum force exists. It should be noted that the cylinder position, DCYL , is considered in the program but not included here for the sake of clarity.

In order to be able to treat the hammer like an open ended one a formula must be derived for the ram velocity at the ports, VFALL:

$$VFALL = \sqrt{2 \frac{G}{W} ((STROKE - DEPIB)(W + PATM(DELA)) + DP)} \quad (3-22)$$

where $DP = PATM (DELA) \frac{DIN^{1.4}}{0.4} (DB^{-0.4} - DE^{-0.4})$,

$$DE = DEPIB - DSTART + DIN,$$

$$DB = STROKE - DSTART + DIN$$

and W and G are ram weight and gravitational acceleration, respectively.

3.2.6 The Air/Steam Hammer Model

The Air/Steam Hammer is much simpler to model than diesel hammers, first, because it has an external power supply and, second, because it has only a few simple hammer components. The ram usually consists of a compact block with a so-called ram point attached to its bottom. The ram point strikes against the capblock. For this reason, the impact is cushioned by a soft material. This in turn allows the ram flexibility to be neglected.

The ram is raised by externally produced air or steam pressure (Figure 3-9b) acting against a piston, housed in the hammer cylinder, which in turn is connected to the ram by a rod. Once the ram is raised a certain distance a valve is activated and the pressure in the chamber is released. At that time the ram has some upward velocity. Therefore it "coasts" up to the maximum height (stroke) and then falls free to impact on the capblock (Figure 3-9a). Upon impact pressure enters again the cylinder. The hammer described

SINGLE ACTING AIR/STEAM HAMMER

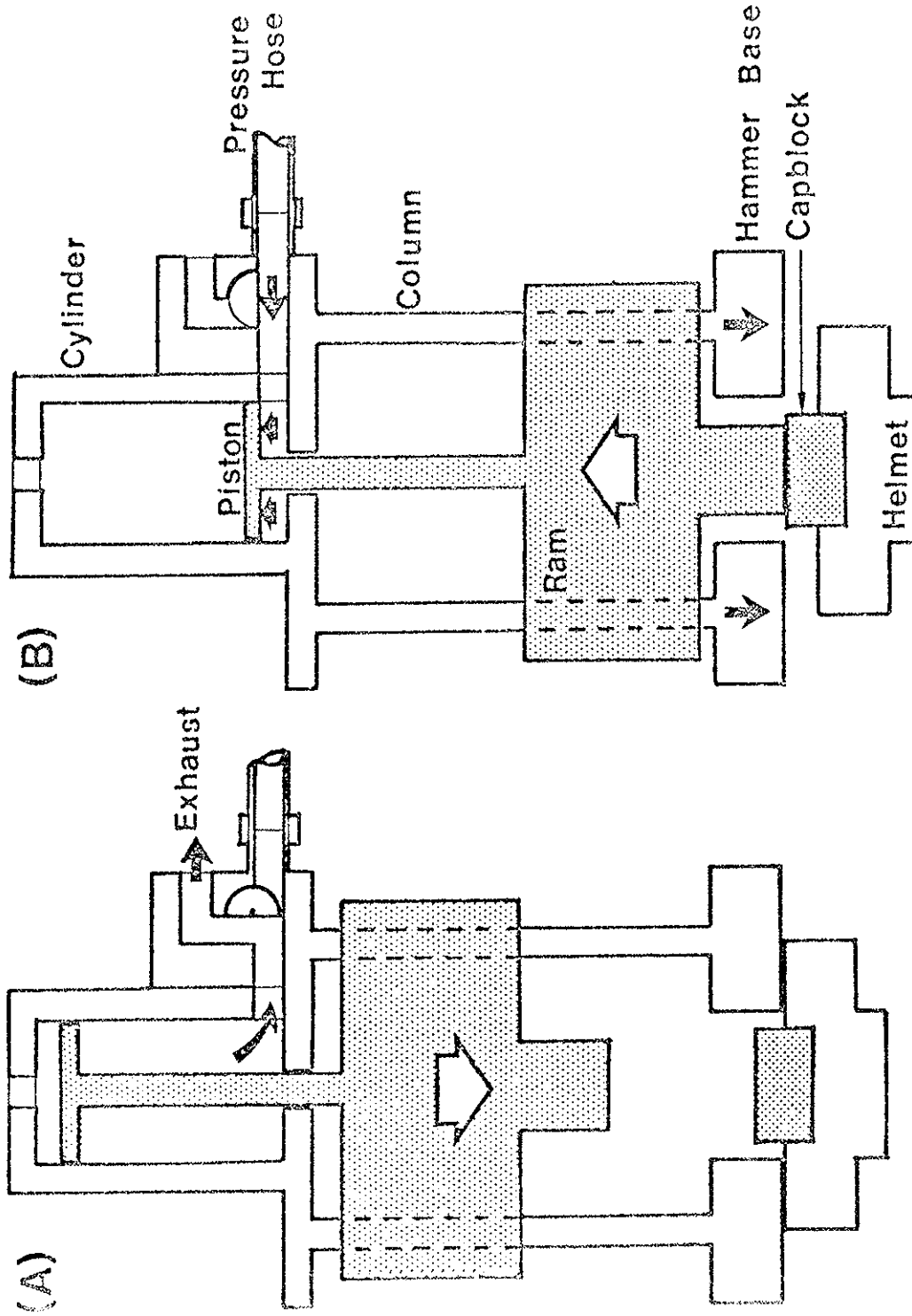


FIGURE 3-9: SIMPLE ACTING AIR/STEAM HAMMER
(A) DURING FALL (B) AFTER IMPACT

above is known as the single acting air/steam hammer.

A double acting hammer is one in which the ram is accelerated during its fall by pressure in addition to gravity. The working principle of one type of double acting hammer, the differential hammer, is illustrated in Figure 3-10.

The pressure applied at the top of the piston acts against the top of the cylinder similar to the bounce chamber pressure in a closed end diesel hammer. Of course, the pressure could lift the hammer assembly, of weight W_A to which the cylinder is connected. Thus, maximum hammer output will be achieved when the pressure is kept at its upper limit ($PLIM$, here gage pressure) at which assembly lift off is incipient.

The maximum energy of a differential acting hammer is

$$E_{MAX} = (W + PLIM(A)) STRM \quad (3-23a)$$

where W is the weight of the ram, A is the effective cylinder area, and $STRM$ is the maximum stroke of the hammer. (Note that the effective cylinder area is equal to that of the smaller (lower) piston). Since

$$A = W_A/PLIM \quad (3-23b)$$

the potential energy of a hammer driven by an actual pressure P_{STEAM} is

$$E = (W + P_{STEAM} \frac{W_A}{PLIM}) STRM. \quad (3-23c)$$

From this relation an effective stroke, $STROKE$, is derived that a weight W ,

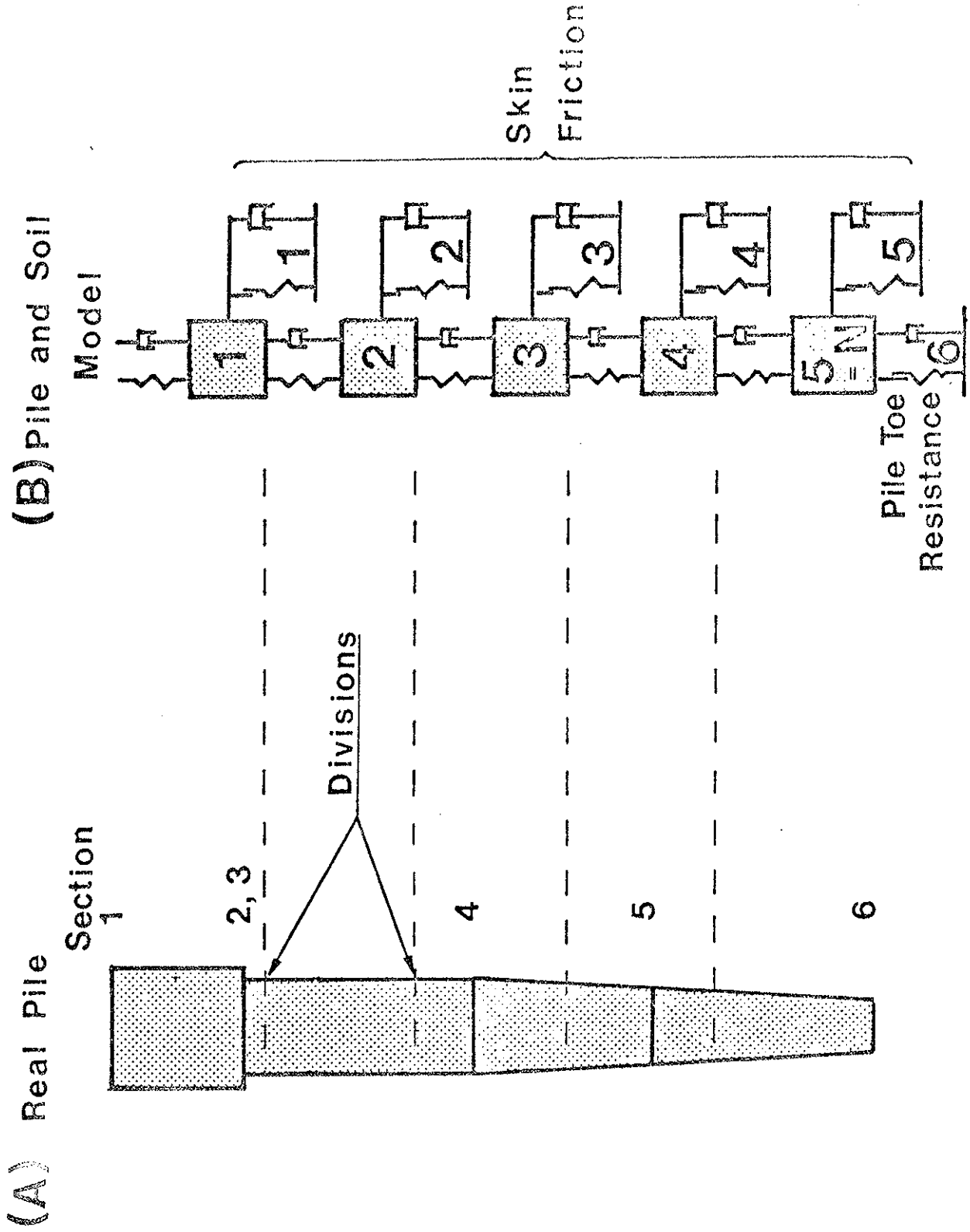


FIGURE 3-1: (A) SCHEMATIC REPRESENTATION OF PILE AND (B) PILE AND SOIL MODEL

with $\overline{R\bar{P}}$ being the average mass density of the pile and $\overline{A\bar{P}}$ the average cross sectional area of the pile. The averages are taken over DL.

Similarly the segment stiffnesses are

$$STP(I) = \frac{\overline{E\bar{P}} (\overline{A\bar{P}})}{DL} \quad (3-29)$$

where $\overline{E\bar{P}}$ is the average elastic modulus over the element length. Obviously, multi-material piles can be treated in this fashion.

A third parameter, the pile damping value can be specified for the pile. Since little is known about the correct structural damping model and since this type of damping produces relatively small forces compared to soil damping, an elaborate model does not seem justified. Thus, viscous damping was assumed with parameters:

$$CDP(I) = \frac{2}{100} \text{IBD} \sqrt{STP(I) PM(I)} \quad (3-30)$$

with IBD being the damping constant in percent of critical. Note that Equation 3-30 is based on the definition of critical damping of the one degree of freedom oscillator.

The damping force between two segments is then

$$PD(I) = CDP(I) (VP(I-1) - VP(I)) \quad (3-31)$$

where $VP(I)$ is the velocity of the I-th segment.

The damping constants IBD are not related to pile length. Thus, if for a particular pile a different number of elements are used with the same damping parameters, then the total damping force is different. This approach is not quite satisfactory and is only tolerated in light of the small effects of pile damping and the limited knowledge of material damping. Further efforts in this area are encouraged.

3.4 Soil

The soil model used offers as an option some differences from the one usually employed. It basically consists of a spring and dashpot (Figure 3-12). The elastic spring yields at a pile segment displacement $QS(I)$ (quake) such that there is no further increase in static resistance with increased displacement ($RS(I) = SU(I)$, SU being the ultimate static resistance at that element). Unloading, i.e. when the pile segment has an upward velocity, follows at a spring rate that is in the usual case equal to that in the loading path.

An option was built into this model allowing the use of a coefficient of restitution, $ESOIL$, which is employed such that

$$\overline{SOK(I)} = \frac{SOK(I)}{ESOIL^2} \quad (3-32)$$

where $SOK(I)$ indicates the soil stiffness at the I -th pile element and where the barred quantity indicates unloading. Note that $ESOIL$ is the same for all pile elements.

In the usual case where the pile experiences an appreciable net set the

energy consumption by the use of a coefficient of restitution is negligible. However, for the very hard driving cases (blow count greater than 120 blows per foot) an energy loss due to an ESOIL - value will improve the validity of the soil model.

The damping model can be chosen according to Smith

$$DAM = J(I) VP(I) RS(I) \quad (3-33)$$

where DAM is a damping force in kips (N) and J(I), VP(I) and RS(I) are the damping factor in sec/ft (sec/m), the pile velocity in ft/sec (m/sec) and the static resistance force in kips (N), respectively; all taken at the same pile segment I.

The second choice is a non-dimensionalized viscous damping for which

$$DAM = JC(I) VP(I) \sqrt{STP(I) PM(I)}. \quad (3-34)$$

Here JC(I) is the Case (Institute of Technology) damping factor of unit dimension. Note that

$$\sqrt{STP(I) PM(I)} = EA/c \quad (3-35)$$

(this is also called pile impedance; Young's modulus, E, times cross sectional area, A, divided by wave speed, c; all in the pile). The use of the expression on the left of Equation 3-35 is preferred as it reflects the average properties at an element. Recalling that viscous damping is defined as

$$DAM = JV(I)VP(I) \quad (3-36)$$

with $JV(I)$ being the viscous damping constant it is apparent that

$$JC(I) = \frac{JV(I)}{\sqrt{STP(I) PM(I)}} = \frac{JV(I)}{EA/c} \quad (3-37)$$

while Smith's damping factor becomes

$$J(I) = \frac{JV(I)}{RS(I)} \quad (3-38)$$

The distribution of damping is handled in the following way: For Smith's damping a constant factor is used along the pile skin and another factor is used at the toe. This actually means that the corresponding viscous damping factor varies proportionally to the static resistance distribution along the skin. In order to have a similar situation for the Case damping approach the input consists here also of skin and toe damping factors which are converted to viscous damping factors by virtue of Equation 3-37. The skin damping factor is then distributed to the segments in proportion to the static resistance.

3.5 Numerical Treatment

There are two aspects in the numerical treatment of the analysis that differ substantially from the TTI or Smith's approach. First the integration equations involve also the acceleration terms, and second a so-called predictor-corrector approach is used.

The integration equations are simply (note that subscripts indicating

element numbers are omitted if not necessary for clarity):

$$VN = \frac{1}{2} (AO + AN) DT + VO \quad (3-39)$$

and

$$DN = \frac{1}{6} (2AO + AN) DT^2 + VO(DT) + DO. \quad (3-40)$$

In these equations A stands for acceleration, V for velocity, D for displacement, O (old) for the beginning and N (new) for the end of the current time increment, DT.

Starting with the hammer and continuing with the pile a prediction of the values DN and VN is made by integrating according the Equations 3-39 and 3-40. Since AN is not known it is set to AO. Then the top (FO) and bottom (FU) spring forces at the I-th element can be computed:

$$FO = (DN(I-1) - DN(I)) ST(I) \quad (3-41a)$$

and

$$FU = (DN(I+1) - DN(I)) ST(I+1) \quad (3-41b)$$

(an H or P after DN and ST would indicate hammer or pile, respectively).

Force contributions due to pile damping at the I-th segment are:

$$FDO = (VNP(I-1) - VNP(I)) * CDP (I) \quad (3-42a)$$

and

$$FDU = (VNP(I+1) - VNP(I)) \cdot CDP(I+1). \quad (3-42b)$$

Static resistance forces, RES, are computed using the soil stiffness SOK

$$RESN = RESO + (DN-DO) SOK \quad (3-43a)$$

with

$$|RESN| \leq SU(I) \quad (3-43b)$$

and for the toe

$$0 \leq RESN \leq SU(N+1) \quad (3-43c)$$

Due to damping resistance

$$DA = (SJ)(VNP) \quad (3-44a)$$

for Case and

$$DA = (SJ)(VNP)(RESN) \quad (3-44b)$$

for Smith's damping.

Including the gravitational acceleration, G, the new acceleration value can now be found:

$$AN = G + (FO + FU + FDO + FDU - RESN - DA)/PM \quad (3-45)$$

(For ram bottom and anvil this equation contains, of course, the gas pressure force too). Integration of AN leads to new VN and DN values.

Once all VN and DN values are corrected the change from the previous quantities is determined and the process is repeated, computing again for all elements the various quantities starting with Equation 3-41a. This process is repeated until both the pile top force and bottom velocity converge. An input directed maximum number of cycles (ITER) is observed.

CHAPTER 4

PROGRAM INPUT INFORMATION

4.1 Introduction

In this chapter the program input information will be described briefly. The input structure was designed so as to minimize the effort required of the engineer in the preparation of data. Therefore, as much routine computation as possible is performed internally. However, in some cases the flexibility of very detailed engineer input is desirable, so options are available which control the form of the input required by the program.

In one particular case, the hammer, it is usually unnecessary for the user to prepare detailed data to describe the hammer since there is a limited number of available hammers. The TTI program has required that each user obtain the hammer details from manufacturers or dealers. A great danger exists that incorrect information is input. In the WEAP program hammer information has been obtained from the manufacturers. Where questions arose the manufacturer was contacted and clarification obtained.

4.2 Open End Diesel Hammer

To call a hammer the user needs only to specify the hammer number (IHAMR) from a list of hammers in the file. There exists an option (IHAMR = 0) that allows the input of all hammer parameters. These parameters are in order:

NAME an alphanumeric string of up to eight characters.
HM(I) weight of the I-th ram segment in kips for all I(I=1,M).
STH(I) ... stiffness of the I-th ram segment in kips/inch for I = 1,
M-1.

DIFFERENTIAL ACTING AIR/STEAM HAMMER

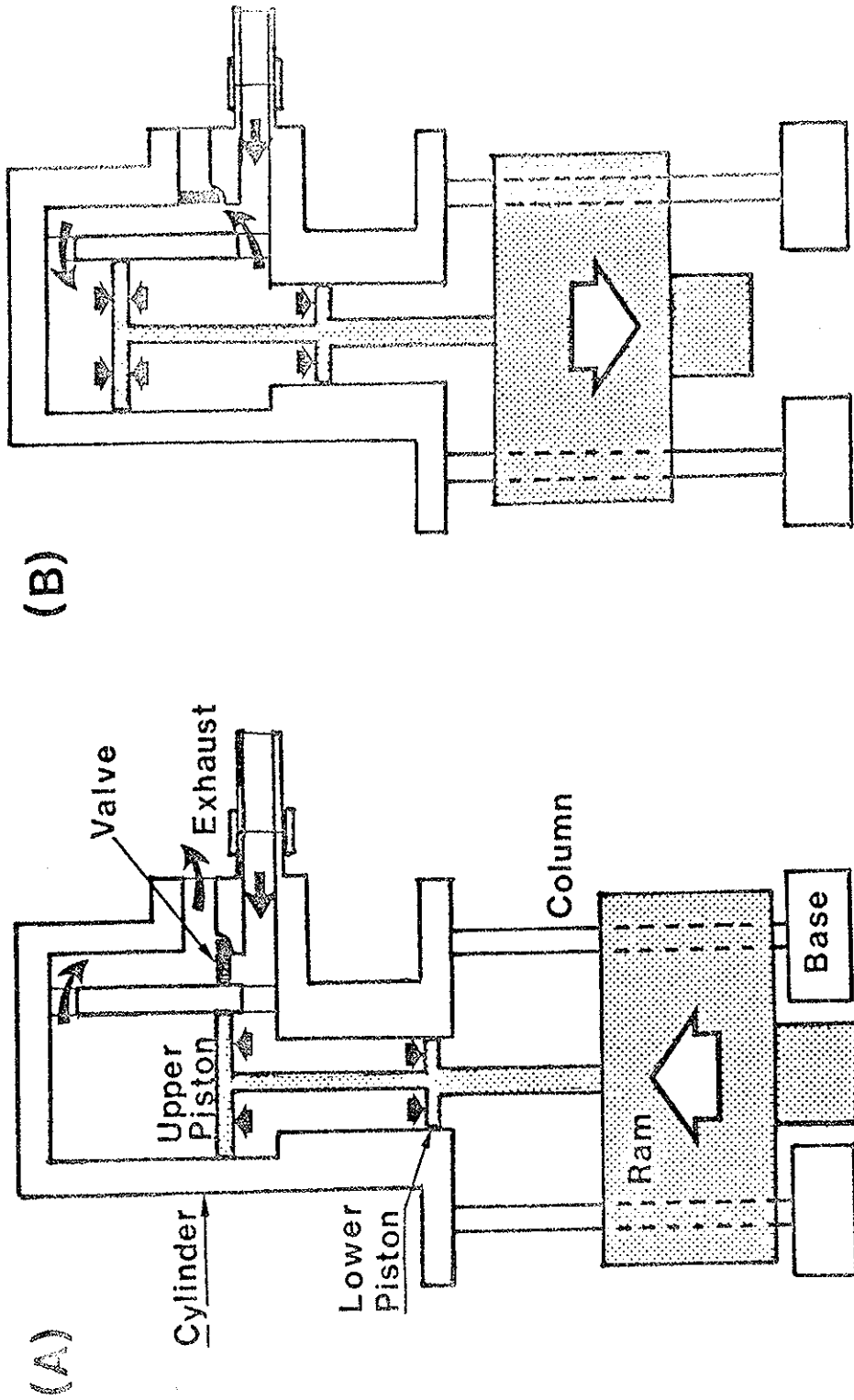


FIGURE 3-10: DIFFERENTIAL ACTING AIR/STEAM HAMMER
(A) AFTER IMPACT (B) DURING FALL

should fall in order to provide a kinetic energy equal to E:

$$\text{STROKE} = \text{STRM} \left(1 + \frac{\text{PSTEAM}}{\text{PLIM}} \frac{\text{WA}}{\text{W}} \right). \quad (3-24)$$

Unfortunately, the quantities STRM and PSTEAM are often lower than specified since the ram might not rise high enough or pressure losses occur and other losses might occur during the fall. For this reason it is common to multiply the effective stroke of both single and double acting hammers by an efficiency, EFFICY, which is a number less than or equal to one.

The ram impact velocity then becomes:

$$\text{VFALL} = \sqrt{(\text{STROKE}) (\text{EFFICY}) 2G} \quad (3-25)$$

(G is the gravitational acceleration).

The mechanical model of the air/steam hammer (Figure 3-11) includes M masses and M-1 stiffnesses to represent the ram. Mass M+1 is the cap mass (capblock plus helmet) and the stiffness M is assigned to the capblock stiffness. In most cases an air/steam hammer is modeled with only one ram mass and no ram stiffness (M = 1). Coefficients of restitution and "DS-values" (see Section 3.2.2) are applicable to the cap cushion, the pile top cushion (if present) and the pile top.

As an addition to this active hammer portion a passive one is considered consisting of MA masses and stiffnesses that represent the hammer assembly. This assembly is assumed to rest on the helmet before impact and to fall freely after impact until an impact of assembly with cap occurs.

AIR / STEAM HAMMER

(A)
Schematic

(B)
Model

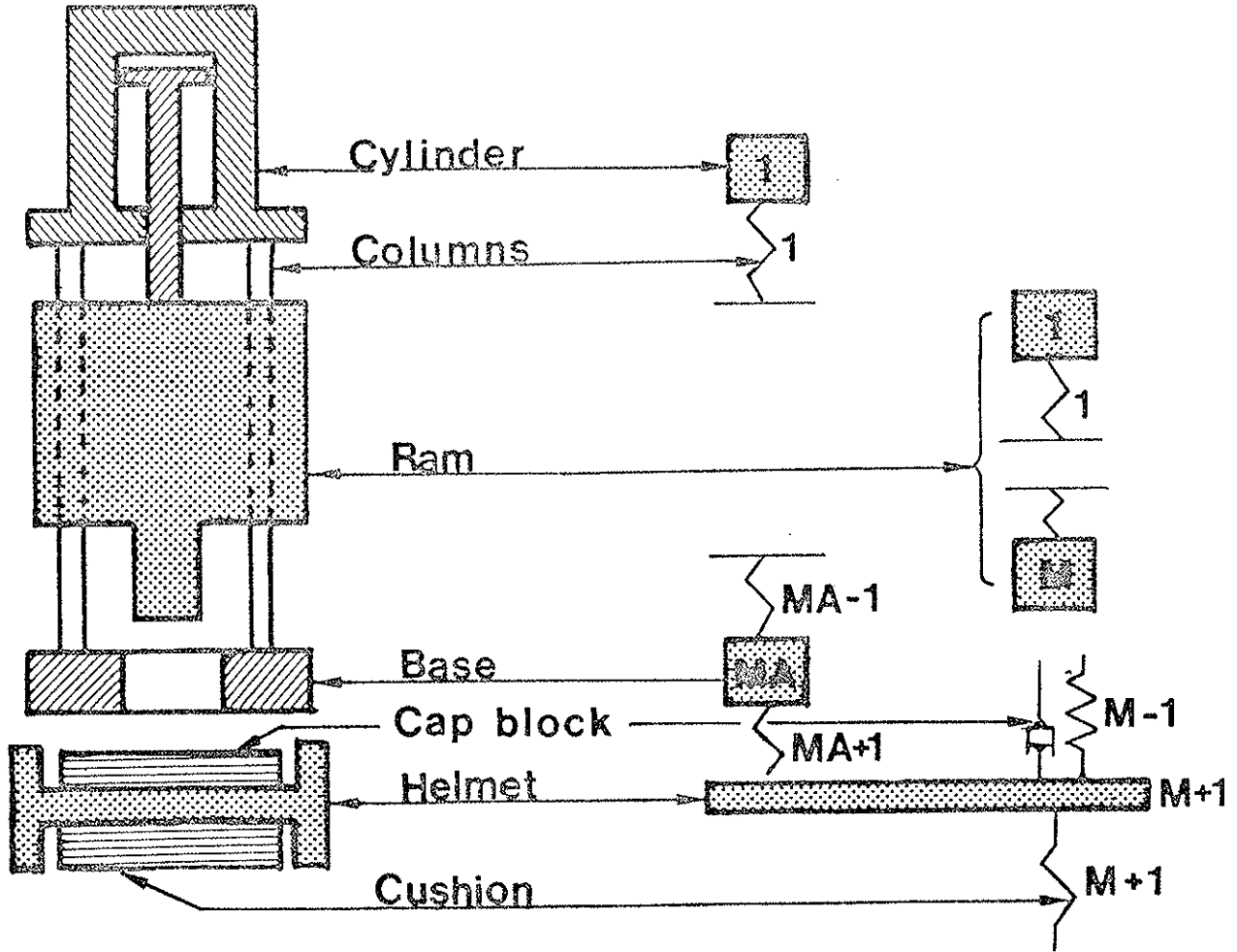


FIGURE 3-11: AIR/STEAM HAMMER (A) SCHEMATIC AND (B) WEAP MODEL

According to Figure 3-11 the model of the assembly will consist of MA masses and MA stiffnesses. Usually, MA equals two. Then the weights correspond to cylinder and base weights and the two stiffnesses should be derived from that of the ram guide bars. Since the impact of the assembly is uncushioned it is recommended that for both springs in this system the same value be used, namely, twice that of all assembly columns. In this way the overall flexibility of the assembly is correctly represented without much sacrifice in accuracy. The flexibilities of both the helmet and lower assembly mass are extremely small and can hardly be modeled as elastic bodies. A coefficient of restitution and a "DS-value" are used in the program to model this assembly drop as realistically as possible.

3.3 Pile

The pile model consists of springs, masses and dashpots (see Figure 3-12). The pile is divided in N segments whose lengths are given by

$$DL = ALPH(I) (XPT) \quad (3-26)$$

where XPT is the total pile length and ALPH(I) is a multiplier which is normalized (by the program) such that:

$$\sum_{I=1}^N (ALPH(I)) = 1.0 \quad (3-27)$$

The mass of the I-th segment is

$$PM(I) = RP \overline{(AP)} DL \quad (3-28)$$

M the number of ram masses. It can be determined by dividing the ram length by 2.5. M should be greater than or equal to two for all Diesel hammers.

HM(M+1) ... weight of anvil in kips

STH(M) ... stiffness of last ram element and anvil combined in kips/inch (see Equation 3-4).

TDEL combustion delay in seconds (values between 0.001 and 0.0025 seconds are normal for regular injection).

VFIN final volume or combustion chamber volume in cubic inches.

DEPIB distance between exhaust ports and impact block (anvil) in inches.

ARAM cross sectional area of cylinder in square inches.

P1, P2, ... combustion pressures for up to five fuel settings in pounds
P5 per square inch. Note that only P1 must be given. Default of P1 causes the use of P1 = 1000 psi.

EFFICY ... a hammer efficiency as defined for the air/steam hammer.

STRM maximum stroke in inches as specified by the manufacturer. This value will be used as a convergence criterion on the stroke. It is assumed that the stroke cannot exceed this value.

EXPP exponent for the expanding, combusted gases. A value between 1.2 and 1.4 is reasonable.

POWSCAV .. should be set to 1 if the scavenging is independent of stroke.

DINJ distance between ram location at which atomized fuel is injected and anvil in feet. DINJ = 0.0 for regular fuel injection.

TS the divation of atomized fuel injection or the divation of ignition in general.

ITYPH hammer type: here=1.001 for open end diesel.

4.3 Closed End Diesel Hammers

The data needed for the complete description of a closed end or a vacuum chamber (VCH) hammer are (in addition to those for open end hammers) as follows:

- ART the cross sectional area of bounce chamber in square feet.
- DEPBBS the distance between the combustion chamber ports and the top of the combustion chamber in feet (not for VCH).
- DSP the distance between the pressure tank ports (only if present) and the top of the bounce chamber in feet. For VCH this is the distance that the ram travels upward before the vacuum chamber pressure starts to decrease.
- DBBT the distance between the anvil and the top of the combustion chamber minus the ram length in feet (not for VCH). Note that $DBBT - DEPBBS = DBCIB$ as used in Equation 3-15.
- RWH the hammer assembly weight, i.e. that force which cannot be exceeded by the bounce chamber (gage) pressure times ART (not for VCH).
- EXPB the exponent used for calculation of the bounce chamber pressure (usually 1.4).
- VCT the volume of the compression tank in cubic feet. For VCH this is the initial vacuum chamber volume (at the time when the pressure starts to decrease).

Note that ITYPH is to be given as 2.001 for all closed end diesel hammers except vacuum chamber hammers which are type ITYPH = 1.001. Note also that STRM is disregarded for ITYPH = 2.001.

4.4 Air Steam Hammers

These hammer types require the following input values:

(a) As for diesel hammers:

NAME

HM(I), I = 1, M

STH(I), I = 1, M-1

M

Pl (Here, maximum (gage) pressure for which uplift just occurs
in psi; for double acting air-steam hammers only).

EFFICY

STRM which is here the regular working stroke in inches.

RWH (for double acting hammers only).

ITYPH (here equal to 3.001, i.e. air-steam hammer).

(b) In addition to this data, hammer assembly information is necessary
if its effect is to be studied:

AM(I) the assembly weights in kips for I = 1, MA.

STA(I) ... the assembly stiffnesses in kips/inch for I = 1, MA.

MA the number of assembly stiffness values. Note that
MA = 0 causes the assembly analysis to be ignored.

4.5 Other Hammer Related Input Information

This data is required (often at option) even if the program stored
hammer information is used.

IOSTR (Ignored for air/steam hammers).

If set equal to 1 it will cause one stroke (either speci-
fied or unspecified) to be analyzed. Thus, no iteration

on stroke will result. If equal to 1, the stroke specified in STROKE (see below) or the program assumed stroke (5.0 feet for open end and STRMAX for closed end hammers) is analyzed and iteration on maximum fuel pressure is performed.

If 0 (or not specified) then the maximum fuel pressure (P1, ..., P5 as specified by IFUEL) is kept constant and iteration on stroke is performed.

Generally speaking, hammers that are operated at a fixed fuel setting should be analyzed with an iteration on stroke. There are hammers, however, (e.g. BSP) which do not have a fixed setting and which, therefore, should be analyzed by assuming a stroke iterating on fuel setting (combustion pressures).

The engineer must be aware that the fixed stroke option may provide unreasonable answers. For example, a stroke specified relatively high together with a low soil resistance may not be achieved in the field.

IFUEL (Ignored for air/steam hammers).

If set to 1, 2, ---, 5 the corresponding pressures P1, P2, ---, P5 will be used. Thus, IFUEL amounts to a fuel setting.

If the corresponding pressure is zero P1 will be used; if P1 is zero 1000 psi is assumed.

IFUEL may or may not correspond physically to fuel settings

at the hammer. Measurements of combustion pressures at reduced fuel settings are only available for a DELMAG D-30 Hammer. Using the results from this test as a guide proportional pressure reductions were determined for other hammers. Use of the IFUEL value should be restricted to parameter studies. In general, it is advisable to determine first the maximum stroke using IFUEL = 0 or 1. If it is necessary to limit stroke (e.g. stress limitations) then a fixed stroke (IOSTR = -1) analysis can be made for a similar stroke.

It is then, when hammers with variable fuel pumps are used, an easy task to adjust the fuel pressure in the field such that the analyzed stroke is actually obtained.

TDEL Is the combustion delay in seconds. It overrides the value set in the diesel hammer data. If negative (hammers with fuel injection have only positive time delays since here TDEL is the delay after injection) preignition will result. As discussed earlier 0.001 to 0.0025 seconds is reasonable.

STROKE ... For diesel hammers: Is a starting value for the ram stroke in feet if IOSTR = 0. It is the stroke analyzed for IOSTR = 1 or -1.
For air/steam hammers this value overrides the hammer information (both program or user supplied).

EFFICY Overrides the efficiency supplied in the hammer information (applicable to all hammer types).

PSTEAM The actual air/steam pressure used. To be supplied for differential acting hammers only. If not given PSTEAM will be set to P1, i.e. the maximum value.

RWH Overrides the hammer assembly weight supplied in the hammer information. Ignored for open end diesels or single acting air/steam hammers.

4.6 Driving Accessories

The term "Driving Accessories" refers to the capblock, the helmet and a possible pile cushion. There are a variety of such accessories available and standardization is hardly possible. The User's Manual contains data for a few frequently encountered systems.

In the case of no capblock the anvil stiffness will be doubled and an equal stiffness assumed between helmet and anvil. This assumption seems justified in light of the uncertainties involved in metal to metal impact. The data to be specified for the accessories is then:

HM(M+2).....Weight of cap (helmet plus capblock including any pile top adapter) in kips.

STH(M+1) ... Stiffness of capblock in kips per inch.

STH(M+2) ... Stiffness of pile cushion in kips per inch.

In addition to the weight and stiffness values, coefficients of restitution should be specified for all non-elastic materials and/or impact interfaces.

Four values are required:

EANV Coefficient of restitution (C.O.R.) of anvil in the
case of diesels or that of the capblock for air/steam
hammers.
ECAP C.O.R. of capblock (diesel) or assembly (air/steam).
EPT C.O.R. of pile top.
ECUS C.O.R. of pile cushion.

Default (i.e. input not specified) results in all coefficients to be set to 0.85 with the exception that ECUS would be set to 1 if no cushion is present.

The User's Manual contains suggested values. Other recommendations are: for the anvil and for steel pile tops 0.85, for pile tops of concrete 0.7 and timber, 0.5. If a pile cushion is used with ECUS < 1 then greater EPT values should be chosen. Note: actually, EANV is a hammer property and should be stated with the hammer. However, since this value is hardly measurable and can only be judged by comparing analysis results with measurements, some flexibility is given here by providing the opportunity for change.

4.7 Pile

There are several options which govern the input of the pile properties.

The two option parameters to be considered are:

NCROSS ... = 0 means uniform pile
>0 means non-uniform pile

and

IPEL = 0 means automatic generation of pile segment parameters

EP(I) elastic modulus in kips per square inch at XP(I)

WP(I) specific weight in pounds per cubic feet at XP(I)

As a last set of values those of the pile bottom $XP(I) = XPT$ must be given.

The extended input, i.e. the input of STP(I) (pile segment stiffness), PM(I) (pile segment weight) and ALPH(I) (relative segment length values) is described in the User's Manual. Computation of stiffnesses and weights should be done according to Equations 3-28 and 3-29. Note that $RP = WP/G$ with G being the gravitational acceleration (32.2 ft/sec^2 or 9.81 m/sec^2).

Another input allows the specification of maximum tension force, SPLICE(I), in any one pile spring. Note that the pile top has a spring that cannot transmit tension. This means that SPLICE(1) is always set to zero. All other spring tension forces are set to -5000 kips. If at certain elements other values are requested then ISPL should be set to the number of springs involved and for each spring a pair of values:

J, SPLICE(J) ... element number, maximum tension force in kips (tension is negative) be specified.

SPLICE(I) can also be specified as a "slack" value in feet. Slack is a distance which a spring can extend without exhibiting any tension force. If SPLICE(I) is given between -0.5 and 0.0 then it is assumed to indicate a slack. More information regarding this input can be found in the User's Manual.

Finally, the pile material damping parameter IBEDAM (IBD in Section 3.3) should be specified as 1 (1% of critical) for steel, 3 for concrete and 5 for timber.

= 1 means automatic generation of pile segment parameters except that the relative element lengths (ALPH(I)) are to be specified.

= 2 means that all parameters are specified as input including the relative element lengths.

N, the number of pile elements to be analyzed must be given for IPEL > 0. (For IPEL = 0, N is determined for a segment length of approximately 5 feet).

For uniform piles (NCROSS = 0) the following data is sufficient:

XPT the total pile length in feet
AP(1) the pile top cross sectional area in square inches
EP(1) the pile top elastic modulus in kips per square inch
WP(1) the pile top specific weight in pounds per cubic feet.

Material properties are contained in the User's Manual.

For non-uniform piles the cross section and the material values must be completely defined. As it is necessary for the computer to interpolate between neighboring defined cross section values this means that discontinuities (sudden changes) have to be defined with two equal depth values the first of which specifies the quantities above the change, and the second one, those below the change. Examples are given in the User's Manual. The program accepts up to nineteen (19) cross sectional values. These are:

XP(I) depth at which values are specified in feet
AP(I) cross sectional area in square inches at XP(I)

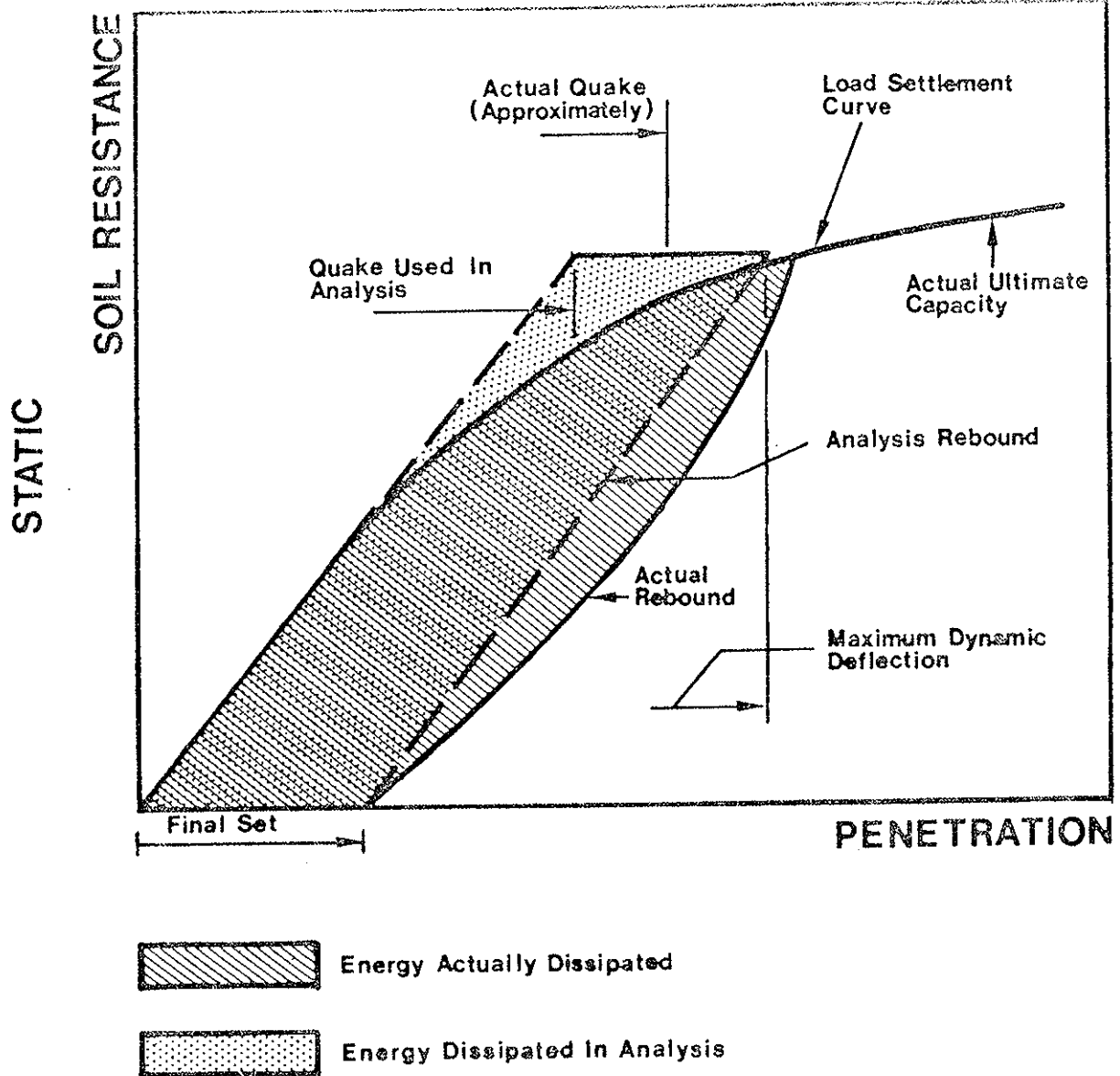


FIGURE 4-1: REAL LOAD DEFLECTION CURVE AND ITS MODEL. IN A CORRECT ANALYSIS QUAKES SMALLER THAN REAL MUST BE USED.

4.8 Soil

Soil data consists of:

quakes,
damping values,
ultimate resistance (optional),
coefficient of restitution of soil and resistance distribution.

Required values are:

QS(1) quake at skin in inches (default: 0.1 inches)

QS(N+1) .. quake at pile bottom (default: 0.1 inches)

Quake values were proposed in the literature between 0.1 and 0.5 inches (7). They are usually based on static load test evaluations. However, it should be noted that the assumed elasto-plastic soil resistance law is only an approximation and should allow the pile to obtain a permanent set under a realistic energy consumption (see Figure 4-1 for an example of a reasonable approximation). The questions arising from these approximations together with those of the damping model are the most serious ones with regard to wave equation applications.

Because of these questions it seems unreasonable to use the static quake values as proposed by Coyle et. al. (7). It is more likely that values between 0.1 and 0.15 inches give satisfactory results. Note that values that are too small produce a soil stiffness that can cause numerical problems.

Damping values impose a much greater problem than quake values because they affect the blow count result to a much larger degree. Recommendations

are given in the User's Manual. Smith's damping values were relatively well proven during several years of use. It is for this reason that Smith's damping should be used whenever no experience with Case Damping exists.

The required damping values are:

SJ(1) skin damping constant in s/ft for Smith
(ISMITH = 1) damping and dimensionless for Case
damping.

SJ(N+1) .. toe damping constant with dimensions as before.

As for pile quantities, all soil parameters QS(I), SJ(I) and SU(I) can be specified for each segment in order to override the automatic conditions. The values are read when the ITYS value is negative.

The total ultimate static resistance value can be specified in three ways:

- (a) RULT > 0 in tons leads to the analysis for the one RULT value specified.
- (b) RULT = 0 (or blank) causes the computer to determine ultimate resistance values based on the pile impedance (EA/c) and an assumed fraction of impact velocity. There will be at most ten (10) analyses performed. If a blow count has exceeded 1200 blows/foot (100 blows/inch - 40 blows/cm) then the analysis will be halted and the results printed.
- (c) RULT < 0 will cause a set of ultimate resistance values (at most 10) to be read and analyzed. These values should be given in an increasing order as the starting values of the

stroke are based on this assumption. Also, the termination condition of (b) hold, again assuming an increasing order of the RULT values.

ESOIL ... is the coefficient of restitution of the soil (default 1.0). If the stroke does not converge for reason of too little energy consumption by the soil (a condition that becomes an unfortunate reality in some cases) then a lower value might be tried. However, values lower than one-half (0.5) should not be taken because of numerical stability.

The distribution of resistance foces, SU(I) can either be taken from a list of ten (10) types which are built into the program (ITYS) or, if this value is 0 or blank, it should be specified as a series of depth and relative magnitude values:

XP(I), DIS(I) ... I-th depth in feet and I-th magnitude (non-dimensional). As in the case of the pile description, discontinuities in the distribution have to be described with two identical XP(I) values. The last XP(I) value must be equal to the pile length, the first one equal to zero.

The proportion of skin to total resistance has to be specified by IPERCS ... the percentage of skin friction of total RULT. This value can also be specified as a negative number. In this case the skin friction force will be determined from the first RULT value (-IPERCS (RULT)) and will be kept at the same level for further RULT analysis. This means that only a gain of toe bearing is assumed.

4.9 Other Program Options

Other input parameters not yet described must be specified in order to

obtain the desired

- (a) Output type and quantity
- (b) Analysis type

Output type can be obtained in printed and plotted form. Basically, the higher the IOU (card 2.000) parameter is specified, the more output is obtained. The IOU = 0 option causes minimum output which is recommended if more than one RULT value is analyzed (typical production run). IOU > 9 causes plots to be produced. The use of these options as well as the more extensive print is discussed in detail in the User's Manual.

It should be noted that the program performs several analyses for one RULT value. For this reason all output quantities have to be temporarily stored until it is known that the stroke or maximum combustion pressure has converged. In order to keep the program small enough for implementation on a variety of computers, the output must be limited to certain types (forces or velocities etc.) and to a maximum number of values (number of pile segments, number of time increments).

The magnitude of the time increments has to be chosen less than critical. This critical time increment is the minimum of:

$$DTCR = \sqrt{PM(I)/STP(I)} \quad (4-1)$$

for all I, or it is the time that the stress wave needs to travel through the shortest element IM

$$DTCR = ALPH(IM) (XPT)/c \quad (4-2)$$

(if only one material is considered) and the actual time increment either for hammer or pile has to be

$$DT \leq DTCR \quad (4-3)$$

In order to accomplish this a percentage value IPHI (%) can be specified which is defined as

$$IPHI = (DTCR/DT)100 \quad (4-4)$$

This IPHI should be always greater or equal to 100. Default or if IPHI was accidentally set to less than 100 results in IPHI = 140 for diesel and 160 for air/steam hammers. It is reasonable to specify IPHI as large as 200. It should be mentioned here that the program involves checks on DTCR which go beyond Equations 4-1 or 4-2. These checks include the magnitude of damping and static soil resistance.

CHAPTER 5

PROGRAM FLOW

Only a small part of the program is actually a wave equation program. It consists of approximately one-third each input and output coding, one-sixth dynamic analysis and the other sixth, in the case of diesel hammers, of "pseudo-dynamic" and simplified dynamic analyses and general control.

By pseudo-dynamic analysis is meant that the total system is not strictly analyzed. This can be done for the rebound portion once the ram has risen a sufficient distance from the impact block. A simplified dynamic analysis is also used for the precompression period.

The following steps are performed by the program for the open end hammer, standard run,(i.e. fixed fuel, variable stroke).

- (a) Read input information
- (b) Assemble hammer data
- (c) Determine pile segment parameters
- (d) Determine soil model parameters
- (e) Find stroke (either input or assumed)
- (f) Determine ram velocity at exhaust ports
- (g) Find initial values just before impact using a simplified dynamic analysis.
- (h) Perform a wave analysis until pile rebounds and ram has risen sufficiently.
- (i) Find velocity (and therefore stroke) at exhaust ports

- (j) For a stroke which is less than 10% different from the assumed one repeat process (h) using the new stroke and modified initial values; for greater differences go to (g).
If the stroke was within 5% of the assumed value, print and plot the required output and continue with (h).
- (k) If a new ultimate resistance value is to be analyzed, determine new stroke based on previous one and continue at (f).

As an alternative (fixed stroke, variable fuel) step (j) is modified in the following manner: For a rebound stroke that is more than 5% (3% for closed end hammer) different from the assumed stroke the maximum combustion pressure value is changed and the process is continued at step (h). Of course, in step (k) no new stroke is assumed. A block diagram indicating the program flow is shown in Figure 5-1.

The steps taken for the closed end hammer (standard run) differ somewhat from those just discussed. Because of the limitations on the stroke in a closed end hammer and since the throttle setting is reduced when uplift occurs, the process is directed such that, if no other stroke is specified, the maximum stroke (Equations 15 or 20) is used for a first analysis together with the maximum fuel setting. If the soil resistance is relatively small, then a rebound stroke will result. In this case iteration is performed on stroke. For higher rebound strokes uplift would occur and, therefore, the fuel setting has to be reduced. Since the question of fuel con-

Block Diagram Of Program Flow

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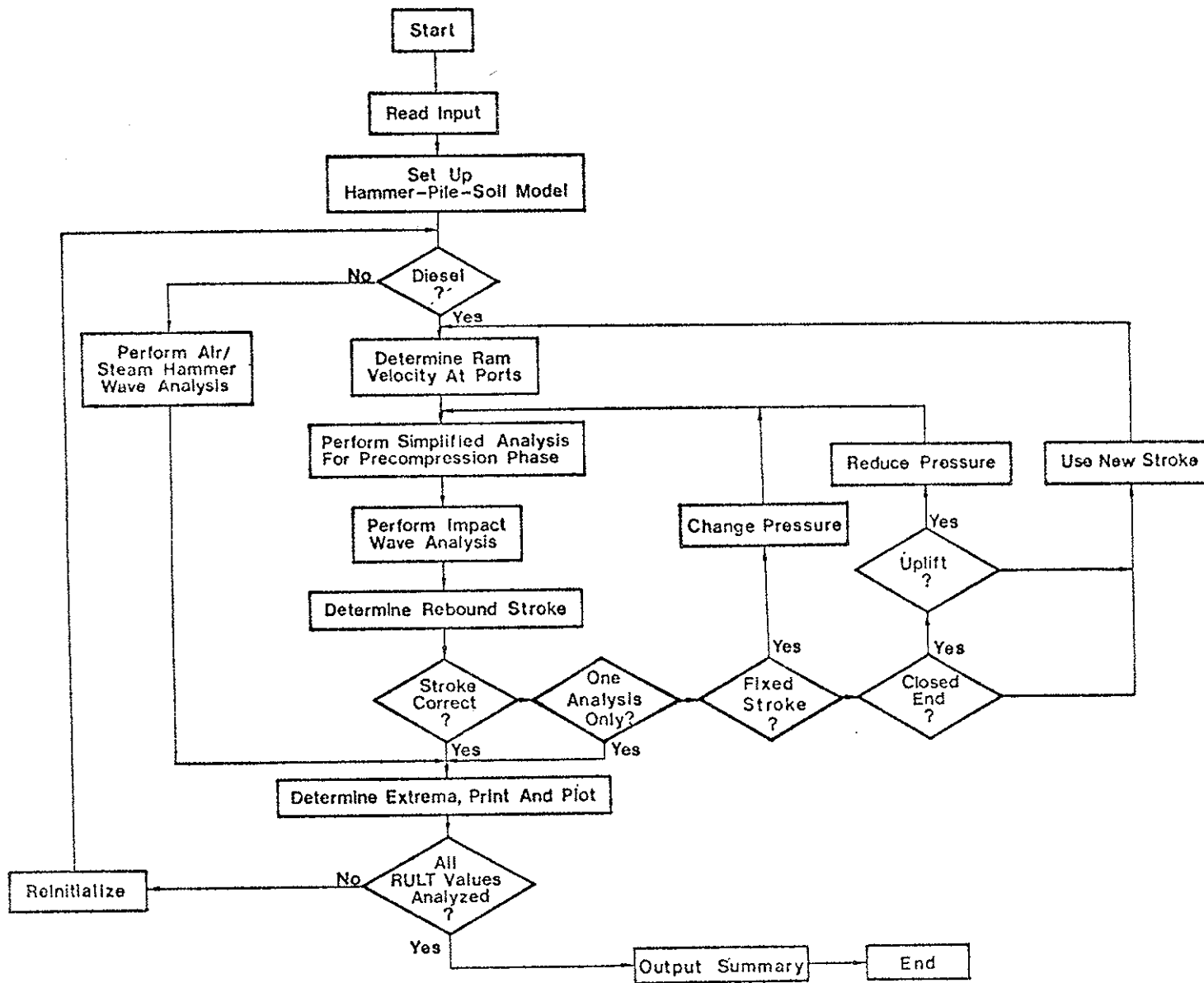


FIGURE 5-1: BLOCK DIAGRAM OF PROGRAM FLOW

sumption is not of importance and since the injection pumps usually employ continuous rather than discrete settings, an arbitrary reduction of 10% of maximum combustion pressure is used. This process is also repeated iteratively.

For closed end hammers (standard run) the program control always causes a decrease in hammer energy supplied and an increase in blow count. In one case this occurs due to a reduced effectiveness of the hammer blow (smaller impact velocity) and in the other case due to a reduced combustion pressure. As the latter effect is probably relatively small, the step size of 10% in pressure reduction seems to be sufficiently small.

In the open end program, iterating on stroke, the initial value for stroke is either as given by the user or taken as five feet. If a second RULT (ultimate resistance) value is used (greater than the previous one), then a stroke 20% higher than the previous result is tried. For a third or further RULT value, an extrapolation over the earlier strokes is employed. If the iterations are done on combustion pressure then the pressure from the previous analysis is starting value for the new RULT value.

A similar process is used for the closed end program except in a case where the previous result was the maximum stroke at a reduced fuel setting. In this case the maximum stroke is again assumed together with a throttle setting that is increased by one step. It should be noted that the program thereafter does not increase fuel settings. Thus, RULT values specified in decreasing order could lead to erroneous results.

The program flow of the air/steam hammer analysis is straight forward. Initial values need not to be determined and the stroke is assumed fixed. A greater energy output for higher soil resistance values has been observed, but no attempt was made here to include the effect of variable stroke.

CHAPTER 6

PROGRAM PERFORMANCE

6.1 Introduction

Both the program and the hammer data were extensively tested against measurements. Measurements consisted of pile-top force and velocity and, in two cases, combustion-chamber pressure records. In addition, blow count and stroke was recorded in many cases. The program testing often proved to be difficult since all of the relevant information was not always available. The information required consisted of three different groups.

(1) Driving system data (cap weight, capblock and cushion stiffness plus all four coefficients of restitution). While the cap weight was usually accurately known, stiffness and coefficient of restitution was either guessed or chosen from a large range of possible values.

(2) Thermodynamic data (combustion delay, combustion pressure, expansion coefficient). The combustion delay was fixed at two milli-seconds for all hammers with regular fuel injection. It was determined for atomized injection by parameter studies. Combustion pressures were measured in some cases and determined from stroke and/or force measurements in other cases. In order to do this properly, the expansion coefficient was fixed at 1.3.

(3) Soil data (skin and toe quake, skin and toe damping and the distribution of the resistance forces). The quakes were mostly chosen as 0.1 inches. However, there were a few exceptional cases where larger values had to be used to achieve satisfactory agreement between computed and measured

values. Damping values were difficult to determine in that they directly affected the stroke and, therefore, the pile top stress and velocity at impact and the blow count. The distribution of the resistance forces affected the shape of force and velocity curves.

Counting the resistance distribution as only one unknown (although it involves all elements) the above list indicates that often as many as 14 different parameters had to be determined for each test case.

Parameter studies performed by the TTI researchers (6) were basically aimed at matching the predicted with the observed blow count. In the current study, however, the attempt was also made to match force, velocity, pressure and stroke. The parameters given in the literature were often found to be insufficient. The current results shed some light on the quality of the predictions of pile force as well as blow count that may be achieved. Since the information regarding hammer performance in the field was often limited, the answers obtained give an idea of the accuracy which can be expected when the program is used. Thus, the comparisons discussed below can be regarded as an assessment of expected program performance. Many of the problems which have been reported for wave equation programs may be partially the result of excessive expectations.

6.2 Data Selection

A vast amount of dynamic and static data was gathered during the course of the Case Western Reserve University piling research projects. In addition, the consulting activity of the authors and the work of other agencies which

also use the Case Method of pile testing have contributed to the data that was available for pile testing.

The results of 16 different pile analyses are presented in this report. The attempt was made to cover a large number of different hammers and pile types. Names of test piles, references and other basic information are given in Table 1.

There are a relatively large number of DELMAG hammers among the hammer types tested. The reason is that a substantial amount of testing, including combustion pressure and stroke measurements, was conducted on these hammers. Stroke is an especially important quantity in this work and it must be emphasized in program testing.

Only one concrete pile is represented in the data. No timber pile data was included. Both concrete and timber introduce additional uncertainties into the evaluation of the program, mainly because of the unknown conditions at the pile top (pile cushion of concrete piles and pile top quality of timber piles).

6.3 Representation of Results

Figures 6-1 through 6-16 contain comparisons of computed with measured forces, velocities and combustion pressures all as a function of time. Velocity comparisons of this kind are certainly a first in Wave Equation test runs, and the large amount of force vs. time curve comparisons is unequalled. For plotting, the curves were shifted in time such that the impact time was in agreement. In addition, the measured forces were shifted vertically such

TABLE 1: DESCRIPTION OF TEST DATA

File No.	Name	Source	Type	Length feet	Hammer	Type
1	FEC 72	C ¹	HBP	40	Delmag D12	OED
2	FEC 72	C	HBP	75	Delmag D12	OED
3	FEC 75	C	HBP	40	Delmag D12	OED
4	Purdue	(10) ²	HBP	50	Delmag D12	OED
5	DTP 3	(11)	18x18 PC	60	Delmag D22	OED
6	FEC 71	C	HBP	70	Delmag D30	OED
7	Bismark	C	HBP	160	Kobe K22	OED
8	GRTPl	C	HBP	60	Kobe K22	OED
9	K25 VP	(12)	HBP	30	Kobe K25	OED
10	Georgia	(15)	Pipe	40	MKT DE30	OED
11	B-N	S ³	HBP	119	MKT DA35B	CED
12	CR 4	(13)	Pipe	90	LB 440	CED
13	DTP 33	C	Pipe	41	LB 660	CED
14	Phila 78	(14)	Pipe	43	Vulcan 01	A/S
15	VO8VP	(12)	HBP	30	Vulcan 08	A/S
16	VO8VP	(12)	HBP	30	Vulcan 08	A/S
17	LaV016	S ⁴	Pipe	200	Vulcan 016	A/S

¹Records from author's consulting practice.

²Numbers in parentheses pertain to references at end of text.

³Special records, were obtained from New York Department of Transportation

⁴Special records, were obtained from Soil Exploration Company.

that the computed and measured precompression forces agreed. This shift was necessary since the precompression force was often subtracted from the measured force curves because of its static nature.

Almost all figures contain two time scales, milliseconds and L/c units. The L/c units are often helpful in recognizing characteristic effects in the stress waves. One figure (6-16) is also included which shows the automatically plotted three dimensional representation of force vs. pile length and time. These plots are instructive where wave propagation considerations are concerned but of less use in the program testing effort.

The figures always show that portion of the record that was analyzed by the accurate analysis portion (excluding the simplified precompression and expansion phases). This produces plots which are spread over time such that the higher frequency components of the curves become apparent.

6.4 Results

Table 2 contains both observed and computed values of the program results, RULT (total static pile bearing capacity) in tons, blow count in blows per foot and stroke (or an equivalent quantity for closed end hammers). In fact, RULT was determined using either the Case Method (1), the CAse Pile Wave Analysis Program (1), or a static load test. Unfortunately, the static load test was usually not performed when stroke measurements were taken. However, it is felt that so many test cases were solved that sufficient confidence in the program is obtained.

TABLE 2: COMPARISON OF COMPUTED WITH OBSERVED
QUANTITIES FOR TESTED DATA

File No.	Bearing Capacity tons	Measured		Predicted	
		Blow Count bl/ft	Stroke ft	Blow Count bl/ft	Stroke ft
1	16 ^C	6	4.0 [±] .2	9	4.0
2	70 ^A	27	6.0 [±] .3	28	5.6
3	60 ^C	24	5.4 [±] .1	26	5.4
4	83 ^L	40	5.5 [±] .1	36	5.6
5	10 ^A	12	4.0 [±] .1	8	4.1
6	125 ^A	N/A	N/A	21	5.5
7	200 ^L	500	7 (1 plug)	102(150)	6.6(5.5 ^P)
8	120 ^A	41	6 (1 plug)	37	5.5 ^P
9	300 ^C	120	8.0 [±] .3	151	8.0
10	130 ^L	107	N/A	114	8.7
11	31 ^A	25	N/A	19	4.0
12	180 ^L	R*	Maximum	R	Maximum
13	350 ^A	R	Maximum	R	Maximum
14	75 ^L	43	Normal	44	--
15	8 ^L	6	Normal	7	--
16	260 ^C	184	Normal	R	--
17	275 ^A	55	Normal	50	--

^PUsing Preignition and reduced fuel pressure

^CCase Method

^ACAPWAP Analysis

^LLoad Test

*Refusal ... no noticeable set

File No. 1

This pile was driven for a hammer performance test. Combustion chamber pressures and stroke were measured in addition to both pile top acceleration and force. Using the Case Method static capacity prediction as the RULT value a WEAP run was made. The resulting pile top force and combustion pressure are shown together with the measured quantities in Figure 6-1. Maximum force is underpredicted by about 8%. The curve behavior follows the measured curve closely except that it is somewhat smoother. It should be mentioned that the record portion after time $2L/c$ after impact is mainly governed by the soil model and is only, to a smaller degree, influenced by the hammer model.

The pressure curves show basically the same behavior except that the measured one is somewhat higher in the beginning and lower later on. Note, however, that the plot is a pressure vs. time plot and that minute changes in ram position appreciably alter the pressure behavior. Since the stroke was determined correctly one can consider the overall pressure behavior to be sufficiently accurate. The force effect on the pile from pressure differences is rather small (100 psi pressure correspond to about 10 kips force).

File No. 2

This pile was tested under similar conditions to Pile No. 1. The difference was that a heavy plate was added at the toe. This additional mass accounts for the third force peak (Figure 6-2). This force curve was very well predicted by the program. (Better than 5% agreement for the maximum).

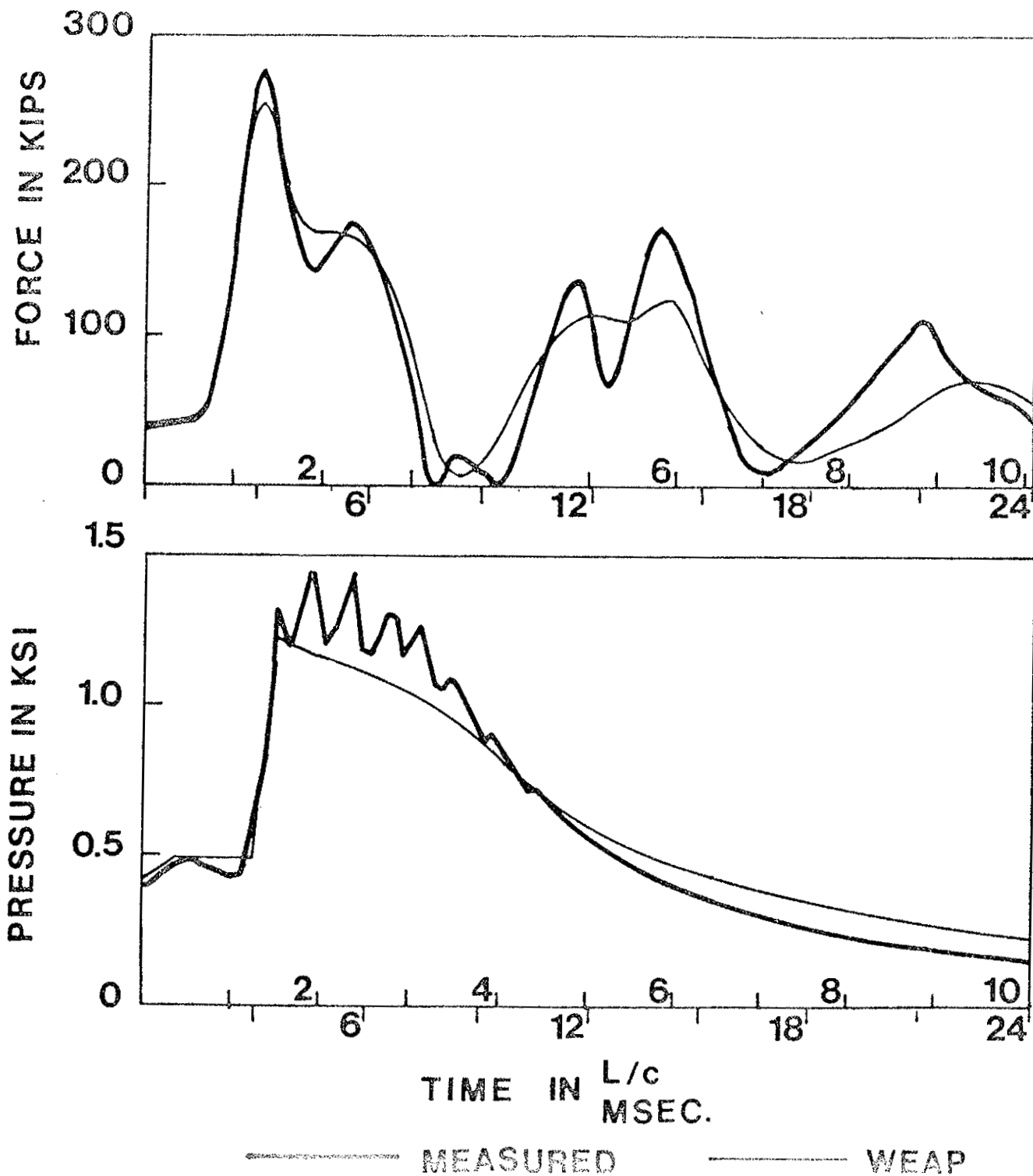


FIGURE 6-1: COMPARISON OF PREDICTED WITH MEASURED PILE TOP FORCES AND COMBUSTION PRESSURES FOR PILE NO.1.

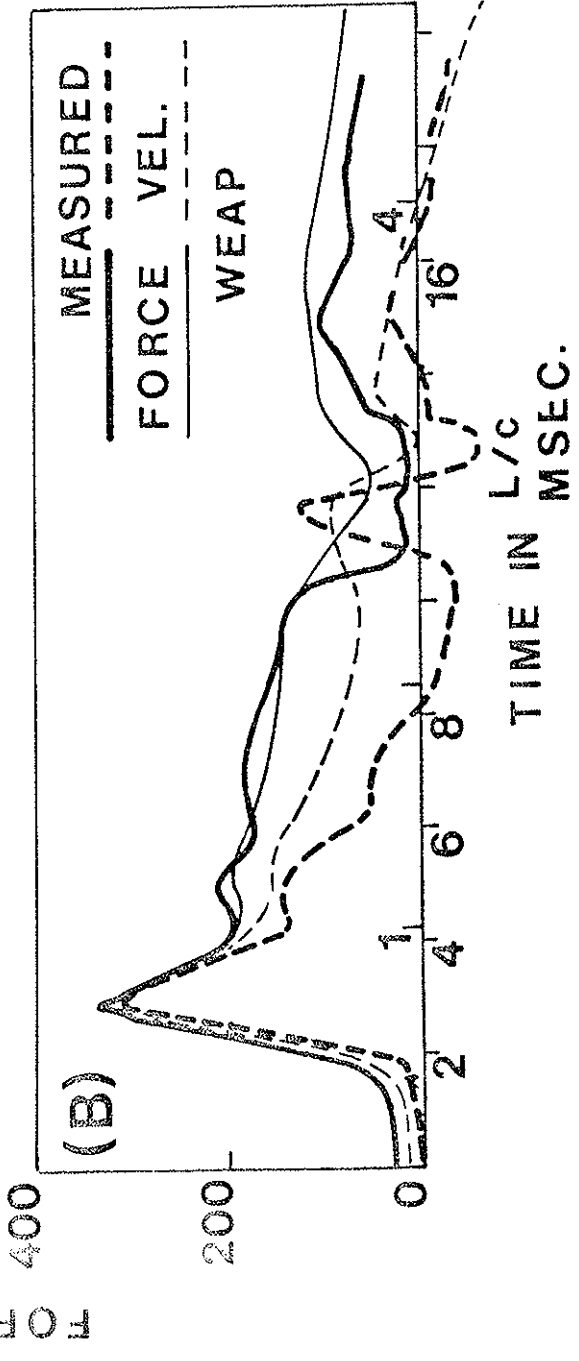
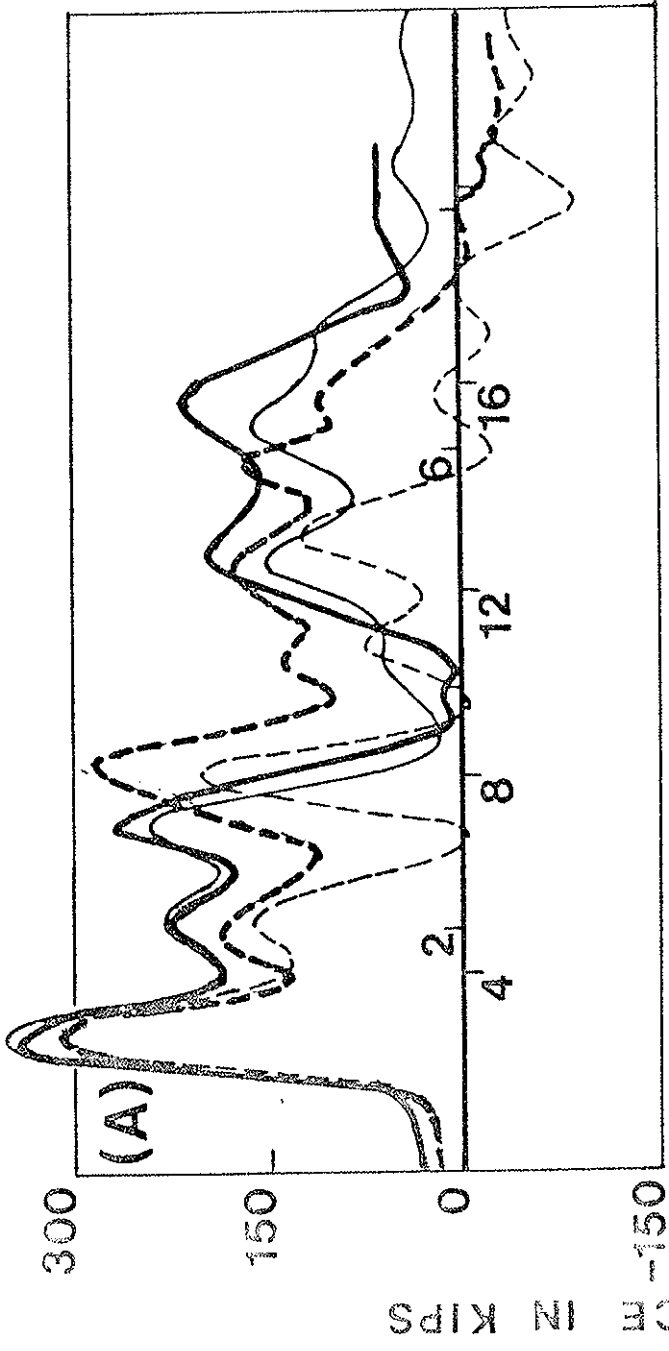


FIGURE 6.-2: COMPARISON OF PREDICTED WITH MEASURED PILE TOP FORCE AND VELOCITY FOR (A) PILE NO. 2 AND (B) PILE NO. 3

The velocity curve (note that velocity curves are plotted after multiplication by the proportionality factor $EP(AP)/c$) however, though similar in character, deviates before time $2L/c$ after impact. This deviation is not considered critical since the blow count was determined rather accurately. Also, the calculated stroke was found to be in good agreement with the measured one.

Pile No. 3

This pile was an extension of Pile No. 1. Again force, (Figure 6-2b) stroke and blow count agreement is very good. It is interesting to note that the force shows a rather smooth behavior while the velocity displays a peak at the time of the wave return (10 milliseconds). The maximum force predicted showed an agreement within 5%.

Pile No. 4

Another well-controlled hammer test was performed on the campus of Purdue University on the occasion of the ASCE Specialty Conference in 1972 (see Reference 10). The force and velocity match of Figure 6-3 is quite good (especially at the time of wave return). The maximum force was over-predicted by 11% and the blow count was underpredicted (36 vs. 40 measured). It may be that a force transducer reduced the force peak to some degree. This transducer was inserted between hammer and pile top and utilized steel plates at top and bottom for attachment. The weight of these plates was not modeled and the additional contact area was not considered in the analysis.

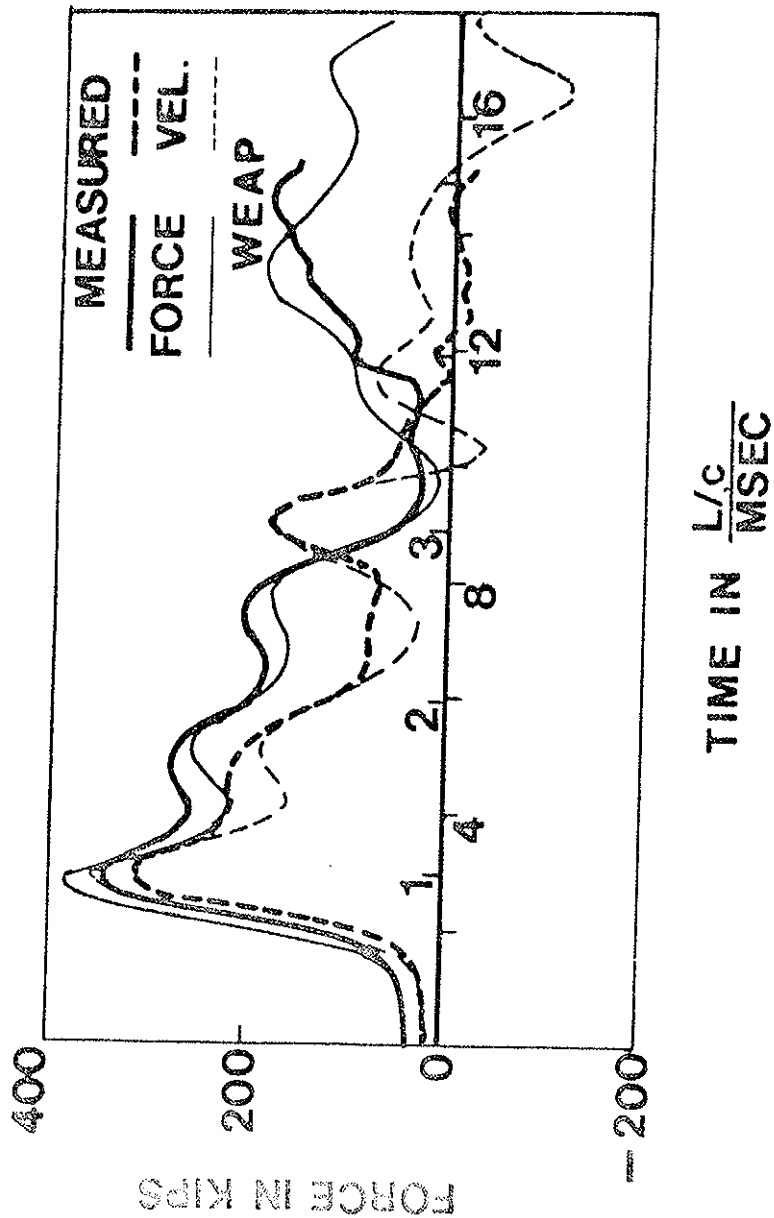


FIGURE 6-3: FORCE AND VELOCITY MATCH FOR PILE NO. 4
(PURDUE)

Under these circumstances the results cannot be expected to be extremely accurate.

Pile No. 5

This pile was of concrete and was tested within a special research project of Case Western Reserve University (11). In order to evaluate whether or not the "easy driving case" can be modeled correctly, an early record was selected with a resistance approximately equal the hammer plus pile weight. This case represents the worst condition regarding tensile stresses. Figure 6-4 shows a very good agreement for the force (8%) considering the uncertainties of cushion properties. The measured velocity shows a higher peak at the time of the wave return which is, perhaps, due to an improper damping assumption. However, for soil resistance values as low as the one used here, a much better agreement seems to be a matter of luck.

Pile No. 6

Another special hammer performance test, this time a DELMAG D-30 hammer, was used for comparison. As can be seen in Figure 6-5 all quantities agree very well, with a maximum force difference that is negligibly small. The computed pressures are somewhat low but for the reasons given for Pile No. 1. These differences are of relatively little concern.

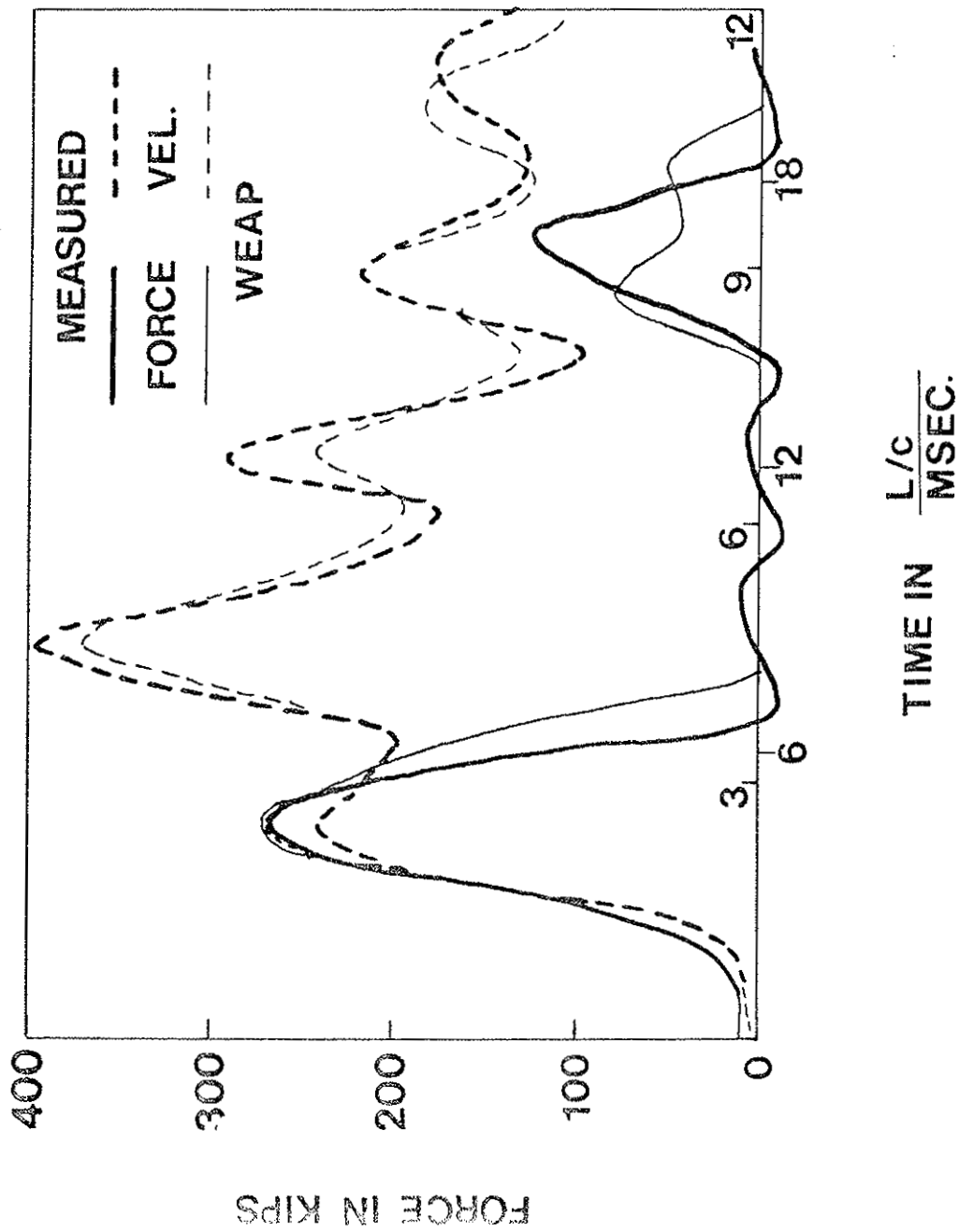


FIGURE 6-4: FORCE AND VELOCITY MATCH FOR PILE NO. 5
(MIAMI DTP 3)

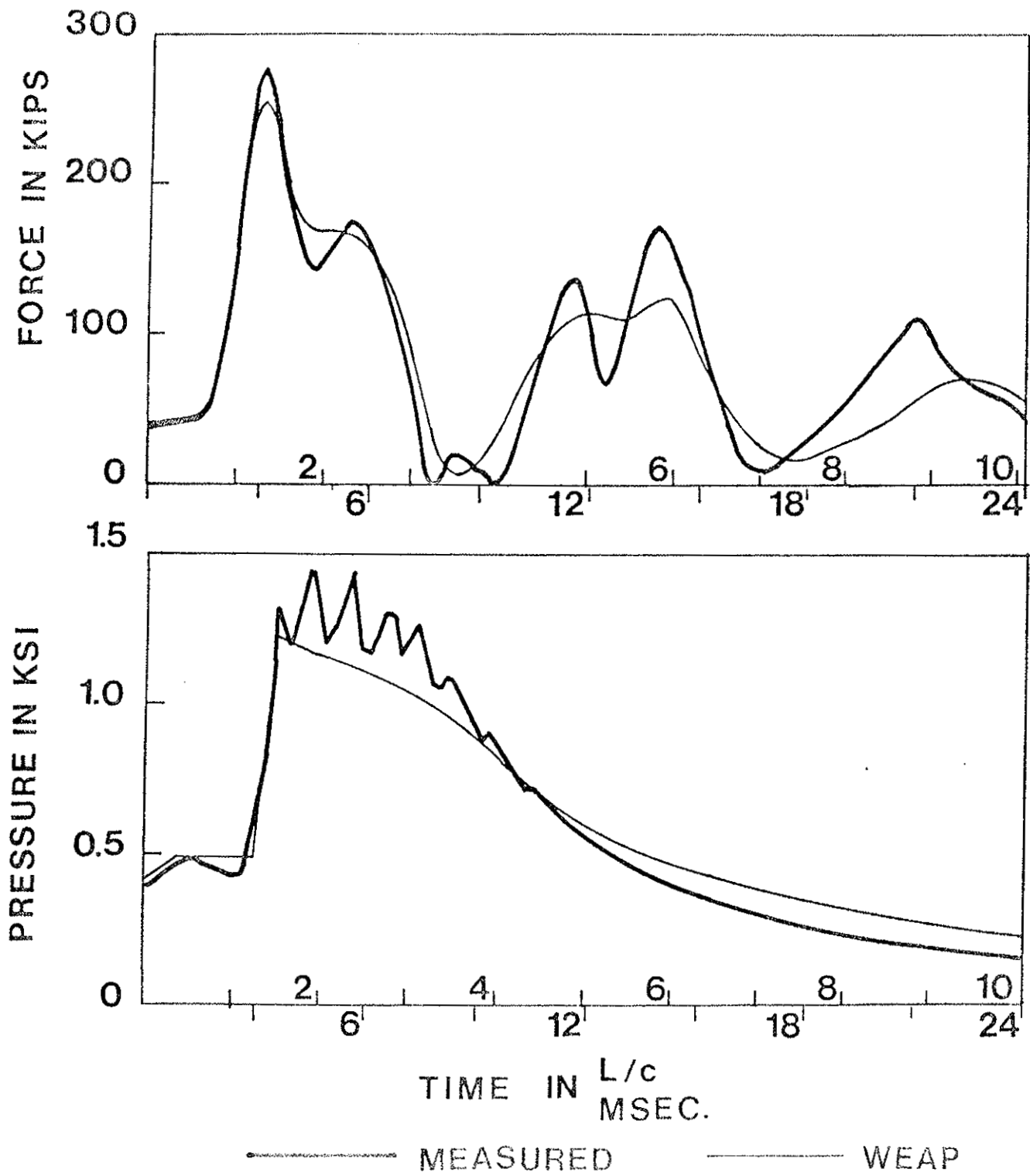


FIGURE 6-5: FORCE, VELOCITY AND PRESSURE MATCH FOR PILE NO. 6

Pile No. 7

This case was selected for two reasons. First, it was a pile driven by a Kobe K22 hammer and second, an exhaust plug was installed into the hammer in order to increase the stroke. The plug produces slower exhausting and, therefore, adds to the pressure in the cylinder after the ram has cleared the ports.

The results shown in Figure 6-6a were produced by a "regular" analysis, i.e. assuming normal hammer performance. It can be seen that the stroke was slightly underpredicted while the force was too high by 18%. Also, the blow count was much smaller than recorded (102 vs. approximately 500). It can also be observed that the measured force increased during impact at a much lower rate.

It was concluded that the hammer both preignited and had a reduced combustion pressure due to poor scavenging. Reanalyzing with a time delay of -0.001 (1 millisecond preignition) and reduced fuel setting (IFUEL = 3) produced the force match of Figure 6-6b. This match is vastly improved and so is the blow count. Since the condition of a plug in the ports cannot be modeled by the program, it is not surprising that the stroke was now smaller than observed (5.5 feet). It can be argued that the effective stroke was indeed about 5.5 feet which indicates a hammer efficiency of 79%.

It is interesting to note that a second plug increased the actual stroke by another 1.5 feet.

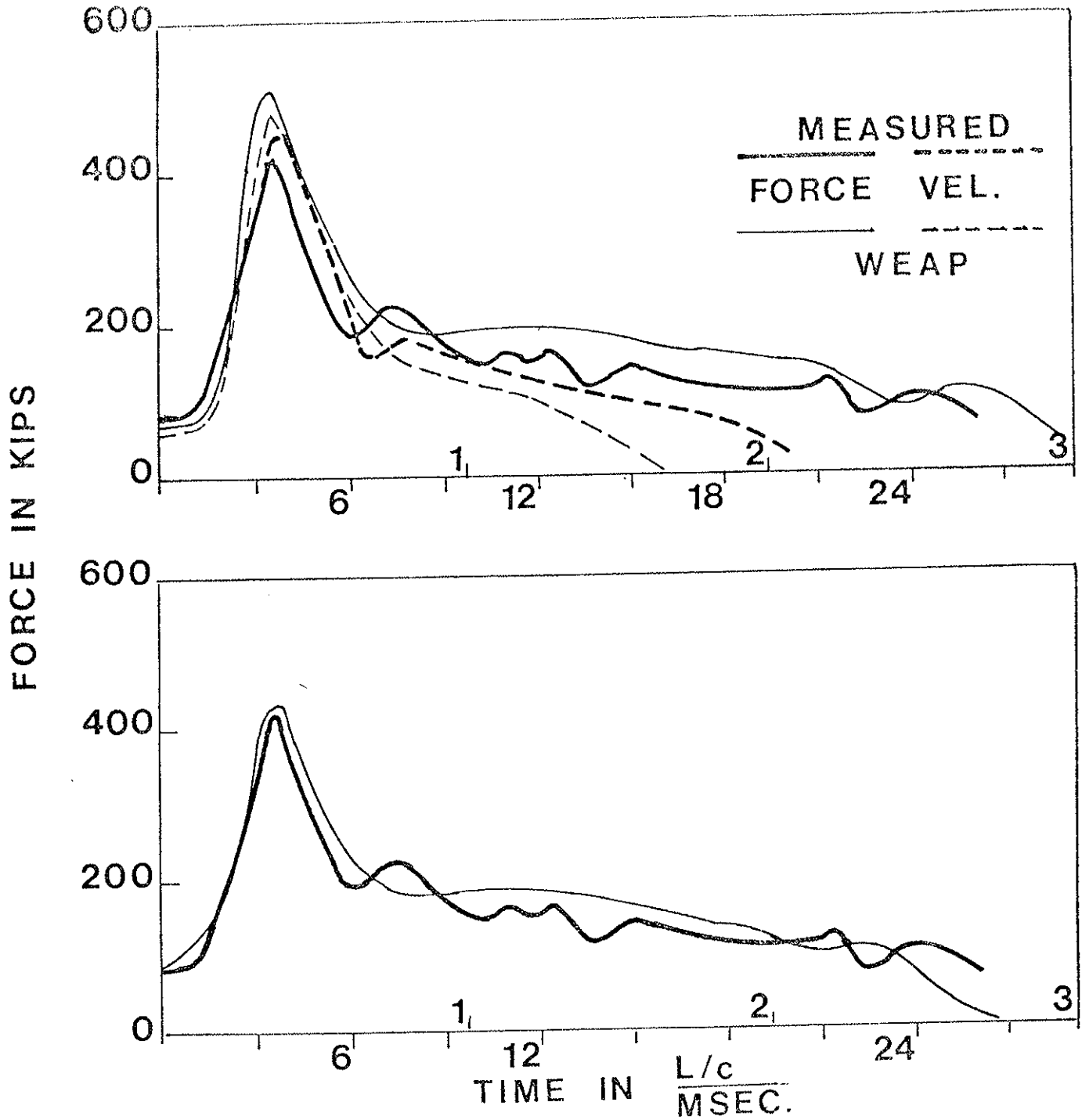


FIGURE 6-6: FORCE AND VELOCITY MATCH FOR PILE NO. 7
 (A) NORMAL PROGRAM PERFORMANCE, (B)
 USING PREIGNITION AND REDUCED FUEL SETTING

Pile No. 8

The data on this pile was used for testing to check a second time on the performance of the K22 with one exhaust port plugged. Using practically the same assumptions as for Pile No. 6 the match of Figure 6-7 was obtained.

The match shows a 9% overprediction in force and a deviation in behavior after the wave returns. Note that the velocity did correlate very well at impact. The later deviation in force and velocity is probably due to an incorrect soil quake (the full resistance acts too early). However, in light of the assumptions necessary to model this hammer, the agreement can be considered sufficient.

Pile No. 9

The match of force and velocity shown in Figure 6-8 was obtained for an H pile driven by a Kobe K25 hammer to rock. Agreement is very good considering that yielding occurred and the pile actually buckled at the pile top.

The measured forces appear to be higher by 17%. However, since strains in the pile were measured, these forces were actually lower and should not have been computed by using a constant elastic modulus throughout the record. The program properly determined that yielding occurred. Blow count and stroke results are very good.

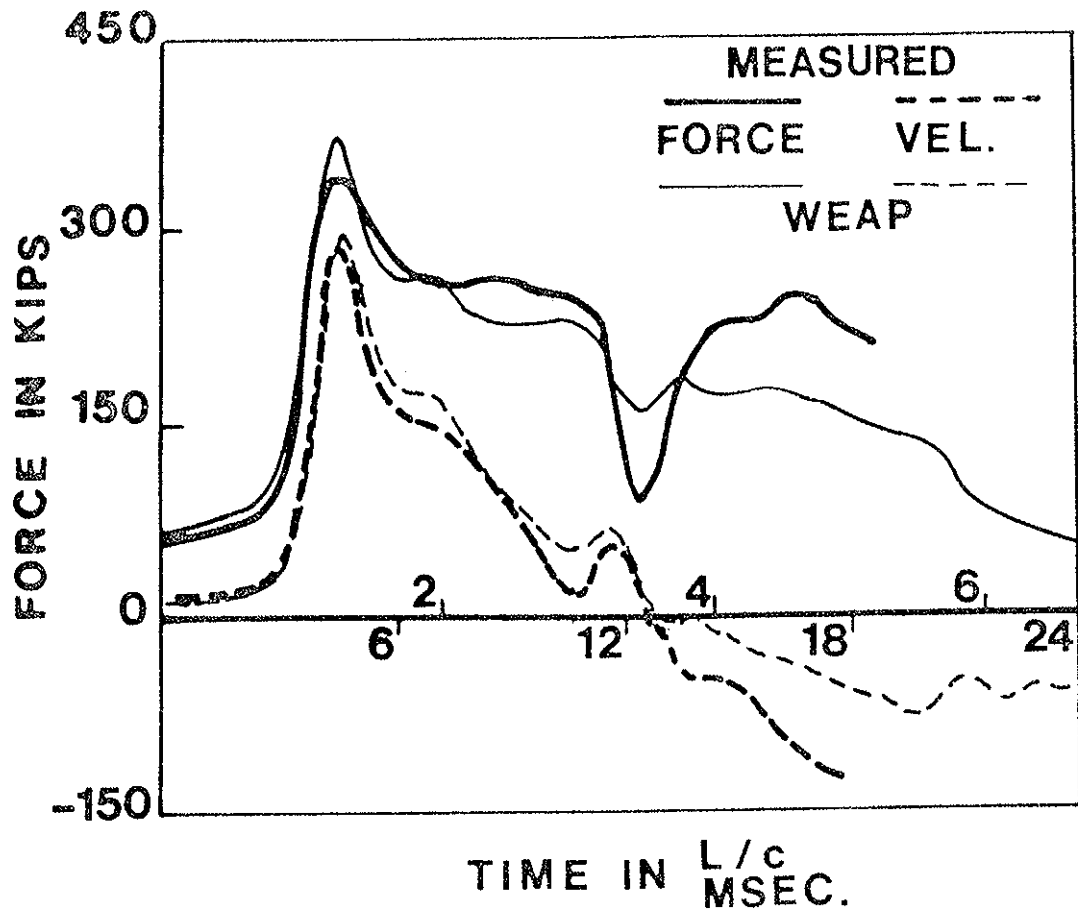


FIGURE 6-7: FORCE AND VELOCITY MATCH FOR PILE NO. 8
 USING PREIGNITION AND REDUCED FUEL SETTING

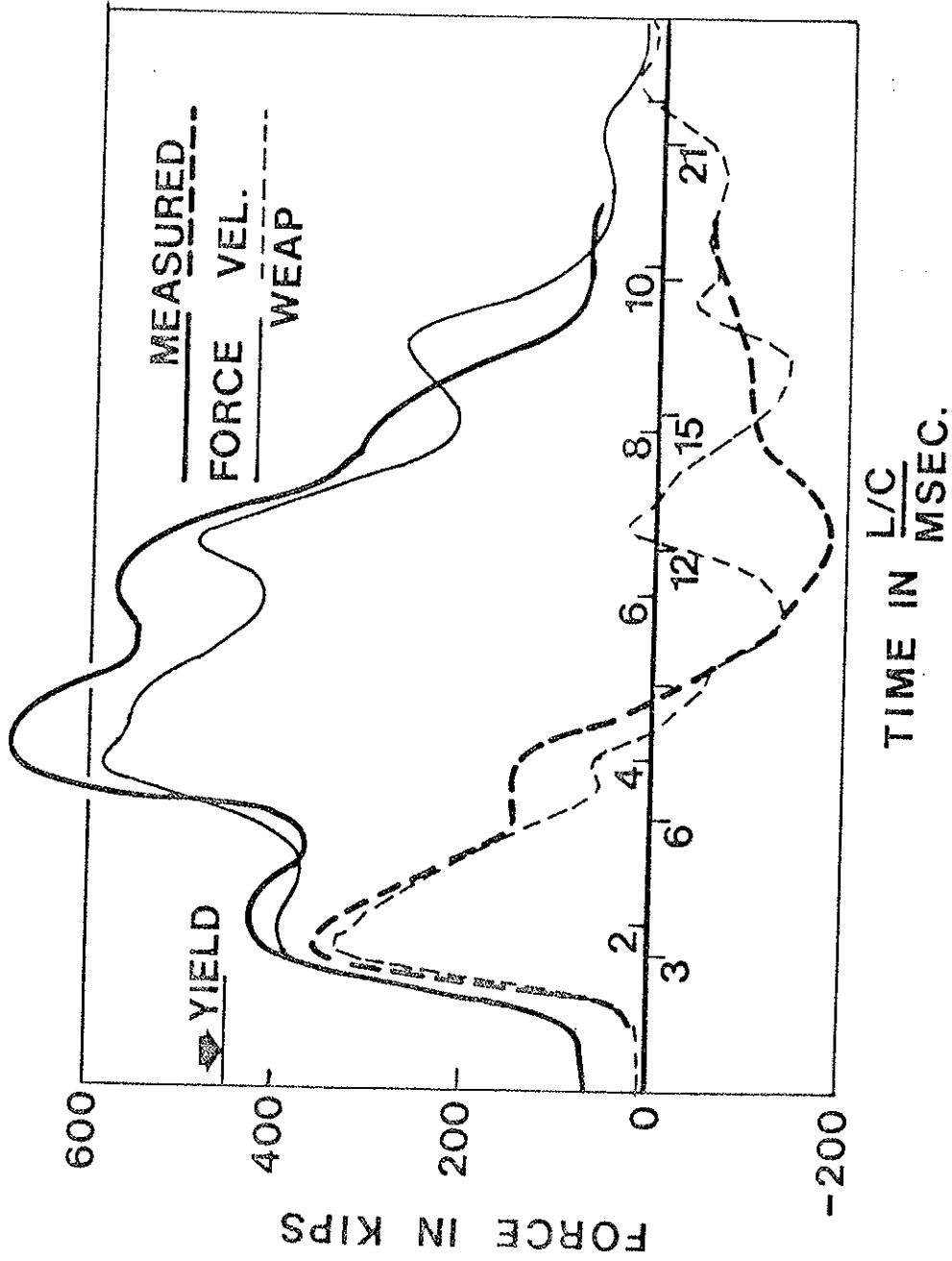


FIGURE 6-8: FORCE AND VELOCITY MATCH FOR PILE NO. 9

File No. 10

This pile was driven and tested as a part of the Case Western Reserve University research project (15). It was selected for analysis because it represented an open end MKT hammer (DE30). Although stroke measurements were not undertaken, it is felt that the results indicate a good agreement (blow count 114 predicted vs. 130 measured).

Figure 6-9 shows the match. The maximum force value was determined very accurately. The deviation of force after the impact peak must be due to some resistance near the top since it does not have an equivalent in the velocity curve.

The velocity match is poor at the time of wave return in a manner similarly observed for earlier piles (e.g. No's 3 and 4). This effect is not fully understood and strangely enough, it only occurs if the force valley at time $2L/c$ after impact is properly matched (Figure 6-6). The authors believe that this high velocity return is due to an improper model of the soil below the pile tip. This problem of a proper soil model has not been solved and should receive attention in further research activities.

File No. 11

The measurements on this pile were obtained by the New York Department of Transportation and are the only ones available for a closed end MKT hammer. The match shown in Figure 6-10 is good with regard to both impact and the time of wave return. Between these two times, however, the predicted force and velocity are both higher than measured. This error can

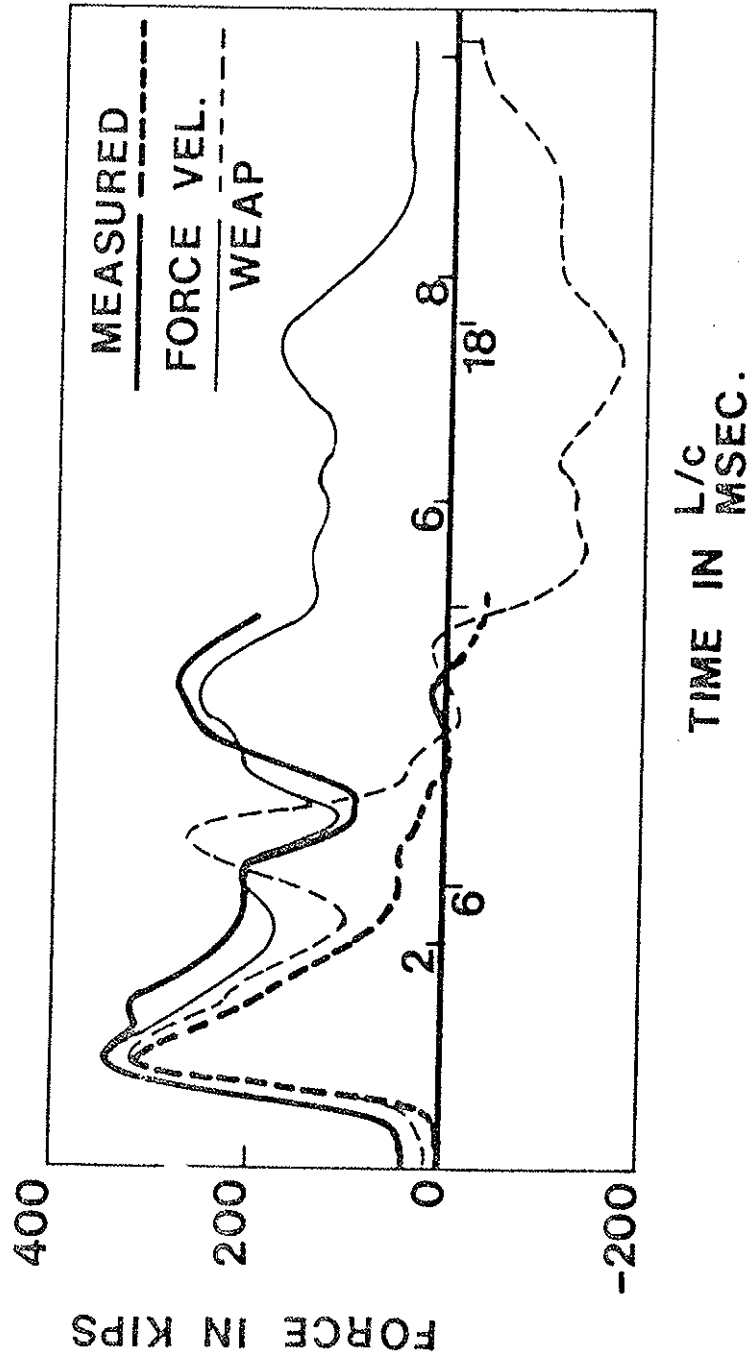


FIGURE 6-9: FORCE AND VELOCITY MATCH FOR PILE NO. 10 (GEORGIA)

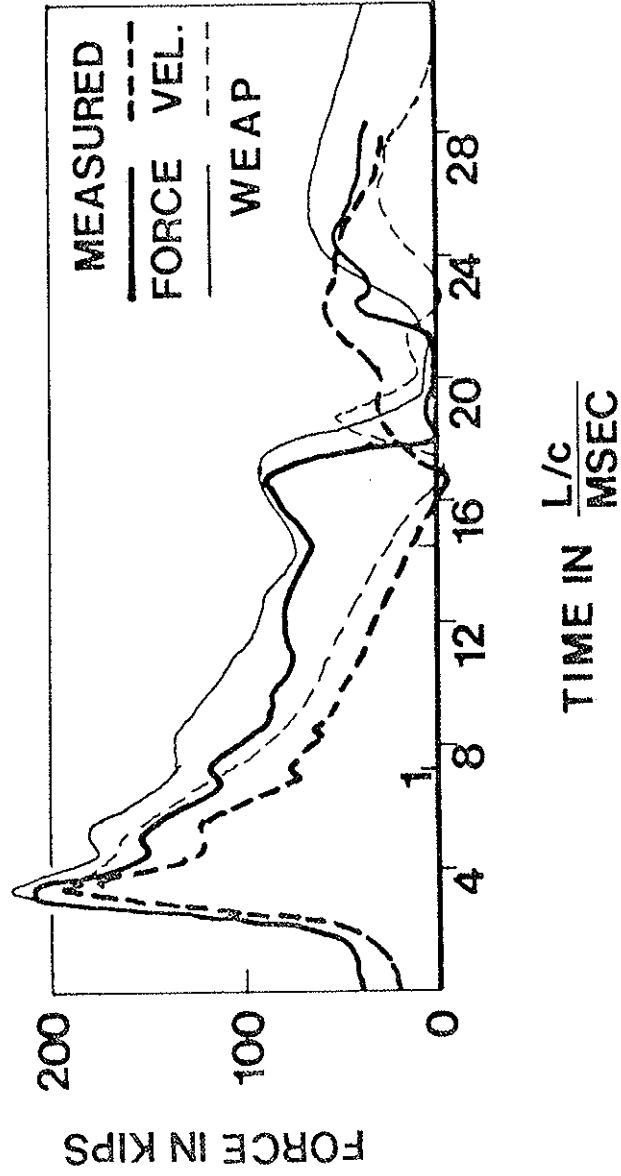


FIGURE 6-10: FORCE AND VELOCITY MATCH FOR PILE NO. 11

be due to a low combustion pressure. (Perhaps because it was in early driving and the hammer did not ignite properly). The fact that the blow was actually higher (25) than computed (19) supports this explanation.

Pile No. 12

This load test pile was restruck, after a waiting period, by a Link Belt 440 hammer. The hammer lifted off and no penetration of the pile was achieved. Both observations were correctly predicted by the program. The maximum stress prediction was within 5%. The velocity match (Figure 6-11) is in this case somewhat better than the force match. Both matches are poor, probably because of inaccurate skin resistance distribution and ignition timing. The latter event is not as well defined for atomized fuel injection as for those hammers which use the impact for atomization.

Pile No. 13

A Link Belt 660 hammer record is shown in Figure 6-12. The match is very good. (Maximum force match within 8%). As in the case of Pile No. 12, no set was observed and the hammer had to be throttled back in order to avoid lift-off.

Pile No. 14

A rather poor match (21% error in maximum force prediction) is shown in Figure 6-13 for a load test pile from the Case research program (14). The hammer was a Vulcan No. 1, powered by compressed air. Unfortunately, capblock

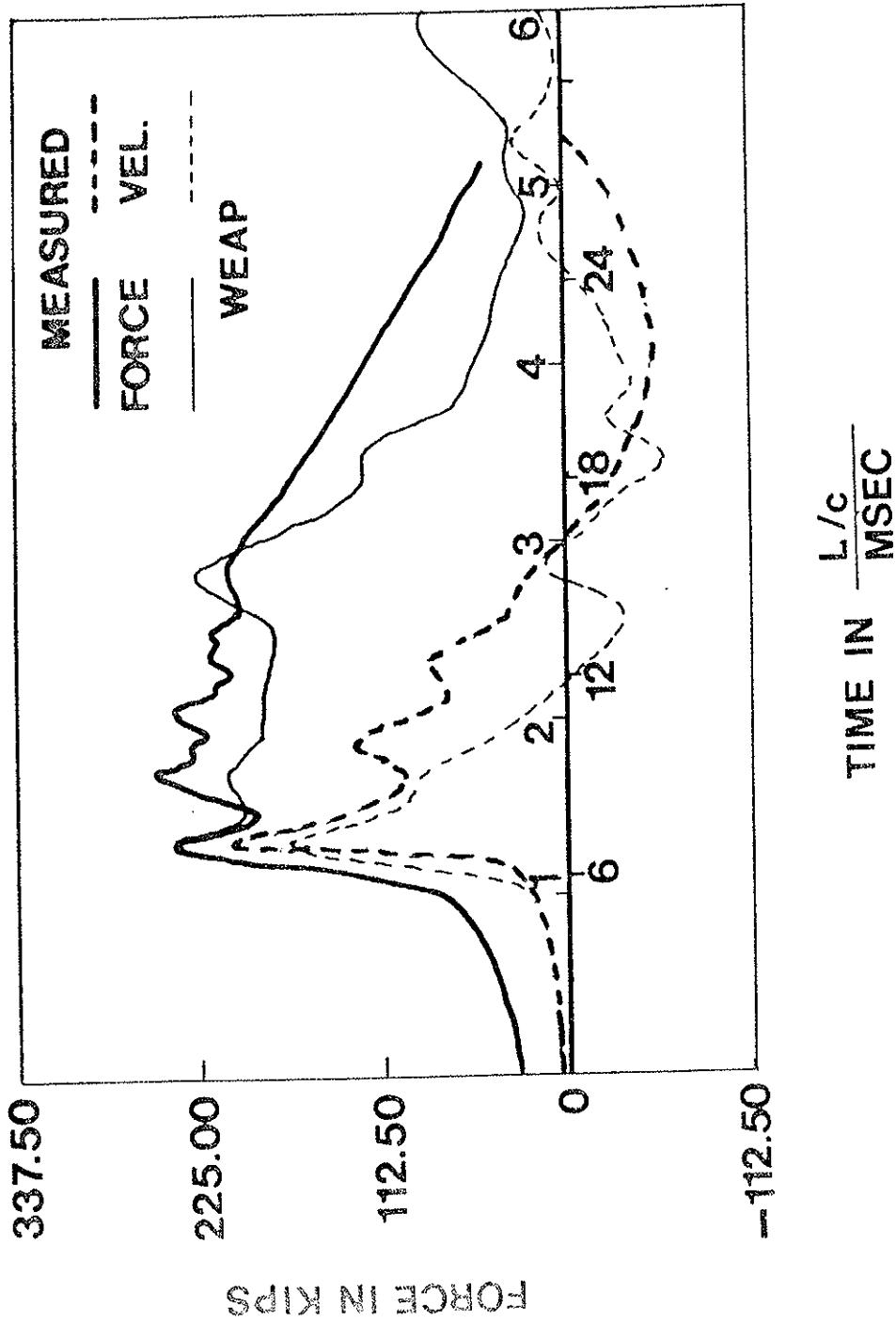


FIGURE 6-11: FORCE AND VELOCITY MATCH FOR PILE NO. 12
(CUYAHOGA RIVER)

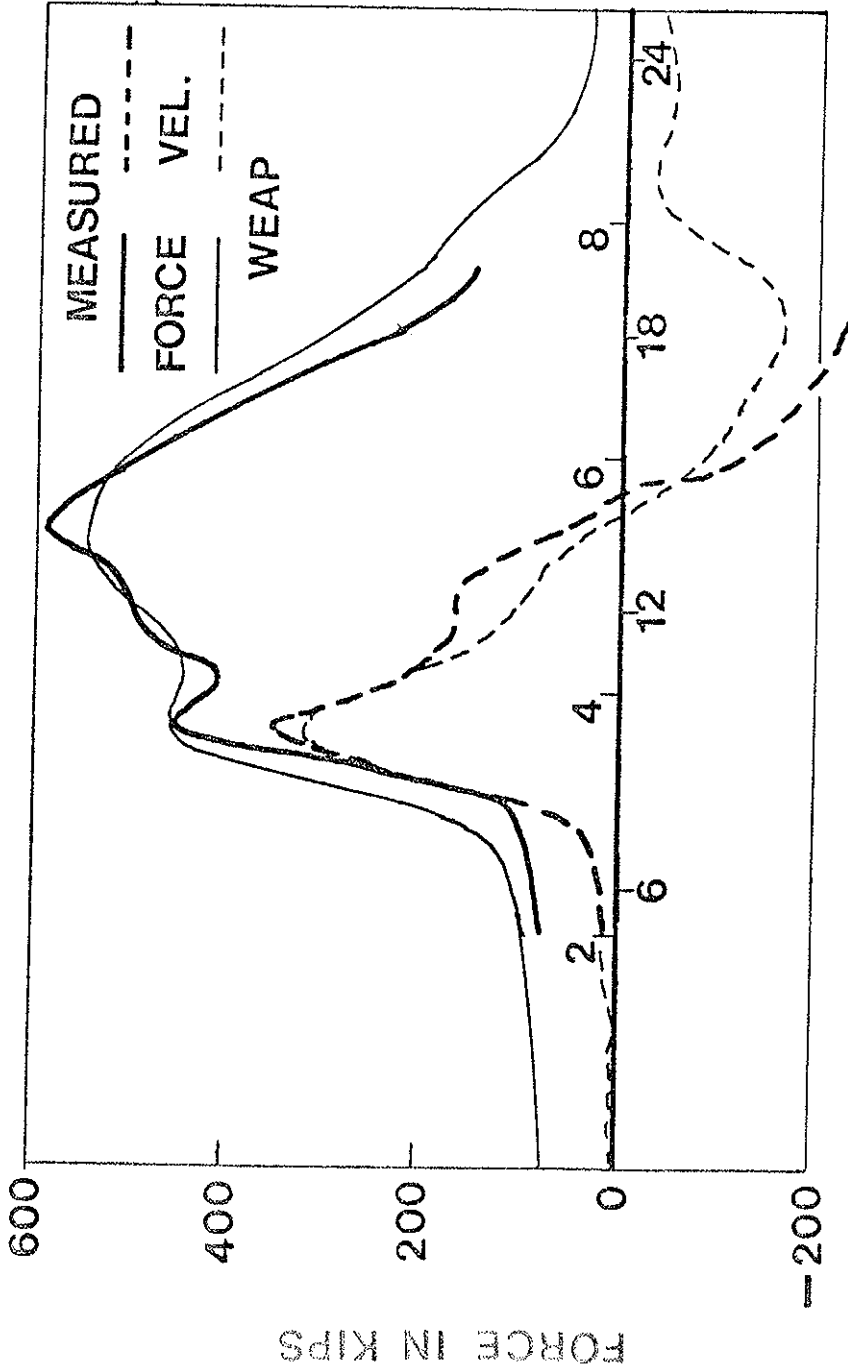


FIGURE 6-1-1: FORCE AND VELOCITY MATCH FOR FILE NO. 13

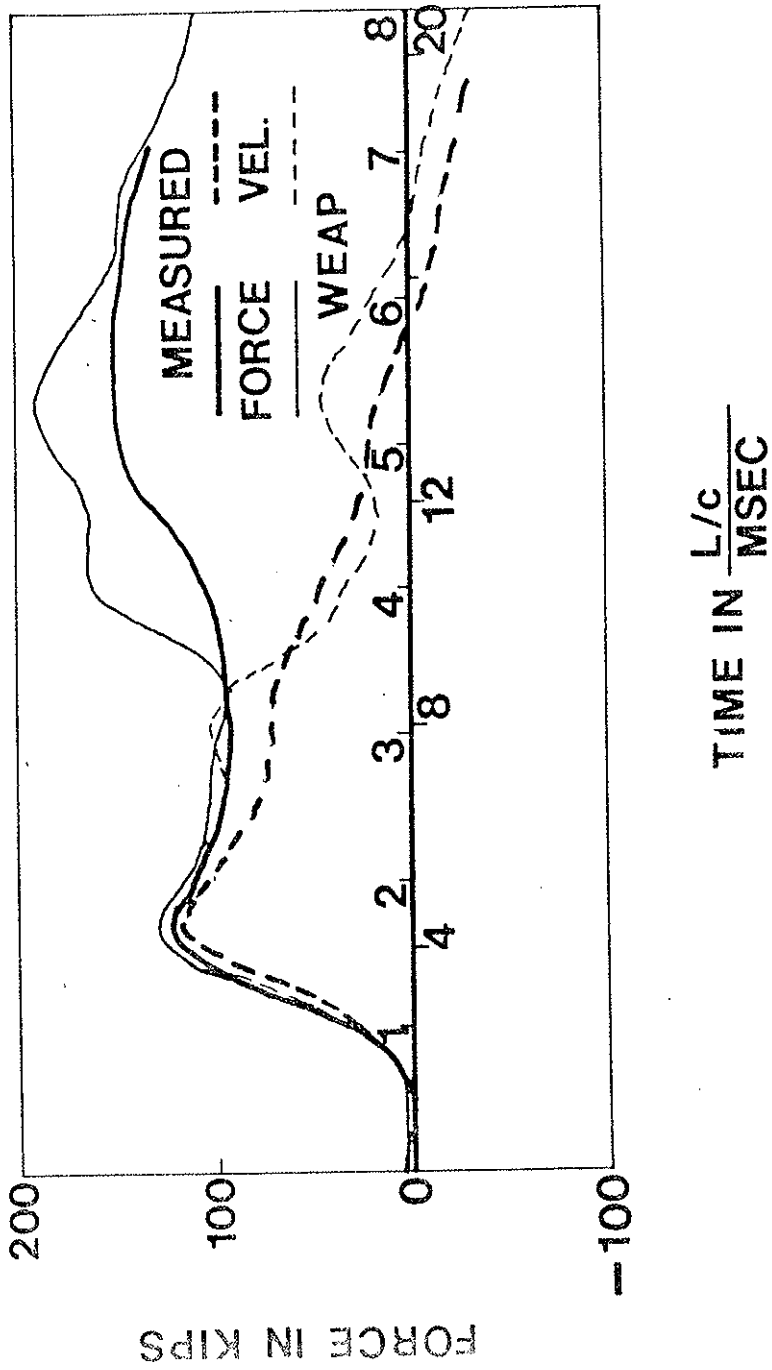


FIGURE 6-13: FORCE AND VELOCITY MATCH FOR PILE NO. 14
(PHILADELPHIA)

and helmet properties were not recorded. The blow count was predicted very accurately.

Piles No. 15 and 16

Both records (Figures 6-14 and 6-15) were actually recorded on the same pile using an air powered Vulcan 08 hammer. The pile was driven at the same site as Pile No. 9, Reference 12.

Both Figure 6-14 which was recorded in easy driving (the capacity of the pile was determined by a pull out test) and Figure 6-15 (which was one of the first records after the pile hit rock) show a very good match. The blow count was accurately determined for the easy driving case; refusal was indicated for hard driving. Note that the measured blow count of 184 is essentially refusal.

Pile No. 17

This was a rather long pile driven by a Vulcan 016 hammer and analysis shows that the assembly drop effect can be predicted rather accurately. Figure 6-16 contains two plots that were obtained on this WEAP run. The lower one is a three dimensional plot of force vs. pile length and time.

Note that the assembly drop creates a wave similar to the impact wave although of lower magnitude.

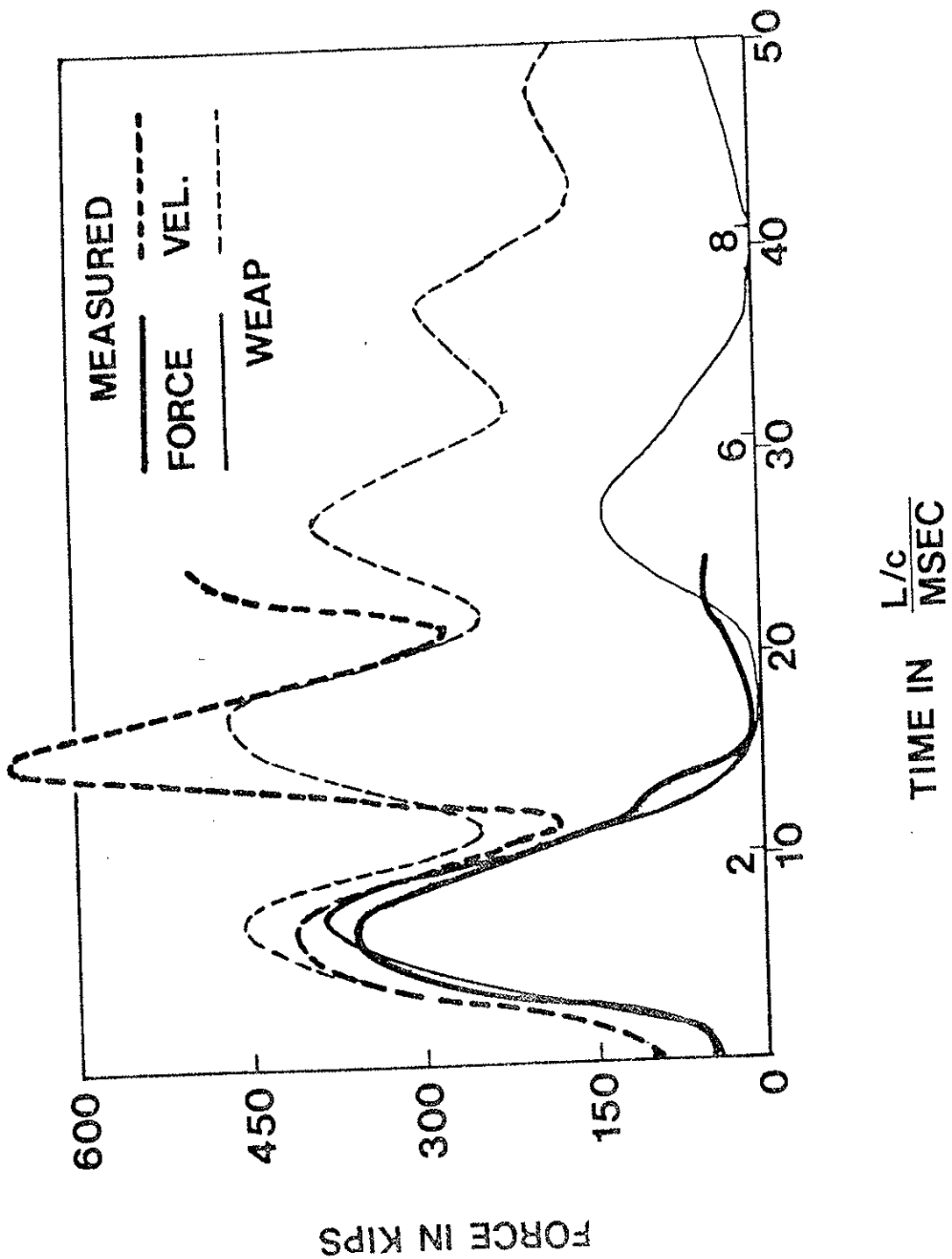


FIGURE 6-14: FORCE AND VELOCITY MATCH FOR PILE NO. 15

CHAPTER 7

CONCLUSIONS AND RECOMMENDATIONS

The foregoing results and the experience obtained with the program to date support the following conclusions:

(1) The program can be used very easily for most of commonly encountered dynamic pile analyses.

(2) Stress predictions will be good (less than 10% error) if the components of the driving system are well-modeled and if the hammer performs normally. The extensive study of program performance versus actual measurements has proven this point. However, it is only when measurements are available that one can be confident of the modeling if the system is at all unusual.

(3) Blow count (or bearing capacity) predictions were within 10% for most of the cases tested. Exceptions were Piles No. 7, 9, and 16 which had blow counts greater than 120 blows per foot. Blow counts becoming greater than this value result in a very small increase in capacity and are often referred to as "at refusal".

It should be mentioned that small penetrations, less than 0.1 inch per blow, may be associated with higher capacities than predicted by the wave equation. This fault lies in the assumption of an elasto-plastic static soil resistance. The ultimate resistance is not activated but a final set is obtained because of the nonlinear soil behavior.

(4) Stress and blow count are both dependent on stroke. The stroke,

on the other hand, was well-determined by the program for all those cases in which the hammer performed properly. Improper conditions were found with a hammer whose exhaust ports were partially plugged. Other problems that may be encountered are preignition or low fuel throttle settings.

Since low fuel throttle settings produce low strokes, their effect is readily recognized in the field. Preignition, on the other hand, produces an ineffective blow at a high stroke and is, therefore, dangerous. It can only be recognized by the examination of field measurements.

The use of the program can help in construction control to avoid problems resulting from such abnormal hammer conditions. The engineer has to evaluate pile driving performance using all analysis results (stroke or bounce chamber pressure, blow count and stresses) simultaneously.

(5) The program is well-suited to solve the driveability problem since it uses the actual hammer potential for a given soil condition. Of course, the program does not solve the principal problem of determination of soil bearing capacity.

(6) The program can be used to establish driving criteria in the conventional way except that it will provide a certain stroke value together with stress and blow count. The stroke should be verified in the field.

(7) The program can be used in the investigation of tension stresses in a pile. These stresses can be predicted accurately by the program if the soil resistance distribution and the driving system properties are accurately known. Further work is required to understand dynamic soil performance.

APPENDIX A

COMBUSTION CALCULATIONS USING COMBUSTION CHARTS

A.1 General Remarks

A variety of combustion charts exist which simplify combustion calculations. Among the charts are those of Hotel, Williams and Satterfield (8), and Newhall and Starkman (9). Unfortunately, these charts are usually devised for four cycle engines or they only consider the combustion of isooctane. Thus, some errors are made in using these charts. However, the uncertainties in basic assumptions usually introduce greater errors than those caused by the use of the charts.

One important question is the degree to which scavenging occurs. Suppose, that the stroke is five times the distance from the exhaust ports to the impact block. Now if the piston has reached the top dead center the gases in the cylinder consist in the worst case of 20% burned products and 80% fresh air. Depending on the geometry of the exhaust ports the down stroke of the piston might either cause all fresh air, all burned products or a fraction of air and fresh products to be exhausted. Thus, the ratio of burned products to fresh air, f , might be anywhere between 0 and 100%.

For the following sample calculation the charts of (9) were used. These charts were set up for isooctane combustion but since the H/C ratio of this fuel is similar to that of diesel fuel, the results should not be affected seriously.

"The Unburned Compression Chart" i.e. the charts that are valid for the compression phase, in (9) were prepared assuming an air-fuel mixture. In the case of diesel hammers, however, pure air (or a mixture of air and combustion products) is compressed. For this reason the unburned charts are not used but rather the results from measurements.

The burned product to fresh air ratio, f , will be assumed at 20% (a very high value compared with the usual engine cycles) together with a constant volume combustion. The latter assumption is justified because the combustion occurs at a very high rate (as evidenced by Figure 6) and because the volume is at the minimum during combustion. This is also apparent from the measurements as the precompression force stays constant during the combustion delay.

Another error is introduced by using the chemical energy value, U_c , as given in (8). This effect can be considered negligible, too.

A.2 Sample Calculation

As an example, consider a DELMAG D-12 hammer with a compression ratio, C.R., of 1:13.5, a fuel charge of 0.00463 lbs. (2.1 grams) per blow and an initial volume of 1308 cubic inches (2.14 cubic decimeters) of air. Assuming an initial temperature of 537°R (25°C) the volume of one pound of air (454 grams) is 13.53 cubic feet (383.1 cubic decimeter). The air to fuel ratio is, therefore

$$A/F = \frac{\frac{1308}{1728} (13.53)}{0.00463} = 12.1$$

The chemically correct mixture has an air/fuel ratio of 15.0 for Diesel fuel (based on the combustion of 1 mol $C_{12}H_{26}$ with 18.50 moles of O_2 and $(18.5)(3.773)$ moles of N_2).

Thus the mixture is $100 \frac{15-12.1}{12.1} = 24\%$ fuel rich. The charts for a 20% fuel rich mixture will be used.

(a) Compression cycle

The measured pressure at impact was on the average 500 psi (34.5 bar). Assuming the pressure volume relation to follow

$$p = p_{atm} \left(\frac{V_{atm}}{V} \right)^{exp} = p_{atm} (C.R.)^{exp}$$

with p_{atm} and V_{atm} being the atmospheric pressure and the corresponding volume respectively, one obtains for the unknown exponent

$$exp = \frac{P}{P_{atm}} \frac{1}{C.R.} = \frac{500}{14.7} \frac{1}{13.5} = 1.36$$

(instead of 1.4 as for the adiabatic and pure air process).

The precompression temperature can then be determined from

$$T = 537 \left(\frac{500}{14.7} \right)^{\frac{1}{1.36}} = 1366 \text{ } ^\circ R$$

$$(T = 759 \text{ } ^\circ K)$$

The internal energy for this temperature is taken from Chart 6. Thus, using the $\phi = 1.2$ curve, one obtains approximately

$$U_2 = 185 \text{ BTU (195 kilo Joule)}$$

(b) Combustion

Going to Chart 3 and using $f = 0.2$ the chemical energy becomes

$$U_c = 0.8 (-60.88) + 0.2 (-1379) = -324 \text{ BTU (-341 kJ)}$$

Thus

$$U_3 = 185 - 324 = -139 \text{ BTU (-146 kJ)}$$

The volume V_3 is the compressed volume which according to the legend of Chart 5 and using $K_\phi = 1.2 = 0.3778$ becomes

$$V_3 = 0.3778 \frac{1366}{500} = 1.03 \text{ ft}^3 \\ (29.17 \text{ dm}^3)$$

and from Chart 3, approximately:

$$p_3 = 1800 \text{ psi (124 bar)}$$

$$T_3 = 4750 \text{ }^\circ\text{R (2639 }^\circ\text{K)}$$

$$S_3 = 2.205 \text{ BTU/}^\circ\text{R (4.19 kJ/}^\circ\text{C)}$$

(c) Expansion

The final volume of expansion is

$$V_4 = 1.03 (13.5) = 13.9 \text{ ft}^3 (393 \text{ dm}^3)$$

and with $S_4 = 2.205 \text{ BTU}/^\circ\text{R}$ ($4.19 \text{ kJ}/^\circ\text{C}$)

one obtains

$$p_4 = 70 \text{ psi (4.83 bar)}$$

$$T_4 = 2420 \text{ }^\circ\text{R (1344 }^\circ\text{K)}$$

$$U_4 = -830 \text{ BTU (788 kJ)}$$

(d) Work

Expansion: $W_e = U_3 - U_4 = -139 - (-830)$

$$W_e = 691 \text{ BTU (729 kJ)}$$

Compression: $W_c = U_2 - U_1 = 185 - 0$

$$W_c = 185 \text{ BTU (195 kJ)}$$

Net Work: $W_n = 691 - 185 = 506 \text{ BTU (534 kJ)}$

Thermal Efficiency

$$\eta_{\text{th}} = \frac{W_n}{(\text{LHV})(\text{Fuel weight})} = \frac{506}{(19,240)(0.056)} = 47\%$$

(Using the weight of diesel fuel per cycle).

A.3 Discussion

The fuel energy per blow converted to work is theoretically

$$W_t = 0.47 (19240)(0.00463) =$$

$$41.9 \text{ BTU (44 kJ)}$$

or

$$W_t = 41.9 (0.778) = 32.6 \text{ k-ft}$$

Note that this corresponds to a stroke of

$$\text{Stroke} = \frac{32.6}{2.75} = 11.9 \text{ ft (3.63 m)}$$

and that the hammer is rated at 22.5 k-ft. (30.5 kJ) for an 8.2 ft (2.50 m) stroke. Such a stroke is more than the maximum possible one of 10.8 ft (3.30 m), e.c. since in general very little combustion energy is consumed through energy losses and, therefore, all of it produces stroke. It can be said that the computed energies are approximately 50% too high. This argument is confirmed by the fact that the combustion pressure was computed much higher than measured.

For this reason it was decided to use the measured combustion pressure as a starting value for the expansion process which was then modeled isentropically using an expansion exponent that is given by the combustion chart. In the current case (D-12, 20% fuel rich) one obtains:

$$\text{exp} = \frac{1800}{70} \frac{1}{C.R.} = 1.25$$

in order to account for some additional losses during the expansion process, the exponent was chosen as 1.3. As a further argument against pressure calculations based on combustion theory it should be mentioned that the strokes predicted by the present wave analysis, using the model just described are in very good agreement with those observed. A limited pressure diesel cycle with measured pressures but a theoretical pressure volume relation would

also produce a much larger net work and cannot be considered a satisfactory approach (see also Figure A1 for a comparison of computed with measured work).

Calculations were also performed in which the combustion product to air ratio, f , was determined assuming a maximum combustion pressure of 1600 psi. The result was $f = 0.48$.

The reason for the differences between theory and measurement probably lie in the inefficient way fuel is mixed with the air. The atomized injection hammers are actually using much lower fuel/air ratios.

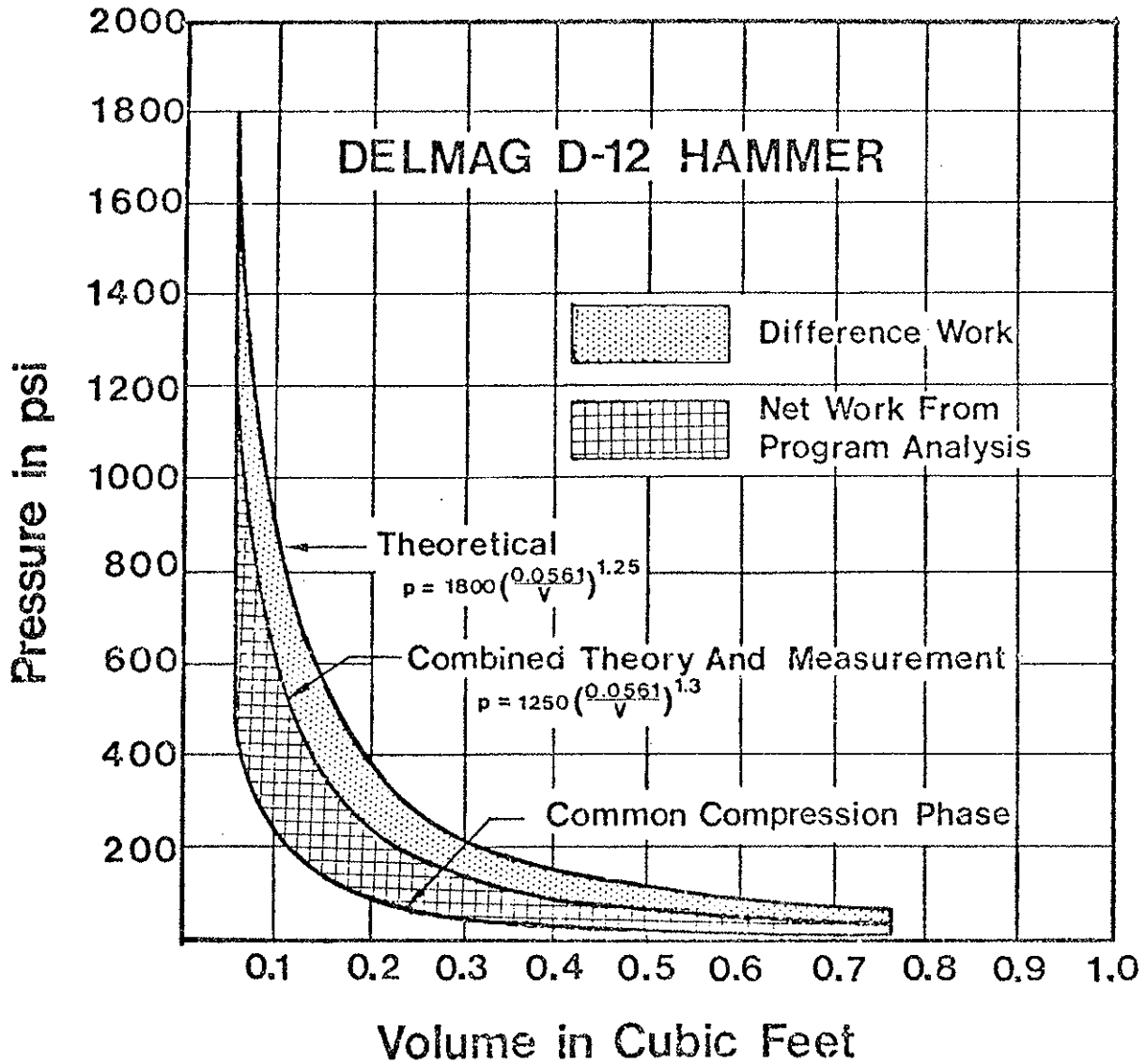


FIGURE A1: COMPARISON OF COMPUTER WITH MEASURED COMBUSTION AND EXPANSION CYCLE.

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METRIC CONVERSION FACTORS

<u>To Convert</u>	<u>To</u>	<u>Multiply By</u>
pounds (lbm)	kilograms (kg)	0.4535924
pounds force (lb)	newtons (N)	4.4482219
kip (1000lb)	kilonewtons (kN)	4.4482219
inches (in.)	centimeters (cm)	2.54
feet (ft)	meters (m)	0.3048
feet per second (fps)	meters per second (m/sec)	0.3048
foot-pounds (ft-lb)	newton-meters (N·m)	0.355818
foot-pounds (ft-lb)	joules (J)	1.355818
Pounds per foot (lb/ft)	kilograms per meter (kg/m)	1.488164
Acceleration of gravity (g)	meters per second squared (m/s ²)	9.806650
Acceleration of gravity (g)	feet per second squared (ft/sec ²)	32.17405
kip-inches (kip-in.)	newton-meters (N m)	112.9848
degrees Centigrade	degrees Fahrenheit	1.8 and add 32

16-742
Installation
Package

WAVE EQUATION ANALYSIS OF PILE DRIVING

NAEAD

PROGRAM

Volume II - User's Manual



U.S. DEPARTMENT OF THE ARMY
CORPS OF ENGINEERS
WATERWAYS EXPERIMENTAL STATION
VICKSBURG, MISSISSIPPI 39180

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16. Abstract <p>A computer program was written and tested that performs a realistic Wave Equation Analysis of Piles driven by any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesels and refined mechanical hammer models.</p> <p>The program development was aimed at providing a simple input and both a flexible and extensive output that includes automatic plotting capabilities. Pile Driving Hammer data were prepared and stored in a file for most of the commonly encountered models. The computer language is FORTRAN IV.</p> <p>The program was extensively tested against measured pile top force and velocity data and against measured diesel combustion pressure and stroke.</p> <p>This volume is the second in a series. The others in the series are:</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <thead> <tr> <th style="text-align: center;"><u>Vol. No.</u></th> <th style="text-align: center;"><u>FHWA No.</u></th> <th style="text-align: center;"><u>Short Title</u></th> <th style="text-align: center;"><u>NTIS (PB) No.</u></th> </tr> </thead> <tbody> <tr> <td style="text-align: center;">1</td> <td style="text-align: center;">IP-76-14</td> <td style="text-align: center;">Background</td> <td></td> </tr> <tr> <td style="text-align: center;">3</td> <td style="text-align: center;">IP-76-14</td> <td style="text-align: center;">Program Documentation</td> <td></td> </tr> <tr> <td style="text-align: center;">4</td> <td style="text-align: center;">IP-76-14</td> <td style="text-align: center;">Narrative Presentation</td> <td></td> </tr> </tbody> </table>						<u>Vol. No.</u>	<u>FHWA No.</u>	<u>Short Title</u>	<u>NTIS (PB) No.</u>	1	IP-76-14	Background		3	IP-76-14	Program Documentation		4	IP-76-14	Narrative Presentation	
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WEAP
WAVE EQUATION ANALYSIS OF PILES
USER'S MANUAL

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CHAPTER 1

INTRODUCTION

WEAP is a computer program written in FORTRAN IV code which performs a Wave Equation Analysis for Piles. This program was developed between July 1975 and July 1976 under a contract with the Federal Highway Administration, Offices of Research and Development. In addition to the program code and this User's Manual, the following publications were prepared:

1. Wave Equation Analysis of Pile Driving, Background
2. Wave Equation Analysis of Pile Driving, Program Documentation
3. Wave Equation Analysis of Pile Driving, Narrative

This User's Manual is intended to provide first, a very basic outline of the various program functions; second, a definition of the input quantities; third, a description of the output and finally, sample problems. The Manual does not give an introduction into the wave equation approach for solving pile driving problems. Also, it does not provide, except for examples, input quantities of pile-soil parameters. The user who is not familiar with the concept of wave equation analyses for piles should, therefore, familiarize himself with such ideas by reading at least Chapter 1 of the Background report and some of the publications referenced there. Other chapters of the Background report should be read when questions regarding the basic approach of the program exist.

The program documentation contains details of the program flow and gives recommendations on the program installation in a computer system. The

"narrative" contains text and illustrations of a tape/slide show.

The use of a discrete pile model, i. e. a string of springs and masses, to analyze the pile driving problem was first proposed by E. A. L. Smith in 1955. Smith himself wrote a program that performed the necessary computations on a digital computer. The apparent advantages of his method prompted various other investigators to expand the concept and to test the idea. As a result, several other program versions were written. The most widely distributed version was prepared by the Texas Transportation Institute (TTI).

The TTI program proved to be a valuable tool both for the design engineer and for construction control. However, as it was developed primarily for air/steam or drop hammers, shortcomings were found when diesel hammer analyses were attempted.

The authors of WEAP intended to contribute to the art in the following ways:

- (a) Provide for a realistic diesel hammer analysis.
- (b) Improve the air/steam hammer analysis by also considering the very heavy assembly of such hammers.
- (c) Test the wave equation analysis results by comparison with force and velocity records, measured at the pile top.
- (d) Make the input as simple as possible by the elimination of "computed input" and by the provision of a data file that contains the data of commonly used pile driving hammers.
- (e) Provide improved output by utilizing automatic plotting capabilities in addition to a very flexible printout,

capable of giving the user many variables versus time in a very instructive manner.

The usual production run that a user must make in order to analyze an actual problem is generally specified with only a few parameters describing pile and soil. Such a run can therefore be coded by using the so-called "Short Input". In unusual cases, i.e. when the user wants to specify quantities which cannot be computed by the program, or when a hammer is to be used that is not contained in the hammer data file, the "Complete Input" has to be utilized.

After a brief discussion of the program flow (which does not replace the corresponding section of the background report) in Chapter 2, Chapter 3 and 4 deal with the description of the Short and Complete Input, respectively. Chapter 5 gives examples that relate to a few selected, frequently encountered input problems. Chapter 6 describes the computer output and Chapter 7 demonstrates first, three basic examples utilizing the short input and second, two hypothetical cases which require the complete input. Forms for both short and complete input are added in the Appendix.

CHAPTER 2

THE BASIC PROGRAM FLOW

In order to facilitate the understanding of the following chapters a few of the ideas behind the program are mentioned. However, this chapter cannot replace the more detailed discussions in the report.

2.1 Input

Input was programmed such that the actual known parameter such as pile length, skin friction distribution, etc. can be used directly. Since hammers require specification of a large number of data, input has been prepared and stored on a file for most of the commonly encountered types.

The program converts the input data to the standard Wave Equation parameters. It is because of this automated data preparation that many of the names that appear in the Texas Transportation Institute (TTI) program are not found in this manual. Also it was intended to make a standard computer run simple while providing for unusual cases. For this reason the input was divided in two parts: Short and Complete Input. In the short input terms known from the TTI work are used as much as possible to identify input quantities. Thus, standard cases can be solved using WEAP by people used to the TTI program without much preparation.

2.1.1 Input of Soil Damping

One parameter should be discussed a little further. It is the skin and toe damping value, usually referred to as J-values. Since the soil damping force in Smith's approach is equal to the product of J times

segment velocity times static soil resistance force, damping is automatically distributed like the static skin resistance. In fact, even though a toe damping parameter may have been specified the toe damping force would be zero if skin friction was 100 percent.

WEAP also allows the use of damping parameters which are independent of the static soil resistance. This type is referred to as viscous damping. The viscous damping force is calculated as damping parameter times segment velocity (the damping constant, therefore, has dimensions kips per foot per second).

In order to make the input comparable for the two damping types, skin and toe damping are again specified. The skin damping parameter for viscous damping is the sum of all segment-skin-damping values. The program apportions this sum according to the skin friction distribution specified (and would be present even for zero skin friction).

To further facilitate the input of the viscous damping parameters, non-dimensionalized skin and toe damping constants are used (so-called Case damping). Non-dimensionalization is accomplished by division by the pile's impedance: $EA/c = \text{Young's Modulus times cross sectional area divided by the wave speed}$. This impedance has the dimensions of a viscous damping parameter. Thus, the input values for Case (or viscous damping) are truly dimensionless.

To clarify further the actual computer program, steps are now listed for the two damping types:

- (a) Smith:
- Input Set each segment's skin damping parameter to the input skin damping parameter.
Set toe segment's damping parameter to the corresponding input constant.
- Analysis Damping force at a segment equals damping parameter times static resistance force times velocity, all for the same segment.
- (b) Viscous (Case):
- Input Apportion input skin damping parameter to all segments according to their static skin friction (still non-dimensional).
Multiply each skin damping parameter by the EA/c value of the corresponding segment.
Multiply input toe damping parameter by the EA/c value of the toe segment.
- Analysis Damping force equals damping parameter times velocity for the same element.

In this context it should be mentioned that the corresponding viscous damping parameters of Smith's approach can become very large (especially at the toe). Since the corresponding viscous parameter is the product of Smith's constant times the static resistance it can vary from analysis to analysis, proportional to the static resistance force. (E.g., 80% toe resistance, $J_{toe} = 0.15$ s/ft, RULT = 50, 100, 150, 200 tons. In the first analysis the corresponding viscous damping constant is $2 \times 50 \times 0.8 \times 0.15 = 12$ kips/ft/sec. In the last analysis it is 48 - this is almost four times the impedance of a twelve inch pile with .178 inch wall).

WEAP reduces the analysis time increment when the damping constant at

any one segment becomes large relative to the pile's impedance (EA/c). Thus, a Smith's damping value that is large at an element which has also a high static resistance force may produce a long and, therefore, costly computer run.

Damping parameters affect the blow count result to a considerable degree. Therefore, it is advisable to use the rather well-documented Smith-values whenever doubts exist as to the proper Case damping values.

2.2 Analysis Cycle

2.2.1 Open End Diesel Hammers

Open end diesel hammers are usually started by assuming a five foot stroke (if the user did not specify another one); a dynamic analysis is then performed and the return stroke is determined. If this return stroke is different by more than five percent from the first assumption, then a new analysis is performed and again a check is made on the convergence of the stroke.

Up to four iterations are usually allowed except for cases where the stroke converges in an alternating mode. Then six trials are permitted.

After the last stroke is analyzed, the extreme values found are printed together with other optional output.

A new ultimate resistance is then analyzed starting with a new stroke. This new stroke is based on previous results assuming an increase in resistance. Of course, iterations are again performed.

This process is continued until all ultimate resistance values are analyzed or until no more permanent set occur. Then a summary of all results is printed and the program is ready to analyze a completely new data set.

An exception to this "standard run" is the "constant stroke" analysis (IOSTR = -1). This option causes the change of combustion pressures until the return stroke equals the input stroke. The use of this option is recommended for hammers that have a variable fuel pump and/or for cases where reduced strokes are to be used. The program does not check on any fuel energy limits when this option is used. Thus, strokes may be accidentally specified which are too large and cannot possibly be obtained in the field (e.g. when the soil resistance is low).

2.2.2 Closed End Diesel Hammer

For this hammer type, a start is made at the maximum stroke at which uplift is just imminent, if the user did not specify another value. Depending on the soil resistance, the stroke will either become smaller or fuel reductions will be necessary to avoid uplift. Again, the program iterates until the proper stroke or fuel setting is found. A maximum of four cycles is allowed.

Fuel setting or stroke are again used as a starting point for further RULT values analyzed and a summary is printed.

The "constant stroke analysis" can also be performed for closed end hammers. Table 1 was provided to facilitate the stroke input for a given

MKTDA35B		MKTDA55B		LB 180		LB 440		LB 520		LB 660	
STROKE,	B.C.P.	STROKE,	B.C.P.	STROKE,	B.C.P.	STROKE,	B.C.P.	STROKE,	B.C.P.	STROKE,	B.C.P.
1.27	-.00	1.41	-.00	.69	2.36	1.35	5.26	.92	2.87	1.64	5.16
1.37	.12	1.51	.40	.79	2.82	1.45	5.82	1.02	3.28	1.74	5.61
1.47	.54	1.61	.85	.89	3.29	1.55	6.39	1.12	3.70	1.84	6.08
1.57	.99	1.71	1.32	.99	3.79	1.65	7.00	1.22	4.15	1.94	6.56
1.67	1.46	1.81	1.81	1.09	4.32	1.75	7.64	1.32	4.61	2.04	7.07
1.77	1.95	1.91	2.34	1.19	4.86	1.85	8.30	1.42	5.09	2.14	7.59
1.87	2.47	2.01	2.89	1.29	5.44	1.95	9.01	1.52	5.60	2.24	8.14
1.97	3.02	2.11	3.47	1.39	6.04	2.05	9.75	1.62	6.12	2.34	8.71
2.07	3.60	2.21	4.09	1.49	6.68	2.15	10.53	1.72	6.67	2.44	9.31
2.17	4.21	2.31	4.75	1.59	7.35	2.25	11.36	1.82	7.25	2.54	9.93
2.27	4.86	2.41	5.44	1.69	8.06	2.35	12.23	1.92	7.85	2.64	10.58
2.37	5.55	2.51	6.18	1.79	8.81	2.45	13.16	2.02	8.48	2.74	11.26
2.47	6.28	2.61	6.97	1.89	9.60	2.55	14.14	2.12	9.14	2.84	11.98
2.57	7.05	2.71	7.80	1.99	10.44	2.65	15.18	2.22	9.84	2.94	12.72
2.67	7.88	2.81	8.70	2.09	11.33	2.75	16.29	2.32	10.57	3.04	13.51
2.77	8.76	2.91	9.66	2.19	12.28	2.85	17.47	2.42	11.34	3.14	14.33
2.87	9.70	3.01	10.69	2.29	13.28	2.95	18.74	2.52	12.15	3.24	15.20
2.97	10.71	3.11	11.79	2.39	14.35	3.05	20.08	2.62	13.00	3.34	16.11
3.07	11.80	3.21	12.98	2.49	15.49	3.15	20.46	2.72	13.90	3.44	17.07
3.17	12.96	3.31	14.26	2.59	16.72			2.82	14.86	3.54	18.08
3.27	14.22	3.41	15.65	2.69	18.03			2.92	15.87	3.64	19.15
3.37	15.57	3.51	17.16	2.79	19.43			3.02	16.94	3.74	20.28
3.47	17.04	3.61	18.80	2.89	20.95			3.12	18.07	3.84	21.49
3.57	18.64	3.71	20.59	2.99	22.58			3.22	19.28		
3.67	20.38	3.81	22.55	3.09	24.35			3.32	20.56		
3.77	22.28	3.91	24.71	3.19	26.13			3.42	21.93		
3.87	24.36	4.01	27.09					3.52	23.40		
3.97	26.66	4.11	29.72					3.62	24.74		
4.07	29.19	4.21	32.66								
4.17	32.01	4.31	35.94								
4.27	35.14	4.41	39.64								
4.37	38.66	4.51	43.82								
4.47	42.63	4.61	48.59								
4.57	47.12	4.71	54.06								
4.67	52.26	4.81	60.40								
4.77	58.18	4.91	67.80								
4.87	65.06	5.01	76.56								
4.97	73.13	5.11	80.11								
5.07	82.70										
5.17	89.35										

TABLE 1: STROKES AND CORRESPONDING BOUNCE CHAMBER PRESSURE FOR CLOSED END HAMMERS
STROKE IS IN FEET; B.C.P. IS IN PSI, GAUGE PRESSURE

bounce chamber pressure. This table lists strokes and corresponding bounce chamber pressures for all closed end hammers which are contained in the hammer file.

2.2.3 Air Steam Hammers

This hammer type is simply analyzed for the stroke and efficiency specified. In contrast to other programs, the motion and impact of the assembly is also modeled. This feature does not influence the overall program flow and no further discussion seems necessary.

2.3 Analysis Details (Diesel Hammers Only)

For a better understanding of the program output a few details of the diesel hammer analysis should be known. The analysis is divided into three parts:

- (a) Precompression
- (b) Impact Analysis
- (c) Ram Rebound

Whenever this manual refers to "the analysis", part (b) is usually meant.

It starts approximately one millisecond before either impact or before ignition and lasts until one of the following two conditions is met:

- (a) TEMAX as specified by the user is exceeded.
- (b) (($2L/c^*$ have at least passed (no impact)) or ($2.3 L/c$ have at least passed since impact)) and ((ram has risen at least 20% of distance to ports and toe has rebounded at least 2%) or (ram has risen at least 10% of distance to ports and toe has rebounded at least 20%)).

* Twice the time that the stress wave needs to travel along the pile of length L.

Output values are collected and extrema are determined at intervals between time increments which depend on the total expected analysis time. The check for stress minima (tension stresses) is limited to a time $3L/c$ after impact. The minimum stress is at least zero.

A further discussion of the analysis is given in Chapters 3, 4 and 5 of the research report.

2.4 Output

Output is made for each RULT analysis and at the end of the job. Variables, like forces, as a function of time have to be stored temporarily since it is not known a priori if the current analysis is for the correct stroke. To keep the program size limited, output is restricted and is given for, at most, two hundred time steps. To cover a sufficiently long time period, the output values stored and later printed are usually not consecutive ones.

When plots are made, the time curves sometimes appear not well rounded. This appearance is caused by the skipping of intermediate values and the plotter's linear interpolation.

CHAPTER 3

SHORT INPUT

The short input consists of the following information: Title, Options and Selections, Cap and Cushion Information, Coefficients of Restitution, Pile Description and Soil Description. Note that an unlimited number of problems can be solved in one run by submitting several input data sets at one time.

The following terminology will be used:

Default:	A value was not specified (left blank). Usually the program inserts a standard value automatically.
Blank or Unspecified:	A data field is left without input.
Integer:	An integer number. Input of integers has to be always right justified within their field, i.e., they may have leading but not trailing blanks.
Real:	A number <u>containing always a decimal point</u> (integer plus fraction). Reals can be input anywhere in their field. Thus, the location of the decimal point is not restricted.
Field:	A number of consecutive spaces in which several characters all belonging to one quantity can be filled.

3.1 Title (Card No. 1.000)

A string of up to 40 alphanumeric characters identifying the problem.

3.2 Options and Selections (Card No. 2.000)

Integers in fields of four spaces, right justified.

The following options are available:

(a) IOUT Used to specify which output level is chosen.
= 0 minimum output consisting of hammer and pile model plus the extreme values found for each RULT. A summary is also printed.
= 1 in addition to the output of the 0 option two forces in the hammer and as many as 13 in the pile are printed as a function of time for each RULT.
= 2 As for 1 but with velocities instead of forces.
= 3 As for 1 but with stresses instead of forces.
= 4 As for 1 but with accelerations instead of forces.
= 5 As for 1 but with displacements instead of forces.
= 6 As for 1 but with selected properties like combustion pressure, sum of resistance forces and force, velocity and displacement being printed at three different pile locations.
If 10 is added to the above option numbers then plots of the corresponding printouts are made. The plots obtained will include one plot of the specified quantities vs. time for each RULT and a summary of resistance, stroke and both maximum and minimum stress vs. blow count.
If 20 is added then in addition to the +10 option output a three-dimensional plot of the printed quantities is made (this is not true for option 26). This plot features quantity vs. pile length and time. If IOUT > 9 card No.s 7.000 and 8.000 have to be supplied (Sections 3.8, 4.13, 4.14).

(b) IJJ This is an option which if equal to 1 requests additional input on Card No. 9.101 (Section 4.15). This input then defines the element numbers whose quantities are printed under output level 1 through 5 (or 11 through 15 or 21 through 25). IJJ = 0 causes these element numbers to be automatically determined.

(c) IHAMR This quantity selects the hammer to be used. It is keyed to Table 2. Accordingly, IHAMR can be any value between 1 and 100. If IHAMR = 0 then input providing various hammer properties has to be provided. (Card No.s 5.201 through 5.205, Sections 4.4 through 4.8).

TABLE 2: HAMMER IDENTIFIER

<u>Identifier</u>	<u>Hammer Name</u>	<u>Identifier</u>	<u>Hammer Name</u>	<u>Identifier</u>	<u>Hammer Name</u>	<u>Identifier</u>	<u>Hammer Name</u>
1	Delmag D-5	26	Kobe KC-25	51	Vulcan 2	76	Vulcan 540
2	Delmag D-12	27	Kobe K-35	52	Vulcan 1	77	Vulcan 360
3	Delmag D-15	28	Kobe KC-35	53	Vulcan 06	78	Vulcan 560
4	Delmag D-22	29	Kobe K-45	54	Vulcan-106 LR	79	Vulcan 3100
5	Delmag D-22-02	30	Kobe KB-60	55	Vulcan-106 HR	80	Vulcan 5100
6	Delmag D-30	31	Kobe K-150	56	Vulcan 08	81	Vulcan 1400-off
7	Delmag D-30-02	32	BSP 15	57	Vulcan 010-on	82	Vulcan 2000-off
8	Delmag D-36	33	BSP 25	58	Vulcan 014-on	83	Vulcan 4000
9	Delmag D-36-02	34	BSP 35	59	Vulcan 016-on	84	Vulcan 6000
10	Delmag D-44	35	BSP 45	60	Vulcan 020-on	85	
11	Delmag D-46	36	Linkbelt 180	61	Vulcan 030-on	86	MKT S-5
12	Delmag D-46-02	37	Linkbelt 440	62	Vulcan 300	87	MKT S-8
13	Delmag D-55	38	Linkbelt 520	63	Vulcan 500	88	MKT S-10
14	MKT DE-10	39	Linkbelt 660	64	Vulcan 650	89	MKT S-14
15	MKT DE-20	40		65	Vulcan 800	90	MKT S-20
16	MKT DE-30	41		66	Vulcan 850	91	MKT No. 7
17	MKT DE-40	42		67	Vulcan 1000	92	MKT 9B3
18	MKT DE-30B	43		68	Vulcan 1400-on	93	MKT 10B3
19	MKT DE-50B	44		69	Vulcan 2000-on	94	MKT 11B3
20	MKT DE-70B	45		70	Vulcan 010-off	95	MKT C-5
21	MKT DA-35B	46		71	Vulcan 014-off	96	MKT C-826
22	MKT DA-55B	47		72	Vulcan 016-off	97	
23	Kobe K-13	48		73	Vulcan 020-off	98	
24	Kobe K-22	49		74	Vulcan 030-off	99	
25	Kobe K-25	50		75	Vulcan 340	100	

- (d) IOSTR If equal to 0, normal program operation, i.e. iteration on stroke (fixed fuel).
 If equal to 1, no iteration.
 If equal to -1, iteration on fuel (fixed stroke).
 For IOSTR = -1 a stroke should be input in Card No. 6.000. (See also Chapters 4 and 5 of the research report.)
- (e) IFUEL If equal to 0 or 1, normal operation. If equal to 2 through 5 other fuel settings corresponding to the P2 through P5 pressures in the hammer data (Section 4.6) will be used. For a discussion of this input see also Chapters 4 and 5 of the research report.
- (f) IPEL Defines how one arrives at the pile segment parameters:
- = 0: automatic determination of pile segments.
 - = 1: automatic determination but with the relative segment lengths (ALPH(I)) being input Card No.s 2.101, ..., Section 4.3.
 - = 2: pile masses and stiffnesses are input (Card No.s 2.101, ... and 2.201, ..., Sections 4.1, 4.2). Corresponding ALPH(I) values are also required (Card No.s 2.301, ..., Section 4.3).
- (g) N Number of pile segments (maximum 98).
 If N = 0 or 1: automatic determination based on an element length of approximately 5 feet. Note that IPEL \neq 0 requires the input of N $<$ 1.
- (h) ISPL The number of springs which can transfer only limited tension forces or for which a slack (extension without tension force) is to be specified. If ISPL is greater than zero, additional input is required (Card No.s 6.501, ..., Section 4.12).
- (i) NCROSS If set to 0 it will be assumed that the pile is uniform and Card No. 5.000, Section 3.5 suffices for pile description. If NCROSS is greater than 0 (usually = 1) then data describing both the variation of pile cross sectional area and pile material vs. depth is required (Card No.s 5.010, ..., Section 3.5-b).

- (j) IBEDAM The pile internal damping value in percent of pile critical damping. Recommended values are for steel 1, concrete 3 and timber 5. These values are usually of minor importance and sufficient information for their choice is not yet available. Default: 1. For 0 pile damping set IBEDAM = -1.
- (k) IPERCS Percent skin friction (between 0 and 100). This value can also be input as a negative number which causes the skin friction as determined for the first analysis (see RULT on Card No. 6.000, Section 3.6) to be kept constant for all further analyses.
For example: IPERCS = -60 and RULT (the total static pile bearing capacity) = 50, 100, 125 tons will produce three analyses with the skin friction always at 30 tons and, therefore, the toe resistance only increasing (20, 70 and 95 tons).
- (l) ISMITH = 1: The damping parameters are assumed to be of the Smith type, i.e., dimension sec/ft.
= 0: The damping parameters are assumed to be of the viscous type but non-dimensionalized by division through EA/c (Section 3.6). This type is also called Case damping.
- (m) ITYS Type of skin friction distribution. If between 1 and 10 a corresponding distribution will be chosen from Figure 1.
If 0 or -1 a description of the distribution (on Card No.s 6.401, ..., Section 3.7) has to be input.
If < -1 QS, SJ and SU must be given for all elements plus the pile point as additional input (Card No.s 6.101, ..., 6.201, ..., 6.301, ...).
If equal to -1 only the damping parameters are read on Card No.s 6.201, ... (See Sections 4.9 through 4.11).
- (n) IPHI Sets the ratio of time increment to critical time increment. It is in percent. Thus, 100 should be the smallest value provided. If left unspecified the IPHI will be set to 140 ($\Delta t / \Delta t_{cr} = 1.4$) for diesel and 160 for air/steam hammers. Normally this value is not specified.

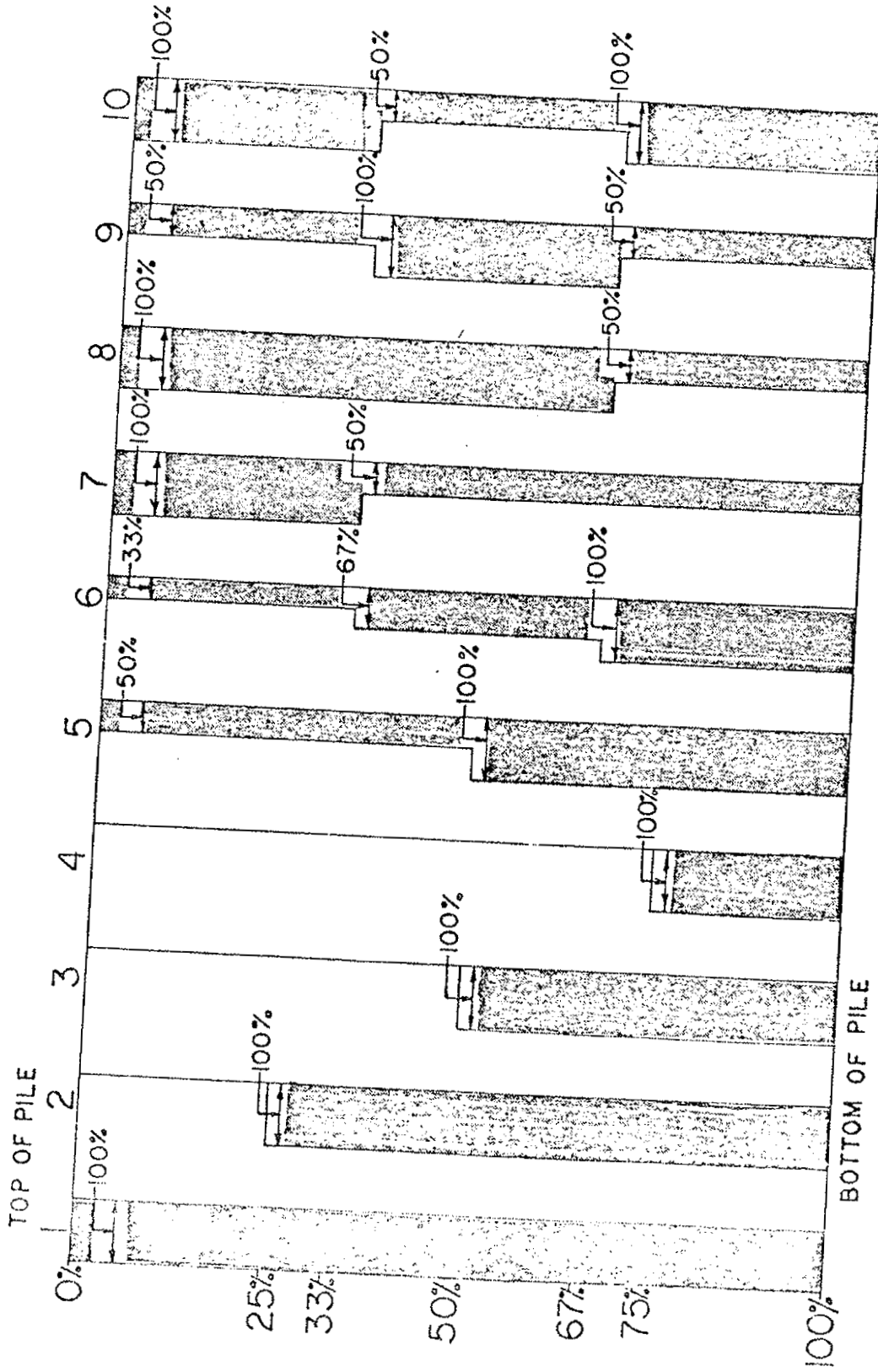


FIGURE 1: PROGRAM BUILT-IN SOIL RESISTANCE DISTRIBUTIONS

In all of the following input, real numbers are required containing a decimal point - each in a field of eight spaces.

3.3 Cap Weight, Cap
Stiffness, Cushion
Stiffness
(Card No. 3.000)

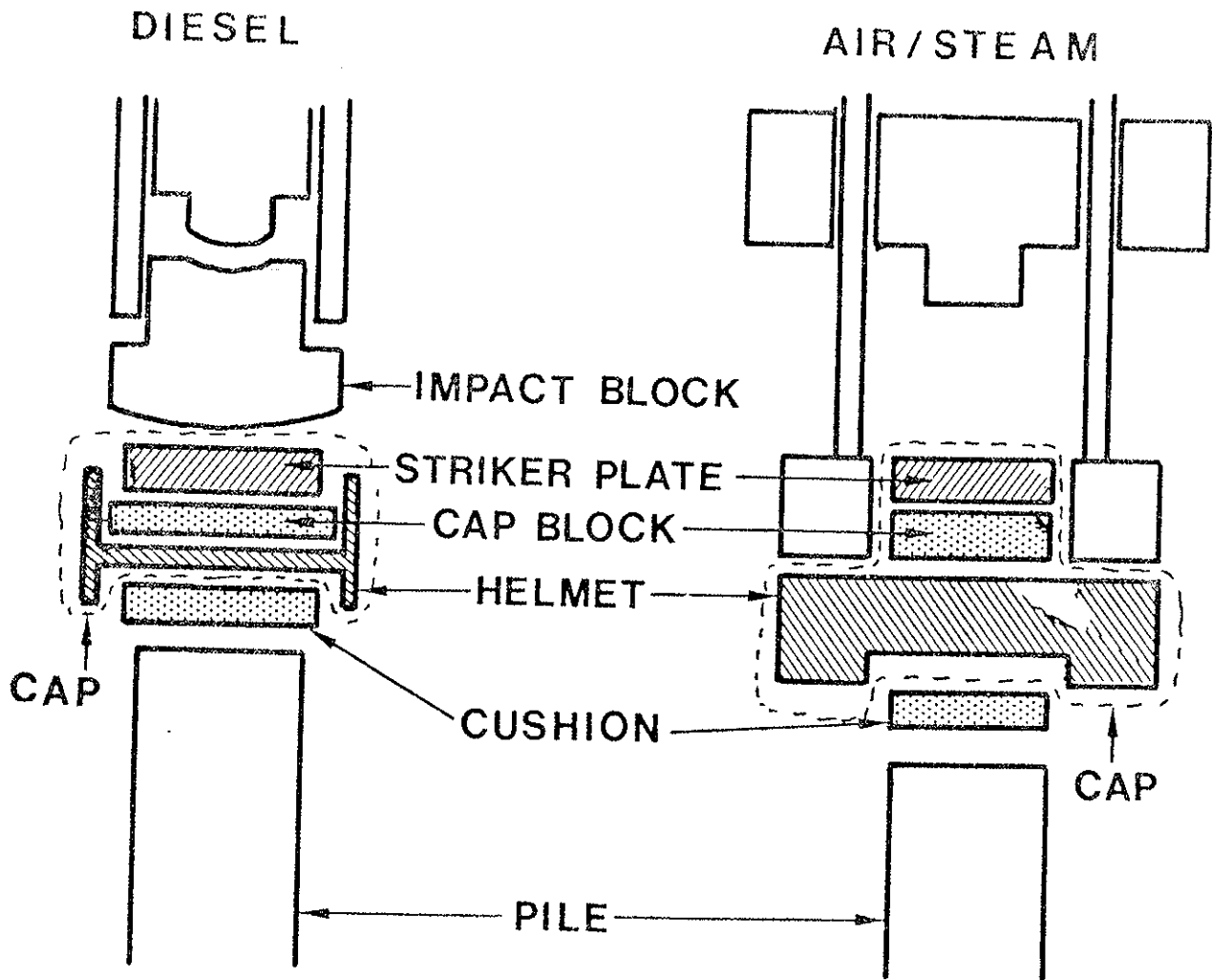
For the definition of "CAP" see Figure 2. Weight of cap in kips, stiffness of capblock in kips per inch and the stiffness of the pile cushion in kips per inch. If either of the two cushions is not present its stiffness should be 0.0 (or blank). A list of typical capblock and cushion properties is given in Table 3.

3.4 Coefficients of
Restitution
TDEL
TEMAX
(Card No. 4.000)

Four coefficients of restitution have to be specified for Anvil, Cap, Pile Top and Pile Cushion for diesel hammers or Cap, Assembly, Pile Top and Pile Cushion for air/steam hammers. If they are not specified a value of 0.85 will be assumed except for ECUS when no cushion is present. Then ECUS will be set to 1.0 no matter how it was specified. It is recommended to use 0.85 for steel to steel impact and lower values should be used where cushioning material is present. Table 3 contains some of these values.

TDEL is the combustion delay after impact or atomized fuel injection in seconds. Specifying TDEL here overrides the data file value or the value specified on Card No. 5.202. It can be specified as a negative value which would cause preignition except for atomized fuel injection where the time delay is always positive and depends on the time of injection. Small positive values can also cause preignition in the case of atomized fuel injection. Usually TDEL is left blank.

TEMAX is the maximum time up to which the analysis is to be carried out (in milliseconds). Note that TEMAX is usually determined by the program such that the stroke is found with sufficient accuracy. Thus, specification of TEMAX might lead to wrong results.



NOTE: THE TERM CAP (CARDS NO. 3.000 AND 4.000) INCLUDES STRIKER PLATE, CAP BLOCK AND HELMET.

FOR STIFFNESS OF CAP USUALLY ONLY THE CAP BLOCK NEEDS TO BE CONSIDERED.

WEIGHT OF CUSHION IS EITHER NEGLECTED OR CAN BE ADDED TO THE CAP.

FIGURE 2: TERMINOLOGY USED FOR DRIVE SYSTEM.

TABLE 3: CAPELOCK AND CUSHION PROPERTIES
AND A FEW HELMET WEIGHTS

TABLE 3(A): BASIC DATA OF CUSHION MATERIALS

Material	Elastic Modulus E (ksi)	Coefficient of Restitution e
Asbestos (1)	45	0.5
Conbest (2)	560	0.80
Ferbon (2)	800	0.85 (est.)
Micarta (1)	450	0.80
Micarta + Aluminum (1)	700	0.80
Oak (1 grain)	40-100	0.50
Oak (// grain)	1000-2000	0.50
Plywood	25-100	0.5
Urethane (3)	350	0.72

TABLE 3(B): WEIGHTS AND STIFFNESSES FOR FEC⁽⁴⁾ DRIVE CAPS

Size	OAK		CONBEST	
	Weight ⁽⁷⁾ kips	Stiffness ⁽⁵⁾ kips/in	Weight ⁽⁷⁾ kips	Stiffness ⁽⁶⁾ kips/in
12"x12"	0.91	28000	0.95	21000
14"x14"	0.96	38000	1.00	28000
16"x16"	1.48	48000	1.53	36000
18"x18"	1.67	60000	1.72	45000
20"x20"	N/A	N/A	2.44	45000
22"x22"	N/A	N/A	2.22	45000
24"x24"	N/A	N/A	2.36	45000

TABLE 3: cont'd

TABLE 3(C): STIFFNESS OF CUSHION USING FOSTERLON AS SPECIFIED BY L.B. FOSTER CO. (8)

Hammer	Size (Nominal)	Thickness in.	Stiffness kips/in
Kobe K13, K22 K25	17"	3.5	62,000
Kobe K32, K42, K35 K45	24"	3.5	126,000
Vulcan 1, 06, 106 50C, 65C	11.5" dia.	6.5	12,200
Vulcan 2, 30C	10.5" dia.	6.5	10,200
Vulcan 0, 0R, 08, 010, 80C	14.0" dia.	8.5	14,000
Vulcan 014, 016, 140C	17.5" dia.	7.0	26,700
Vulcan 020, 200C	19.75" dia.	7.5	31,900
MKT DE30, DA35	19" dia.	2.5	88,400
MKT DE40	23" dia.	2.5	130,000

TABLE 3(D): OAK CUSHIONS AS RECOMMENDED FOR KOBE HAMMERS⁽⁹⁾

Hammer	K13	K25	KC25	K35	KC35	K45	KB60	K150
Thickness (in)	4	6	6	6	6	8	8	11
Stiffness (kips/in)	102,000	100,000	100,000	141,000	141,000	138,000	183,000	242,000

TABLE 3: cont'd

TABLE 3: FOOTNOTES

- (1) Taken from Table A p. 14 of Manual of TTI, 1973.
- (2) As specified by manufacturer.
- (3) Determined by New York DOT
- (4) FEC ... Foundation Equipment Co., Newcomerstown, Ohio. Distributor of DELMAG hammers; manufacturer of pile driving accessories.
- (5) Based on an $E = 1000$ ksi (load parallel to grain).
- (6) Based on an $E = 560$ ksi as specified by manufacturer.
- (7) Does not include adaptors.
- (8) L.B. Foster Company, Pittsburgh, Pa.,
Data taken from L.B. Foster Brochure.
- (9) Taken from Specifications as supplied by the L. E. Foster Co.,
using $E_{\text{oak}} = 1420$ ksi.

3.5 (a) Pile Description
 (Card No. 5,000)
L pile, A pile top
E pile top, W pile
top

(For pile material data see Table 4.)
 i.e. Pile length, and for the pile top cross-sectional area, elastic modulus and specific weight, respectively. In the simplest case which is a uniform pile these four quantities are sufficient to describe the pile.

Table 4: Pile Material Properties

Material	Elastic Modulus EP ksi	Specified Weight RP lbs/ft ³	Wave Speed c ft/msec
Steel	29,500-31,000	492	16.7-17.0
Concrete	3,000-7,000	150	9.6-14.7
Timber	1,300-2,500	40-70	Dependent on Ep and Rp

(b) For NCROSS > 0:
Card No.s 5.101,...
XP(I), AP(I), EP(I),
WP(I)

i.e. depth, cross sectional area, elastic modulus and specific weight, the latter three at the XP(I) depth. The program interpolates properties linearly between consecutive XP(I) values. Stepwise changes of cross section (or changes of material) have to be identified by two cards with identical XP(I) values first giving the pile properties just above the change and second just below that section.

Any combination of linear with straight sections and with any type of material is possible.

The program recognizes the last set of input values by comparing XP(I) with Lpile. It is, therefore, imperative that the last set of XP(I), AP(I), ..., specifications starts with an XP(I) value that is greater than or equal to Lpile.

Example 1: A steel pile (EP = 30000 ksi, WP = 492 lbs/ft³) consists of 45 feet of uniform pipe with a cross sectional area of 10 square inches.

Below is added a tapered 45 foot pipe that decreases from 10 to 7.5 in², linearly. The input would be as in Figure 3(a).

Example 2: A concrete pile (EP = 5000 ksi, WP = 150 lbs/ft³) 14 by 14 inches square is 60 feet long and has a steel tip of 15 feet length and 15.5 square inch cross section. The data for this case is given in Figure 3(b).

3.6 Soil Information
Hammer Optional
Input
(Card No. 6,000)

The necessary soil information is basically supplied in terms of the commonly known wave equation parameters. The following need to be specified (for soil damping values see Table 5).

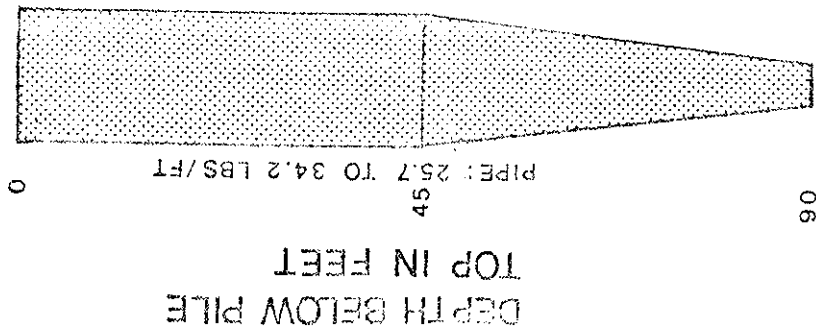
Table 5: Recommended Damping Values

	Smith*		Case	
	Skin	Toe	Skin	Toe
Gravel	0.05	0.05-0.1	0.1-0.2	0.1
Sand	0.05	0.1-0.2	0.2-0.4	0.1-0.2
Silt	0.10-0.15**	0.1-0.5**	0.5-1.5	0.4-0.8
Clay	0.2	0.01-1.0**	1.0-2.0	0.5-0.8

*These values are based on the usually employed distribution of static resistance forces.

** The TTI recommendations are the lower bound values.

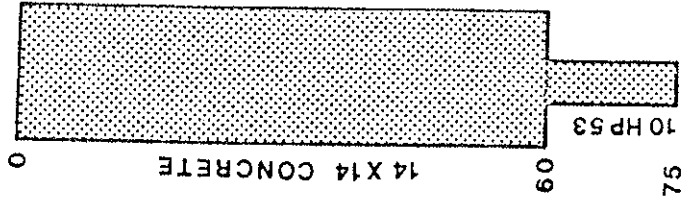
(A)



DATA

XP	AP	EP	WP
FT	IN ²	KSI	LBS/FT ³
0.0	10.0	3000.0	492.0
45.0	10.0		
90.0	7.5		

(B)



DATA

XP	AP	EP	WP
FT	IN ²	KSI	LBS/FT ³
0.0	196.0	5000.0	750.0
60.0	196.0		
60.0	15.5	30000.0	492.0
75.0	15.5		

FIGURE 3: (A) EXAMPLE 1 (B) EXAMPLE 2 FOR NON-UNIFORM PILE DATA INPUT

Quake Skin, Quake
Toe, Damping Skin,
Damping Toe, RULT
ESOIL

Quakes are to be specified in inches (usually 0.1). Damping parameters can be given in either of two ways. If ISMITH (Card No. 2.000) was chosen as 1, meaning that Smith damping is to be employed, the damping values have the usual s/ft dimension. If ISMITH was equal to zero then the damping values are considered dimensionless (for a conversion from Smith to viscous damping see the report). RULT is the total static capacity in tons for which an analysis is to be performed. If RULT is zero (or not specified) various values (at most 10) will be assumed by the computer and all analyses will be performed unless the blow count exceeds 1200 blows/ft. If RULT is given as a negative value (e.g. -1.0) an additional card (No. 9.000, Section 3.9) with up to 10 RULT values, (in tons), must be supplied.

ESOIL is a coefficient of restitution that is applied to all static soil resistance calculations. ESOIL left blank will cause the computer to use 1.0 (usual case).

Stroke, Efficiency
Steam Pressure and
Reaction Weight

The Stroke value (in feet) is used as a starting value in a diesel hammer analysis (if IOSTR = 1 or = -1 then only this stroke will be analyzed). For air/steam hammer it replaces the -usual- stroke value either program or user supplied.

The Efficiency can be left blank which causes the use of a hammer efficiency of 0.95 for diesel hammers or the one specified in the hammer data for air/steam hammers.

The Steam Pressure (in psi) is disregarded for all except differential acting steam hammers. If it is chosen greater than the maximum allowed by the manufacturer, it is ignored. If not supplied, the maximum value specified by the manufacturer is used.

The Reaction Weight (in kips) is only used for closed end diesel or differential acting air/steam hammers. It replaces - if specified - the values supplied by the data file or user.

3.7 Skin Friction Dis-
tribution
(Card No.s 5.401,
...)

For skin friction distributions that do not conform to Table 2, i.e., when ITYS = 0, the following cards - at most 20 - can be used. These specifications consist of pairs of values.

XP(I), DIS(I)

XP(I) is the depth in feet (in increasing order) and DIS(I) is a relative, dimensionless quantity. Only the skin friction is affected by the distribution choice. The amount of skin friction is a certain percentage of the total ultimate resistance RULT and was specified by IPERCS. As for the pile data on Card No.s 5.101, ..., it is imperative that the last depth value, XP(I), be greater than or equal to Lpile.

Example 1: The data for a triangular distribution over the bottom half of a 60 foot pile is given in Figure 4(a). Note that DIS(3) could have been given as any other value (it is only relative). Also, as in the case of pile specifications (Card No. 5.101, ...) the end of distribution specifications is recognized by the appearance of an XP value which is equal to or greater than Lpile.

Example 2: For the pile of Example 1 the soil resistance is of rectangular shape from 20 to 30 feet and then starting with one-half of the value above increasing linearly to twice that value at 50 feet. It is to be zero below that point. The required data is given in Figure 4(b). Note that 20 is the maximum number of pairs of values (or cards).

An additional example showing the computations performed by the computer to derive the SU(I) values from DIS(I) is shown in Figure 5.

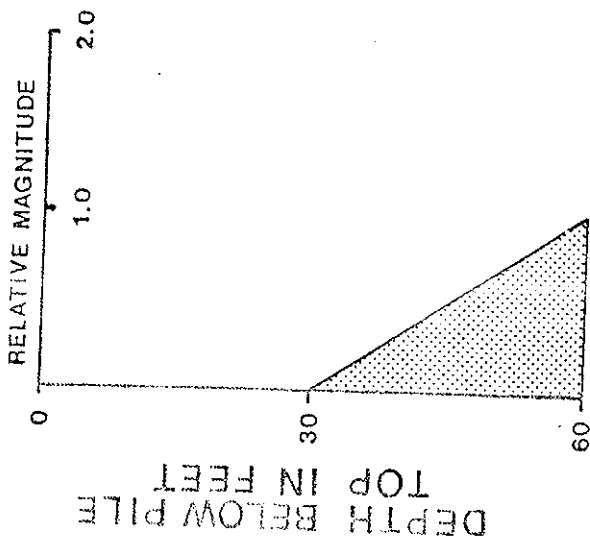
3.8 Plotter Infor-
mation
(Card No.s 7.000
and 8.000)

Two cards specifying the scale and the quantities to be plotted are to be given if IOUT > 9. A discussion of this data is given in Sections 4.13 and 4.15. The most important plotter output is provided and a scale is automatically chosen, if the two cards are left blank.

3.9 RULT Values
(Card No. 9.000)

For RULT < 0 on Card No. 6.000 up to 10 RESULT (I) values can be specified.

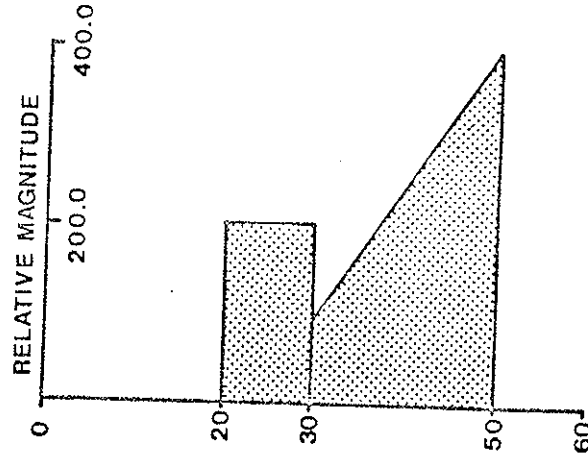
(A)



DATA

XP	DIS
0.0	0.0
30.0	0.0
60.0	1.0

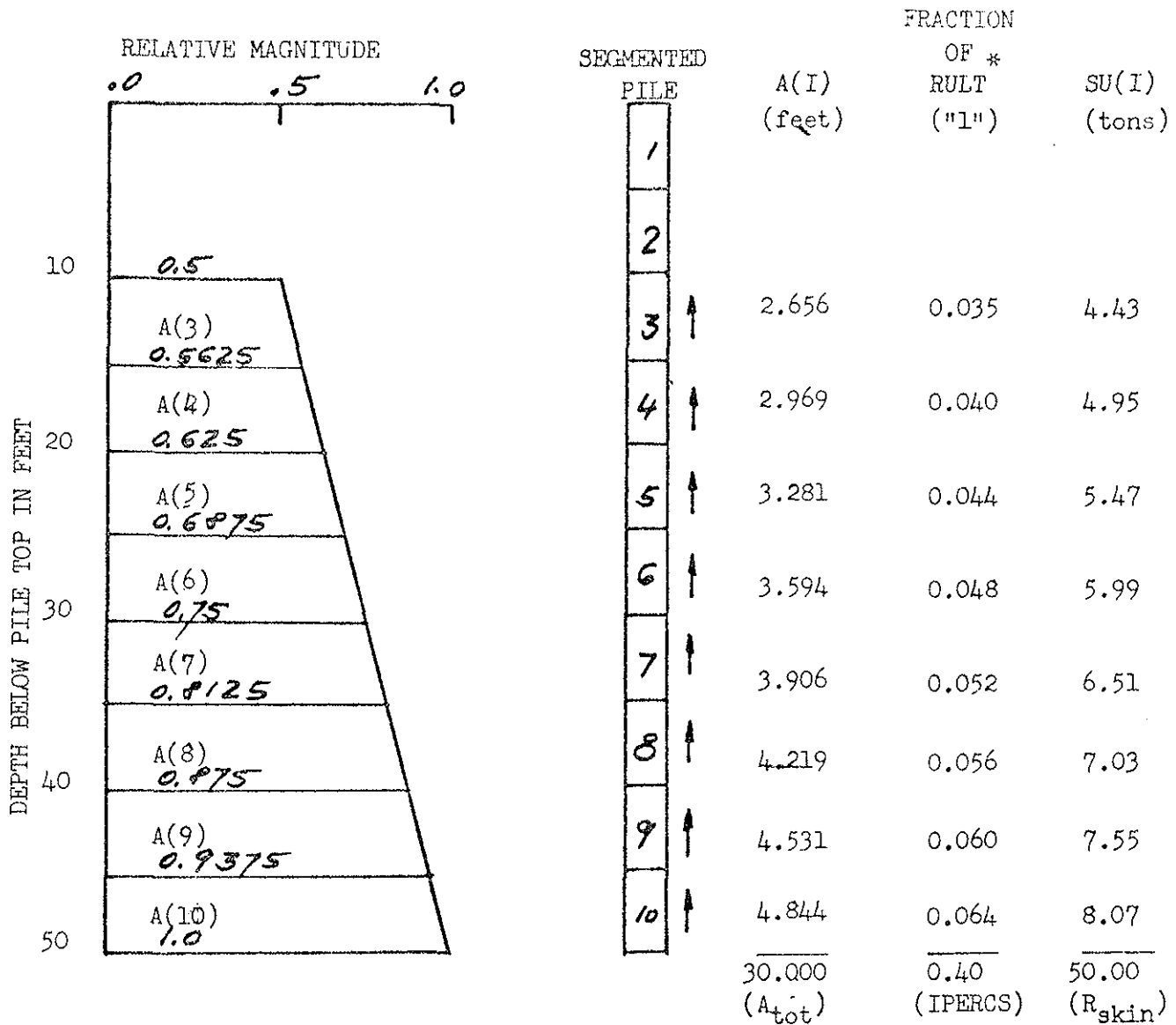
(B)



DATA

XP	DIS
0.0	0.0
20.0	0.0
20.0	200.0
30.0	200.0
30.0	100.0
50.0	400.0
50.0	0.0
60.0	0.0

FIGURE 4: (A) EXAMPLE 1 (B) EXAMPLE 2 FOR INPUT OF SKIN FRICTION DISTRIBUTION



* FRACTION OF RULT = $\frac{A(I)}{A_{tot}} \left(\frac{IPERCS}{100} \right)$

SU(I) = (FRACTION OF RULT) x (RULT)

EXAMPLE FOR THE COMPUTATION OF ELEMENT RESISTANCE FORCES:

IPERCS = 40 (%), RULT = 125 (tons)

N = 10 XPT = 50 (feet)

Segments are of equal length (5 feet)

Total Skin Resistance = 125 (40) / 100 = 50 (tons)

Total Area under resistance curve = $(0.5 + 1.0) \frac{1}{2} (50 - 10) = 30.0$ (feet)

Toe Resistance = SU(11) = $(100 - 40)(125) / 100 = 75$ (tons)

FIGURE 5: EXAMPLE COMPUTATION FOR THE DISTRIBUTION OF SKIN RESISTANCE FORCES. (THIS COMPUTATION IS USUALLY PERFORMED BY THE COMPUTER)

RESULT (1), RESULT
(2), ...

RESULT(I) is the I-th ultimate resistance (RULT) in tons. The values should be given in increasing order. The first occurrence of a zero (or blank field) terminates the search for further RULT values. Also, no additional analysis is performed if the blow count of the last RULT value has exceeded 1200 blows/foot.

CHAPTER 4

COMPLETE INPUT

The following data can be specified in addition to the short input.

The following cards have to be inserted according to card number sequence - see also complete input form. All values are real numbers (with decimal points) inserted in fields of 8 spaces except in Card No.s 6.501, ..., 8.000 and 8.101.

- 4.1 For IPEL = 2:
STP(I), I = 1, N
Card No.s 2.101, ... Pile segment stiffnesses in kips per inch for all N elements. Since 10 values are read per card there might be up to 10 cards. (N = 98 = maximum) if the I-th value is the same as the (I-1) - th, it can be left blank. The first value must be given.
- 4.2 For IPEL = 2:
PM(I), I = 1, N
Card No.s 2.201, ... Pile segment weights in kips for all N elements. Otherwise as for STP(I).
- 4.3 For IPEL ≠ 0:
ALPH(I), I = 1, N
Card No.s 2.301, ... Relative segment lengths (any relative - real - numbers). Otherwise as for STP(I). (Note, ALPH(I) has to be provided even though STP(I) and PM(I) are given since it is needed to determine the spacing in the 3-dimensional plot).
- 4.4 For IHAMR = 0 NAME 8 alphanumeric characters describing the hammer.
Card No. 5.201
HM(1), consecutive pairs of hammer weights in kips
STH(1) and stiffnesses in kips per inch.
.
.
.
HM(5) If there are less than 5 values to be specified then the others are left blank.
- 4.5 For IHAMR = 0 STH(5), Continuation of ram weights and stiffnesses. Note,
Card No. 5.202 HM(6) that there will be always one stiffness less
STH(6), than weights.
HM(7)

M The number of ram segments to be used (given as a real number, e.g., 3.0). $M \geq 3$ for Diesels.

HM(M+1) Anvil weight in kips (diesel only).

STH(M) Combined anvil and last ram spring stiffness (diesel only).

TDEL Combustion delay in seconds (diesel hammers only). For a description of this value see 3.4.

TS The duration of fuel injection or ignition in seconds. For atomized injection (DINJ > 0 on Card No. 5.204) TS should correspond to the duration of injection. For regular injection 0.001 sec. is a reasonable value.

4.6 For IHAMR = 0 VFIN Combustion chamber volume in cubic inches.
Card No. 5.230 (Diesel hammers)

DEPIB Distance between exhaust ports and anvil top in inches. (Diesel hammers)

ARAM Inside cross section of cylinder in square inches. (Diesel hammers)

P1, P2, .. Up to five maximum combustion pressures as
..., P5 measured. (Diesel and double acting air/steam hammers). For diesel hammers IFUEL = 1, 2, ..., 5 will cause these pressures to be used as maxima. For Double Acting air/steam hammers P1 is the pressure at which hammer is rated.

EFFICY Hammer efficiency. (All hammers) (Default 0.95). Will be overridden by the value on Card 6.000.

STRM Maximum stroke (diesel) or stroke to be used (air/steam) in inches.

4.7 For IHAMR = 0 EXPP Exponent for the expansion process (diesel
Card No. 5.204 hammers).

ART Bounce chamber cross sectional area in square feet (closed end diesel hammers).

DEPBE Distance between bounce chamber exhaust ports and cylinder top in feet (closed end diesel hammers).

DSF Distance between compression tank ports and cylinder top in feet. (Closed end diesel hammer with compression tank). For vacuum chamber hammers DSF is the distance between ram bottom and anvil at which the pressure in the vacuum hammer starts to decrease.

DBBT Distance between anvil and cylinder top minus ram length in feet (closed end diesel hammers).

POWSCA 0.0 most cases. 1.0 where power scavenging is employed (e.g. Linkbelt) (diesel hammers).

RWH Reaction weight of closed end diesel or air/steam hammer in kips. Will be overridden by the value of Card 6.000.

EXPB Exponent for bounce chamber expansion - compression (usually 1.4) for closed end diesel hammers.

DINJ Distance of ram bottom from anvil at which atomized fuel is injected in feet. Should be zero for non-atomized fuel (diesel hammers).

VCT Volume of compression tank in cubic feet (closed end diesel hammers). For vacuum chamber hammers it is the volume of the chamber when pressure starts to decrease.

RAM For vacuum chamber hammers only the ram weight in kips (serves as identifier for this hammer type).

For IHAMR = 0: AM(I), Assembly segment weights (kips) and stiffnesses
 Card No. 5.205 STA(I) (kips/inch) for up to three assembly segments.
 (Air/steam hammers only).

MA Number of assembly segments (MA = 2 or 3) given as a real number, e.g., 2.0. (Air/steam hammers only).

- | | | |
|------|--|---|
| | <u>ITYPH</u> | Hammer type |
| | | 1.01 open end diesel hammer |
| | | 2.01 closed end diesel hammer |
| | | 3.01 air/steam hammer |
| 4.9 | For ITYS < -1:
Card No.s
6.101,... | <u>QS(I)</u> ,
<u>I = 1</u> ,
<u>N+1</u> |
| | | Soil quake in inches for all elements plus pile point, up to 99 values may be required. (Otherwise as for 2.101). |
| 4.10 | For ITYS < 0:
Card No.
6.201,... | <u>SJ(I)</u> ,
<u>I = 1</u> ,
<u>N+1</u> |
| | | Soil damping parameters. For ISMITH = 1 in s/ft, the usual values as given in Table 5 can be used.
For ISMITH = 0 as a fraction of the EA/c value of the corresponding segment. Otherwise as for 6.101. If a zero value is to be specified and the previous values were non-zero a 0.0002 should be given. Note that Case damping values are not related to static soil resistance. Therefore, they can be specified even though the static resistance is zero. However, the values recommended in Table 5 represent the <u>sum</u> of all skin damping parameters. Input on Card No. 6.201 represents the constant for each individual element. For example: For N = 15, a 0.2 Case skin damping factor corresponds to 0.2/15 = 0.0133 for each element if a uniform damping distribution is desired. |
| 4.11 | For ITYS < -1:
Card No. 6.301,
... | <u>SU(I)</u> ,
<u>I = 1</u> ,
<u>N+1</u> |
| | | Relative magnitudes of ultimate static soil resistance values. Note that this input overrides IPERCS, i.e., the point resistance is determined using SU(N+1). The program normalizes the SU(I) values. Therefore, only relative magnitudes need to be given. As in 6.201 a blank field indicates that the previous value is to be used. Thus, zero input should be made as 0.0002. At least one value should be greater than zero. |
| 4.12 | For ISPL > 0:
Card No.
6.501,... | <u>J</u> ,
<u>SPLICE</u>
<u>(J)</u> |
| | | The number of cards to be provided is equal to ISPL. On each card the element number J (a right justified integer in a field of 4 spaces) and the corresponding splice value SPLICE(J) (a real number with 8 space field). SPLICE(J) is the largest tension force (negative) that spring J can transmit, if SPLICE(J) = 0.0 or < -0.5 (kips). SPLICE(J) specified between -0.5 and 0.0 (feet) means a <u>slack</u> value (0 tension force within an extension of SPLICE(J) feet). See Example 5.7. |

The following two cards are always required when plots are desired; they both may be blank.

4.13 For IOUT > 9 YMAX
Card No. 7.000

The maximum value to which the positive axis in the plotter routine should be drawn. For example, if YMAX = 100.0 and if forces were plotted then the positive axis would extend to 100 kips (often four inches). There would be no problem if YMAX would be slightly smaller than the actually occurring maximum value. However, if YMAX was accidentally set too small by a full order of magnitude or more the program would automatically change this scale. Note that YMAX is dimensional and should correspond to the output variables selected by IOUT. Thus, the dimensions for YMAX are

 kips for IOUT = 11 or 21
 ft/s for IOUT = 12 or 22
 ksi for IOUT = 13 or 23
 g's for IOUT = 14 or 24
 inches for IOUT = 15 or 25
 psi for IOUT = 16 or 26 and ISEG(1) = 0
 kips for IOUT = 16 or 26 and ISEG(1) = 1
 ft/s for IOUT = 16 or 26 and ISEG(1) = 2
 kips for IOUT = 16 or 26 and ISEG(1) = 3

(ISEG(I) is specified on the next card).
If YMAX is not specified then the scale is automatically selected.

4.14 For IOUT > 9: ISEG(I),
Card No. 8.000 I = 1,
 15

Integers in fields of 4 spaces.
Selects pile segments to be plotted as a function of time. If ISEG(I) is specified equal to J then the

 J=1 corresponds to ram bottom

 J=2 corresponds to anvil

and J=3 through J=15 correspond either to the pile segments specified by INP(1) through INP(13) (next card) or to the automatically selected INP(I) values. An example will be given below.

If IOUT = 16 or 26 then the variable vs. time plot would be combustion pressure for ISEG(1) = 0, pile top force and pile bottom force, for

ISEG(1) = 1, pile top and bottom velocity
for ISEG(1) = 2, and pile top force plus the
proportional pile top velocity (multiplied by
EA/c) for ISEG(1) = 3. If ISEG(2) through
ISEG(4) are also specified all of these plots
can be obtained.

Viz ISEG(1) = 0, ISEG(2) = 1,
ISEG(3) = 2, ISEG(4) = 3

plots all four combinations. Note that ISEG(5)
through ISEG(15) are ignored if IOU = 16 or 26.
If all ISEG(I) values are zero then the ISEG(3),
i.e., the pile top quantity in the usual case
will be selected for plotting except for IOU
= 16 or 26 where combustion pressure would be
selected. Examples will be given below.

4.15 For IJJ = 1 and IOU
≠ 6, 16 or 26:
INP(I), I = 1, 13
Card No. 9.101

Integers in fields of 4 spaces.

At most 15 variables can be printed due to the
width limitations of the paper. Since two
hammer variables are usually included a se-
lection of variables has to be made if $N > 13$.
This is conveniently done automatically (IJJ =
0). If the user wants to specify certain ele-
ments, he can do this by specifying the 13 pile
segments for which he desires output. For ex-
ample, if $N = 40$ a uniform spacing would be
given using the following string of numbers:

1, 4, 7, 10, 13, 16, 20, 24, 27, 30, 33, 36, 40
which would include the pile top (1) and bottom
(40).

Examples: (a) IOU = 3, N = 10, IJJ = 1

Since no plot is desired, YMAX and ISEG(I) would not be read.
Because of IJJ = 1 INP(I) would have to be specified which
would be: 1, 2, 3, ..., 9, 10 and three blank fields. The
output would be stresses in the ram, anvil and in all pile
elements. The INP(I) values thus specified would be the
same as the automatically selected ones.

(b) IOU = 15, N = 10, IJJ = 0

YMAX may be left blank and ISEG(I) may be chosen as 3, 12, 0,
0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, in which case all 10 pile

displacements plus the two hammer displacements would be printed and a plot vs. time be constructed showing the pile top and pile bottom displacement as a function of time.

(c) IOUT = 16, N = 50, IJJ = 0

YMAX and ISEEG(I) may be left blank. In addition, INP(I) is not read and IOUT = 16 calls for output of a variety of variables. The result would be a plot of combustion pressure vs. time.

(d) A pile is 120 feet long and N is given to be 16. A plot and print of stresses at pile top, middle and bottom are desired with the fixed scale of 50 ksi = maximum. Set IJJ = 1, IOUT = 13 (or 23 which would also provide a three dimensional plot of stresses), YMAX = 50.0 and ISEEG(I) = 3, 10, 15, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, 0, (I = 1, 15) INP(I) = 1, 2, 4, 5, 6, 7, 8, 9, 10, 12, 14, 15, 16 (I = 1, 13). This would produce a print of stresses in ram and anvil in addition to those in pile springs identified by INP(I). The stress vs. time plot would contain the stresses in pile spring No. 1, 9 and 16.

CHAPTER 5

ADDITIONAL INPUT EXAMPLES

5.1 Specifying a Soil Plug

Suppose it is intended to describe a steel pipe pile (A_S, E_S, W_S) which was driven open ended and within which a soil plug has been formed. A plug exists when the soil does not move relative to the pile. It may be assumed that the soil has a negligible stiffness compared to the pile. Thus, the pile area and modulus specified must yield the pile stiffness and, therefore, only WP is affected. This value can be calculated as follows:

$$WP = W_{\text{soil}} \frac{A_{\text{plug}}}{A_S} + W_S$$

where WP is the unit weight of steel plus soil combined, W_{soil} is the unit weight of the plug and A_{plug} is the cross sectional area of the plug.

For a 24 inch outside diameter pipe with 60 feet length, 1 inch wall and 5 foot plug, one would obtain

$$A_S = 23 (1) = 72.27 \text{ in}^2$$

$$W_S = 492 \text{ \#/cu ft}$$

$$E_S = 30,000 \text{ ksi}$$

$$W_{\text{soil}} = 110 \text{ \#/cu ft (assumed)}$$

$$A_{\text{plug}} = 22^2 / 4 = 380.13 \text{ in}^2$$

and therefore

$$WP = 110(380.13)/72.27 + 492 = 1071 \text{ lb/cu ft}$$

The input would be:

Card #	Lpile/XP	AP	EP	WP
5.000	60.	72.26	3000.0	492.0
5.001	55.	72.26		
5.002	55.	72.26		1071.0
5.003	60.	72.26		1071.0

5.2 Specifying a Pile Point

Suppose the input is to be given for a 10HP42 ($AP = 42/3.42 = 12.28 \text{ in}^2$) pile of 40 feet length with a 75 pound pile point that extends 3 inches below the bottom of the H-pile. Since the effective cross sectional area of the pile point, A_{point} , is not known it is sufficiently accurate to find

$$A_{\text{point}} = \frac{1728 \text{ WT}_{\text{point}}}{(W_{\text{point}}) (T_{\text{point}})} \frac{(\text{in}^3 / \text{ft}^3)(\text{lbs})}{(\text{lbs}/\text{ft}^3)(\text{in})}$$

with WT_{point} being the total weight, W_{point} the unit weight, T_{point} the maximum thickness of the pile point.

In the given example one obtains:

$$A_{\text{point}} = 1728(75)/(492(3)) = 87.80 \text{ in}^2$$

The input would be

Card #	Lpile/XP	AP	EP	WP
5.000	40.25	12.28	30000.	492.
5.001	40.0	12.28		
5.002	40.0	87.8		
5.003	40.25	87.8		

5.3 Specifying a Single Acting Air/Steam Hammer

This is the simplest hammer to specify and, incidentally, a drop hammer would be specified in the same manner.

As an example, consider a hammer with 5.000 kips ram weight, efficiency 0.8, normal stroke 36.0 inches. The assembly is to be disregarded.

Since the air/steam ram is very compact its stiffness is extremely high and can be neglected. Thus, the necessary input is as follows:

Card #

```
5.201      NAME = AIRSTEAM, HM(1) = 5.0
5.202      M = 1.0
5.203      EFFICY = 0.8, STRM = 36.0
5.204      blank
5.205      ITYPH = 3.001
```

Should the assembly be considered the following must be determined:

Weight of the assembly top (supported by columns), weight of hammer base, stiffness of all columns. Suppose the assembly top weighs 3.0 kips and the base 2.0 kips. There are four columns of 4-1/2 inch diameter and 90 inch length. Therefore, the assembly stiffness is

$$STA = 4(4.5^2) \pi / 4 (30000) / 90 = \frac{21206}{\text{kips/inch}}$$

The two assembly springs have twice this stiffness. Thus, the following data is to be given on card 5.205: AM(1) = 3.0, STA(1) = 42212., AM(2) = 2.0, STA(2) = 42212., MA = 2.0, ITYPH = 3.01.

5.4 Specifying an Open End Diesel Hammer

In this example a hammer is considered having the following properties:

	Ram	Anvil
Weight:	2,750	810 lbs
Length:	95	19 inch
Diameter:	12.5	12.5 inch

Compression ratio 12:1

Chamber volume 120 cu in

Combustion delay approximately 2 Msec.

Maximum combustion pressure on full throttle 1150 psi
(Values for other throttle settings not known).

Maximum stroke 102 inch

Expansion exponent not determined, assume 1.3

No power scavenging

No atomized fuel injection

The ram cross sectional area is

$$A_{RAM} = \frac{\pi}{4} 12.5^2 = 122.7 \text{ in}^2$$

The volume displaced by the ram is

$$V_R = 12(120) - 120 = 1320 \text{ cu in}$$

The distance between exhaust ports and anvil is therefore (assuming cylinder area equal ram cross section):

$$DEPIB = \frac{1320}{122.7} = 10.76 \text{ in}$$

Choosing three ram elements (equal length) their weights are

$$HM(I) = \frac{2.75}{3} = 0.917 \text{ kips, } I = 1, 2, 3$$

The segment stiffnesses are ($E = 30000 \text{ ksi}$):

$$STH(I) = 3 \frac{30000(122.7)}{95} = 116000 \text{ kips/inch, } I = 1, 2$$

The anvil's stiffness is

$$STA = \frac{30000 (122.7)}{19} = 194000 \text{ kips/inch}$$

Combining STA with the ram bottom stiffness one obtains ($M = 3$):

$$STH(3) = \frac{194000 (116000)}{194000 + 116000} = 72600 \text{ kips/inch}$$

The corresponding input is:

Card #

5.201: NAME = HYPOTHET, HM(1) = 0.917 = HM(2) = HM(3), STH(1) = 116000. = STH(2)
5.202: M = 3.0, HM(M+1) = 0.81, STH(M) = 72600., TDEL = 0.002
5.203: VFIN = 120., DEPIP = 10.76, ARAM = 122.72, P1 = 1150.0, STRM = 102.0
5.204: EXPP = 1.3
5.205: ITYPH = 1.01

A computer run demonstrating the use of this hammer data is discussed in Example 7.4.

5.5 Additional Specifications for a Double Acting Air/Steam Hammer

If the hammer of Section 5.3 were double acting the following additional specifications would be needed:

Pl (maximum operating pressure, i.e., at lift-off)
RWH (hammer reaction weight)

Note that these two values are related by the effective cylinder area, A_e , during downstroke ($P_l = RWH/A_e$). For example, if the maximum rated energy for the hammer of Section 5.3 was given as 27,000 ft-lbs then 12,000 ft-lbs would be from the pressure on the downward stroke. The reaction weight is equal to the maximum pressure force (at rated pressure):

$$\text{Pressure force} = \text{Energy/stroke} = 12/3 = 4 \text{ kips}$$

The equation used for calculating reduced equivalent strokes in the program assumes that uplift occurs at the rated pressure force. If $A_e = 50 \text{ in}^2$

$$RWH = 4.0 \text{ kips}$$

$$P_l = (4.0/50) 1000 = 80 \text{ psi}$$

Note, assembly weights and reaction weight need not agree numerically.

5.6 Additional Specifications for Closed End Diesel Hammers

Suppose that the hammer of Section 5.4 had a closed cylinder top with the following known data:

Ram top starts to compress bounce chamber when the ram has moved 5 inches up from the anvil. The ram is uniform and there is a distance of 4-1/4 feet that the ram can move upward from the time the bounce chamber compression begins before it hits the top. No compression tank and, therefore, safety chamber exists.

Thus the following can be determined:

$$\text{ART} = \text{ARAM} = 122.72/144 = 0.8522 \text{ ft}^2$$

$$\text{DEPBB} = 4.5 \text{ ft}$$

$$\text{DBBT} = \text{DEPBB} + 5 \text{ inch}$$

$$= 4.5 + 5/12 = 4.9167 \text{ ft}$$

(Of course the maximum stroke would be smaller than in Section 5.4).

Assuming an adiabatic bounce chamber compression/expansion ($\text{EXPB} = 1.4$) all of the additional data to be given on Card 5.204 is found. On Card 5.205 the hammer type is to be changed to 2.01.

5.7 Specifying Slack or Splice

Suppose that a 100 foot pile had two different splices. First, 32 feet above the pile bottom a connection was made which cannot transmit more than 100 kips tension force. 66 feet above the toe a second connection allows a two inch (0.167 feet) extension before it can transmit tension.

In order to properly assign JSPLICE values, N should be specified, say $N = 20$. Then the sixth spring would correspond to the upper and the 13-th spring to the lower splice.

The input would be

Card No.	Element	
	J	SPLICE(J)
6.501:	6	-0.167
6.502:	13	-100.

CHAPTER 6

OUTPUT DESCRIPTION

In this section a discussion will be given in input check print and headings used in the printed output. Also a discussion of possible message printout will be given.

6.1 Input Check

The first page always contains listings of important input values. After the job name the pile description is printed as it was given on Cards 5.000 and 5.101, ...

For air/steam hammers the following is added: The stroke actually used, efficiency and derived impact velocity, for double acting air/steam hammers also the maximum and actual pressure and ram and reaction weights. Next, the hammer model including cap and cushion are printed. The headings are self-explanatory.

Under the heading "PILE PROPERTIES" follows a summarized description of the pile and its material (pile top properties). Then for each pile segment the following values are given:

Weight	...	the weight of the segment in kips
Stiffness	...	the stiffness of each spring kips/inch
Pdamp	...	the damping constant for each <u>pile</u> dashpot in kips/ft/sec
Splice	...	the minimum force value (compression is positive) which can be transferred by the springs in kips.
Soil-S	...	the static soil resistance force acting at an element as a fraction of the total static capacity (RULT)

Soil-D ... the soil damping parameters either in kips/ft/sec (Case damping) or in sec/ft (Smith's damping)

Quake ... the quake of each elasto-plastic soil spring in inches

L.B.T. ... the length below the top to which the bottom of a segment extends

After the print out for all N elements the three soil properties Soil-S, Soil-D and Quake for the pile toe and the soil's coefficient of restitution are printed. For a negative IPERCS a message is then printed indicating that the skin friction is to be kept at a constant value. Next follows a recap of options and specifications:

PHI ... the ratio of the critical to the time increment used

S-Damping ... whether Smith's or whether viscous damping was used

P-Damping ... the pile damping parameter in percent of critical

J SKIN ... the soil skin damping parameter (dimensions as in the pile table under Soil-D)

J TOE ... the soil toe damping parameter (dimensions as for J SKIN)

TEMAX ... the maximum analysis time allowed in milliseconds (input value)

IFUEL, i.e., the fuel setting

IOUT, i.e., the output option

Soil Dist. No., i.e., ITYS

IOSTR, i.e., stroke option

RWT (kips), i.e., the hammer reaction weight actually used

TDEL (sec), i.e., the combustion delay (input value)

EFFICIENCY; i.e., the efficiency used in the case of diesel hammers

TIME INCR. (MS) the analysis time increment

6.2 Output

After the input recap a heading is given indicating the total static soil resistance analyzed and the portion acting at the bottom. For print options which are not 0, 10 or 20 selected variables are then printed vs. time. The selection of variables was discussed in the input information. The headings used indicate the type of variable and its dimension (except for the 6, 16, 26 option), the hammer variables (RAM M means the ram bottom), and the pile variables. TOP stands for the first segment while the following numbers refer to the corresponding pile segments.

JP is a time counter and TIME is in milliseconds. TIME is usually started one millisecond before impact.

For the 6, 16, 26 option the following headings are used:

JP	...	Time counter
TIME	...	Time in milliseconds
P	...	Combustion pressure for diesel hammers
SEA	...	Distance between assembly and cap for air/steam hammers
D	...	Indicating displacement for RAM, ANV (any11), TOP (pile top), MID (at midlength) and TOE (pile bottom segment)
V	...	Indicating velocity with subscripts TOP, MID and TOE

F ... Indicating forces in the pile with subscript TOP, MID and TOE (for the first, middle and last pile spring, respectively)

SUM ST ... Is the sum of all activated static soil resistance

SUM DP ... Is the sum of all activated dynamic soil resistance

RT TOE ... Is the activated static toe resistance

In all cases a table follows with extreme values for each pile element and their time of occurrence. Time of occurrence is given in terms of JP (JP time the time increment is the time since the start of the analysis).

The headings in this table are ELEM for element or segment, FMAX and FMIN for maximum and minimum pile force, respectively; MINSTR and MAXSTR for minimum and maximum pile stress, respectively; VELMX AND DISMX for maximum pile velocity and displacement. The table contains the extreme values followed by the time counter in parentheses. Appended to these tables is the maximum energy transferred to the pile and the strokes analyzed. Note that the last stroke listed is the rebound stroke. Also for the fixed stroke run, the total change of maximum combustion pressure is given as necessitated for stroke convergence.

If more than one RULT value is analyzed further output starting with the new total and toe static resistance is given. After the last analysis and its respective table of extreme values a summary table is then printed containing the analysis No., RULT value, Blow Count, Stroke and the minimum and maximum (of all elements) stress. For all diesel hammers the blow rate (blows per minute) is also given. For all closed end hammers a number in

parentheses is appended to stroke indicating how many fuel reductions had to be taken and the maximum bounce chamber pressure is given.

6.3 Messages

A number of messages may or may not be printed during the execution of the program. Four different types of them exist.

- P ... Regular output giving certain indications about the program's performance.
- S ... Stop messages. Program termination message.
- I ... Interrupt message. An unexpected condition causes the program to skip to the next RULT or next data set.
- W ... Warning. The program's performance is not affected, however, the user is cautioned about the results.

Message No.	Message Type Subroutine	Message Text and Explanation
1	I Main	<u>Damping grtr. critical, T.I. required lss .01 msec. (xxx)</u> The damping parameter became so large that an analysis would have become infeasible. Damping should either be spread over more elements or reduced.
2	I Main	<u>Analysis was terminated as no permanent set resulted.</u> The condition encountered was no permanent set (infinite blow count) and no increase in stroke in the case of diesel hammers. Therefore, no additional RULT values were analyzed since they are assumed to be even larger.
3	P IPTN	<u>Insufficient plotter information, therefore values are assumed.</u> YMAX and/or ISEG were not specified (Card

numbers 7.000 and 8.000) although IOUW was greater than 9. Zero input was automatically assumed.

- 4 P Hammer will not run.
STARTC For the given parameters no impact occurred or the ram did not get sufficiently close to the anvil for a reasonable analysis. The current analysis is, therefore, skipped and a higher RULT value is analyzed if RULT was specified \leq 0 in Card No. 6.000. Otherwise the next set of data is read.
- 5 P Fuel setting reduced by ten percent.
Main Message given for closed end hammers that reach lift off and, therefore, require a reduced fuel setting.
- 6 P No Fuel Conversion.
Main After four fuel reductions lift off was still imminent. No new fuel reduction is analyzed. Results are in general acceptable.
- 7 P No Stroke Conversion.
Main After four analyses the rebound stroke was still different from the input stroke. No new stroke is analyzed. Depending on the last rebound stroke it should be decided whether an additional analysis would be necessary using the last stroke value as an input.
- 8 P Large Damping Requires New Time Increment =
Main xxx Msec.
The damping parameters had become greater than critical. To avoid numerical instability time increments were reduced.
- 9 W ***Caution Ram Might Blow Out***
Main For open end diesel hammers if the stroke exceeds the maximum stroke specified by the manufacturer. Note that at most the maximum stroke will be analyzed.
- 10 S Negative Volume in Vac Chamber.
VACHAM Similarly to message No. 18, this was caused by improper vacuum hammer specifications.

- 11 S
 IPTN Masses and Stiffnesses for First Elements are not Specified.
 Extended input requires at least the first segment specifications to be greater than zero.
- 12 S
 IPTN Pile Properties at Bottom of Pile not Specified.
 During the specification of AP, EP and RP an XP value had been encountered that was less than earlier ones. This is interpreted as out of range or as a later input card. Thus, pile bottom specifications were not found.
- 13 S
 IPTN Pile Masses or Stiffnesses Too Small, Check Input Info.
 At least one of the pile stiffnesses or masses were less than 100 kips/in or 10^{-7} kips/ft/sec², respectively. Either the pile specifications or the extended input were in error.
- 14 S
 HAEL Insufficient Hammer Info. IHAMR = xxx.
 The number of ram segments specified was less than one. Either the hammer data file, or the user supplied data cards were in error, or the user has specified a hammer identifier corresponding to an empty file.
- 15 S
 STARTC Negative Velocity in Start.
 Due to some abnormal condition the ram velocity was upwards at the exhaust ports. Check hammer data.
- 16 S
 IPTN Unknown Hammer Type, Sorry.
 IHAMR was specified greater than the available number of hammers on file.
- 17 S
 IPTN ***N was not set, extended input not possible***
 If IPEL is given greater than zero, N has to be specified to match pile segment specifications.

- 18 S Improper Condition Met When Determining
VACHAM Ram Velocity at Ports, VFALL = xxx.
Geometric properties were such for the
vacuum chamber hammer that a complex im-
pact velocity resulted.
- 19 S Hammer or Pile T.I. Equal to Zero, Check
IPTN Input.
Critical time increments were not sufficient-
ly large. Probably input error.
- 20 S Ram Still Moves Downward at End of Blow.
UP End of blow is here defined as the end of
the impact analysis. Causes may be in-
sufficient combustion pressure, extremely
low soil resistance or incorrect hammer or
pile data. It may also indicate that the
hammer does not run under the given conditions.
- 21 S Data Error.
IPTN Message was probably caused by improper data
format.

CHAPTER 7

WAVE EQUATION EXAMPLES

7.1 Open End Diesel Hammer - Determination of Bearing Graph

7.1.1 Situation

A 45 ton (design) pile is to be driven through a soft compressible layer into a dense, coarse sand with gravel. The contractor wants to use 10HP53 profiles and a D-12 hammer. He uses a standard 12x12 inch cap with 4-1/2 inches of conest.

7.1.2 Problem

Determine the blow count/bearing capacity relation.

7.1.3 Approach

Using a safety factor of two (2) the pile has to be driven to an ultimate capacity of 90 tons. A curve can be constructed for the desired range if capacities of 30, 60, 90 and 120 tons are analyzed.

7.1.4 Solution

The short input form is sufficient for solving this problem.

Card No. 1.000		Inserting a proper title
Card No. 2.000		IOUT = 10 for a printed and plotted summary only.
IJJ	...	leave it blank (no output vs. time)
IHAMR=2	...	for DELMAG D12 (see Table 2)
IOSTR	...	leave it blank (stroke iteration allowed)
IFUEL	...	leave it blank (full combustion pressure)
IPWL	...	leave it blank (computer determines pile elements)

N ... leave it blank (automatically determined)

ISPL ... leave it blank (no splices or slacks)

NCROSS ... leave it blank (uniform pile, neglect the existence of an endplate)

IBEDAM ... leave it blank (steel)

IPERCS ... leave it blank (endbearing only)

ISMITH ... leave it blank for Case damping input.

ITYS-6 ... for skin resistance distribution type 6 of Figure 1. (Note that since toe bearing only was assumed this distribution will pertain to the viscous damping parameters).

IPHI ... leave it blank (normal)

Card No. 3.000

Weight of cap95 kips (Table 3B - Conbest)

Stiffness of cap ... 21,000 kips/inch (Table 3B - Conbest)

Stiffness of Cushion ... leave it blank (no cushion)

Card No. 4.000

C.O.R. Cap .. 0.80 (Table 3A)

C.O.R. Anvil . 0.80 (a little more conservative than the usual 0.85)

C.O.R. Pile Top ... 0.80

C.O.R. Cushion ... leave it blank (no cushion)

TEMAX ... leave it blank (normal)

TDEL ... leave it blank (normal)

Card No. 5.000 L PILE ... 40.0 ft

 A PILE ... $53 \frac{144}{492} = 15.5 \text{ inch}^2$

 E PILE ... 30000 ksi (Table 4)

 W PILE ... 492 lbs/ft^3 (Table 4)

Card No. 5.101,... Do not use, since NCROSS = 0.

Card No. 6.000 Quake Skin .. 0.1 inch (standard)

 Quake Toe ... 0.1 inch (standard)

 Damping Skin.. 0.3 (Table 5 - Case Damping, assume sand governs)

 Damping Toe.. 0.15 (Table 5 - Case Damping, assume sand governs)

 RULT ... -1.0 (more than one value is to be input)

 Coeff. of Rest. of Soil. leave it blank (means 1.0)

 Stroke ... leave it blank (uses 5.0 feet as a starting value)

 Hammer Efficiency .. leave it blank (normal)

 Steam Pressure ... leave it blank (for A/S hammers only)

 Reaction Weight ... leave it blank (for closed diesel or double acting A/S hammers only)

Card No. 6.401, ... Do not insert (ITYS = 6)

Card No. 7.000 YMAX ... leave it blank (no plots vs. time)

Card No. 8.000 ISEG ... leave it blank (no plots vs. time)

Card No. 9.000 Ultimate Resistance... 30.0, 60.0, 90.0, 120.0

The input form is shown in Form 1. The output is shown in Figure 6 and Form 2.

7.1.5 Discussion of Results

It can be concluded that a design load of 45 tons (90 tons with FS = 2) requires a blow count of 40 blows/ft. The stroke should at this time be 5.7 feet. Then the maximum compressive stress would be 23.4 ksi.

7.2 Closed End Hammer - Driveability Study

7.2.1 Situation

A step tapered pipe pile (20.4 feet of 14 inch O.D., .203 inch wall, 23 feet of 11.5 inch OD, .219 inch wall, rest 10 inch OD, .219 inch wall) of 79 feet length with an additional 11 inch diameter toe plate of 1 inch thickness is to be driven to a depth of 74 feet. The soil consists of silt with some sand.

The soils expert has determined that the pile would be able to transfer as much as 160 tons load to the soil, mainly (70%) in skin friction. However, he expects a loss of 25% of this capacity during driving due to dynamic effects (the "static" capacity during driving would, therefore, be only 120 tons). He recommends to use high soil damping parameters as are appropriate for clays.

7.2.2 Problem

The contractor who must drive the pile to penetration wants to use an LB 520 hammer. He is not sure, however, whether this hammer will do the job

WAVE EQUATION ANALYSIS for PILES Short Input Form, page 1

1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0

1.000 Title (40 characters) ISPL NCROSS
 EXAMPLE 7.V OPEN END DIESEL HAMMER No. of 0 =

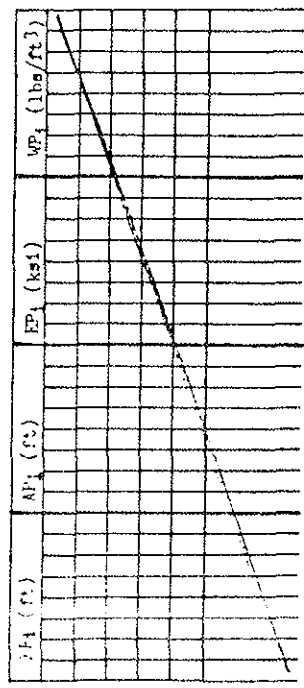
2.000 ISHMITH
 IOUT IJW IHAMR IOSTR IFUEL IPEL N Splices Uniform IBEEDAM IPERCS I-Smith ITYS IPHI
 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0 1 2 3 4 5 6 7 8 9 0

3.000 ISPL NCROSS
 Cap and Cushion
 Wt. Cap Stiff. Cap Stiff. Cushion
 (kips) (k/in) (k/in)
 0.75 21000 10000

4.000 ISPL NCROSS
 Coefficients of Restitution
 Anvil (Diesel) Cap (Diesel) Pile Top Cushion TDELT
 Cap(Air/Steam) Assembly(A/S) (msec.) (sec.)
 0.8 0.8 0.8 0.8 0.8 0.8

5.000 ISPL NCROSS
 L Pile A Pile Top E Pile Top W Pile Top
 (ft) (in²) (ksi) (lbs/ft³)
 40. 157.5 30000. 492.

If NCROSS = 1 and IPEL ≠ 2 only

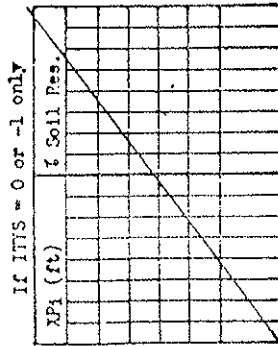


5.101
 5.102
 .
 .
 .
 5.119

WAVE EQUATION ANALYSIS for PILES Short Input Form, page 2

Quake Sidn (in)	Quake Toe (in)	Damping Sidn*	Damping Toe*	RULT (tons)	Coeff. of Rest. Of Soil	Stroke (ft)	Hammer Efficiency	Steam Pressure (psi)	Reaction wt. (klps)
6.000	.1	.3	.15	1.0					

* Damping Parameters - sec/ft for Smith; Dimensionless for Case



6.401
6.402
.
.
6.420
50

If IOUT > 9

IYAX	Plotter Scale
------	---------------

7.000

If ICUT > 9 only

ISST(I), I = 1, 2, ..., 15 (Plotted Variables)

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15
---	---	---	---	---	---	---	---	---	----	----	----	----	----	----

8.000

If RULT < 0 only:

50	60	70	80	90	100	110	120	130	140	150	160	170	180	190	200
----	----	----	----	----	-----	-----	-----	-----	-----	-----	-----	-----	-----	-----	-----

Ultimate Resistance - tons (at next 10 values)

9.000

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20
---	---	---	---	---	---	---	---	---	----	----	----	----	----	----	----	----	----	----	----

EXAMPLE 7.1 OPEN END DIESEL HAMMER

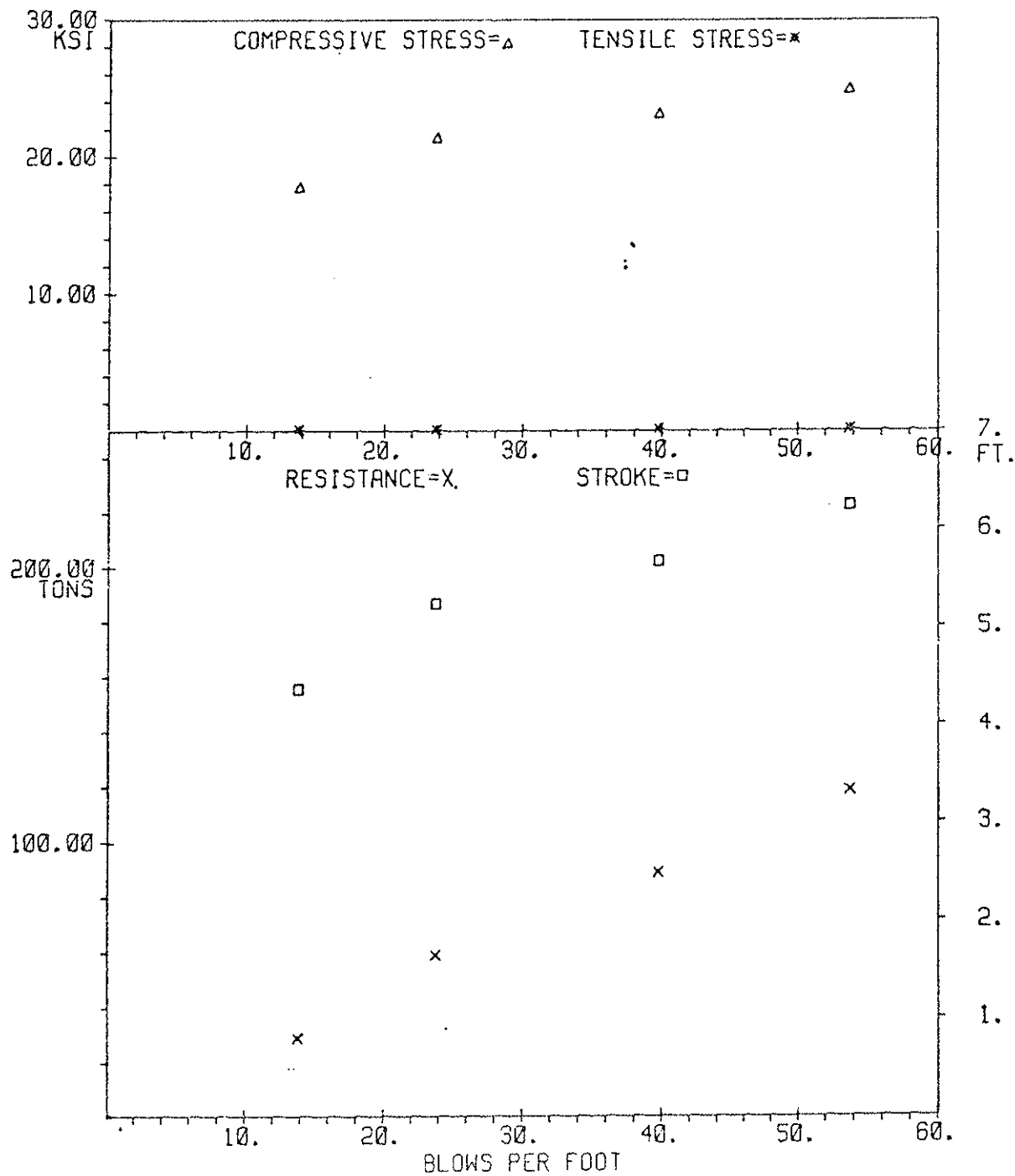


FIGURE 6: BEARING GRAPH AS OBTAINED FOR EXAMPLE 7.1

W E A P - WAVE EQUATION ANALYSIS FOR PILES
 THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
 BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.1 OPEN END DIESEL HAMMER
 PILE DESCRIPTION

X BEL. TOP (FT) .0 40.0
 A (SQ. IN.) 15.5 15.5
 E (KSI) 30000. 30000.
 GAMMA (LB/CU FT) 492.0 492.0

ELEMENT NUMBER	HAMMER MODEL DEL.D-12		COEFF. RESTITUTION
	WEIGHT (KIPS)	STIFFNESS (K/IN)	
1	.688		
2	.688	121250.0	
3	.688	121250.0	
4	.688	121250.0	
ANVIL	.810	66833.3	.800
CAP	.950	21000.0	.800
CUSHION		.0	1.000
PILE TOP			.800

PILE PROPERTIES

PILE LENGTH= 40. FT., AREA(AT TOP)= 15.5 S-IN
 E. MODUL(AT TOP)=30000. KSI., SPEC. WT.(AT TOP)= 492. LBS/CU FT

NO.	WEIGHT (KIPS)	STIFFN. (K/IN)	PDAMP. (KS/FT)	SPLICE (KIPS)	SOIL-S (PCT.)	SOIL-D (KS/FT)	QUAKE (IN.)	L.B.T. (FT.)
1	.235	8719.	.55	0.	.000	.516	.100	4.4
2	.235	8719.	.55	-5000.	.000	.516	.100	8.9
3	.235	8719.	.55	-5000.	.000	.516	.100	13.3
4	.235	8719.	.55	-5000.	.000	1.032	.100	17.8
5	.235	8719.	.55	-5000.	.000	1.032	.100	22.2
6	.235	8719.	.55	-5000.	.000	1.032	.100	26.7
7	.235	8719.	.55	-5000.	.000	1.547	.100	31.1
8	.235	8719.	.55	-5000.	.000	1.547	.100	35.6
9	.235	8719.	.55	-5000.	.000	1.547	.100	40.0
TOE					1.000	4.150	.100	

COEFFICIENT OF RESTITUTION OF SOIL 1.000

OPTIONS AND SPECIFICATIONS				
PHI	1.40	S-DAMPING	VISCOUS	RWT (KIPS) .00
IOUT	10	P-DAMPING	1	SOIL DIST. NO. 6
IFUEL	1	J SKIN	.30	TDEL (SEC.) .0000
IOSTR	0	J TOE	.15	TEMAX (MS) .00
		EFFICIENCY	.950	
		TIME INCR. (MS)	.087	

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RULT= 30.0, AT TOE= 30.0 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	280.7(36)	.0(0)	.0(0)	18.1(36)	9.14(37)	.989(285)
2	279.0(39)	.0(0)	.0(0)	18.0(39)	9.05(40)	.983(286)
3	278.5(42)	.0(0)	.0(0)	18.0(42)	8.92(43)	.977(293)
4	279.0(45)	.0(0)	.0(0)	18.0(45)	8.74(46)	.971(297)
5	275.1(48)	.0(0)	.0(0)	17.8(48)	9.51(78)	.966(299)
6	271.7(51)	.0(0)	.0(0)	17.5(51)	9.22(77)	.960(301)
7	265.7(54)	.0(0)	.0(0)	17.1(54)	8.51(57)	.954(304)
8	240.4(57)	.0(0)	.0(0)	15.5(57)	10.10(63)	.948(306)
9	187.5(58)	.0(0)	.0(0)	12.1(58)	11.99(64)	.942(308)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 9.6 K-FT

STROKES ANALYZED AND LAST RETURN (FT) 5.0 4.4 4.4

RULT= 60.0, AT TOE= 60.0 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	336.7(33)	.0(0)	.0(0)	21.7(33)	11.00(34)	.692(176)
2	335.6(36)	.0(0)	.0(0)	21.6(36)	10.87(38)	.682(181)
3	334.8(39)	.0(0)	.0(0)	21.6(39)	10.71(41)	.674(186)
4	336.0(43)	.0(0)	.0(0)	21.7(43)	10.49(44)	.665(187)
5	332.3(46)	.0(0)	.0(0)	21.4(46)	10.24(47)	.655(188)
6	329.1(49)	.0(0)	.0(0)	21.2(49)	9.96(50)	.644(189)
7	324.0(52)	.0(0)	.0(0)	20.9(52)	9.96(54)	.632(192)
8	299.4(55)	.0(0)	.0(0)	19.3(55)	10.92(58)	.620(197)
9	250.4(57)	.0(0)	.0(0)	16.2(57)	12.91(61)	.608(198)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 9.7 K-FT

STROKES ANALYZED AND LAST RETURN (FT) 5.2 5.1

RULT= 90.0, AT TOE= 90.0 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	362.3(32)	.0(0)	.0(0)	23.4(32)	11.86(33)	.555(139)
2	361.6(35)	.0(0)	.0(0)	23.3(35)	11.73(36)	.537(141)
3	361.8(38)	.0(0)	.0(0)	23.3(38)	11.53(39)	.518(141)
4	363.2(41)	.0(0)	.0(0)	23.4(41)	11.25(42)	.498(136)
5	358.6(44)	.0(0)	.0(0)	23.1(44)	10.96(46)	.478(135)
6	355.5(48)	.0(0)	.0(0)	22.9(48)	10.67(49)	.458(153)
7	351.4(51)	.0(0)	.0(0)	22.7(51)	10.55(52)	.439(152)
8	329.4(53)	.0(0)	.0(0)	21.3(53)	11.15(56)	.419(149)
9	291.0(56)	.0(0)	.0(0)	18.8(56)	12.34(60)	.400(120)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 9.3 K-FT

STROKES ANALYZED AND LAST RETURN (FT) 5.7 5.8

RULT= 120.0, AT TOE= 120.0 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	392.0(30)	.0(0)	.0(0)	25.3(30)	12.82(31)	.504(118)
2	391.1(33)	.0(0)	.0(0)	25.2(33)	12.65(34)	.484(116)
3	390.6(36)	.0(0)	.0(0)	25.2(36)	12.44(38)	.461(115)
4	391.9(40)	.0(0)	.0(0)	25.3(40)	12.17(41)	.443(106)
5	388.0(43)	.0(0)	.0(0)	25.0(43)	11.86(44)	.424(106)
6	384.6(46)	.0(0)	.0(0)	24.8(46)	11.50(47)	.400(106)
7	380.7(49)	.0(0)	.0(0)	24.6(49)	11.29(51)	.375(105)
8	361.8(52)	.0(0)	.0(0)	23.3(52)	11.56(54)	.349(100)
9	335.4(56)	.0(0)	.0(0)	21.6(56)	11.88(58)	.323(98)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 9.6 K-FT

STROKES ANALYZED AND LAST RETURN (FT) 5.9 6.2 6.2

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EXAMPLE 7.1 OPEN END DIESEL HAMMER

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI	BLOWS/ MINUTE
1	30.0	14.	4.35	.00	18.11	56.3
2	60.0	24.	5.22	.00	21.72	51.4
3	90.0	40.	5.66	.00	23.43	49.5
4	120.0	54.	6.23	.00	25.29	47.2

and is worried about the driving stresses in this relatively thin walled pile. He would also like to know if an interruption would affect the driveability. (The contractor supplied the following data: helmet (cap) weight = 450 lbs., stiffness 10,000 kips/inch.)

7.2.3 Solution

First, a sketch of the pile geometry is made on page 1 of the short input form. Similarly a sketch is made of an assumed skin friction distribution (for both adhesion and friction) on page 2 of the short input form.

- Card No. 2.000 It was decided to print forces as a function of time (IOUT = 1); IHAMR was found in Table 2 as 38 and NCROSS was set to 1 (non-uniform pile). IPERCS was set to 70. No other data were needed on Card No. 2.000.
- Card No. 3.000 The cap parameters were inserted as given by the contractor. No cushion is present.
- Card No. 4.000 Standard coefficients of restitution were chosen (blank card).
- Card No. 5.000 The pile length includes the toe plate. A pile top was computed from wall thickness and diameter E pile top (as for the rest of the pile) was set at the pipe manufacturers value of 29000 ksi. Standard steel weight was inserted for W Pile Top.
- Card No. 5.101,... There were three points of discontinuity which had to be described at depths of 20.4, 43.4 and 79.0 feet. Also the cross sectional area of the end plate had to be given at the pile bottom. Thus, seven cards were necessary. Note that EP and WP were not repeated.

Card No. 6.000

Of interest on this card are only the damping values. Since Case damping parameters were chosen (ISMITH blank) value from the right part of Table 5 are chosen. Since the recommendation was to use high values (in order to investigate the worst case) values of 2.0 and .8 for skin and toe, respectively, were chosen.

(It is interesting to make a comparison of the Case damping values with the corresponding Smith parameters). The average pile impedance (EA/c) is about 14 kips/ft/sec (using 8 inch² as an average steel area). Thus the total skin viscous damping constant is $14 \times 2 = 28$ kips/ft/sec. The static skin resistance at 120 tons is $0.7 \times 240 = 168$ kips. Thus, the corresponding Smith damping parameter is $28/168 = 0.167$ (somewhat less than the usually recommended 0.2). The toe damping constant used here corresponds to about 0.17. This latter value seems rather small, however, it is actually a large damping value since 30% end bearing were used more than could usually be expected in clay).

Cards No. 6.401,...

A trapezoidal distribution was chosen assuming that the pile top was 5 feet above grade and that the skin friction would be twice as high at the bottom as it was at 10 feet below the pile top.

Card No. 9.000

The capacities to be investigated are 120 and 160 tons.

The two pages of input are shown in Form 3. The output is reproduced in Form 4.

7.2.4 Discussion of Results

The important results can be found in the summary, the last page of Form 4. It is found there that the blow count for RULT = 120 is 221 more than 18 blows per inch. The stroke (see also B.C.P. = Bounce Chamber Pressure) is at the maximum and in order to avoid lift-off, the fuel setting

WAVE EQUATION ANALYSIS for PILES

Short Input Form, page 1

1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70

Title (40 characters)

1.000 EXAMPLE 7.2 DRIVENABILITY STUDY LB SZO

ISPL NCROSS
No. of 0 =

ISMITH
l-Smith ITYS IPHI

2.000 IOUT IUJ IHAMR IOSTR IFUEL IPEL N Splices Uniform IBEDAM IPERCS

Cap and Cushion

Wt. Cap Stiff. Cap Stiff. Cushion
(kips) (K/in) (K/in)

3.000 .45 10000.

Coefficients of Restitution

Anvil (Diesel) Cap (Diesel) TEMA TDEL
Cap(Air/Steam) Assembly(A/S) Pile Top Cushion (msec.) (sec.)

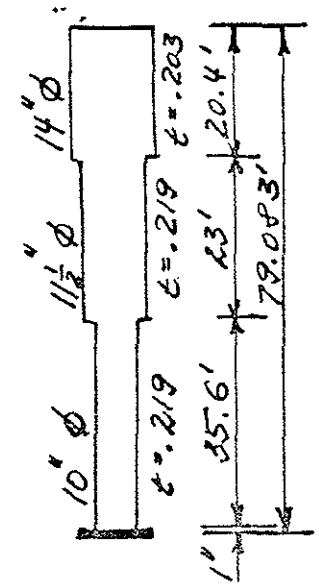
4.000

L Pile A Pile Top E Pile Top W Pile Top
(ft) (in²) (ksi) (lbs/ft³)

5.000 79.083 7.80 29000. 492.

If NCROSS = 1 and IPEL ≠ 2 only

	L ₁ (ft)	A ₁ (in ²)	E ₁ (ksi)	W ₁ (lbs/ft ³)
5.101	20.4	7.8		
5.102	20.4	7.8		
.	43.4	7.8		
.	43.4	7.8		
.	79.0	7.8		
5.119	79.0	95.0		
	79.0+3	95.0		



FORM 3: EXAMPLE 7.2

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WAVE EQUATION ANALYSIS for PILES Short Input Form, page 2

Queue Sdn (in)	Queue Toe (in)	Damping Sdn*	Damping Toe*	RULT (tons)	Coeff. of Rest. Of Soil	Stroke (ft)	Hammer Efficiency	Steam Pressure (psi)	Reaction wt. (kips)
1.1	1.1	2.0	1.8	-1.0					

6.000

If ITTS = 0 or -1 only

XP1 (ft)	% Soil Res.
0.0	0.0
5.0	0.0
10.0	1.0
79.083	2.0

6.401
6.402

7.000

If IOVT > 9

IPAX	Plotter Scale

7.000

If IOVT > 9 only

ISS(I), I = 1, 2, ..., 15 (Plotted Variables)

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15

8.000

If RULT < 0 only:

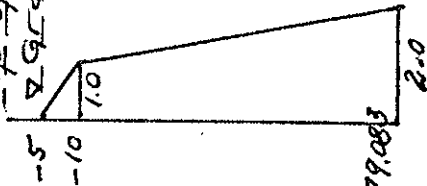
Ultimate Resistance - tons (at least 10 values)

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15
120.	169.													

9.000

* Damping Parameters - sec/ft for Smith; Dimensionless for Case

Top of Pile
▽ grade



1	2	3	4	5	6	7	8	9	10	11	12	13	14	15

W E A P - WAVE EQUATION ANALYSIS FOR PILES

THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.2 DRIVEABILITY CHECK LB 520
PILE DESCRIPTION

X BEL. TOP (FT)	.0	20.4	20.4	43.4	43.4	79.0	79.0	79.1
A (SQ. IN.)	8.8	8.8	8.4	8.4	7.4	7.4	95.0	95.0
E (KSI)	29000.	29000.	29000.	29000.	29000.	29000.	29000.	29000.
GAMMA (LB/CU FT)	492.0	492.0	492.0	492.0	492.0	492.0	492.0	492.0

HAMMER MODEL LB 520			
ELEMENT NUMBER	WEIGHT (KIPS)	STIFFNESS (K/IN)	COEFF. RESTITUTION
1	1.753		
2	1.692	247000.0	
3	1.600	239000.0	
ANVIL	2.058	81000.0	.850
CAP	.450	10000.0	.850
CUSHION		.0	1.000
PILE TOP			.850

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PILE PROPERTIES

PILE LENGTH= 79. FT., AREA(AT TOP)= 8.8 S-IN
 E. MODUL(AT TOP)=29000. KSI., SPEC. WT.(AT TOP)= 492. LBS/CU FT

NO.	WEIGHT (KIPS)	STIFFN. (K/IN)	PDAMP. (KS/FT)	SPLICE (KIPS)	SOIL-S (PCT.)	SOIL-D (KS/FT)	GUAKE (IN.)	L-B.T. (FT.)
1	.149	4303.	.31	0.	.000	.000	.100	4.9
2	.149	4303.	.31	-5000.	.016	.695	.100	9.9
3	.149	4303.	.31	-5000.	.034	1.487	.100	14.8
4	.149	4303.	.31	-5000.	.036	1.591	.100	19.8
5	.143	4131.	.30	-5000.	.038	1.626	.100	24.7
6	.142	4107.	.29	-5000.	.041	1.715	.100	29.7
7	.142	4107.	.29	-5000.	.043	1.813	.100	34.6
8	.142	4107.	.29	-5000.	.045	1.911	.100	39.5
9	.138	3989.	.29	-5000.	.048	1.954	.100	44.5
10	.125	3618.	.26	-5000.	.050	1.857	.100	49.4
11	.125	3618.	.26	-5000.	.052	1.943	.100	54.4
12	.125	3618.	.26	-5000.	.055	2.030	.100	59.3
13	.125	3618.	.26	-5000.	.057	2.117	.100	64.3
14	.125	3618.	.26	-5000.	.059	2.203	.100	69.2
15	.125	3618.	.26	-5000.	.062	2.290	.100	74.1
16	.150	3675.	.29	-5000.	.064	2.622	.100	79.1
TOE					.300	11.464	.100	

COEFFICIENT OF RESTITUTION OF SOIL 1.000

OPTIONS AND SPECIFICATIONS		VISCOUS		RWT (KIPS)		SOIL DIST. NO.	
PHI	1.40	S-DAMPING	1				6.30
IOUT	1	P-DAMPING	2.00				0
IFUEL	1	J SKIN	.80				.0000
IOSTR	0	J TOE	.950				.00
		EFFICIENCY	.094				
		TIME INCR.	(MS)				

RULT= 120.0, AT TOE= 36.0 TONS

JP	TIME (MS)	HAMMER		HAMMER AND PILE FORCES(KIPS)												
		RAM M	ANVIL	PILE ELEMENTS												
			TOP	2	3	4	6	7	8	10	11	13	14	15	16	
3	.3	96.5	126.5	79.7	78.1	71.8	61.5	44.3	37.3	31.2	21.4	17.7	12.0	9.8	8.1	6.7
6	.6	30.1	133.3	83.3	81.7	75.3	64.4	46.3	38.9	32.5	22.3	18.4	12.4	10.2	8.4	6.9
9	.8	149.5	140.2	87.1	85.4	78.9	67.5	48.4	40.7	34.0	23.2	19.2	12.9	10.6	8.7	7.2
12	1.1	100.2	148.1	91.1	89.4	82.7	70.9	50.7	42.6	35.6	24.3	20.0	13.5	11.0	9.0	7.5
15	1.4	67.9	158.0	95.3	93.5	86.6	74.5	53.3	44.7	37.3	25.4	20.9	14.0	11.5	9.4	7.8
18	1.7	138.6	168.1	99.7	97.7	90.6	78.5	56.0	46.9	39.1	26.6	21.9	14.7	12.0	9.8	8.1
21	2.0	184.6	287.7	104.6	102.1	94.9	82.5	58.9	49.4	41.1	27.9	22.9	15.4	12.5	10.3	8.4
24	2.3	274.5	459.6	110.4	107.1	99.5	86.7	62.1	52.0	43.2	29.3	24.0	16.1	13.1	10.7	8.8
27	2.5	448.7	574.4	118.5	113.2	104.4	91.0	65.5	54.8	45.6	30.8	25.2	16.9	13.8	11.2	9.2
30	2.8	412.1	577.3	131.2	121.5	110.4	95.8	69.1	57.8	48.0	32.4	26.6	17.7	14.4	11.8	9.7
33	3.1	276.9	448.6	150.5	134.0	118.5	101.7	73.1	61.0	50.7	34.2	28.0	18.6	15.2	12.4	10.1
36	3.4	197.6	271.3	176.3	153.1	130.7	109.6	77.6	64.6	53.5	36.0	29.5	19.6	15.9	13.0	10.6
39	3.7	101.5	189.0	204.8	178.8	149.1	121.0	83.1	68.6	56.7	38.1	31.1	20.7	16.8	13.7	11.2
42	4.0	104.7	188.6	229.0	207.7	174.0	138.0	90.3	73.5	60.2	40.2	32.9	21.8	17.7	14.4	11.8
45	4.2	166.5	185.2	243.8	233.3	202.5	161.3	100.4	80.1	64.6	42.6	34.8	23.0	18.7	15.2	12.4
48	4.5	115.2	182.1	249.1	249.6	228.4	188.3	114.9	89.6	70.4	45.3	36.8	24.3	19.8	16.0	13.1
51	4.8	98.9	178.8	248.6	256.1	246.0	213.4	134.9	103.5	78.9	48.6	39.2	25.8	20.9	17.0	13.8
54	5.1	154.0	175.4	246.6	256.3	253.4	231.0	158.8	122.2	91.4	53.0	42.1	27.3	22.1	17.9	14.6
57	5.4	109.4	172.3	246.1	254.2	253.7	239.1	182.3	144.4	108.8	59.4	46.0	29.1	23.5	19.0	15.5
60	5.6	95.7	169.2	248.2	253.1	251.5	240.4	200.2	166.7	129.8	68.7	51.6	31.3	25.0	20.2	16.4
63	5.9	143.6	166.3	251.9	254.4	250.4	239.0	210.2	184.7	150.8	81.8	59.9	34.3	26.9	21.5	17.4
66	6.2	103.9	163.6	255.8	257.9	251.8	238.3	213.1	195.0	167.9	98.3	71.4	38.6	29.5	23.2	18.6
69	6.5	87.9	161.1	259.0	262.1	255.4	239.7	212.5	198.1	178.5	116.7	86.0	45.0	33.3	25.5	20.1
72	6.8	132.2	158.8	261.4	265.6	259.6	243.2	211.8	197.7	182.3	133.5	102.3	53.8	38.9	28.8	22.1
75	7.1	109.4	156.7	262.4	267.8	263.1	247.4	212.7	197.1	182.0	145.0	118.0	65.2	46.7	33.7	25.1
78	7.3	78.5	154.7	262.6	268.7	265.3	250.8	215.6	198.0	181.8	150.5	130.2	78.0	56.7	40.6	29.6
81	7.6	124.8	152.9	262.3	268.7	266.1	252.8	219.4	200.9	183.6	152.8	136.6	90.5	68.0	49.5	35.9
84	7.9	108.9	151.3	261.7	268.0	265.7	253.6	223.0	205.3	187.2	154.2	138.8	100.6	79.1	59.7	44.1
87	8.2	79.4	149.8	261.0	266.8	264.7	253.6	226.1	209.8	191.7	156.1	140.5	107.8	88.4	69.8	53.3
90	8.5	112.6	148.5	260.0	265.5	263.6	253.0	228.2	213.3	196.6	159.6	143.1	112.2	94.9	78.3	62.4
93	8.7	112.3	147.3	258.7	264.3	262.5	252.4	229.5	215.8	200.5	164.6	146.6	114.7	99.0	84.0	69.6
96	9.0	79.1	146.3	257.2	263.0	261.5	252.1	230.2	217.5	203.0	169.4	151.3	117.1	102.0	87.2	74.1
99	9.3	109.7	145.4	255.5	261.5	260.7	251.9	230.7	218.4	204.7	173.5	156.7	124.3	105.0	89.1	76.3
102	9.6	109.7	144.7	253.7	260.0	259.8	251.4	230.9	218.9	206.2	177.2	161.4	126.4	108.5	91.2	77.4
105	9.9	79.3	144.0	251.8	258.4	258.5	250.7	230.9	219.5	207.2	180.1	164.8	131.2	112.6	94.5	78.9
108	10.2	102.5	143.5	250.0	256.5	256.8	249.7	231.0	220.1	208.1	182.1	167.8	135.2	117.1	98.9	81.8
111	10.4	110.6	143.0	248.2	254.4	254.8	248.2	231.0	220.5	209.2	183.7	170.2	138.6	121.4	103.7	85.8
114	10.7	84.2	142.7	246.5	252.3	252.7	246.5	230.6	220.9	210.1	185.3	171.6	141.5	125.3	108.2	90.1
117	11.0	102.2	142.4	244.5	250.2	250.5	244.7	230.0	221.0	210.6	186.4	172.6	143.8	128.4	111.8	94.2
120	11.3	106.5	140.8	242.3	248.0	248.4	243.0	229.3	220.7	210.7	187.0	173.9	145.9	130.8	114.7	97.6
123	11.6	84.6	139.3	240.1	245.6	246.4	241.3	228.2	219.9	210.6	187.6	175.2	147.6	132.6	116.8	100.2
126	11.9	97.2	137.8	237.7	243.3	244.3	239.6	226.9	218.9	209.8	188.3	176.1	148.9	134.1	118.5	102.3
129	12.1	100.3	136.3	235.5	241.0	242.0	237.8	225.5	217.7	208.7	188.6	176.8	150.0	135.4	119.9	103.7
132	12.4	86.4	135.1	233.3	238.6	239.7	235.6	223.9	216.2	207.8	188.5	177.1	150.9	136.4	120.9	103.9
135	12.7	93.2	133.8	230.8	236.1	237.2	233.2	222.1	214.9	207.0	188.1	176.9	151.4	136.9	121.1	103.7
138	13.0	97.8	132.7	228.0	233.3	234.5	230.8	220.2	213.6	205.9	187.4	176.4	151.4	136.8	120.6	103.5
141	13.3	84.4	131.6	225.0	230.2	231.5	228.1	218.3	212.1	204.6	186.4	175.7	150.7	135.9	119.9	103.2
144	13.5	87.9	130.6	221.7	226.7	228.2	225.3	216.4	210.2	203.0	185.4	174.7	149.4	134.7	119.2	102.7

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELOCITY FT/S	DISMX INCH	DISMX INCH	DISMX INCH	DISMX INCH	DISMX INCH	DISMX INCH	DISMX INCH			
147	93.8	129.7	218.3	223.0	224.7	222.4	214.2	208.2	201.2	184.1	173.3	147.7	133.5	118.3	102.0
150	84.0	128.9	214.8	219.2	221.5	219.4	211.7	206.1	199.2	182.3	171.4	146.1	132.2	117.1	103.1
153	84.0	128.9	211.1	215.5	217.5	215.1	209.1	203.7	197.2	180.0	169.2	144.5	130.7	115.9	100.1
156	92.1	127.4	207.3	211.6	213.7	212.5	206.3	201.1	194.6	177.4	166.8	142.8	129.1	114.5	99.0
159	85.1	126.7	203.4	207.4	209.6	208.8	203.2	198.2	191.8	174.7	164.5	140.9	127.5	113.1	97.8
162	82.0	125.0	199.3	203.0	205.2	204.8	199.8	195.0	188.6	172.2	162.1	138.9	125.8	111.7	96.7
165	85.9	125.4	195.0	198.2	200.8	200.8	196.3	191.5	185.4	169.7	159.7	136.9	124.0	110.2	95.4
168	85.1	124.8	190.3	193.3	195.8	196.3	192.4	187.9	182.3	167.1	157.3	134.8	122.0	108.7	94.2
171	85.1	124.8	185.3	188.1	190.6	191.6	188.4	184.5	179.2	164.5	155.0	132.7	120.3	107.0	92.8
174	87.7	123.7	180.1	182.6	185.2	186.6	184.3	181.0	176.1	162.0	152.7	130.8	118.5	105.3	91.4
177	85.0	123.1	174.7	176.6	179.4	181.3	180.3	177.4	173.0	159.5	150.4	128.9	116.7	103.7	89.9
180	80.5	122.6	169.0	170.5	173.3	175.9	176.3	173.8	169.8	157.1	148.2	127.1	115.0	102.1	88.5
183	85.6	122.0	163.0	164.2	167.2	170.6	172.1	170.1	166.6	154.7	146.1	125.3	113.4	100.6	87.2
186	84.5	121.4	156.6	157.7	161.2	165.2	167.9	166.5	163.4	152.3	144.1	123.7	111.9	99.3	85.9
189	79.7	120.8	150.2	151.2	155.1	159.8	163.6	162.8	160.2	149.9	142.0	122.1	110.5	98.0	84.9
192	83.4	120.2	143.7	144.7	148.9	154.3	158.4	159.1	157.0	147.5	140.0	120.6	109.2	97.0	84.3
195	83.7	119.5	137.3	138.0	142.5	148.7	155.1	155.4	153.8	145.2	138.0	119.2	108.1	96.3	83.9
198	86.6	118.7	131.0	131.4	136.1	143.1	150.9	151.7	150.6	142.9	136.0	117.9	107.3	95.8	83.4
201	80.6	117.9	124.7	124.8	129.8	137.4	146.6	148.0	147.4	140.5	134.1	116.8	106.6	95.2	82.6
204	82.1	117.1	118.6	118.5	123.6	131.8	142.3	144.3	144.2	138.2	132.2	115.8	105.8	94.3	81.7
207	77.6	116.1	112.5	112.2	117.5	126.3	137.9	140.5	141.0	136.0	130.5	114.8	104.7	93.2	80.7
210	78.1	115.1	106.6	106.1	111.6	120.9	133.6	136.7	137.8	133.9	128.8	113.5	103.4	92.1	79.7
213	79.9	114.1	101.0	100.1	105.8	115.7	129.4	133.0	134.6	131.9	127.2	112.0	102.1	90.9	78.7
216	76.0	112.9	95.5	94.4	100.0	110.6	125.3	129.4	131.6	129.8	125.3	110.5	100.8	89.8	77.7
219	75.4	111.7	90.2	89.2	94.4	105.6	121.4	126.0	128.6	127.6	123.3	108.9	99.4	88.6	76.7
222	77.2	110.3	85.2	84.0	89.0	100.6	117.7	122.8	125.8	125.1	121.1	107.4	98.1	87.5	75.8
225	73.9	109.0	80.5	79.0	84.0	95.8	114.1	119.7	123.0	122.7	119.0	105.8	96.8	86.4	74.9
228	72.4	107.5	76.0	74.4	79.4	90.9	110.5	116.6	120.2	120.3	116.9	104.3	95.6	85.4	74.0
231	73.9	106.0	71.7	70.3	75.1	86.3	106.9	113.4	117.2	118.0	114.8	102.8	94.3	84.4	73.2
234	71.3	104.4	67.8	66.5	70.9	82.0	103.2	110.0	114.3	115.8	112.9	101.4	93.1	83.4	72.3
237	69.1	102.7	64.2	62.6	67.1	78.0	99.4	105.7	111.4	113.7	111.1	100.0	91.9	82.4	71.5
240	70.2	101.0	60.9	59.0	63.5	74.3	95.7	103.4	108.6	111.7	109.4	98.7	90.8	81.4	70.7
243	68.2	99.2	57.8	55.9	60.1	70.5	92.0	100.2	105.0	109.7	107.8	97.5	89.6	80.5	69.9
246	66.6	97.4	54.9	53.1	56.9	66.8	88.5	97.1	103.1	107.7	106.1	96.4	88.8	79.7	69.2
249	68.4	95.5	52.3	50.3	53.8	63.4	85.2	94.1	100.5	105.8	104.5	95.3	87.9	78.9	68.5
252	64.1	93.7	49.8	47.5	50.9	60.4	81.9	91.3	98.0	104.0	103.0	94.2	87.0	78.2	67.9
255	63.2	91.7	47.5	44.9	48.2	57.7	78.7	88.6	95.7	102.2	101.4	93.2	86.2	77.5	67.3
258	64.0	89.8	45.3	42.8	46.0	55.3	75.9	85.9	93.4	100.5	100.0	92.1	85.3	76.7	66.7
261	60.4	87.9	43.1	41.0	44.1	53.1	73.4	83.5	91.2	98.9	98.6	91.1	84.4	76.0	66.1
264	59.3	85.9	41.2	39.3	42.5	51.2	71.0	81.4	89.2	97.3	97.2	90.1	83.5	75.2	65.5
267	59.8	84.0	39.5	37.6	40.8	49.5	68.7	79.4	87.4	95.8	96.0	89.1	82.7	74.5	64.9
270	56.6	82.0	38.1	36.1	39.2	47.8	66.8	77.6	85.8	94.4	94.7	88.2	81.9	73.9	64.3
273	55.3	80.1	36.8	34.7	37.7	46.2	65.3	75.8	84.3	93.2	93.5	87.3	81.1	73.2	63.8
276	55.7	78.2	35.6	33.6	36.5	44.8	63.9	74.2	82.8	92.1	92.5	86.5	80.4	72.6	63.3
279	52.8	76.3	34.5	32.6	35.4	43.7	62.5	72.8	81.6	91.1	91.7	85.7	79.8	72.1	62.8

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELOCITY FT/S	DISMX INCH	DISMX INCH	DISMX INCH	DISMX INCH
1	262.6(78)	.0(0)	.0(0)	29.8(78)	9.83(48)	.826(135)			
2	268.7(78)	.0(0)	.0(0)	30.5(78)	9.38(51)	.771(138)			
3	266.1(81)	.0(0)	.0(0)	30.2(81)	8.92(54)	.717(141)			

FORM 4: CONTINUED

4	253.6(84)	.0(0)	.0(0)	.0(0)	28.8(84)	8.45(57)	.664(141)
5	241.7(96)	.0(0)	.0(0)	.0(0)	28.6(96)	7.91(60)	.610(144)
6	231.0(108)	.0(0)	.0(0)	.0(0)	27.5(108)	7.38(63)	.557(147)
7	221.0(117)	.0(0)	.0(0)	.0(0)	26.3(117)	6.90(69)	.506(147)
8	210.7(120)	.0(0)	.0(0)	.0(0)	25.1(120)	6.47(72)	.458(150)
9	199.7(120)	.0(0)	.0(0)	.0(0)	24.4(120)	5.99(75)	.410(153)
10	188.6(129)	.0(0)	.0(0)	.0(0)	25.5(129)	5.43(78)	.361(159)
11	177.1(132)	.0(0)	.0(0)	.0(0)	23.9(132)	4.85(81)	.316(165)
12	164.6(135)	.0(0)	.0(0)	.0(0)	22.2(135)	4.21(84)	.275(168)
13	151.4(138)	.0(0)	.0(0)	.0(0)	20.5(138)	3.61(87)	.238(174)
14	136.9(135)	.0(0)	.0(0)	.0(0)	18.5(135)	3.04(90)	.206(180)
15	121.1(135)	.0(0)	.0(0)	.0(0)	16.4(135)	2.53(93)	.178(183)
16	103.9(132)	.0(0)	.0(0)	.0(0)	11.7(132)	2.14(96)	.154(186)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 13.5 K-FT

RULT= 160.0, AT TOE= 48.0 TONS

JP	TIME (MS)	HAMMER		HAMMER AND PILE FORCES(KIPS)												
		RAM M	ANVIL	PILE ELEMENTS												
				TOP	2	3	4	6	7	8	10	11	13	14	15	16
3	.3	97.9	128.3	81.2	79.7	72.4	60.9	42.1	34.7	28.4	18.5	14.9	9.6	7.7	6.3	5.1
6	.6	30.5	135.1	85.1	83.6	76.0	63.9	44.1	36.3	29.7	19.4	15.6	10.0	8.1	6.5	5.3
9	.8	151.5	142.1	89.3	87.6	79.9	67.1	46.3	38.1	31.1	20.3	16.3	10.5	8.4	6.8	5.6
12	1.1	101.5	149.9	93.6	91.8	84.0	70.5	48.6	40.0	32.7	21.3	17.1	11.0	8.8	7.1	5.8
15	1.4	68.1	159.0	98.0	96.3	88.4	74.2	51.1	42.0	34.3	22.3	17.9	11.5	9.2	7.4	6.1
18	1.7	140.8	193.4	102.8	101.0	92.9	78.1	53.8	44.2	36.1	23.4	18.8	12.0	9.6	7.8	6.3
21	2.0	228.6	343.0	108.1	105.9	97.5	82.4	56.7	46.6	38.0	24.6	19.7	12.6	10.1	8.1	6.6
24	2.3	327.0	513.3	114.7	111.2	102.3	87.1	59.7	49.1	40.0	25.9	20.8	13.3	10.6	8.5	7.0
27	2.5	457.9	597.8	124.4	117.8	107.7	91.9	63.0	51.7	42.1	27.3	21.9	13.9	11.2	9.0	7.3
30	2.8	390.7	553.7	139.4	127.6	114.4	97.2	66.6	54.6	44.5	28.7	23.0	14.7	11.7	9.4	7.7
33	3.1	245.6	393.6	161.6	142.7	124.0	103.6	70.5	57.7	46.9	30.3	24.3	15.5	12.3	9.9	8.1
36	3.4	151.5	224.2	189.7	164.7	138.4	112.6	75.0	61.2	49.6	32.0	25.6	16.3	13.0	10.4	8.5
39	3.7	96.5	181.4	218.5	192.7	159.5	126.2	80.6	65.1	52.6	33.8	27.0	17.2	13.7	11.0	8.9
42	4.0	121.7	178.4	240.5	221.9	186.8	146.1	88.5	70.1	56.1	35.8	28.6	18.1	14.5	11.6	9.4
45	4.2	141.4	174.8	251.8	245.6	216.1	171.7	100.4	77.0	60.4	37.9	30.2	19.2	15.3	12.2	9.9
48	4.5	99.5	171.4	254.2	258.7	240.5	199.3	117.6	87.6	66.5	40.5	32.1	20.3	16.1	12.9	10.5
51	4.8	115.6	168.0	252.1	262.2	254.7	223.0	139.9	103.1	75.8	43.6	34.3	21.4	17.1	13.6	11.0
54	5.1	130.0	164.6	249.9	260.3	258.8	237.9	164.6	124.0	89.5	48.1	37.0	22.8	18.1	14.4	11.7
57	5.4	97.1	161.5	249.9	257.6	257.1	243.2	187.4	147.9	108.0	54.9	40.9	24.4	19.2	15.3	12.3
60	5.6	107.5	158.6	252.3	257.0	254.5	242.3	203.6	169.9	130.1	64.9	46.8	26.4	20.5	16.3	13.1
63	5.9	123.8	155.8	255.8	259.0	253.8	240.2	211.0	185.6	151.6	78.6	55.6	29.3	22.3	17.4	13.9
66	6.2	89.1	153.4	259.3	262.5	255.7	240.0	211.7	193.8	167.3	95.1	67.5	33.7	24.8	18.9	14.9
69	6.5	101.1	151.1	262.2	266.0	259.4	242.2	210.4	195.3	175.1	112.5	82.0	40.3	28.6	21.1	16.3
72	6.8	118.9	149.0	263.8	269.0	263.3	245.7	210.2	193.8	177.4	128.0	97.2	49.4	34.3	24.4	18.2
75	7.1	89.4	147.2	264.0	270.8	266.2	249.4	211.7	193.4	177.0	138.7	110.8	60.4	42.2	29.4	21.3
78	7.3	89.5	145.5	263.8	271.0	267.6	252.3	214.7	195.4	176.7	143.1	121.2	72.1	51.8	36.3	25.8
81	7.6	117.1	144.0	263.3	270.1	267.8	254.0	218.7	198.9	179.1	144.0	127.1	82.5	62.0	44.9	32.2
84	7.9	89.1	142.7	262.5	269.0	266.9	254.2	222.2	203.0	183.7	145.7	129.0	90.1	71.2	54.0	40.0
87	8.2	86.4	141.6	261.4	267.9	265.5	253.6	224.7	207.5	188.5	148.8	129.9	94.8	78.1	62.4	48.3
90	8.5	110.3	140.6	260.2	266.5	264.3	252.9	226.5	211.0	192.6	152.7	133.2	97.6	82.2	68.5	55.6
93	8.7	93.2	139.8	259.0	264.9	263.2	252.3	227.7	212.9	196.4	157.4	138.1	100.2	84.3	71.9	60.6
96	9.0	86.2	139.2	257.3	263.5	262.1	251.9	228.1	214.1	198.9	162.7	142.8	103.7	86.1	73.3	63.1
99	9.3	104.5	138.6	255.4	262.2	261.1	251.6	228.2	215.0	200.2	166.9	147.6	108.3	88.6	74.0	63.6
102	9.6	93.6	138.3	253.5	260.5	260.1	251.1	228.6	215.4	201.2	169.8	152.4	113.3	92.4	75.3	63.6
105	9.9	85.5	138.0	251.7	258.5	258.8	250.2	228.8	215.8	202.5	172.4	156.1	117.8	97.0	78.0	64.5
108	10.2	101.2	137.9	249.9	256.7	256.8	248.9	228.5	216.6	203.5	174.8	158.5	121.7	101.4	81.8	66.7
111	10.4	95.7	137.9	248.2	254.7	254.6	247.4	228.3	217.1	204.2	176.2	160.2	124.9	105.2	85.6	69.8
114	10.7	90.3	137.9	246.4	252.5	252.6	245.6	228.1	217.1	205.2	177.2	161.5	127.3	108.6	88.8	72.8
117	11.0	99.6	138.1	244.3	250.3	250.5	243.8	227.4	217.0	205.8	178.1	162.6	129.1	111.2	91.4	75.0
120	11.3	96.2	137.3	242.1	248.1	248.3	242.1	226.2	216.8	205.5	178.8	163.5	130.7	112.6	93.4	76.7
123	11.6	89.2	136.5	239.9	245.8	246.2	240.5	225.2	215.9	204.8	179.1	164.4	131.8	113.6	94.7	78.0
126	11.9	97.8	135.7	237.8	243.4	244.1	238.6	224.0	214.3	203.9	179.3	165.0	132.3	114.4	95.5	79.0
129	12.1	93.8	135.1	235.4	241.1	241.8	236.6	222.2	212.9	202.8	179.1	165.0	132.7	115.0	96.3	79.6
132	12.4	84.6	134.5	233.1	238.7	239.4	234.5	220.3	211.6	201.6	178.5	164.5	132.9	115.4	97.0	80.0
135	12.7	96.0	134.1	230.6	235.9	236.8	232.0	218.4	210.0	200.3	177.4	163.7	132.8	115.6	97.4	80.4
138	13.0	93.2	133.7	228.0	232.9	233.8	229.3	216.5	208.2	198.9	176.0	162.6	132.3	115.6	97.5	80.6
141	13.3	88.3	133.4	225.0	229.6	230.3	226.4	214.4	206.4	197.1	174.6	161.3	131.7	115.1	97.5	80.5
144	13.5	92.6	133.1	221.5	225.9	226.7	223.5	212.2	204.4	195.1	173.1	160.1	130.8	114.5	97.1	80.2

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147	13.8	91.1	132.9	217.6	221.6	223.1	220.3	209.8	202.1	193.1	171.6	158.8	129.7	113.6	96.3	79.7
150	14.1	88.5	132.8	213.5	217.2	219.0	216.8	207.1	199.7	191.1	170.0	157.4	128.5	112.5	95.3	78.9
153	14.4	91.9	132.7	209.0	212.7	214.3	212.9	204.1	197.3	189.0	168.4	155.9	127.4	111.3	94.2	78.0
156	14.7	91.7	132.6	204.4	207.7	209.5	208.6	201.0	194.7	186.9	166.8	154.5	126.1	110.2	93.2	77.4
159	15.0	88.6	132.6	199.5	202.2	204.3	203.9	197.8	192.0	184.6	165.1	153.0	124.8	109.1	92.4	76.8
162	15.2	90.7	132.5	194.1	196.4	198.7	199.1	194.3	189.1	182.1	163.4	151.5	123.7	108.2	91.8	76.0
165	15.5	91.6	132.5	188.3	190.4	192.8	194.1	190.6	186.0	179.5	161.5	149.9	122.7	107.5	91.1	75.2
168	15.8	88.9	132.5	182.0	183.9	186.8	188.9	186.8	182.7	176.8	159.5	148.3	121.9	106.7	90.1	74.2
171	16.1	90.2	132.4	175.5	177.1	180.6	183.5	182.8	179.2	173.8	157.6	146.8	120.9	105.8	89.2	73.3
174	16.4	91.5	132.3	168.9	170.2	173.9	177.9	178.6	175.7	170.7	155.6	145.3	119.8	104.7	88.3	72.4
177	16.6	89.1	132.2	162.1	163.1	167.1	171.9	174.3	172.0	167.7	153.6	143.7	118.5	103.6	87.3	71.6
180	16.9	89.6	132.0	155.3	155.9	160.2	165.8	169.8	168.2	164.6	151.5	141.9	117.2	102.5	86.4	70.8
183	17.2	91.0	131.7	148.4	148.6	153.3	159.6	165.3	164.5	161.5	149.3	140.0	115.9	101.4	85.5	70.1
186	17.5	88.9	131.4	141.4	141.4	146.2	153.5	160.6	160.7	158.3	147.0	138.0	114.5	100.3	84.6	69.4
189	17.8	88.7	130.9	134.4	134.1	139.3	147.3	156.0	156.8	155.1	144.6	136.0	113.1	99.2	83.8	68.7
192	18.1	89.9	130.3	127.5	126.8	132.4	141.2	151.5	152.8	151.6	142.2	134.0	111.8	98.2	83.0	68.2
195	18.3	88.1	129.6	120.6	119.7	125.6	135.4	147.0	148.8	148.0	139.7	132.0	110.6	97.2	82.2	67.6
198	18.6	87.1	128.8	113.9	112.8	119.0	129.6	142.5	144.8	144.6	137.3	130.1	109.3	96.2	81.4	67.0
201	18.9	88.0	127.8	107.4	106.1	112.6	123.9	138.0	140.9	141.3	134.9	128.1	108.1	95.3	80.7	66.4
204	19.2	86.4	126.7	101.1	99.8	106.2	118.2	133.6	137.1	138.1	132.6	126.2	106.9	94.4	80.0	65.9
207	19.5	84.8	125.4	95.2	93.8	99.8	112.7	129.4	133.5	135.0	130.5	124.4	105.7	93.4	79.3	65.3
210	19.8	85.3	124.0	89.6	88.0	93.6	107.1	125.3	130.0	132.0	128.4	122.6	104.5	92.4	78.5	64.7
213	20.0	83.8	122.5	84.3	82.2	87.9	101.6	121.4	126.7	129.2	126.4	121.0	103.3	91.5	77.7	64.0
216	20.3	81.8	120.8	79.3	77.0	82.7	96.2	117.6	123.5	126.5	124.5	119.4	102.2	90.5	76.9	63.3
219	20.6	81.7	119.0	74.5	72.5	77.9	91.0	113.8	120.4	123.9	122.7	117.9	101.1	89.5	76.0	62.6
222	20.9	80.3	117.1	70.0	68.3	73.4	86.3	110.1	117.3	121.4	120.9	116.4	100.1	88.6	75.3	62.0
225	21.2	78.0	115.1	66.1	64.1	69.3	82.0	106.5	114.2	118.9	119.2	115.0	99.1	87.8	74.5	61.3
228	21.4	77.4	112.9	62.5	60.2	65.4	78.1	103.2	111.3	116.3	117.5	113.6	98.1	87.0	73.9	60.7
231	21.7	76.0	110.7	59.2	56.8	61.8	74.6	100.0	108.5	113.8	115.8	112.2	97.2	86.2	73.2	60.1
234	22.0	73.6	108.4	56.2	54.0	58.8	71.2	96.8	105.8	111.5	114.1	110.8	96.3	85.4	72.5	59.6
237	22.3	74.7	106.0	53.5	51.5	56.2	68.1	93.8	103.2	109.3	112.4	109.5	95.4	84.7	71.9	59.0
240	22.6	72.5	103.6	51.1	49.1	53.7	65.4	91.0	100.7	107.2	110.9	108.1	94.5	84.0	71.3	58.5
243	22.9	68.6	101.1	49.0	46.8	51.3	62.9	88.5	98.4	105.2	109.4	106.8	93.5	83.2	70.7	58.0
246	23.1	69.5	98.6	47.2	44.8	49.1	60.5	86.1	96.3	103.3	108.0	105.6	92.6	82.5	70.1	57.5
249	23.4	67.4	96.1	45.6	43.1	47.2	58.3	83.8	94.4	101.6	106.7	104.5	91.8	81.7	69.5	57.1
252	23.7	63.7	93.6	44.1	41.7	45.6	56.6	81.6	92.5	100.0	105.4	103.4	91.0	81.1	69.0	56.6
255	24.0	64.3	91.1	42.7	40.4	44.4	55.2	79.6	90.8	98.5	104.3	102.4	90.3	80.5	68.4	56.2
258	24.3	62.2	88.7	41.5	39.4	43.4	53.9	77.9	89.2	97.1	103.3	101.5	89.7	79.9	68.0	55.8
261	24.6	58.9	86.2	40.5	38.5	42.4	52.9	76.4	87.8	95.8	102.4	100.8	89.1	79.5	67.6	55.5
264	24.8	59.1	83.8	39.6	37.8	41.6	51.9	75.1	86.6	94.7	101.5	100.1	88.6	79.0	67.2	55.1
267	25.1	57.2	81.4	38.9	37.0	40.8	51.1	74.1	85.6	93.8	100.7	99.4	88.1	78.6	66.9	54.9
270	25.4	54.2	79.1	38.4	36.4	40.2	50.3	73.2	84.9	93.1	100.1	98.9	87.8	78.3	66.6	54.6
273	25.7	54.2	76.9	37.9	36.0	39.7	49.7	72.4	84.2	92.6	99.6	98.4	87.4	78.0	66.3	54.3
276	26.0	52.5	74.6	37.5	35.7	39.3	49.2	71.8	83.7	92.2	99.3	98.1	87.1	77.7	66.0	54.1
279	26.2	49.8	72.5	37.1	35.5	39.2	48.9	71.3	83.3	91.8	99.0	97.8	86.8	77.5	65.8	53.9
282	26.5	49.7	70.4	36.8	35.3	39.0	48.8	71.0	83.0	91.5	98.9	97.7	86.6	77.3	65.6	53.7
285	26.8	48.2	68.3	36.6	35.2	39.0	48.7	70.8	82.8	91.4	98.8	97.6	86.5	77.1	65.5	53.6

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE							
ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH	
1	264.0(75)	.0(0)	.0(0)	30.0(75)	9.68(48)	.745(126)	

FORM 4: CONTINUED

2	271.0(78)	.0(0)	.0(0)	30.8(78)	9.21(51)	.688(126)
3	267.8(81)	.0(0)	.0(0)	30.4(81)	8.75(54)	.632(129)
4	254.2(84)	.0(0)	.0(0)	28.9(84)	8.25(57)	.577(129)
5	240.7(99)	.0(0)	.0(0)	28.5(99)	7.70(60)	.521(132)
6	228.8(105)	.0(0)	.0(0)	27.2(105)	7.14(63)	.467(132)
7	217.1(111)	.0(0)	.0(0)	25.8(111)	6.58(66)	.416(132)
8	205.8(117)	.0(0)	.0(0)	24.5(117)	6.10(69)	.367(135)
9	192.7(120)	.0(0)	.0(0)	23.6(120)	5.57(72)	.320(135)
10	179.3(126)	.0(0)	.0(0)	24.2(126)	4.84(78)	.271(138)
11	165.0(129)	.0(0)	.0(0)	22.3(129)	4.17(78)	.226(141)
12	149.5(129)	.0(0)	.0(0)	20.2(129)	3.56(81)	.185(141)
13	132.9(132)	.0(0)	.0(0)	18.0(132)	3.02(84)	.149(144)
14	115.6(135)	.0(0)	.0(0)	15.6(135)	2.49(87)	.117(144)
15	97.5(138)	.0(0)	.0(0)	13.2(138)	2.00(90)	.091(147)
16	80.6(138)	.0(0)	.0(0)	9.1(138)	1.60(93)	.069(150)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 12.2 K-FT
ANALYSIS WAS TERMINATED AS NO PERMANENT SET RESULTED

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FORM 4: CONTINUED

EXAMPLE 7.2 DRIVEABILITY CHECK LB 520

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI	BLOWS/ MINUTE	B.C. PR. PSI
1	120.0	221.	3.60(1)	.00	30.54	73.9	24.7
2	160.0	999999.	3.60(2)	.00	30.80	73.9	24.7

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was once reduced (a computational process that is not related to a physical process).

For RULT = 160 no permanent set resulted.

It must be concluded that even if driving is not interrupted and therefore no set up occurs during the driving process, the blow count is already too high to be economical. Note that it is not very easy to keep the stroke exactly at maximum thus it can be expected that driving is even harder. The maximum stresses were about 31 ksi and occurred at the top (see list of maxima for individual elements in Form 4). Thus, the danger of yield, which will be greatest at the points of cross sectional change, is only limited. A recommendation based on these results is to increase the wall thickness rather than the hammer size. (Stiffer piles drive better).

7.3 Tension Stress Check

7.3.1 Situation

Using a Vulcan 80 C hammer a 14x14 inch prestressed concrete pile is to be driven through very soft material. The pile length is 50 feet. The soil engineer has estimated that there will be only 10 tons of skin resistance (no tip resistance) in the early stages of driving (20 feet penetration) and that skin damping (Smith) is equal to 0.2.

7.3.2 Problem

It is expected that tension stresses will develop in the pile during

the early driving stages. These tension stresses should at no point exceed 1.0 ksi (prestress). How many sheets of 3/4 inch plywood should be put on the pile top to sufficiently protect the pile? The plywood sheets are to be reused. Capblock properties are given as: weight = 1.5 kips, stiffness = 10,000 kips/inch and coefficient of restitution = 0.8.

7.3.3 Solution

The following input was made solving first the case of a cushion consisting of 3 plywood sheets.

Card No. 2.000 IOU = 3 for a print of stresses
 IHAMR = 65 from Table 2
 IPERCS = 100 no toe resistance
 ISMITH = 1 for Smith's damping
 IBEDAM = 3 for concrete

Card No. 3.000 Wt. Cap = 1.5 kips
 Stiff. Cap = 10000, kips/inch
 Stiff. Cushion = 3360; This value was derived from
 AE/L with -
 A = 196 inch² (equal to the pile
 cross sectional area)
 E = 30 ksi (estimated for
 plywood)
 L = 3 x 0.58 inch (assuming that
 the plywood was compressed to one
 and three-quarter inches during use)

Card No. 4.000 C.O.R. of Cap = 0.8, of pile top = 1.0, of Cushion =
 0.5, of assembly = blank (normal).

Card No. 5.000 L Pile = 50. feet
 A Pile Top = 196. square inch
 E Pile Top = 5000. ksi (assumed)
 W Pile Top = 150. lbs. per cu. ft. (standard)

Card No. 6.000 Quakes = 0.1 inch
 Skin Damping = 0.2 sec/ft

Toe Damping	Not specified since there is no toe resistance force.
RULT = 10.0	tons
	All other data are left blank for standard performance.

Card No. 6.401,... From 0 to 30 feet below the pile top no skin friction. From 30 to 50 feet uniform skin friction.

The filled-in input form is shown in Form 5. The corresponding output is reproduced in Form 6.

7.3.4 Discussion of Results

The output shows that the tension stresses are greater than 1.0 ksi at several segments (note that the output gives tension as a negative stress). Driving the pile with only 3 cushion sheets is therefore not advisable.

7.3.5 Additional Computer Analysis

It is concluded that more cushion sheets should be used and a second computer run is made for six cushion sheets. Thus the cushion stiffness on Card No. 3.000 is merely divided by 2. The first and last page of the output is reproduced in Form 7. Obviously the maximum tension stress was reduced such that tension cracks are less likely to occur. Driving with reduced pressures in the early phases of driving is recommended for a better and more effective means of getting the pile safely into the ground. The program could have equally well been used to find PSTREAM which would also limit tension in easy driving.

WAVE EQUATION ANALYSIS for PILES Short Input Form, page 1

1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40

Title (40 characters)

EXAMPLE 7.3 TENSION STRESS CHECK 3-PLY

ISPL NCROSS ISMITH
 No. of 0 =
 Splices Uniform IBEDAM IPERCS I-Spith ITYS IPHI

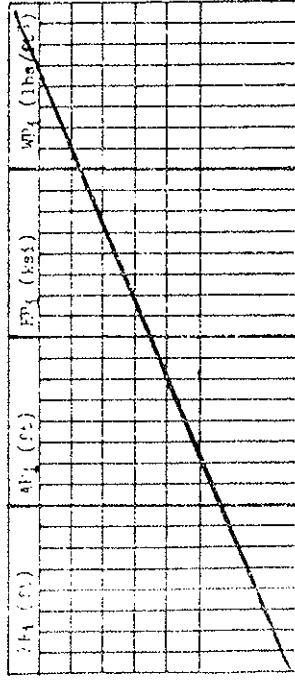
ICUT IUJ IHAMR IOSTR IFUEL IPEL N 3 100 1

Cap and Cushion
 Wt. Cap (Wt.) Stiff. Cap (K/in) Stiff. Cushion (K/in)
 1.5 1000. 3360.
 Coefficients of Restitution
 Anvil (Diesel) Cap (Diesel) Pile Top Cushion TDEL (sec.)
 .8 1.0 .5

For 6 plywood sheets, cushion stiffness is 1,680 k/in

W File A File Top E File Top W File Top
 (ft) (ln²) (ksl) (lbs/ft²)
 50. 196. 5000. 150.

If HCR-SS = 1 and IPEL ≠ 2 only



5.101
 5.102
 .
 .
 .
 5.119

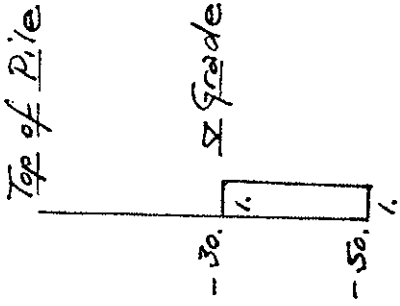
WAVE EQUATION ANALYSIS for PILES Short Input Form, page 2

6.000	Quake Sfdn (In)	Quake Toe (In)	Damping Sfdn*	Damping Toe*	RULT (tons)	Coeff. of Rest. Of Soil	Stroke (ft)	Hammer Efficiency	Steam Pressure (psi)	Reaction wt. (kips)
	0.1	0.1	0.2	10.						

*Damping Parameters - sec/ft for Smith; Dimensionless for Case

If IUTS = 0 or -1 only

XPI (ft)	% Soil Res.
0.	0.
50.	0.
30.	1.
50.	1.



If IOUT > 9

7MAX	Plotter Scale

If IOUT > 9 only

ISDC(1), I = 1, 2, ..., 15 (Plotted Variables)

If RULT < 0 only:

Ultimate Resistance - tons (at least 10 values)

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30
---	---	---	---	---	---	---	---	---	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----

FORM 5: CONTINUED

W E A P - WAVE EQUATION ANALYSIS FOR PILES

THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.3 TENSION STRESS CHECK 3-PLY
PILE DESCRIPTION

X BEL. TOP (FT) .0 50.0
A (SQ. IN.) 196.0 196.0
E (KSI) 5000. 5000.
GAMMA (LB/CU FT) 150.0 150.0
STROKE (EQUIV.) 3.1 FT, EFFICIENCY .80 , IMPACT VELOCITY 12.5 FT/S
ACTUAL/ MAX. PRESSURE 120.0/ 120.0 PSI, REACT./RAM WEIGHT 9.8/ 8.0 KIPS

HAMMER MODEL VULC.80C

ELEMENT NUMBER	WEIGHT (KIPS)	STIFFNESS (K/IN)	COEFF. RESTITUTION
1	8.000		
CAP	1.500	10000.0	.800
CUSHION		3360.0	.500
PILE TOP			1.000

PILE PROPERTIES

PILE LENGTH= 50. FT., AREA(AT TOP)= 196.0 S-IN
 E. MODUL(AT TOP)= 5000. KSI., SPEC. WT.(AT TOP)= 150. LBS/CU FT

NO.	WEIGHT (KIPS)	STIFFN. (K/IN)	PDAMP. (KS/FT)	SPLICE (KIPS)	SOIL-S (PCT.)	SOIL-D (S/FT)	QUAKE (IN.)	L.B.T. (FT.)
1	.928	17967.	4.73	0.	.000	.200	.100	4.5
2	.928	17967.	4.73	-5000.	.000	.200	.100	9.1
3	.928	17967.	4.73	-5000.	.000	.200	.100	13.6
4	.928	17967.	4.73	-5000.	.000	.200	.100	18.2
5	.928	17967.	4.73	-5000.	.000	.200	.100	22.7
6	.928	17967.	4.73	-5000.	.000	.200	.100	27.3
7	.928	17967.	4.73	-5000.	.091	.200	.100	31.8
8	.928	17967.	4.73	-5000.	.227	.200	.100	36.4
9	.928	17967.	4.73	-5000.	.227	.200	.100	40.9
10	.928	17967.	4.73	-5000.	.227	.200	.100	45.5
11	.928	17967.	4.73	-5000.	.227	.200	.100	50.0
TOE					.000	.000	.100	

COEFFICIENT OF RESTITUTION OF SOIL 1.000

OPTIONS AND SPECIFICATIONS					
PHI	1.60	S-DAMPING	SMITH	RWT (KIPS)	9.78
IOUT	3	P-DAMPING	3	SOIL DIST. NO.	0
IFUEL	1	J SKIN	.20	TDEL (SEC.)	.0000
IOSTR	0	J TOE	.00	TEMAX (MS)	.00
		TIME INCR. (MS)	.184		

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RULT= 10.0 AT TOE= .0 TONS

HAMMER AND PILE STRESSES (KSI)

JP	TIME (MS)	HAMMER		PILE ELEMENTS										
		RAM M	ANVIL	TOP	2	3	4	5	6	7	8	9	10	11
1	.2	.00	.00	.06	.06	.06	.06	.06	.05	.05	.04	.03	.03	.02
2	.4	.00	.00	.06	.06	.06	.06	.06	.05	.05	.04	.03	.03	.02
3	.6	.00	.00	.06	.06	.06	.06	.06	.05	.05	.04	.03	.03	.01
4	.7	.00	.00	.06	.06	.06	.06	.05	.05	.05	.04	.03	.02	.01
5	.9	.00	.00	.06	.06	.06	.06	.05	.05	.04	.04	.03	.02	.01
6	1.1	.00	.00	.06	.05	.05	.05	.05	.05	.04	.04	.03	.02	.01
7	1.3	.00	.00	.06	.05	.05	.05	.05	.05	.04	.04	.03	.01	.01
8	1.5	.00	.00	.06	.05	.05	.05	.05	.05	.04	.03	.02	.01	.01
9	1.7	.00	.00	.06	.05	.05	.05	.05	.04	.04	.03	.02	.01	.01
10	1.8	.00	.00	.06	.05	.05	.04	.04	.04	.03	.03	.02	.01	.01
11	2.0	.00	1.23	.06	.05	.05	.04	.04	.04	.03	.02	.01	.01	.01
12	2.2	.00	3.09	.06	.05	.05	.04	.04	.03	.03	.02	.01	.01	.00
13	2.4	.00	5.38	.07	.06	.05	.04	.03	.03	.02	.02	.01	.01	.00
14	2.6	.00	7.85	.08	.07	.05	.04	.03	.02	.02	.01	.01	.01	.00
15	2.8	.00	10.28	.10	.09	.05	.04	.03	.02	.01	.01	.01	.01	.00
16	2.9	.00	12.39	.14	.13	.07	.04	.02	.01	.01	.01	.01	.01	.00
17	3.1	.00	13.97	.21	.18	.09	.04	.02	.01	.01	.01	.01	.00	.00
18	3.3	.00	14.89	.31	.26	.12	.05	.02	.01	.01	.01	.00	.00	.00
19	3.5	.00	15.13	.45	.35	.18	.08	.03	.01	.01	.00	.00	.00	.00
20	3.7	.00	14.51	.65	.49	.25	.11	.04	.02	.01	.00	.00	.00	.00
21	3.9	.00	13.16	.90	.66	.35	.17	.07	.03	.01	.00	.00	-.00	-.00
22	4.0	.00	11.33	1.20	.87	.48	.24	.11	.04	.02	.01	.00	-.00	-.00
23	4.2	.00	9.33	1.52	1.12	.65	.35	.17	.07	.03	.01	.00	-.00	-.00
24	4.4	.00	7.46	1.84	1.41	.86	.48	.24	.11	.04	.02	.00	-.00	-.00
25	4.6	.00	5.93	2.12	1.71	1.11	.65	.35	.17	.07	.03	.01	.00	-.00
26	4.8	.00	4.83	2.34	2.01	1.40	.86	.48	.25	.11	.04	.02	.00	.00
27	5.0	.00	4.15	2.51	2.26	1.70	1.11	.66	.35	.17	.07	.03	.01	.00
28	5.1	.00	3.86	2.62	2.46	1.99	1.39	.87	.49	.25	.12	.05	.02	.01
29	5.3	.00	3.90	2.66	2.59	2.24	1.68	1.11	.66	.36	.18	.08	.03	.01
30	5.5	.00	4.19	2.66	2.66	2.45	1.97	1.39	.87	.50	.26	.12	.05	.02
31	5.7	.00	4.76	2.56	2.67	2.59	2.23	1.67	1.11	.67	.37	.18	.08	.03
32	5.9	.00	5.61	2.40	2.63	2.66	2.44	1.96	1.38	.88	.51	.26	.12	.05
33	6.1	.00	6.74	2.19	2.52	2.67	2.58	2.22	1.67	1.12	.68	.37	.18	.07
34	6.2	.00	8.09	1.99	2.36	2.62	2.65	2.43	1.95	1.39	.89	.51	.26	.11
35	6.4	.00	9.59	1.80	2.16	2.51	2.66	2.57	2.21	1.67	1.12	.68	.37	.15
36	6.6	.00	11.13	1.65	1.95	2.36	2.61	2.65	2.42	1.95	1.38	.88	.49	.22
37	6.8	.00	12.56	1.55	1.76	2.17	2.51	2.66	2.57	2.20	1.66	1.11	.65	.29
38	7.0	.00	13.78	1.49	1.61	1.96	2.36	2.61	2.65	2.42	1.93	1.35	.83	.38
39	7.2	.00	14.66	1.48	1.52	1.77	2.18	2.51	2.67	2.57	2.18	1.61	1.02	.49
40	7.3	.00	15.15	1.50	1.48	1.62	1.98	2.37	2.62	2.65	2.38	1.86	1.23	.60
41	7.5	.00	15.24	1.54	1.48	1.52	1.80	2.19	2.52	2.67	2.52	2.07	1.43	.72
42	7.7	.00	14.82	1.60	1.52	1.48	1.64	2.00	2.37	2.62	2.60	2.23	1.61	.83
43	7.9	.00	13.97	1.68	1.58	1.49	1.54	1.82	2.20	2.50	2.59	2.33	1.75	.93
44	8.1	.00	12.79	1.77	1.65	1.53	1.49	1.66	2.01	2.34	2.50	2.34	1.82	.99
45	8.3	.00	11.44	1.86	1.74	1.59	1.49	1.55	1.81	2.14	2.35	2.26	1.81	1.01
46	8.4	.00	10.07	1.94	1.83	1.66	1.53	1.49	1.64	1.92	2.13	2.09	1.71	.97
47	8.6	.00	8.79	2.00	1.92	1.75	1.59	1.47	1.50	1.69	1.86	1.84	1.53	.88
48	8.8	.00	7.71	2.05	1.99	1.83	1.66	1.49	1.40	1.46	1.55	1.52	1.27	.73

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49	9.0	.00	6.86	2.07	2.04	1.91	1.73	1.53	1.33	1.24	1.23	1.15	.94	.54
50	9.2	.00	6.24	2.06	2.07	1.98	1.79	1.56	1.29	1.05	.90	.75	.57	.31
51	9.4	.00	5.86	2.01	2.07	2.02	1.84	1.59	1.25	.88	.58	.35	.19	.08
52	9.5	.00	5.69	1.91	2.03	2.03	1.87	1.59	1.21	.74	.29	-.03	-.18	-.15
53	9.7	.00	5.67	1.79	1.96	2.00	1.87	1.57	1.14	.60	.04	-.37	-.49	-.34
54	9.9	.00	5.76	1.65	1.84	1.93	1.82	1.51	1.05	.46	-.16	-.62	-.73	-.48
55	10.1	.00	5.93	1.50	1.69	1.81	1.72	1.42	.92	.32	-.31	-.79	-.88	-.57
56	10.3	.00	6.14	1.35	1.50	1.63	1.57	1.27	.77	.18	-.41	-.86	-.93	-.59
57	10.5	.00	6.37	1.22	1.31	1.41	1.36	1.08	.60	.05	-.47	-.84	-.87	-.55
58	10.6	.00	6.59	1.09	1.10	1.15	1.09	.84	.41	-.07	-.47	-.73	-.73	-.46
59	10.8	.00	6.76	.97	.89	.87	.78	.55	.21	-.16	-.43	-.57	-.53	-.32
60	11.0	.00	6.86	.87	.69	.56	.43	.24	.00	-.22	-.35	-.36	-.28	-.16
61	11.2	.00	6.86	.77	.48	.26	.08	-.08	-.20	-.25	-.22	-.13	-.03	.01
62	11.4	.00	6.73	.68	.29	-.03	-.27	-.38	-.38	-.25	-.08	.09	.20	.16
63	11.6	.00	6.46	.58	.09	-.30	-.58	-.66	-.53	-.24	.08	.30	.39	.28
64	11.7	.00	6.03	.49	-.09	-.54	-.84	-.88	-.64	-.21	.21	.47	.53	.36
65	11.9	.00	5.48	.40	-.25	-.74	-1.04	-1.03	-.70	-.18	.32	.59	.61	.39
66	12.1	.00	4.81	.30	-.39	-.89	-1.16	-1.10	-.72	-.15	.37	.64	.63	.38
67	12.3	.00	4.06	.20	-.50	-.98	-1.19	-1.09	-.69	-.12	.38	.63	.59	.34
68	12.5	.00	3.27	.10	-.58	-1.00	-1.15	-1.01	-.62	-.11	.33	.54	.50	.28
69	12.7	.00	2.48	.02	-.59	-.96	-1.04	-.87	-.52	-.10	.24	.40	.37	.20
70	12.8	.00	1.72	.00	-.54	-.84	-.86	-.68	-.41	-.11	.11	.21	.20	.12
71	13.0	.00	1.04	.00	-.43	-.65	-.64	-.48	-.29	-.13	-.04	-.00	.02	.02
72	13.2	.00	.48	.00	-.27	-.41	-.38	-.27	-.18	-.16	-.19	-.21	-.16	-.08
73	13.4	.00	.07	.00	-.09	-.13	-.10	-.06	-.09	-.20	-.33	-.40	-.34	-.18
74	13.6	.00	.00	.00	.11	.17	.18	.12	-.02	-.23	-.45	-.55	-.48	-.27
75	13.8	.00	.00	.00	.29	.45	.43	.29	.03	-.26	-.53	-.66	-.59	-.35
76	13.9	.00	.00	.00	.44	.68	.66	.42	.08	-.28	-.56	-.72	-.66	-.40
77	14.1	.00	.00	.00	.53	.85	.82	.53	.12	-.27	-.56	-.72	-.67	-.41
78	14.3	.00	.00	.00	.58	.93	.92	.61	.17	-.23	-.51	-.67	-.63	-.39
79	14.5	.00	.00	.00	.56	.92	.95	.66	.23	-.15	-.43	-.57	-.55	-.33
80	14.7	.00	.00	.00	.50	.84	.89	.67	.30	-.05	-.30	-.43	-.41	-.24
81	14.9	.00	.00	.00	.40	.68	.77	.65	.37	.09	-.13	-.26	-.25	-.14
82	15.0	.00	.00	.00	.27	.48	.60	.59	.44	.25	.06	-.05	-.06	-.03
83	15.2	.00	.00	.00	.13	.26	.40	.49	.50	.41	.28	.18	.14	.08
84	15.4	.00	.00	.00	-.01	.04	.19	.38	.53	.57	.52	.43	.33	.18
85	15.6	.00	.00	.00	-.13	-.15	-.01	.26	.53	.71	.75	.66	.51	.28
86	15.8	.00	.00	.00	-.22	-.30	-.17	.14	.52	.82	.95	.88	.66	.36
87	16.0	.00	.00	.00	-.28	-.40	-.29	.04	.48	.90	1.11	1.06	.80	.42
88	16.2	.00	.00	.00	-.30	-.44	-.35	-.03	.44	.93	1.21	1.18	.90	.48
89	16.3	.00	.00	.00	-.28	-.42	-.36	-.06	.41	.92	1.24	1.24	.96	.52
90	16.5	.00	.00	.00	-.23	-.35	-.31	-.05	.38	.86	1.20	1.23	.99	.54
91	16.7	.00	.00	.00	-.15	-.24	-.21	.00	.37	.78	1.09	1.16	.96	.53
92	16.9	.00	.00	.00	-.06	-.10	-.06	.10	.37	.68	.94	1.02	.87	.50
93	17.1	.00	.00	.00	.03	.06	.11	.21	.38	.57	.75	.82	.73	.44
94	17.3	.00	.00	.00	.12	.22	.29	.35	.40	.47	.55	.59	.55	.34
95	17.4	.00	.00	.00	.21	.38	.47	.47	.42	.37	.34	.34	.32	.21
96	17.6	.00	.00	.00	.29	.51	.62	.59	.44	.28	.16	.09	.08	.06
97	17.8	.00	.00	.00	.34	.62	.74	.67	.46	.21	-.01	-.14	-.16	-.10
98	18.0	.00	.00	.00	.39	.68	.81	.72	.47	.15	-.14	-.35	-.38	-.24
99	18.2	.00	.00	.00	.40	.71	.82	.72	.46	.10	-.25	-.50	-.55	-.36
100	18.4	.00	.00	.00	.40	.68	.78	.69	.43	.06	-.32	-.61	-.66	-.43
101	18.5	.00	.00	.00	.36	.61	.70	.61	.37	.02	-.36	-.65	-.70	-.45
102	18.7	.00	.00	.00	.30	.50	.57	.49	.29	-.02	-.36	-.63	-.66	-.42

103	18.9	.00	.00	.00	.00	.21	.36	.40	.35	.18	-.06	-.34	-.56	-.35
104	19.1	.00	.00	.00	.00	.11	.19	.22	.17	.06	-.11	-.30	-.43	-.25
105	19.3	.00	.00	.00	.00	.01	.01	.01	-.02	-.08	-.16	-.23	-.27	-.12
106	19.5	.00	.00	.00	.00	-.10	-.17	-.20	-.21	-.22	-.19	-.14	-.09	.00
107	19.6	.00	.00	.00	.00	-.20	-.34	-.40	-.41	-.35	-.22	-.05	.09	.12
108	19.8	.00	.00	.00	.00	-.28	-.49	-.59	-.58	-.46	-.23	.04	.24	.21
109	20.0	.00	.00	.00	.00	-.34	-.61	-.75	-.73	-.54	-.22	.13	.37	.27
110	20.2	.00	.00	.00	.00	-.39	-.70	-.87	-.83	-.59	-.20	.19	.48	.31
111	20.4	.00	.00	.00	.00	-.41	-.75	-.93	-.88	-.60	-.17	.23	.47	.31
112	20.6	.00	.00	.00	.00	-.42	-.77	-.94	-.87	-.58	-.15	.23	.44	.28
113	20.7	.00	.00	.00	.00	-.41	-.73	-.89	-.81	-.52	-.13	.20	.36	.22
114	20.9	.00	.00	.00	.00	-.37	-.66	-.78	-.70	-.44	-.12	.14	.25	.15
115	21.1	.00	.00	.00	.00	-.32	-.54	-.62	-.54	-.35	-.11	.04	.11	.06
116	21.3	.00	.00	.00	.00	-.24	-.39	-.43	-.36	-.24	-.13	-.06	-.05	-.03
117	21.5	.00	.00	.00	.00	-.14	-.21	-.21	-.17	-.14	-.15	-.17	-.20	-.11
118	21.7	.00	.00	.00	.00	-.02	-.01	.02	.02	-.05	-.16	-.27	-.34	-.19
119	21.8	.00	.00	.00	.00	.10	.19	.23	.19	.03	-.17	-.36	-.46	-.24
120	22.0	.00	.00	.00	.00	.22	.38	.42	.33	.10	-.17	-.41	-.53	-.28
121	22.2	.00	.00	.00	.00	.32	.53	.58	.44	.16	-.16	-.42	-.56	-.30
122	22.4	.00	.00	.00	.00	.39	.65	.68	.51	.21	-.12	-.40	-.55	-.29
123	22.6	.00	.00	.00	.00	.43	.71	.74	.55	.25	-.07	-.34	-.48	-.27
124	22.8	.00	.00	.00	.00	.44	.71	.74	.57	.29	.00	-.24	-.38	-.22
125	22.9	.00	.00	.00	.00	.40	.65	.69	.55	.32	.00	-.10	-.23	-.16
126	23.1	.00	.00	.00	.00	.32	.54	.60	.52	.36	.19	.05	-.06	-.08
127	23.3	.00	.00	.00	.00	.22	.39	.47	.46	.39	.31	.22	.12	.02
128	23.5	.00	.00	.00	.00	.11	.23	.32	.39	.42	.43	.39	.31	.11
129	23.7	.00	.00	.00	.00	.00	.06	.17	.31	.44	.54	.56	.50	.29
130	23.9	.00	.00	.00	.00	-.09	-.09	.03	.23	.46	.64	.72	.67	.52
131	24.0	.00	.00	.00	.00	-.16	-.21	-.09	.16	.46	.72	.85	.81	.64
132	24.2	.00	.00	.00	.00	-.21	-.29	-.18	.10	.45	.77	.94	.92	.73
133	24.4	.00	.00	.00	.00	-.22	-.32	-.22	.06	.44	.79	.98	.98	.79
134	24.6	.00	.00	.00	.00	-.20	-.30	-.22	.04	.41	.78	1.00	1.00	.44
135	24.8	.00	.00	.00	.00	-.16	-.24	-.17	.05	.39	.74	.96	.97	.78
136	25.0	.00	.00	.00	.00	-.10	-.14	-.09	.09	.37	.67	.88	.89	.72
137	25.1	.00	.00	.00	.00	-.02	-.02	.03	.15	.36	.59	.76	.77	.62
138	25.3	.00	.00	.00	.00	.06	.11	.16	.24	.35	.50	.61	.60	.49
139	25.5	.00	.00	.00	.00	.13	.24	.30	.32	.35	.40	.44	.42	.34
140	25.7	.00	.00	.00	.00	.20	.36	.42	.41	.36	.30	.26	.22	.17
141	25.9	.00	.00	.00	.00	.26	.45	.53	.49	.36	.22	.10	.02	.00
142	26.1	.00	.00	.00	.00	.30	.52	.60	.54	.36	.14	-.05	-.16	-.09
143	26.2	.00	.00	.00	.00	.32	.55	.64	.56	.35	.08	-.16	-.31	-.18
144	26.4	.00	.00	.00	.00	.32	.55	.64	.55	.33	.03	-.25	-.42	-.25
145	26.6	.00	.00	.00	.00	.29	.51	.59	.51	.28	-.01	-.30	-.49	-.30
146	26.8	.00	.00	.00	.00	.25	.44	.50	.43	.22	-.04	-.31	-.50	-.32
147	27.0	.00	.00	.00	.00	.19	.34	.38	.31	.15	-.07	-.30	-.48	-.31
148	27.2	.00	.00	.00	.00	.12	.21	.23	.18	.06	-.10	-.27	-.41	-.26
149	27.3	.00	.00	.00	.00	.04	.07	.07	.03	.04	-.12	-.22	-.31	-.19
150	27.5	.00	.00	.00	.00	-.04	-.08	-.11	-.13	.14	-.14	-.16	-.19	-.10
151	27.7	.00	.00	.00	.00	-.12	-.22	-.28	-.22	.24	-.16	-.10	-.17	-.10
152	27.9	.00	.00	.00	.00	-.20	-.35	-.43	-.42	.33	-.18	-.03	-.06	-.01
153	28.1	.00	.00	.00	.00	-.26	-.46	-.56	-.54	.40	-.19	-.03	.07	.08
154	28.3	.00	.00	.00	.00	-.31	-.55	-.66	-.63	.46	-.19	.08	.18	.16
155	28.4	.00	.00	.00	.00	-.34	-.60	-.72	-.68	.49	-.19	.12	.26	.21
156	28.6	.00	.00	.00	.00	-.35	-.62	-.75	-.70	.49	-.18	.14	.33	.24

157	28.6	.00	.00	.00	.34	.60	.72	.68	.47	.16	.13	.30	.33	.21
158	29.0	.00	.00	.00	.31	.55	.66	.61	.42	.15	.10	.24	.26	.16
159	29.2	.00	.00	.00	.26	.47	.56	.52	.35	.13	.06	.15	.16	.10
160	29.4	.00	.00	.00	.20	.36	.42	.39	.27	.12	.00	.04	.05	.03
161	29.5	.00	.00	.00	.13	.22	.26	.25	.18	.11	.08	.07	.07	.04
162	29.7	.00	.00	.00	.05	.08	.09	.09	.09	.11	.15	.19	.18	.11
163	29.9	.00	.00	.00	.04	.07	.08	.06	.01	.10	.15	.19	.27	.16
164	30.1	.00	.00	.00	.12	.21	.25	.20	.07	.10	.26	.36	.34	.20
165	30.3	.00	.00	.00	.19	.34	.39	.32	.14	.08	.29	.41	.38	.23
166	30.5	.00	.00	.00	.26	.44	.50	.41	.20	.06	.28	.41	.39	.23
167	30.6	.00	.00	.00	.30	.51	.57	.47	.25	.02	.25	.38	.36	.22
168	30.8	.00	.00	.00	.32	.54	.61	.50	.28	.03	.18	.31	.31	.19
169	31.0	.00	.00	.00	.32	.54	.60	.51	.31	.10	.09	.21	.22	.14
170	31.2	.00	.00	.00	.29	.49	.55	.48	.34	.17	.03	.09	.12	.08
171	31.4	.00	.00	.00	.24	.41	.47	.44	.36	.26	.16	.06	.01	.01
172	31.6	.00	.00	.00	.18	.31	.37	.39	.39	.35	.30	.21	.14	.07
173	31.8	.00	.00	.00	.10	.19	.26	.33	.39	.44	.44	.37	.44	.14
174	31.9	.00	.00	.00	.02	.07	.15	.27	.40	.52	.56	.51	.29	.21
175	32.1	.00	.00	.00	.05	.04	.05	.21	.41	.58	.67	.63	.50	.27
176	32.3	.00	.00	.00	.11	.13	.04	.16	.41	.63	.76	.73	.58	.32
177	32.5	.00	.00	.00	.14	.19	.09	.13	.41	.66	.81	.80	.64	.35
178	32.7	.00	.00	.00	.15	.21	.12	.11	.40	.67	.83	.82	.66	.37
179	32.9	.00	.00	.00	.14	.20	.11	.10	.39	.66	.82	.81	.65	.36
180	33.0	.00	.00	.00	.11	.15	.07	.12	.37	.62	.77	.76	.61	.34
181	33.2	.00	.00	.00	.06	.07	.01	.15	.35	.56	.69	.68	.54	.30
182	33.4	.00	.00	.00	.00	.02	.08	.19	.34	.49	.59	.57	.45	.25
183	33.6	.00	.00	.00	.07	.12	.24	.24	.32	.41	.46	.43	.34	.18
184	33.8	.00	.00	.00	.13	.22	.28	.30	.31	.33	.33	.28	.21	.11
185	34.0	.00	.00	.00	.18	.31	.37	.35	.30	.24	.19	.13	.08	.04
186	34.1	.00	.00	.00	.22	.38	.44	.39	.29	.16	.06	.02	.05	.04
187	34.3	.00	.00	.00	.25	.42	.48	.42	.27	.09	.06	.16	.17	.10
188	34.5	.00	.00	.00	.25	.44	.50	.42	.25	.04	.15	.27	.26	.16
189	34.7	.00	.00	.00	.24	.42	.48	.40	.21	.01	.22	.34	.33	.20
190	34.9	.00	.00	.00	.22	.37	.42	.35	.17	.05	.26	.39	.37	.22
191	35.1	.00	.00	.00	.18	.30	.34	.28	.12	.08	.27	.39	.38	.23
192	35.2	.00	.00	.00	.12	.21	.23	.18	.06	.10	.26	.37	.35	.21
193	35.4	.00	.00	.00	.07	.10	.10	.06	.02	.12	.23	.31	.30	.18
194	35.6	.00	.00	.00	.00	.01	.03	.06	.09	.13	.18	.23	.22	.13
195	35.8	.00	.00	.00	.06	.13	.17	.18	.17	.14	.13	.14	.12	.07
196	36.0	.00	.00	.00	.12	.23	.30	.30	.24	.15	.07	.04	.02	.01
197	36.2	.00	.00	.00	.18	.33	.41	.40	.31	.16	.02	.05	.08	.05
198	36.3	.00	.00	.00	.23	.41	.50	.48	.36	.16	.02	.13	.16	.11
199	36.5	.00	.00	.00	.26	.46	.56	.53	.39	.16	.06	.19	.22	.15
200	36.7	.00	.00	.00	.28	.49	.59	.56	.40	.16	.07	.22	.26	.17

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	522.3(29)	.0(0)	.0(0)	2.7(29)	11.56(73)	3.426(272)
2	524.2(31)	-115.4(69)	-.6(69)	2.7(31)	10.58(73)	3.425(272)
3	522.4(33)	-196.7(68)	-1.0(68)	2.7(33)	9.21(78)	3.424(272)
4	521.9(35)	-234.0(67)	-1.2(67)	2.7(35)	9.89(82)	3.423(272)

5	522.1(37)	-215.6(66)	-1.1(66)	2.7(37)	10.34(83)	3.421(272)
6	523.0(39)	-140.6(66)	-.7(66)	2.7(39)	11.04(61)	3.418(272)
7	523.1(41)	-54.1(76)	-.3(76)	2.7(41)	11.13(61)	3.415(272)
8	508.8(42)	-110.7(76)	-.6(76)	2.6(42)	10.60(59)	3.412(272)
9	458.3(44)	-168.4(56)	-.9(56)	2.3(44)	9.85(56)	3.408(272)
10	356.0(44)	-181.7(56)	-.9(56)	1.8(44)	11.40(51)	3.406(272)
11	197.9(45)	-115.3(56)	-.6(56)	1.0(45)	12.55(51)	3.405(272)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 14.2 K-FT

T6

EXAMPLE 7.3 TENSION STRESS CHECK 3-PLY

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI
1	10.0	4.	3.06	-1.19	2.67

FORM 6: CONTINUED

W E A P - WAVE EQUATION ANALYSIS FOR PILES

THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.3 TENSION STRESS CHECK 6-PLY
PILE DESCRIPTION

X BEL. TOP (FT) .0 50.0
A (SQ. IN.) 196.0 196.0
E (KSI) 5000. 5000.
GAMMA (LB/CU FT) 150.0 150.0
STROKE (EQUIV.) 3.1 FT; EFFICIENCY .80 , IMPACT VELOCITY 12.5 FT/S
ACTUAL/ MAX. PRESSURE 120.0/ 120.0 PSI; REACT./RAM WEIGHT 9.8/ 8.0 KIPS

HAMMER MODEL VULC.80C

ELEMENT NUMBER	WEIGHT (KIPS)	STIFFNESS (K/IN)	COEFF. RESTITUTION
1	8.000		
CAP	1.500	10000.0	.800
CUSHION		1680.0	.500
PILE TOP			1.000

92

EXAMPLE 7.3 TENSION STRESS CHECK 6-PLY

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI
1	10.0	4.	3.06	-.62	2.10

7.4 Hypothetical Hammer Input

7.4.1 Situation

A contractor has decided to build his own hammer. He supplied the data as used in Input Sample 5.4. A pile with 12 3/4 inch OD pipe with 1/4 inch wall thickness has to be driven to 90 ton ultimate capacity. The length of the pile is 60 feet including a one inch toe plate.

7.4.2 Problem

Determine whether this new hammer will drive the pile assuming that the 90 ton resistance will be reached at a depth of 50 feet where the otherwise loose sand becomes dense.

7.4.3 Solution

Since the hammer being analyzed is not contained in the data file the Complete Input Form must be used. A sketch is made of the assumed skin friction distribution on Page 3 of the input form

Card No. 2.000	IOUT = 0 for minimum output IHAMR = 0, hammer data to be input NCROSS = 1 nonuniform pile (toe plate) IPERCS = 10 (loose sand on skin, dense sand at toe)
Card No. 3.000	Wt. Cap = .95 kips Stiff. Cap = 21000 kips/inch Stiff. Cushion = leave blank (no cushion)
Card No. 4.000	C.O.R. of Anvil = 0.8 of Cap = 0.8 of Pile Top = 0.8

Card No. 5.000 L Pile = 60.0 feet
 A Pile Top = $9.82 = (12.75 - 0.25)\pi(0.25)$ inch²
 E Pile Top = 30000 ksi
 W Pile Top = 492 lbs/cu.ft.

Card No. 5.101,... The pile is uniform until the end plate, so only three cards are required to specify the discontinuity and cross sectional area of the end plate ($12.75^2\pi/4 = 127.7$ inch²).

Card No. 5.201
 through 5.205 Since IHAMR was set to 0, the next 5 cards must be input. The data is as in 5.4

Card No. 6.000 Quakes = 0.1
 Skin Damping = 0.3
 Toe Damping = 0.15
 RULT = 90 tons
 Stroke = 6.0 feet - assumed stroke is input as a starting value.

Card No. 6.401,... The skin resistance distribution was based on SPT values of 2, between grade and 20 feet depth, 4 between 30 and 40 feet depth, and values increasing to 6 just above the dense sand. Note that these SPT readings could have been inserted directly in the % Soil Res. column with no difference in the resulting resistance distribution. The input is listed in Form 8. The corresponding output is reproduced in Form 9.

7.4.4 Discussion of Results

The summary found on the last page of the output shows all the important information. With RULT = 90, the blowcount is 28, or 2.3 blows per inch. The stroke is 6.0 feet as assumed. The maximum stress is almost 27 ksi. The hammer is running at a speed of 48 blows per minute, driving the pile at a rate of 1.7 feet per minute. It can be concluded that this hammer would be a reasonable choice for driving the pile.

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 1

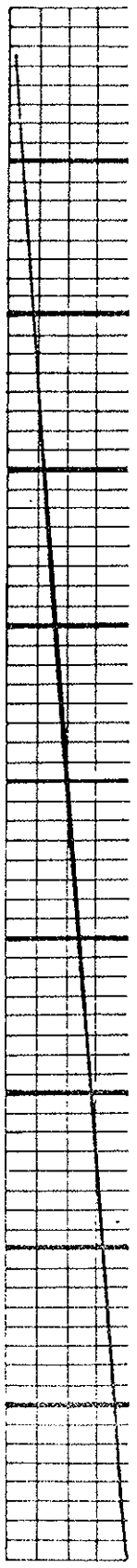
6 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100

Title (40 characters)

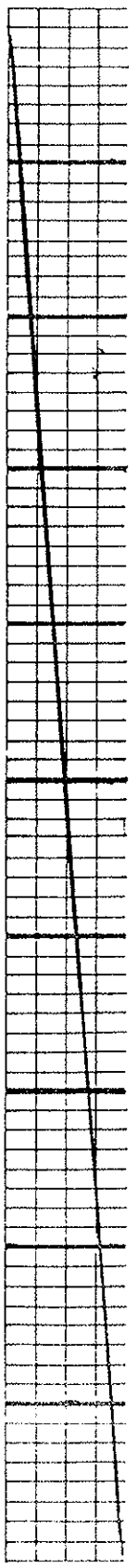
1.000 EXAMPLE 7.14 HYPOTHETICAL HAMMER IMPACT

2.000 IOUT LJN IHAVR IOSTR IFEEL IPEL N ISPL NCROSS ISMITH IPHI
 No. of 0 = Splices Uniform IBEDAM IPERGS I-Smith ITIS

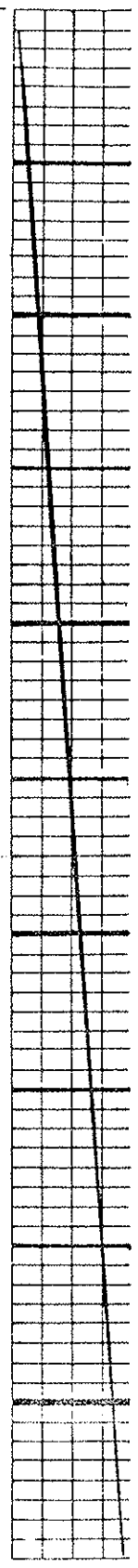
If IPEL = 2
 ST(I), I = 1, 2, ..., N (Pile Segment Stiffnesses - k/in)



If IPEL = 2
 P(I), I = 1, 2, ..., N (Pile Segment Weights - kips)



If IPEL > 2
 ALPH(I), I = 1, 2, ..., N (Relative Lengths of Pile Segments)



3.000 Wt. Cap (kips) 21000
 Cap and Cushion Stiff. Cap (k/in)
 Stiff. Cushion (k/in)

Coefficients of Restitution

4.000 Anvil (Diesel) Cap (Diesel) TBMX TDEL
 Cap (Stress) Assembly(A/S) Pile Top Cushion (sec)

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 2

L pile (ft) 50. A pile top (ksi) 9.182 F pile top (ksi) 30000. W pile top (lbs/ft²) 482.

If NCROSS = 1 and IPEL # 2 only

NP1 (ft)	API (ft)	EPI (ksi)	WP1 (lbs/ft ²)
5.101	9.182		
5.102	127.7		
.	127.7		
5.119			

Farmer Information (if IHAVR = 0 only)

LAVE HM(1) (kips) 917 STM(1) (k/in) 116000. HM(2) (kips) 917 STM(2) (k/in) 116000. HM(3) (k/in) 917 STM(3) (k/in) 116000. HM(4) (kips) 917 STM(4) (k/in) 116000. HM(5) (kips) 917 STM(5) (k/in) 116000.

Farmer Information (if IHAVR = C only)

STM(5) (k/in) 917 HM(6) (kips) 917 STM(6) (k/in) 116000. HM(7) (kips) 917 STM(7) (k/in) 116000. M 3.001 BM(N+1) (kips) 917 STM(N) (k/in) 116000. TDEL (sec) 0.002

Farmer Information (if IHAVR = 0 only)

YIN (in) 120. DEEP (in) 10.76 ARM (in²) 122.74 PL (psi) 1150. P2 (psi) 1150. P3 (psi) 1150. P4 (psi) 1150. P5 (psi) 1150. RFF(OT) 102. STM (in) 102.

Farmer Information (if IHAVR = 0 only)

EXP 1.3 ART (ft²) 1.3 DETB (ft) 1.3 DSP (ft) 1.3 IBBT (ft) 1.3 POWSCA (1) 1.3 EUPB 1.3 DIMJ (ft) 1.3 VCT (ft³) 1.3

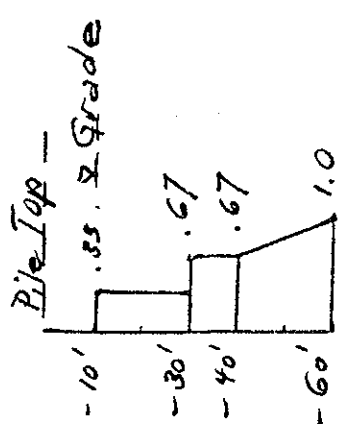
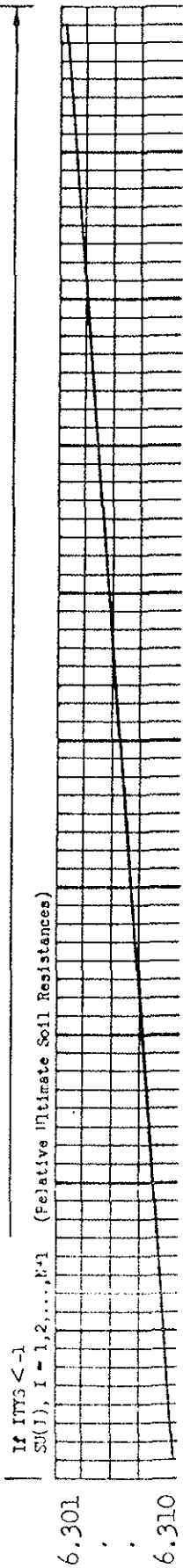
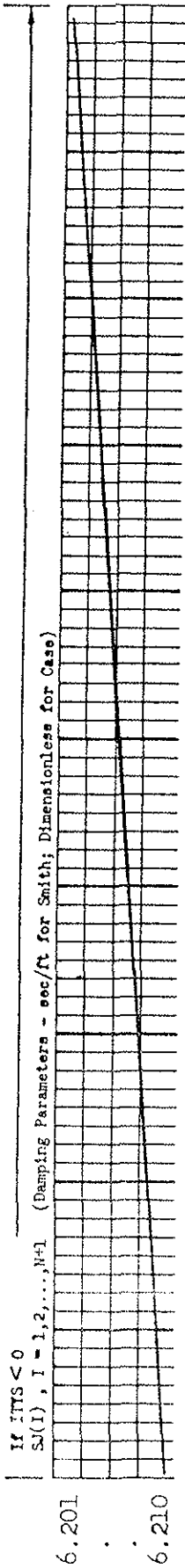
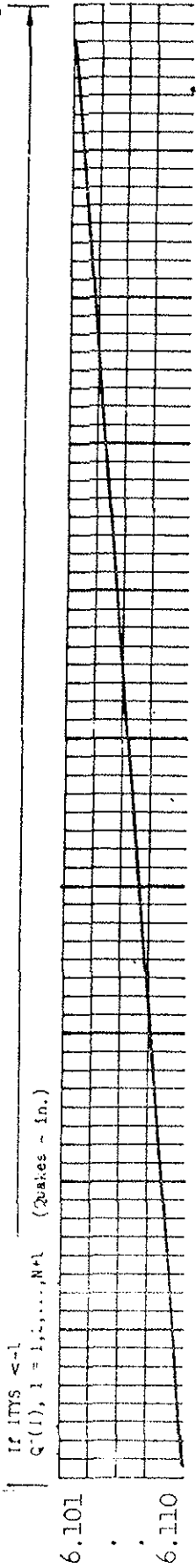
Farmer Information (if IHAVR = 0 only)

RAV (ft) 1.3 AM(1) (kips) 1.3 STA(1) (k/in) 1.3 AM(2) (kips) 1.3 STA(2) (k/in) 1.3 AM(3) (kips) 1.3 STA(3) (k/in) 1.3 ITPR 1.3

Quake Sdr (in) 0.1 Quake Toe (in) 0.1 Damping Sdr# 0.3 Damping Toe# 0.5 RULT (Tons) 0.1 Coeff. of Rest. of Soil 0.1 Stroke (ft) 0.1 Hammer Efficiency 0.1 Steam Pressure (psi) 0.1 Reaction wt. (kips) 0.1

*Damping Parameters - sec/ft for Smith; Dimensionless for Case

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 3

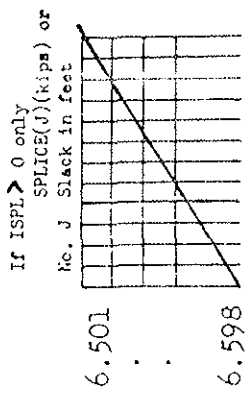


If ITYS = 0 or -1 only

XP1 (ft)	f Soil Res.
0.	0.
10.	0.
10.	0.55
30.	0.67
30.	0.67
40.	0.6
40.	0.6
60.	1.0

6.401
 6.402
 .
 .
 6.420

Note: Splice force = 0.0 kips or less than -0.5 kips.
 Slack between -0.5 and 0.0 feet.
 Both Splice and Slack are to be given as negative numbers.



WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 4

7.000 If ICUT > 9
 Flotter Scale
 MAX For dimension of scale see Input Information

8.000 If ICUT > 9 only
 ISER(I), I = 1, 21, ..., 15 (Printed Variables)

9.000 If RULT < 0 only:
 Ultimate Resistance - tone (at most 10 values)

9.101 If IJJ > 0
 INP(I), I = 1, 2, ..., 13 (Printed Variables)

1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100

W E A P - WAVE EQUATION ANALYSIS FOR PILES

THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.4 HYPOTHETICAL HAMMER INPUT
PILE DESCRIPTION

X BEL. TOP (FT)	.0	59.9	59.9	60.0
A (SQ. IN.)	9.8	9.8	127.7	127.7
E (KSI)	30000.	30000.	30000.	30000.
GAMMA (LB/CU FT)	492.0	492.0	492.0	492.0

HAMMER MODEL HYPOTHET			
ELEMENT NUMBER	WEIGHT (KIPS)	STIFFNESS (K/IN)	COEFF. RESTITUTION
1	.917		
2	.917	116000.0	
3	.917	116000.0	
ANVIL	.810	72700.0	.800
CAP	.950	21000.0	.800
CUSHION		.0	1.000
PILE TOP			.800

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PILE PROPERTIES

PILE LENGTH= 60. FT., AREA(AT TOP)= 9.8 S-IN
 E. MODUL(AT TOP)=3000. KSI., SPEC. WT.(AT TOP)= 492. LBS/CU FT

NO.	WEIGHT (KIPS)	STIFFN. (K/IN)	PDAMP. (KS/FT)	SPLICE (KIPS)	SOIL-S (PCT.)	SOIL-D (KS/FT)	QUAKE (IN.)	L.B.T. (FT.)
1	.155	5319.	.35	0.	.000	.000	.100	4.6
2	.155	5319.	.35	-5000.	.000	.000	.100	9.2
3	.155	5319.	.35	-5000.	.004	.222	.100	13.8
4	.155	5319.	.35	-5000.	.005	.267	.100	18.5
5	.155	5319.	.35	-5000.	.005	.267	.100	23.1
6	.155	5319.	.35	-5000.	.005	.267	.100	27.7
7	.155	5319.	.35	-5000.	.008	.405	.100	32.3
8	.155	5319.	.35	-5000.	.010	.542	.100	36.9
9	.155	5319.	.35	-5000.	.010	.545	.100	41.5
10	.155	5319.	.35	-5000.	.011	.593	.100	46.2
11	.155	5319.	.35	-5000.	.012	.655	.100	50.8
12	.155	5319.	.35	-5000.	.014	.717	.100	55.4
13	.187	5406.	.39	-5000.	.015	.862	.100	60.0
TOE					.900	2.913	.100	

COEFFICIENT OF RESTITUTION OF SOIL 1.000

OPTIONS AND SPECIFICATIONS		RVW (KIPS)	SOIL DIST. NO.
PHI	S-DAMPING	1.40	0
IGUT	P-DAMPING	0	0
IFUEL	J SKIN	1	.0000
IOSTR	J TOE	0	.00
	EFFICIENCY		.950
	TIME INCR. (MS)		.102

RULT= 90.0, AT TOE= 81.0 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	258.3(26)	.0(0)	.0(0)	26.3(26)	13.39(28)	.861(130)
2	258.6(28)	.0(0)	.0(0)	26.3(28)	13.44(30)	.836(130)
3	263.7(32)	.0(0)	.0(0)	26.9(32)	13.18(32)	.810(130)
4	262.5(34)	.0(0)	.0(0)	26.7(34)	13.23(36)	.782(132)
5	261.3(38)	.0(0)	.0(0)	26.6(38)	13.02(38)	.755(136)
6	261.1(40)	.0(0)	.0(0)	26.6(40)	12.84(42)	.730(138)
7	259.3(44)	.0(0)	.0(0)	26.4(44)	12.63(44)	.703(140)
8	259.7(46)	.0(0)	.0(0)	26.4(46)	12.22(48)	.675(142)
9	254.3(48)	.0(0)	.0(0)	25.9(48)	12.03(50)	.645(144)
10	253.4(52)	.0(0)	.0(0)	25.8(52)	11.61(52)	.613(146)
11	250.0(54)	.0(0)	.0(0)	25.5(54)	11.30(56)	.586(132)
12	243.0(58)	.0(0)	.0(0)	24.7(58)	11.18(58)	.557(134)
13	235.1(60)	.0(0)	.0(0)	19.8(60)	11.70(62)	.528(134)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 11.9 K-FT

STROKES ANALYZED AND LAST RETURN (FT) 6.0 6.0

EXAMPLE 7.4 HYPOTHETICAL HAMMER INPUT

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI	BLOWS/ MINUTE
1	90.0	28.	6.00	.00	26.85	48.0

7.5 Pile Segment and Damping Input

7.5.1 Situation

A timber pile, (See also Figure 7 for details) has to be driven through a soil of stratified clay and sand to a dense gravel layer. The timber pile has a length of 36 feet and 2 inch. Its cross sectional area varies from 128.7 at the top to 56.2 (square inches) at the bottom. It has to be driven by a Link Belt 440 hammer.

A soils investigation resulted in the following data: At a depth of 25 feet and 8 inches the pile point will have penetrated into the gravel such that a total ultimate bearing of 75 tons (90% at the toe) is obtained. The Smith damping factors are 0.15 sec/ft in the sand and in the gravel (toe) and 0.20 sec/ft in the clay .

7.5.2 Problem

The hammer should be run at a bounce chamber pressure of 15 psi (gauge) to avoid pile damage. To what blow count must the pile be driven to insure the 75 ton bearing capacity?

7.5.3 Solution

The Complete Input Form must be used since the damping factors are different along the pile skin. For the purpose of demonstration only, the element masses and stiffnesses are also calculated and input in the Complete Form. They can be determined automatically in this case.

Form 10 lists the input parameters. They are:

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 1

0 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49

Title (40 characters)

1.000 EXAMPLE 7.5 PILE SEGMENT + DATA / WAVE

ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS
ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS
ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS	ISPL	NCROSS

2.000

If IPEL = 2
 SU(I), I = 1, 2, ..., N (Pile Segment Stiffnesses - k/in)

6592.	65157.	6176.	5700.	5445.	5100.	4769.	4490.	4120.	3823.
-------	--------	-------	-------	-------	-------	-------	-------	-------	-------

2.101

If IPEL = 2
 P(I), I = 1, 2, ..., N (Pile Segment Weights - kips)

.142	.129	.118	.111	.104	.098	.091	.085	.079	.073
------	------	------	------	------	------	------	------	------	------

2.201

If IPEL > 0
 ALPHA(I), I = 1, 2, ..., N (Relative Lengths of Pile Segments)

3.176	3.0	3.	3.	3.	3.	3.	3.	3.	3.
-------	-----	----	----	----	----	----	----	----	----

2.301

Cap and Cushion

Wt. Cap (kips)	Cap Stiff. (k/in)	Cushion Stiff. (k/in)
3000	3000	3000

3.000

Coefficients of Restitution

Arril (Diesel) Cap (Diesel)	Pile Top	Cushion	TRMAX (mscc)	TRDEL (mscc)
0.5	0.5	0.5	0.5	0.5

4.000

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 2

L pile A pile top E pile top W pile top
 (ft) (ft) (ft) (ft)
 5.000 36.67 128.67 2000. 57.

If NCROSS = 1 and IPFL # 2 only

NP1 (ft)	AP1 (ft)	EP1 (ksf)	WP1 (lbs/ft ³)
5.101	128.67		
5.102	112.65		
.	97.33		
.	83.13		
5.119	78.04		
	58.12		

5.201 Hammer Information (if IHAMR = 0 only)
 STH(1) (kips) STH(2) (k/in) HM(2) (kips) STH(3) (k/in) HM(3) (kips) STH(4) (k/in) HM(4) (kips) STH(5) (kips)

5.202 Hammer Information (if IHAMR = 0 only)
 STH(5) (k/in) HM(5) (kips) STH(6) (k/in) HM(6) (kips) STH(7) (kips) HM(7) (kips) TDEL (sec)

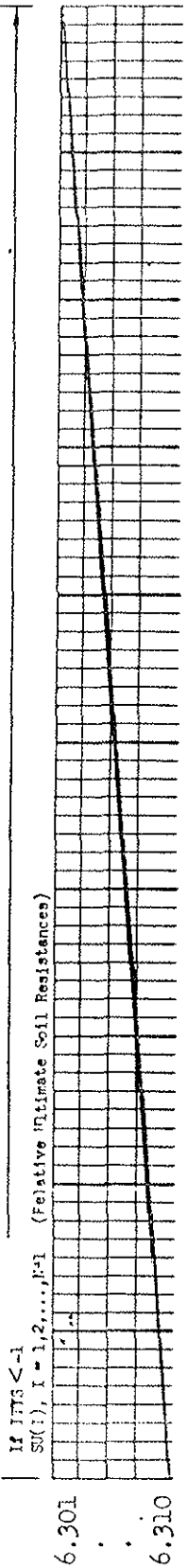
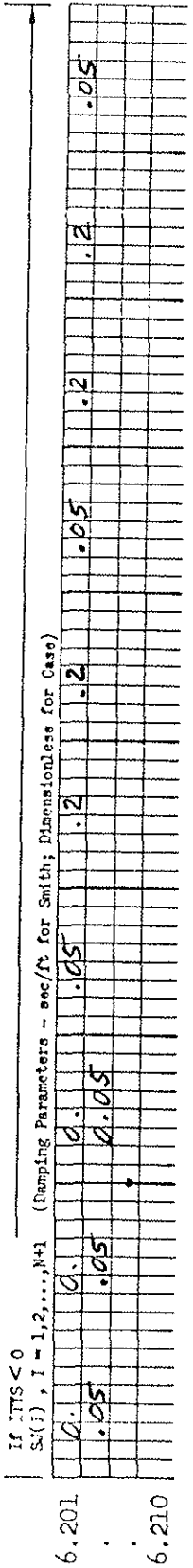
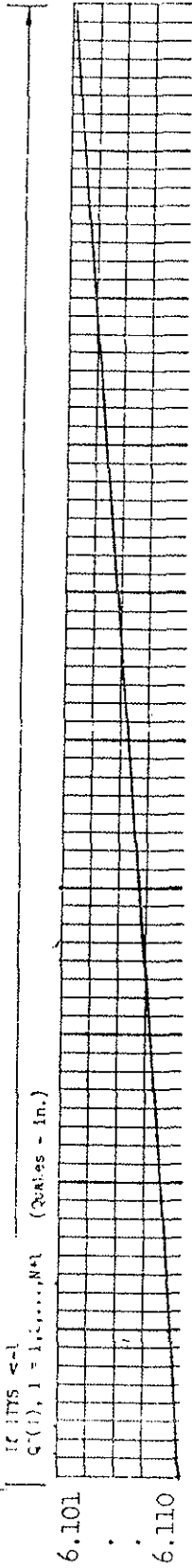
5.203 Hammer Information (if IHAMR = 0 only)
 WAIN (in²) DECEL (in) P1 (psi) P2 (psi) P3 (psi) P4 (psi) P5 (psi) EFFICI

5.204 Hammer Information (if IHAMR = 0 only)
 DEFP (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft) DEFPB (ft)

5.205 Hammer Information (if IHAMR = 0 only)
 AM(1) (kips) STA(1) (k/in) AM(2) (kips) STA(2) (k/in) AM(3) (kips) STA(3) (k/in) MI

6.000 Quake Skin (in) Quake Toe (in) Damping Skin* Damping Toe* RULT (tons) Coeff. of Rest. of Soil (ft) Stroke (ft) Hammer Efficiency Reaction wt. (kips)

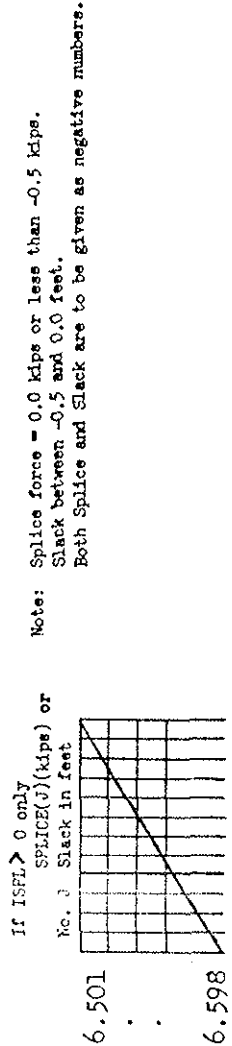
WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 3



If $ITVS = 0$ or -1 only

XPI (ft)	f Soil Pas.
0.	0.
10.5	0.
10.5	1.
36.157	1.

6.401
6.402
.
.
6.420



Note: Splice force = 0.0 kips or less than -0.5 kips.
 Slack between -0.5 and 0.0 feet.
 Both Splice and Slack are to be given as negative numbers.

WAVE EQUATION ANALYSIS for PILES Complete Input Form, page 4

If IOUT > 9
 Plotter Scale

MAX	
-----	--

For dimension of scale see Input Information

7.000

If IOUT > 9 only
 ISE(1), I = 1, 21, ..., 15

(Printed Variables)

8.000

If RUL7 < 0 only:

Ultimate Resistance - tons (at most 10 values)

9.000

If IJJ > 0
 INP(1), I = 1, 2, ..., 13

(Printed Variables)

9.101

1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90	91	92	93	94	95	96	97	98	99	100
---	---	---	---	---	---	---	---	---	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	-----

Card No. 1.000

A Title

Card No. 2.000

IOUT = 0 for minimum output
 IHAMR = 37 according to Table 2
 IOSTR = -1 to effect a constant stroke analysis
 IPEL = 2 to allow the input of element masses and
 stiffnesses
 N = 12 Because of IPEL \neq 0 a choice of the number
 of elements had to be made. N = 12 results
 in segments of approximately 3 feet length
 which is a length much smaller than
 necessary
 NCROSS = 1 The pile is non-uniform
 IBEDAM = 5 Pile Damping for Timber
 IPERCS = 10 According to the soil investigation
 ISMITH = 1 As specified in 7.5.1
 ITYS = -1 To allow the input of damping parameters.
 for individual elements.

Cards No. 2.101, 2.102 Pile Segment stiffnesses, STP(I), for I = 1, ... ,12.
 They are computed using the formula

$$STP(I) = \frac{\overline{AP(I)} \overline{EP(I)}}{ALPH(I) (L)}$$

where $\overline{AP(I)}$ is the average pile cross sectional area,
 $\overline{EP(I)}$ is the average elastic modulus, both averages
 are to be taken over the element length ALPH(I) (L);
 ALPH(I) is the normalized, relative element length
 of the I-th segment and L is the total pile length.
 (See Figure 7 for a sample calculation).

Cards No. 2.201, 2.202 Pile Segment Weights, PM(I), for I = 1, ... ,12.
 They are computed using the formula

$$PM(I) = \overline{AP(I)} \overline{WP(I)} ALPH(I) (L)$$

where $\overline{WP(I)}$ is the average specific weight of the
 pile material over the segment length (See
 Figure 7 for a sample calculation).

Cards No. 2.301, 2.302 Relative Length values of pile segments, ALPH(I)
 for I = 1, ... ,12. It was convenient to use the

PILE DESCRIPTION		STIFFN. kips/in	PILE MODEL			SOIL PROFILE	DAMPING J sec/ft	STATIC % Soil Res.
DEPTH ft	A _p in ²		WEIGHT kips	SEGMT. No.	DEPTH below top feet			
0.167	128.67	6592	.142	1	3.167		0.0	1.0
7.0	112.65	6557	.126	2	6.167		0.0	
		6176	.118	3	9.167	10.5 GRADE	0.0	
		5800	.111	4	12.17	13.0 SAND	0.05	
14.0	97.33	5445	.104	5	15.17		0.20	
15.17	94.92					CLAY		
18.17	88.87	5100	.098	6	18.17	18.2	0.20	
21.0	83.13	4764	.091	7	21.17	21.2 SAND	0.05	
		4440	.085	8	24.17		0.20	
		4128	.079	9	27.17	27.2 CLAY	0.20	
28.0	70.04	3823	.073	10	30.17		0.05	
		3534	.068	11	33.17	SAND	0.05	
		3257	.065	12	36.16	36.2 SAND	0.05	
36.17	56.20						0.05	TOE:

EXAMPLE CALCULATION: $STP(6) = \frac{2000(94.92 + 88.87)}{2(3.0)12} = 5100 \text{ kips/inch}$

$PM(6) = \frac{94.92 + 88.87}{2(144)1000} 51(3.0) = .098 \text{ kips}$

FIGURE 7: DETAILS OF EXAMPLE 7.5

actual element lengths. (Note that the input could have been simplified, since equal and consecutive values need not be repeated).

Card No. 3.000

Weight of Cap = .7 kips (assumed)
Stiffness of Cap = 30000 kips/inch (assumed)
Cushion Stiffness = blank (no cushion)

Card No. 4.000

Coefficients of restitution
Cap = 0.8
Pile Top = 0.5 (timber)
Other C.O.R.'s are left blank for computer determination.

Card No. 5.000

L Pile = 36.167 feet
A Pile Top = 128.67 inch²
E Pile Top = 2000 ksi
W Pile Top = 51 lbs/ft³
Note, these values have to be provided although the element masses and stiffnesses are specified. The program utilizes these values for printing and for stress calculations.

Card No. 5.101,
and 5.102, ...

The pile geometry is input on these cards according to Figure 7. The note of card 5.000 is applicable.

Card No. 6.000

Quakes = 0.1 inch
Skin and Toe damping values are left blank (Damping has to be specified on cards No. 6.201 and 6.202).
RULT = 75 tons
Stroke = 2.63 feet (for 15.0 psi bounce chamber pressure, see Table 1).

Cards No. 6.201,
and 6.202

N+1 = 13 damping parameters are given as shown in figure 7. The 13th value is the toe damping factor.

Card No. 6.401,
and 6.402, ...

A uniform skin friction distribution is specified as shown in Figure 7.

The output is reproduced in Form 11.

W E A P - WAVE EQUATION ANALYSIS FOR PILES

THIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTRATION
BY GOBLE & ASSOCIATES, CLEVELAND, OHIO

EXAMPLE 7.5 PILE SEGMENT + DAMPING INPUT
PILE DESCRIPTION

X BEL. TOP (FT)	.0	.2	7.0	14.0	21.0	28.0	36.2
A (SQ. IN.)	128.7	128.7	112.6	97.3	83.1	70.0	56.2
E (KSI)	2000.	2000.	2000.	2000.	2000.	2000.	2000.
GAMMA (LB/CU FT)	51.0	51.0	51.0	51.0	51.0	51.0	51.0

HAMMER MODEL LB 440			
ELEMENT NUMBER	WEIGHT (KIPS)	STIFFNESS (K/IN)	COEFF. RESTITUTION
1	2.106		
2	.987	235000.0	
3	.943	111000.0	
ANVIL	1.160	70000.0	.850
CAP	.700	30000.0	.800
CUSHION		.0	1.000
PILE TOP			.500

III

PILE PROPERTIES

PILE LENGTH= 36. FT., AREA(AT TOP)= 128.7 S-IN
 E. MODUL(AT TOP)= 2000. KSI., SPEC. WT.(AT TOP)= 51. LBS/CU FT

NO.	WEIGHT (KIPS)	STIFFN. (K/IN)	PDAMP. (KS/FT)	SPLICE (KIPS)	SOIL-S (PCT.)	SOIL-D (S/FT)	QUAKE (IN.)	L.B.T. (FT.)
1	.142	6592.	1.87	0.	.000	.000	.100	3.2
2	.126	6557.	1.76	-5000.	.000	.000	.100	6.2
3	.118	6176.	1.65	-5000.	.000	.000	.100	9.2
4	.111	5800.	1.55	-5000.	.006	.050	.100	12.2
5	.104	5445.	1.45	-5000.	.012	.200	.100	15.2
6	.098	5100.	1.37	-5000.	.012	.200	.100	18.2
7	.091	4764.	1.27	-5000.	.012	.050	.100	21.2
8	.085	4440.	1.19	-5000.	.012	.200	.100	24.2
9	.079	4128.	1.10	-5000.	.012	.200	.100	27.2
10	.073	3823.	1.02	-5000.	.012	.050	.100	30.2
11	.068	3534.	.95	-5000.	.012	.050	.100	33.2
12	.065	3257.	.89	-5000.	.012	.050	.100	36.2
TOE					.900	.050	.100	

COEFFICIENT OF RESTITUTION OF SOIL 1.000

OPTIONS AND SPECIFICATIONS			
PHI 1.40	S-DAMPING	SMITH	RWT (KIPS) 5.21
IOUT 0	P-DAMPING	5	SOIL DIST. NO. 0
IFUEL 1	J SKIN	.00	TDEL (SEC.) .0000
IOSTR -1	J TOE	.15	TEMAX (MS) .00
	EFFICIENCY	.950	
	TIME INCR. (MS)	.075	

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RULT= 75.0; AT TOE= 67.5 TONS

TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURRENCE

ELEM. NO.	FMAX KIPS	FMIN KIPS	MINSTR KSI	MAXSTR KSI	VELMX FT/S	DISMX INCH
1	153.6(242)	.0(0)	.0(0)	1.2(242)	6.26(164)	.581(268)
2	154.8(246)	.0(0)	.0(0)	1.2(246)	6.26(168)	.560(270)
3	155.7(248)	.0(0)	.0(0)	1.2(248)	6.26(170)	.538(272)
4	155.9(250)	.0(0)	.0(0)	1.4(250)	6.21(172)	.513(274)
5	157.9(220)	.0(0)	.0(0)	1.4(220)	6.17(176)	.487(276)
6	162.4(216)	.0(0)	.0(0)	1.7(216)	6.09(178)	.459(278)
7	165.3(214)	.0(0)	.0(0)	1.7(214)	6.01(182)	.429(282)
8	164.9(212)	.0(0)	.0(0)	2.0(212)	5.86(184)	.398(284)
9	160.9(212)	.0(0)	.0(0)	1.9(212)	5.58(186)	.364(288)
10	158.9(220)	.0(0)	.0(0)	1.9(220)	5.08(188)	.328(290)
11	157.7(200)	.0(0)	.0(0)	2.3(200)	4.20(190)	.289(292)
12	159.4(198)	.0(0)	.0(0)	2.3(198)	3.12(200)	.248(296)

THE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS 4.8 K-FT

RETURNED STROKES AND STROKE ANALYZED, (FT) 3.1 2.7 2.6 2.6
 TOTAL PRESSURE CHANGE -22.6 %

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EXAMPLE 7.5 PILE SEGMENT + DAMPING INPUT

SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI	BLOWS/ MINUTE	B.C. PR. PSI
1	75.0	81.	2.63(2)	.00	2.28	82.6	14.7

7.5.4 Discussion of Results

The relatively complicated input should be checked by comparing Figure 7 with the pile model table on the second page of the output. The skin and toe damping parameters in the "Options and Specifications" output are zero since they were not specified on Card No. 6.000.

The summary output shows that a blow count of 83 blows per foot will drive the pile to 75 tons capacity if the hammer runs at 15 psi bounce chamber pressure. The summary lists B.C.P. as 14.2 psi which was the bounce chamber pressure on the last return stroke. The corresponding stroke was considered sufficiently accurate and no additional analysis was performed to obtain better agreement.

The concentrated toe resistance resulted in a relatively high stress of 2.3 ksi. Actually this stress is higher yet however, the area of the bottom element was only roughly approximated by the area specification just above the bottom (70.04 inch^2). A better job of stress computation is done by the computer if element masses and stiffnesses are automatically determined. The reader is encouraged to rerun this problem with IPEL = 0 and correspondingly no input on Cards No. 2.101 to 2.302.

METRIC CONVERSION FACTORS

<u>To Convert</u>	<u>To</u>	<u>Multiply By</u>
pounds (lbm)	kilograms (kg)	0.4535924
pounds force (lb)	newtons (N)	4.4482219
kips (1000lb)	kilonewtons (kN)	4.4482219
inches (in.)	centimeters (cm)	2.54
feet (ft)	meters (m)	0.3048
feet per second (fps)	meters per second (m/sec)	0.3048
foot-pounds (ft-lb)	newton-meters (N·m)	0.355818
foot-pounds (ft-lb)	joules (J)	1.355818
Pounds per foot (lb/ft)	kilograms per meter (kg/m)	1.488164
Acceleration of gravity (g)	meters per second squared (m/s ²)	9.806650
Acceleration of gravity (g)	feet per second squared (ft/sec ²)	32.17405
kip-inches (kip-in.)	newton-meters (N·m)	112.9848
degrees Centigrade	degrees Fahrenheit	1.8 and add 32

76-14.3

76-14.3

Implementation Package

WAVE EQUATION ANALYSIS OF PILE DRIVING

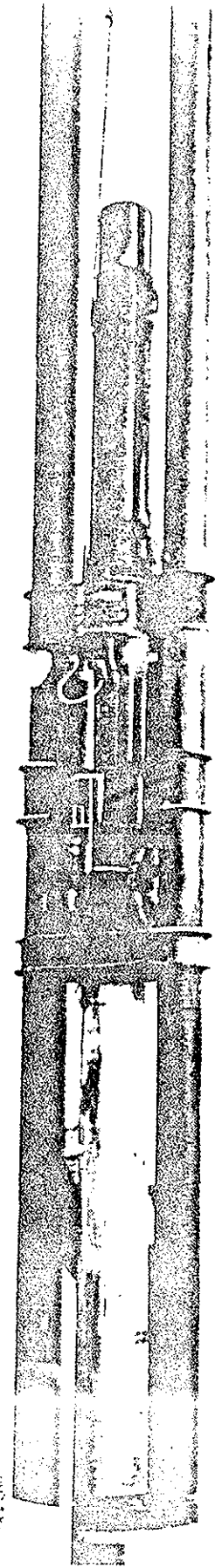
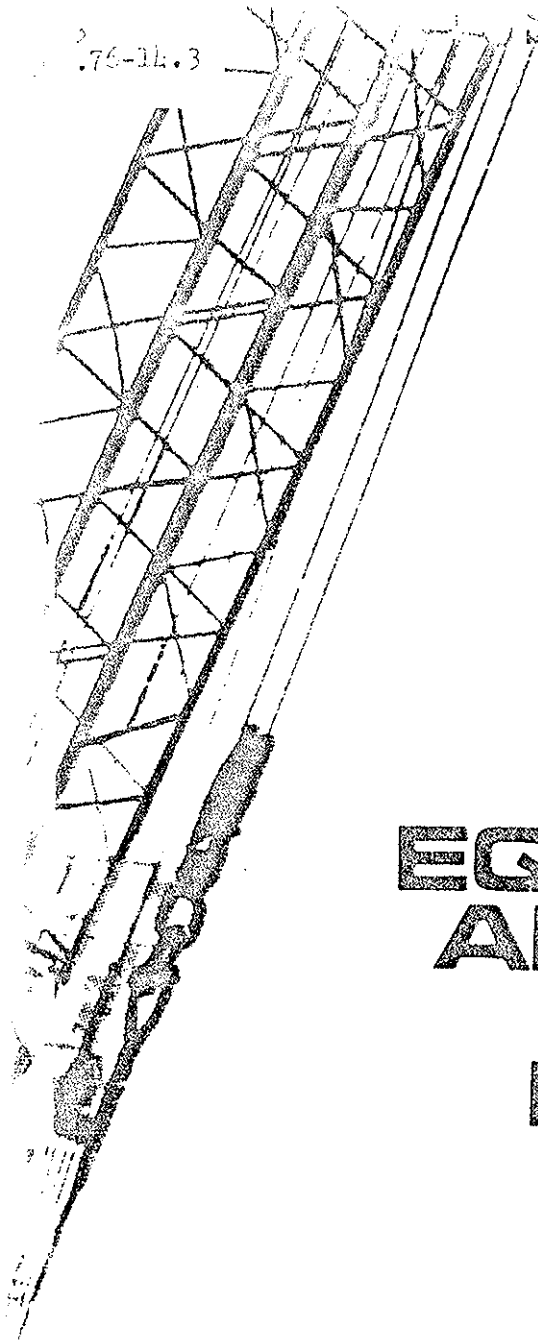
WEAP PROGRAM

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				11. Contract or Grant No. DOT-FH-11-8830																	
12. Sponsoring Agency Name and Address U.S. Department of Transportation Federal Highway Administration Offices of Research and Development Washington, D. C. 20590				13. Type of Report and Period Covered Final Report																	
				14. Sponsoring Agency Code																	
15. Supplementary Notes FHWA Contract Manager: Chien-Tan Chang (HDV-22)																					
16. Abstract A computer program was written and tested that performs a realistic Wave Equation Analysis of Piles driven by any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesels and refined mechanical hammer models. The program development was aimed at providing a simple input and both a flexible and extensive output that includes automatic plotting capabilities. Pile Driving Hammer data were prepared and stored in a file for most of the commonly encountered models. The computer language is FORTRAN IV. The program was extensively tested against measured pile top force and velocity data and against measured diesel combustion pressure and stroke. This volume is the third in a series. The others in the series are: <table border="1"> <thead> <tr> <th><u>Vol. No.</u></th> <th><u>FHWA No.</u></th> <th><u>Short Title</u></th> <th><u>NTIS (PB) No.</u></th> </tr> </thead> <tbody> <tr> <td>1</td> <td>IP-76-14</td> <td>Background</td> <td></td> </tr> <tr> <td>2</td> <td>IP-76-14</td> <td>User's Manual</td> <td></td> </tr> <tr> <td>4</td> <td>IP-76-14</td> <td>Narrative Presentation</td> <td></td> </tr> </tbody> </table>						<u>Vol. No.</u>	<u>FHWA No.</u>	<u>Short Title</u>	<u>NTIS (PB) No.</u>	1	IP-76-14	Background		2	IP-76-14	User's Manual		4	IP-76-14	Narrative Presentation	
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17. Key Words COMBUSTION, COMPUTERS, DESIGN, DIESEL, DYNAMICS, FOUNDATIONS, IMPACT, PILE DRIVING, SOIL MECHANICS, WAVE EQUATION			18. Distribution Statement No restrictions. Copies of this volume are available from: National Technical Information Service Springfield, Virginia 22161																		
19. Security Classif. (of this report) Unclassified		20. Security Classif. (of this page) Unclassified		21. No. of Pages 127	22. Price																

FOREWARD

This program documentation is one of four volumes of the description of WEAP, Wave Equation Analysis of Piles. The other three volumes contain:

1. Background
2. User's Manual
3. A Narrative Report (tape/slide show).

The WEAP program was written under contract with the Federal Highway Administration. It performs a dynamic analysis of impact driven piles. The results of the analysis consist, among others, of pile stresses, velocities and penetrations and are provided by the program in printed and plotted form. The program is used in foundation design, construction control and pile driving preparation. Potential users include soil and foundation engineers, construction supervisors and pile driving contractors.

WEAP
WAVE EQUATION ANALYSIS FOR PILES
PROGRAM DOCUMENTATION

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1. GENERAL INSTRUCTIONS

1.1 Program

There are a few changes that usually have to be made in the program before it can be compiled and loaded into a machine. The changes pertain to the I/O device numbers that are referenced in the program.

The following I/O devices are used:

- a) Card Reader
- b) Line Printer
- c) Sequentially Reading Tape or Disc File

It is easily possible to convert the program for use with a terminal type I/O device having only 80 print columns.

The following program changes have to be made in any case:

- a) In SUBROUTINE IPTN set IW to the card reader's or terminal's device number.
- b) In SUBROUTINE IPTN set IR to the line printer's or terminal's device number.
- c) For terminal use set ICOL = 0 in IPTN (ICOL = 132 otherwise).
- d) In SUBROUTINE HAEL set IF to the tape or disc file's device number.

If a plotter is to be used, additional work will have to be done to adjust the SUBROUTINE OUTPUT to the particular software of

the machine to be used. Since plotter routines are not standardized advice cannot be given here. For no plotter use the calls to OUTPUT should be removed from the main program. Conversion routines are available for standard CALCOMP software.

1.2 File

Two steps are involved when loading the hammer data file.

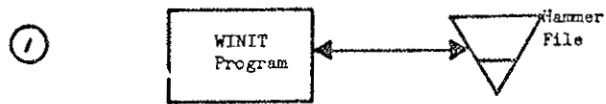
(i) Run the initialization program provided. It initializes the file in the proper format. Note that IF has to be set in this program to the proper file device number before compilation.

(ii) Run the file update program with the data provided. Before compiling it IW, IR and IF have to be set (as in 1.1). Any change of data can be accomplished by running this program with the new hammer data sets. The new hammer data sets have to be supplied with their respective identifier number and in ascending order of identifier number. Also the total number of hammers to be loaded has to be given as a first input (a print option can be set on this card too). This input data is also explained on the file loading form.

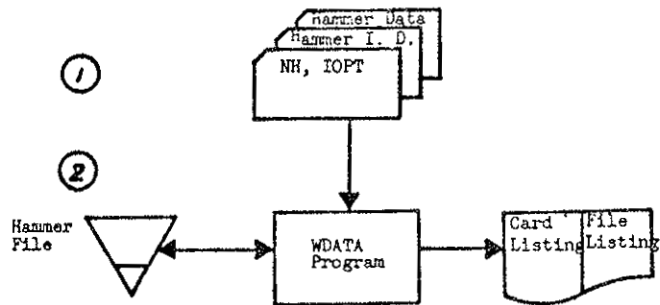
2. OPERATOR'S FLOW CHART

2.1 Graphic

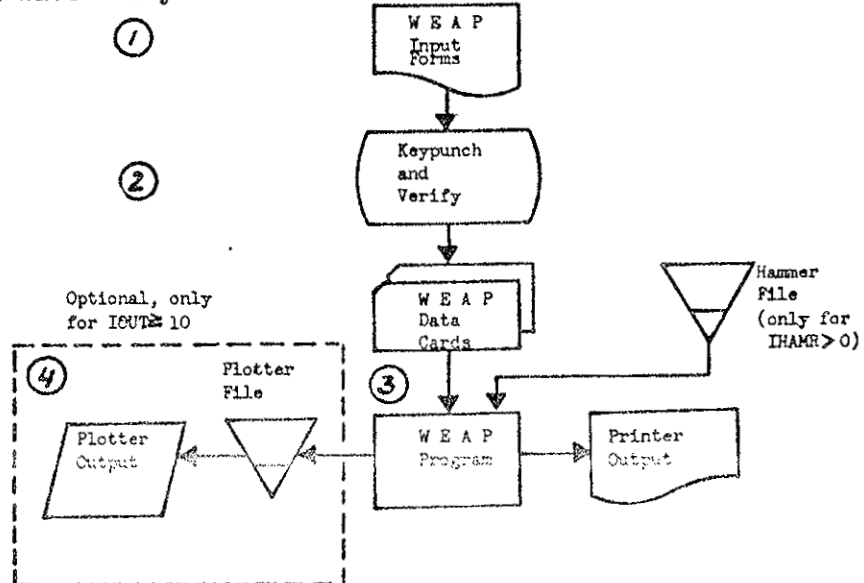
2.1.1 Initialize Hammer Data File



2.1.2 Load Hammer Data File



2.1.3 Perform Wave Analysis



2.2 Narrative

2.2.1 INITIALIZE HAMMER FILE

1. Initialize file (either tape, disc or drum) using program WINIT. This program loads zeroes and blanks in the proper format in the file.

2.2.2 LOAD HAMMER DATA

1. Obtain hammer data cards from engineering department consisting of:
First card: NH, IOPT ... number of hammers to be loaded,
print option (2I4).
and NH times
1 card: IHAMR ... hammer number (I4)
5 data cards: ... as per input forms. (Total number of cards $6(NH) + 1$).
2. Process using WDATA program which replaces data on file with corresponding card data. Input cards from step 1 and hammer file. Output to hammer file and printer. Printed output is the card data and, if IOPT was greater than 0, the total file contents (non-zero data).

2.2.3 WAVE ANALYSIS

1. Obtain input forms from engineering department.

2. Punch and verify input cards with format as indicated in forms.

3. Process WEAP program.

Input: data cards from step 2; hammer file - only
if IHAMR > 0 (card No. 2.000, cc 9-12).

Output: Printed Results

Plotter Data if IOUT > 9 (card No. 2.000
cc 1-4).

4. Plot Plotter Data from Step 3.

3. DESCRIPTION OF SUBROUTINES

Subroutine Name	Subroutine primary function and calls
IPTN	Read input from cards. Assemble and initialize data; calls HAEL, PIEL, SOILN, OUTIN, JJNP, VACHAM.
HAEL	Supply hammer information and determine hammer critical time increment.
PIEL	Determine pile segment properties.
SOILN	Determine distribution of static soil resistance; reads from cards.
OUTIN	Determine input information.
JJNP	Determine output segment numbers.
DOWN	Determine maximum stroke and velocity at exhaust ports for closed end diesel hammers; calls FTR.
FTR	Determine force on ram top.
STARTC	Using stroke determine initial values; calls FTR, VACHAM.
VACHAM	Determine velocity of ram at ports or vacuum force if vacuum chamber hammer is analyzed.
DIESEL	Carry out the analysis of a diesel hammer; calls PILEAN, FILL, FILL2, INTEGR, STIFF.
PILEAN	Analyze the pile for a certain pile increment; calls INTEGR, SRESN.
INTEGR	Integrate assuming linear acceleration to obtain displacement.
STIFF	Find stiffness of a spring with quadratic force-deflection relation.
SRESN	Find static soil resistance.

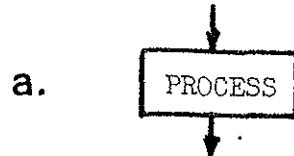
UP	Find rebound stroke from pressure and ram velocity; calls FTR, VACHAM.
OUT2	Print segment variables as function of time and extreme values for each pile element.
OUTPUT	Plot segment variables vs. time, vs. time and pile length and plot summary. Calls FPLOT, FMOVE, FCHAR, FSCHAR, FSCALE; writes to plotter file.
FILL	Fills up to output arrays depending on output option.
FILL2	Fills up output arrays for options 6, 16, 26.
FPLOT*	Plots to a point.
FMOVE*	Moves to a point
FCHAR*	Plots a character or string of characters.
FSCHAR*	Plots a special character.
FSCALE*	Sets up origin and scale.

* Only contained in certain program versions; may be computer library routines. The F ... routines call standard CALCOMP routines like NEWPEN, NUMBER, PLOT, PLOTS, QPLOT, SYMBOL.

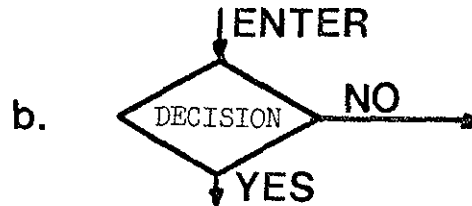
4. FLOW CHARTS

NOTES ON SUBROUTINE GRAPHS:

- (i) The programs and subroutines are listed in order of occurrence in the program flow.
- (ii) The flow charts contain three different symbols.



A process may involve arithmetic, input/output or any other data manipulation.



Decisions lead the program in either of two branches. The rule used in the following branches was always

YES = downwards

NO = sidewise

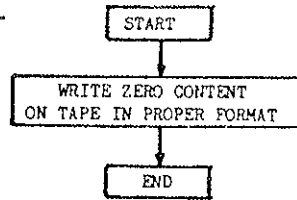
The text in the decision diamond may or may not be identical with the corresponding IF statement in the program code. In general, it was attempted to provide a text that explains the action taken rather than a copy of the actual code.



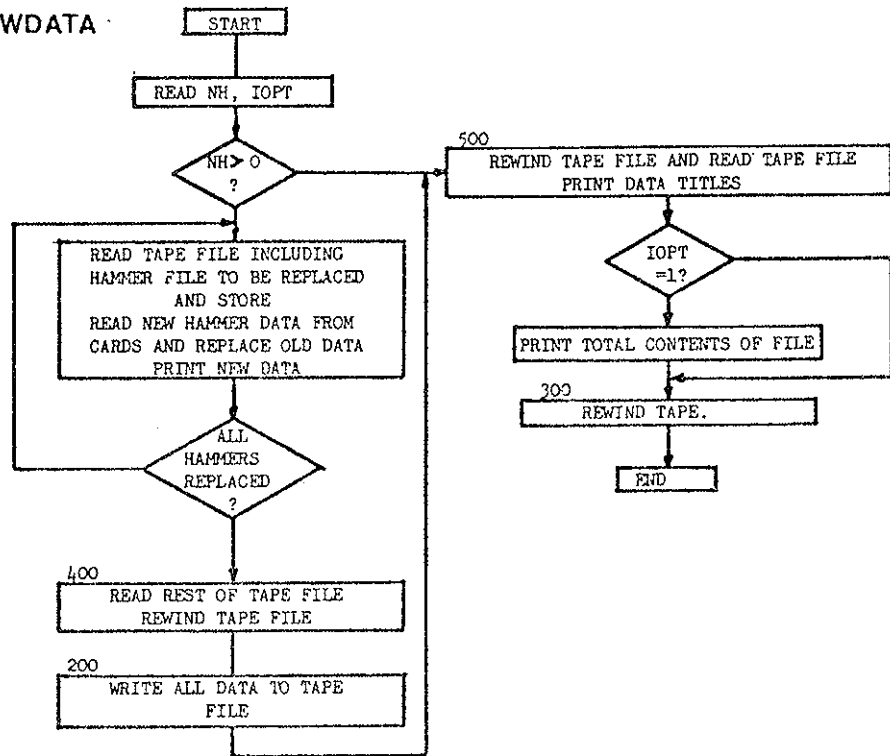
If the program flow chart is interrupted by an end of page then flow lines are terminated and restarted by the off-page connector using corresponding numbers.

- (iii) Statement numbers were added to the flow charts wherever possible.
- (iv) The reference to tape file includes tape simulated discs or tape simulated drums.

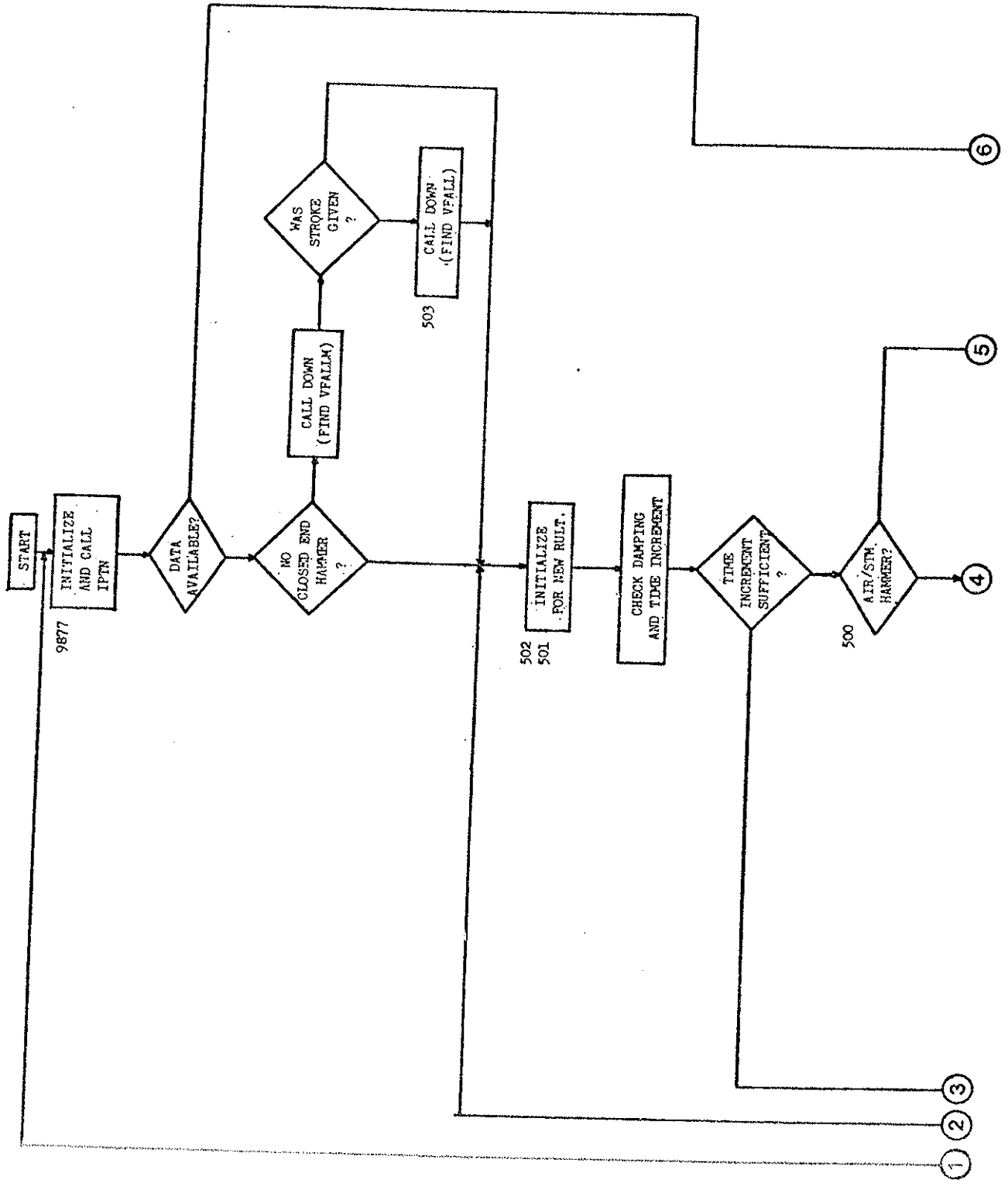
PROGRAM WINIT

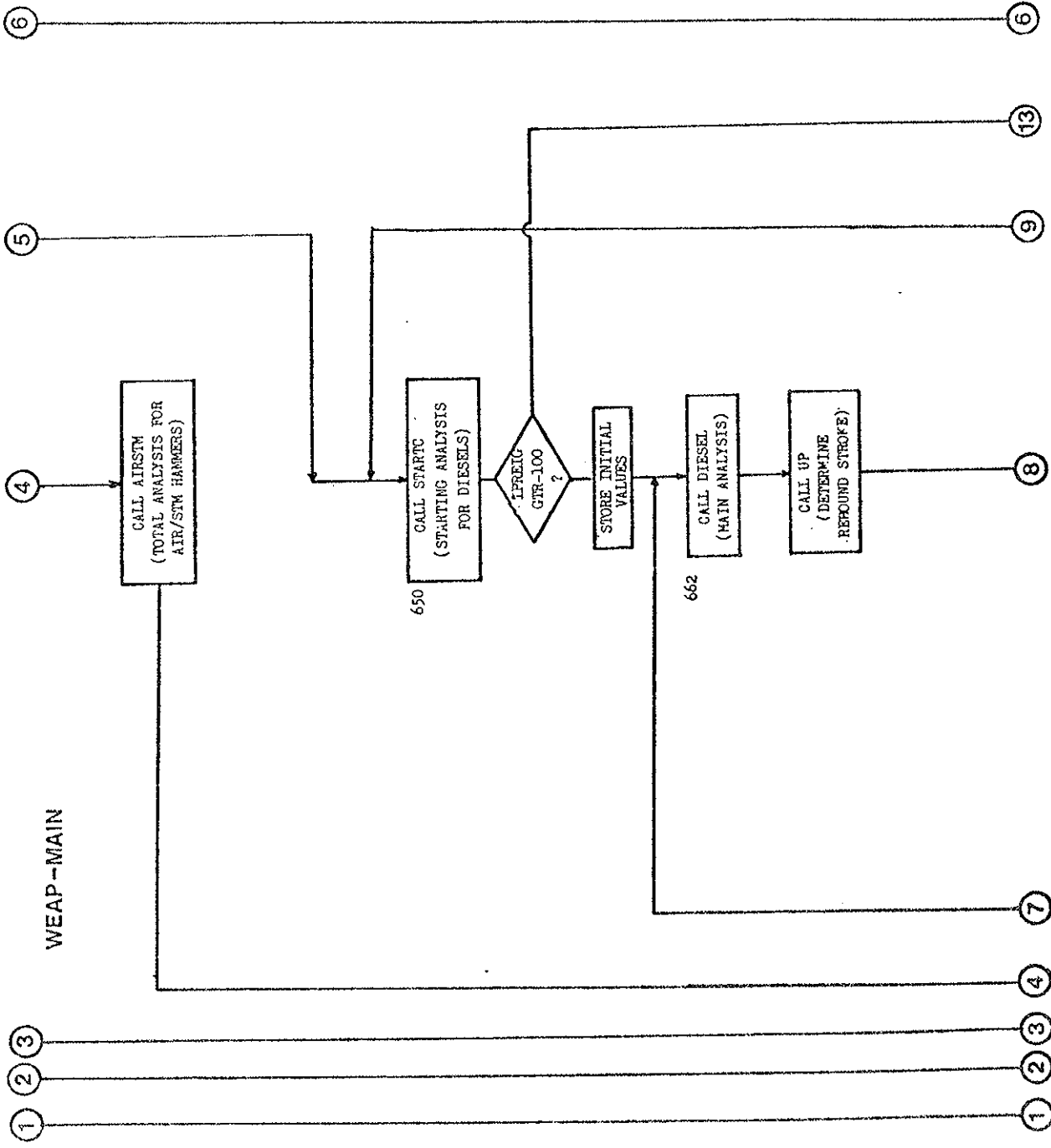


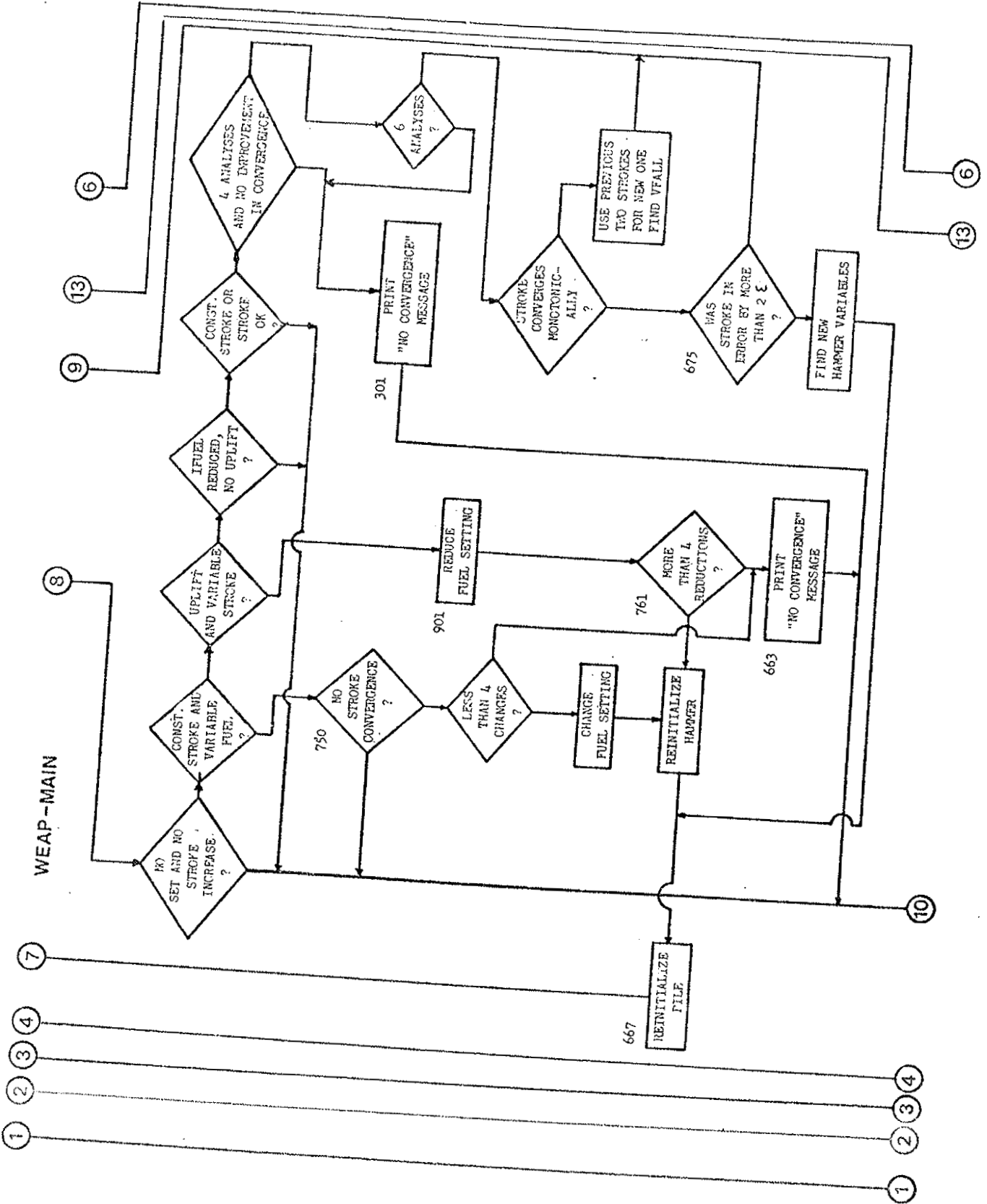
PROGRAM WDATA

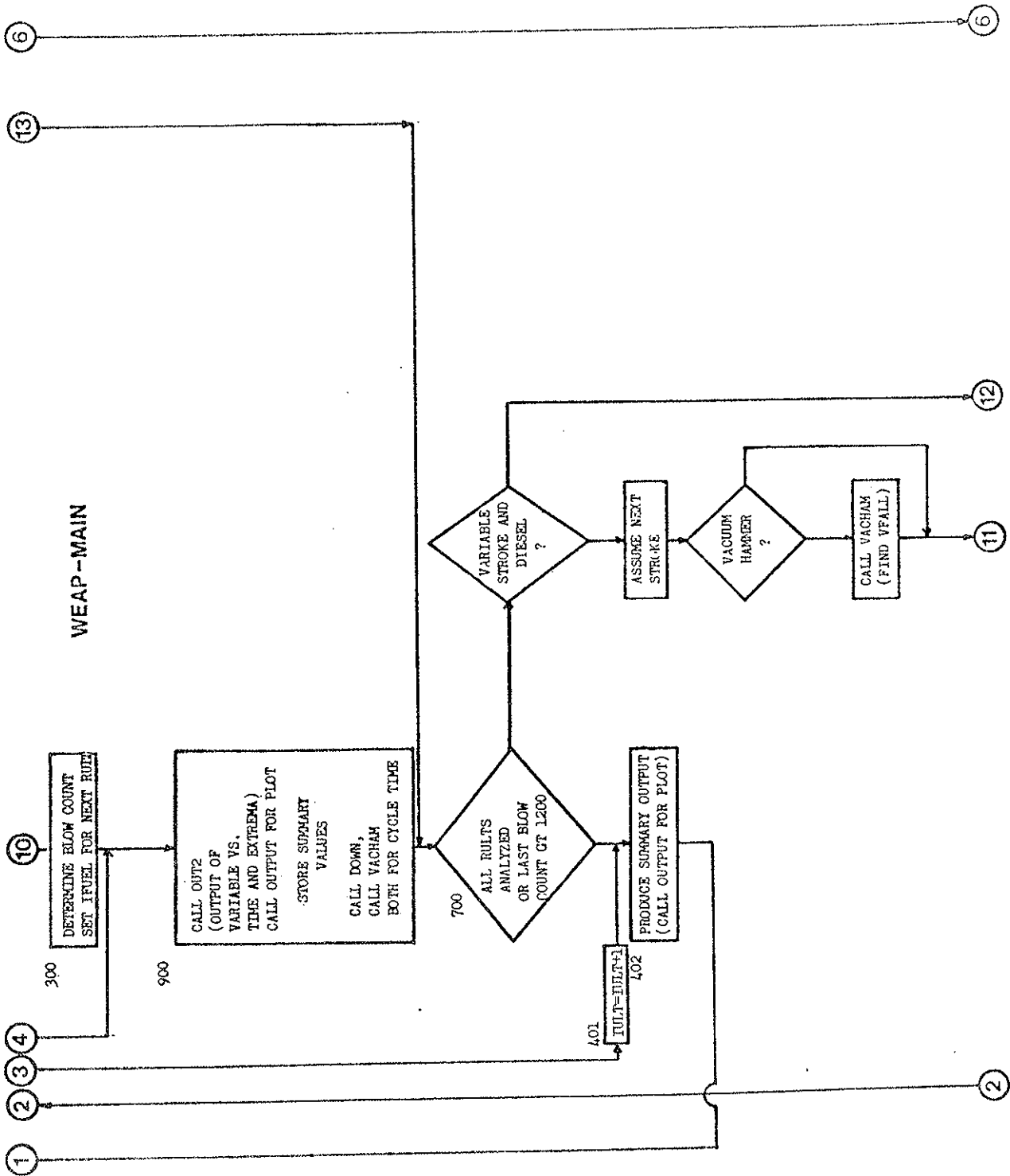


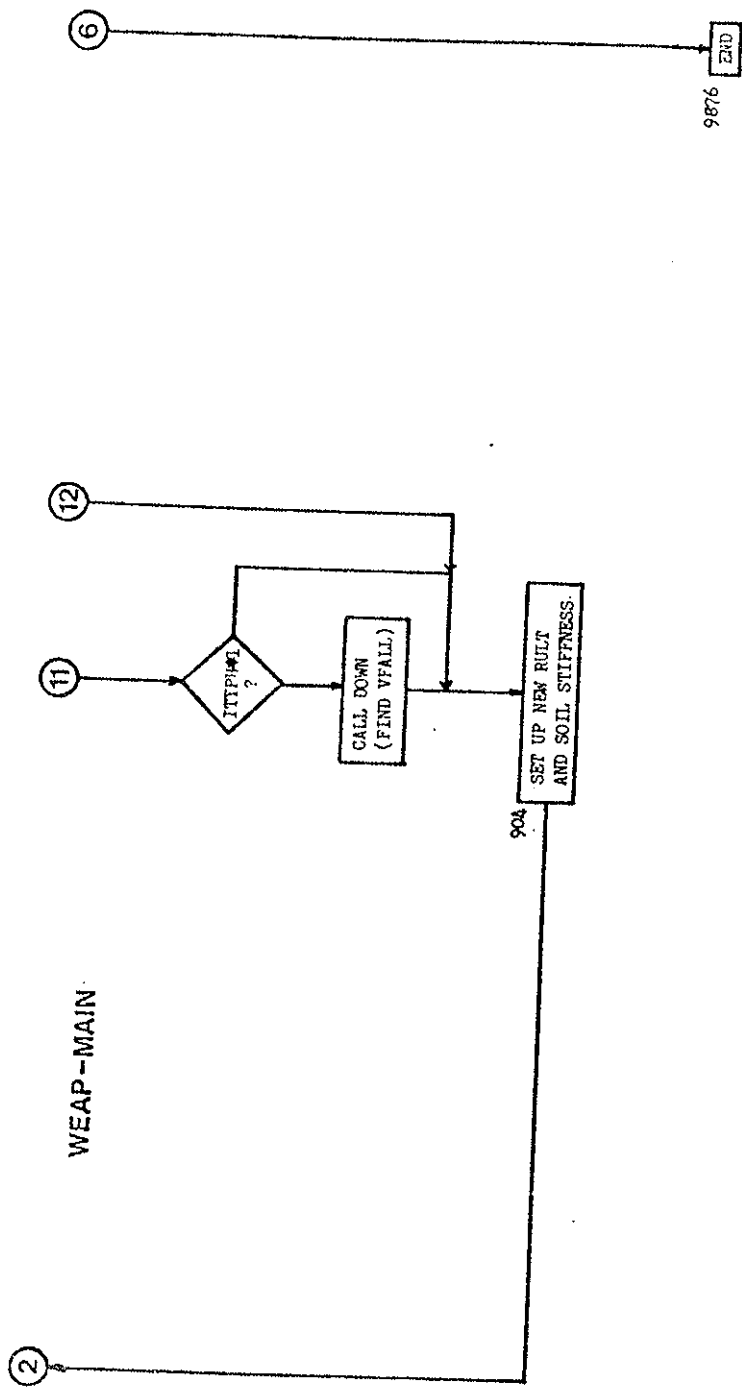
WEAP--MAIN



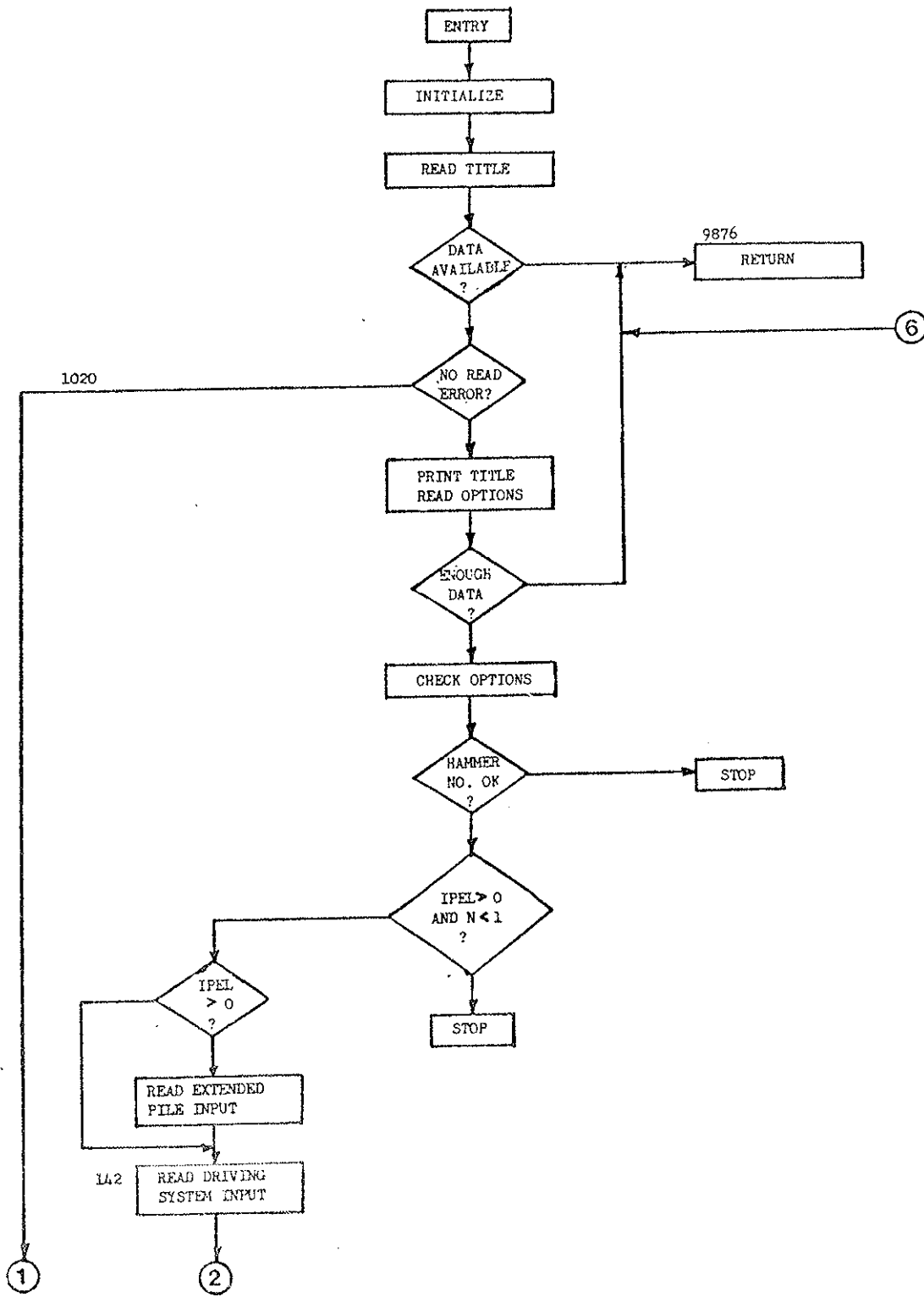




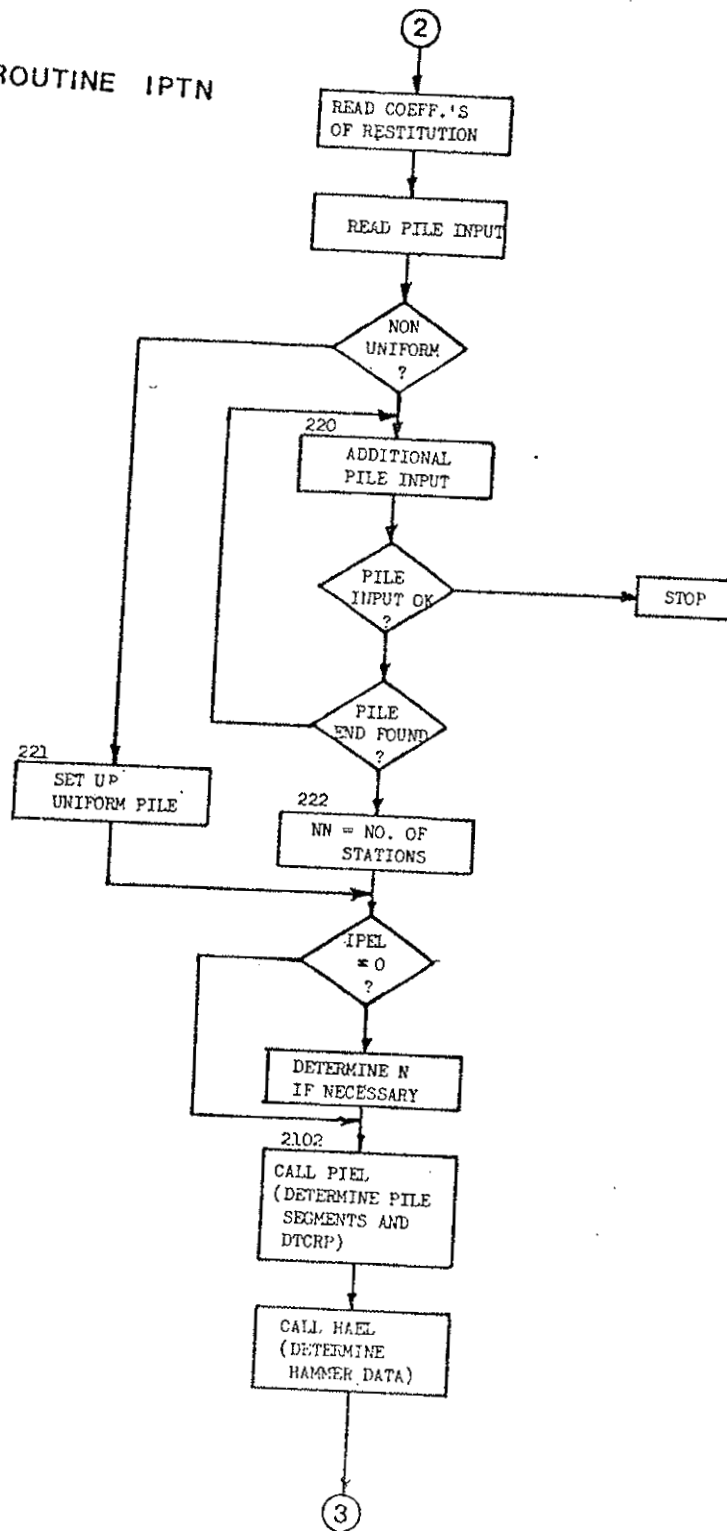




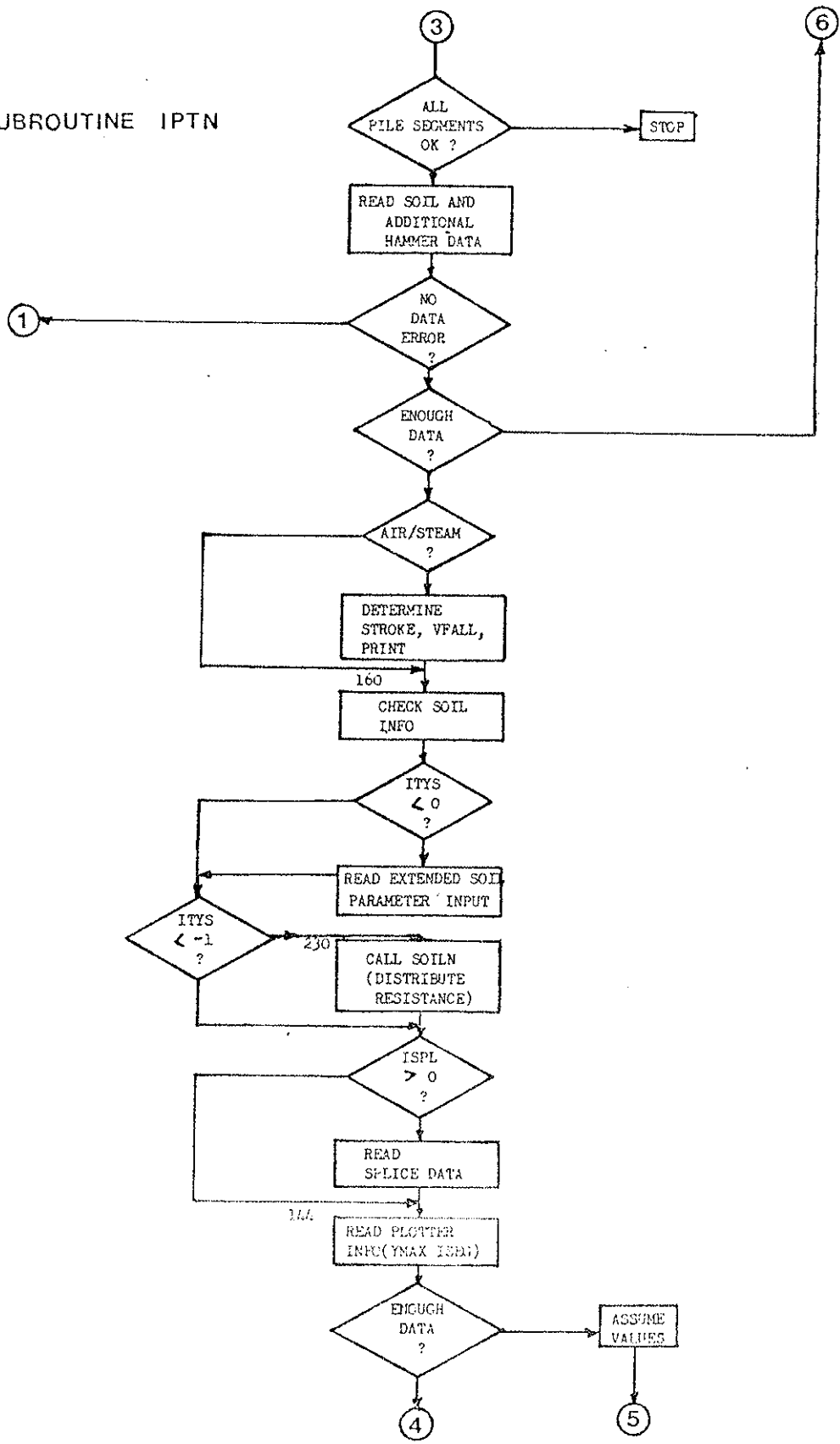
SUBROUTINE IPTN



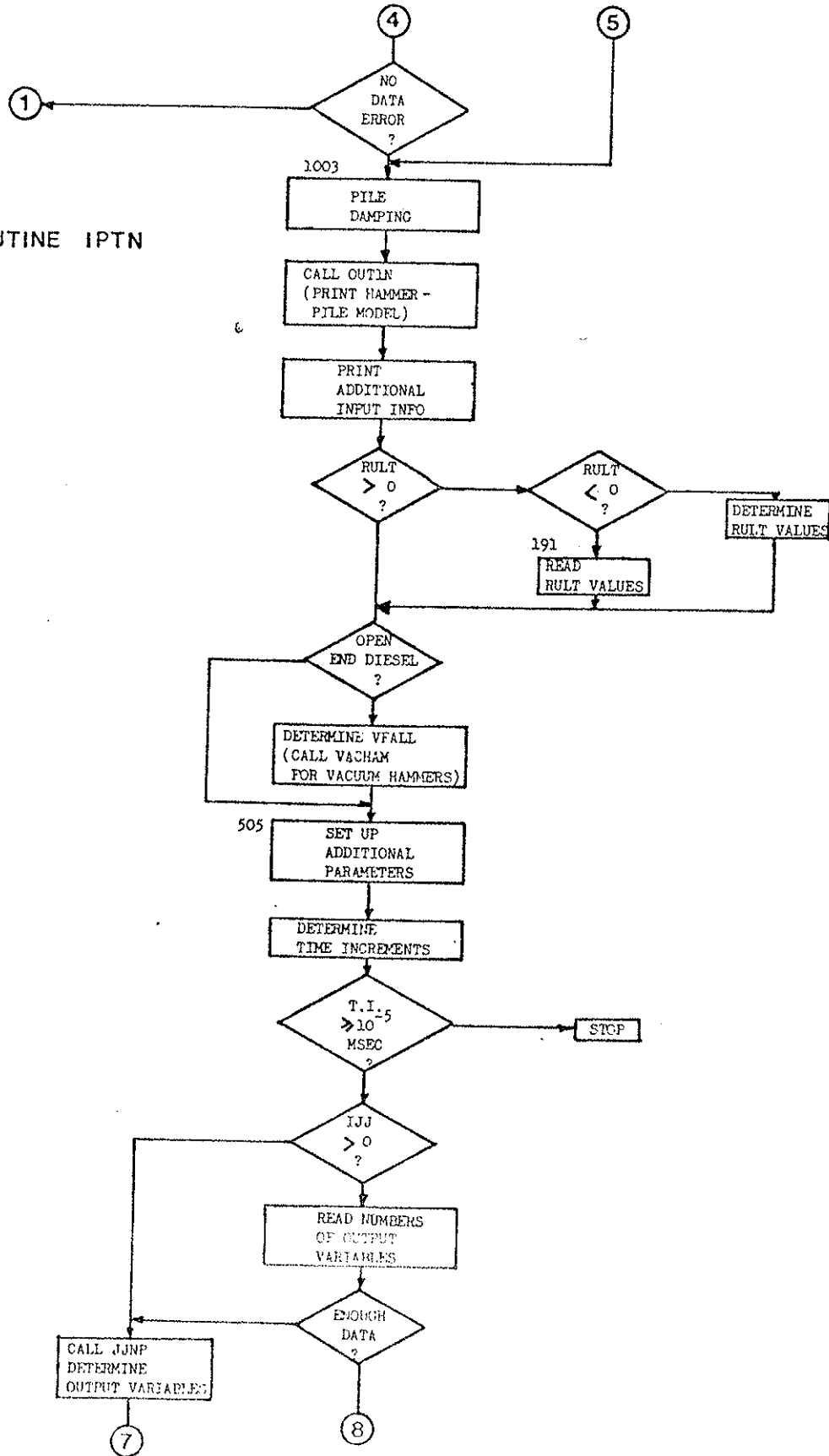
SUBROUTINE IPTN



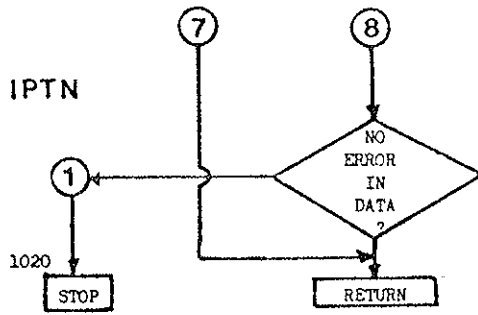
SUBROUTINE IPTN



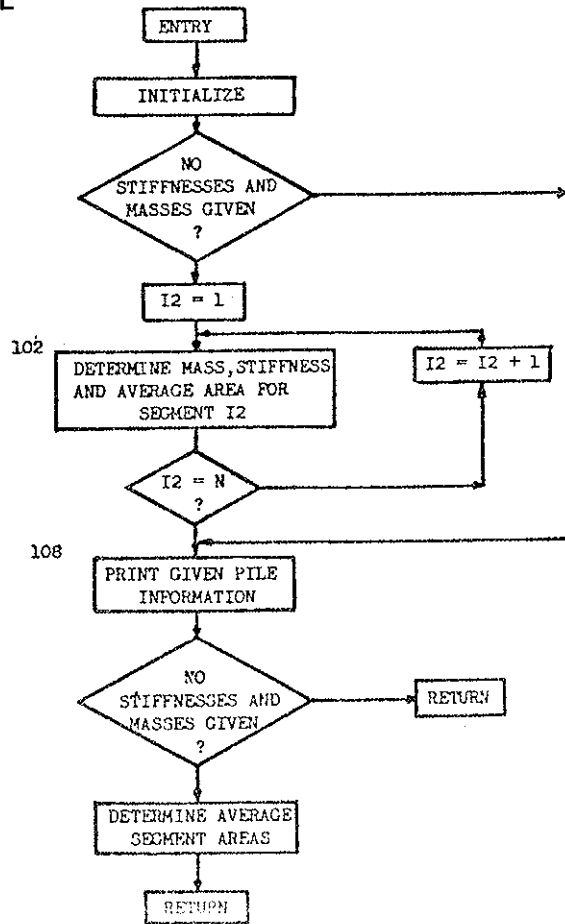
SUBROUTINE IPTN



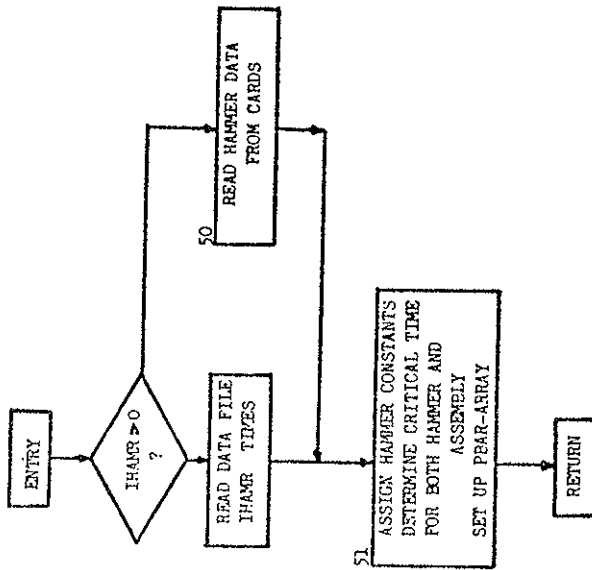
SUBROUTINE IPTN



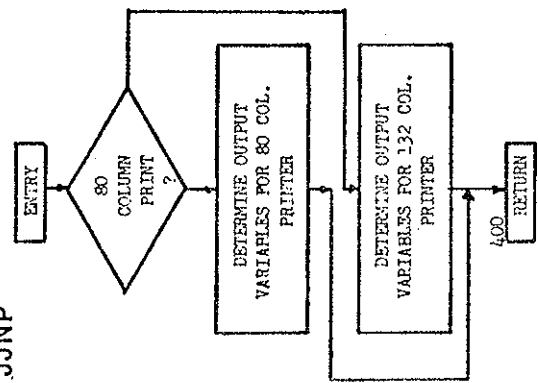
SUBROUTINE PIEL



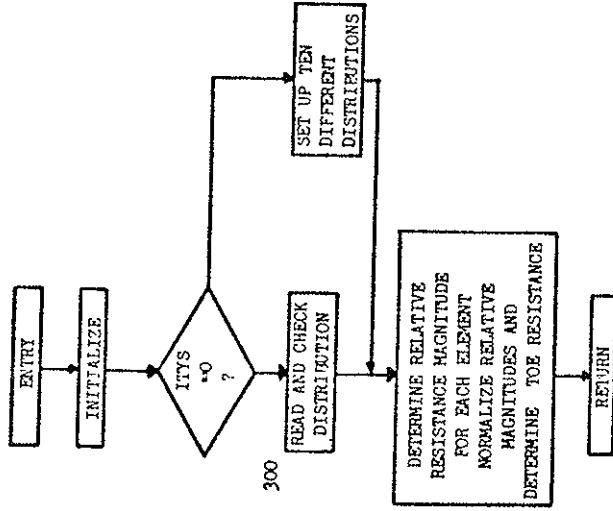
HAEL



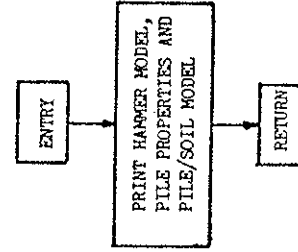
JUNP

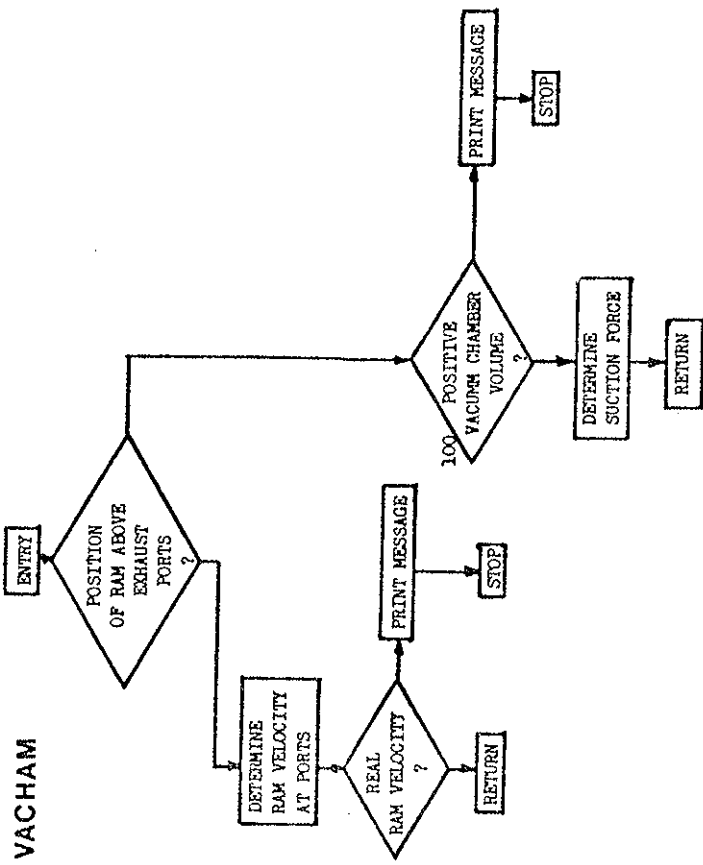


SOILN

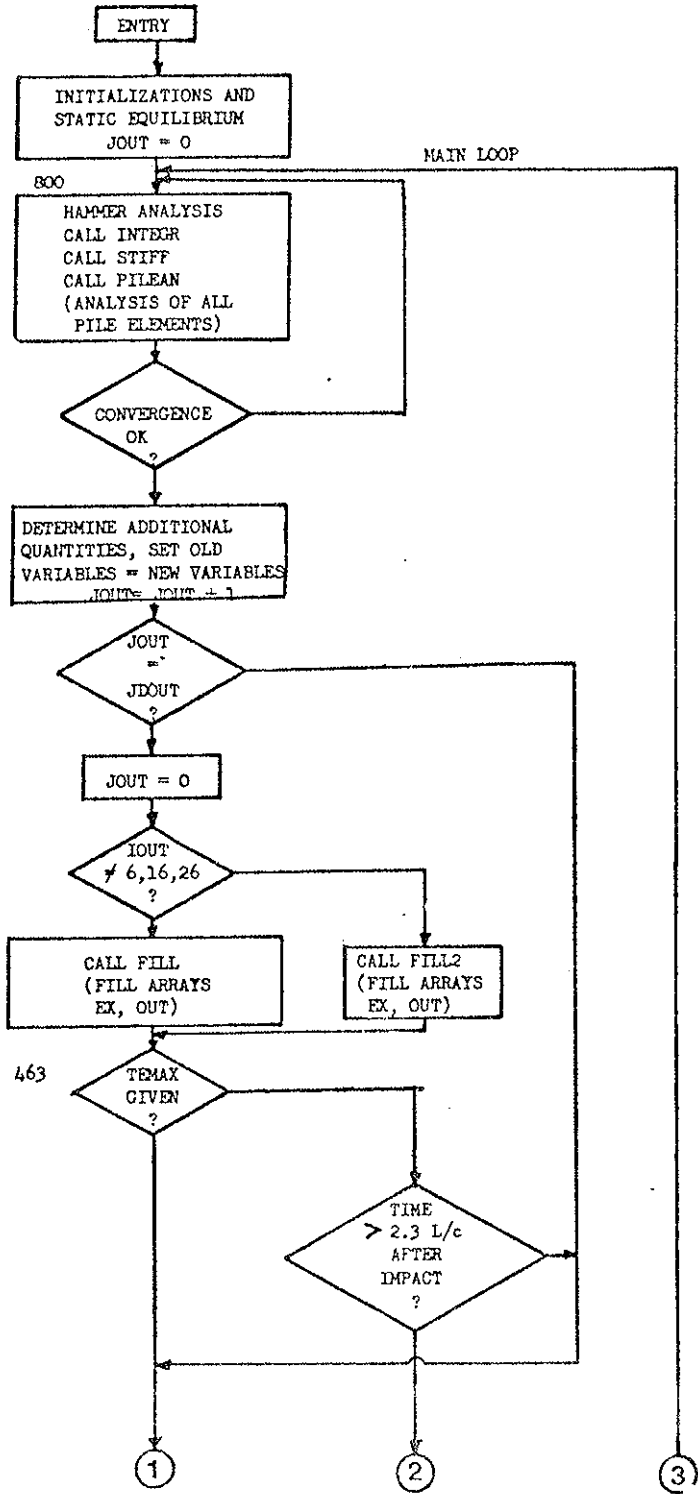


OUT1N

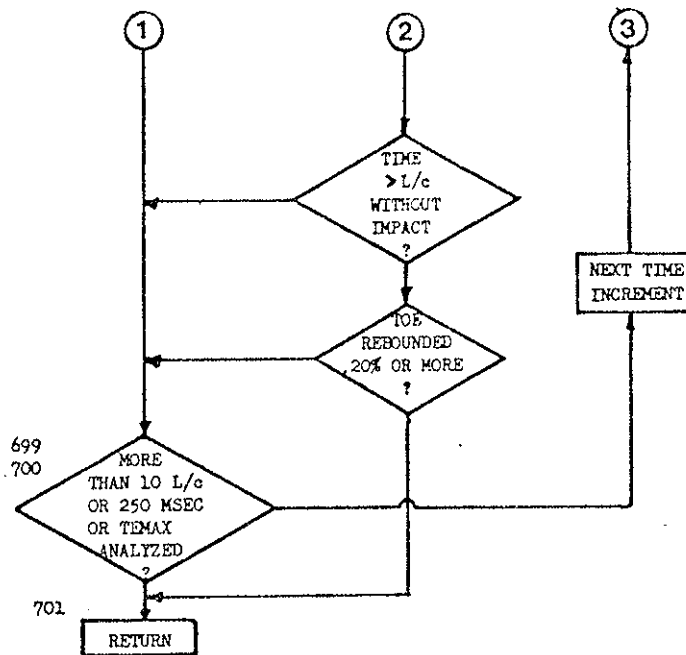




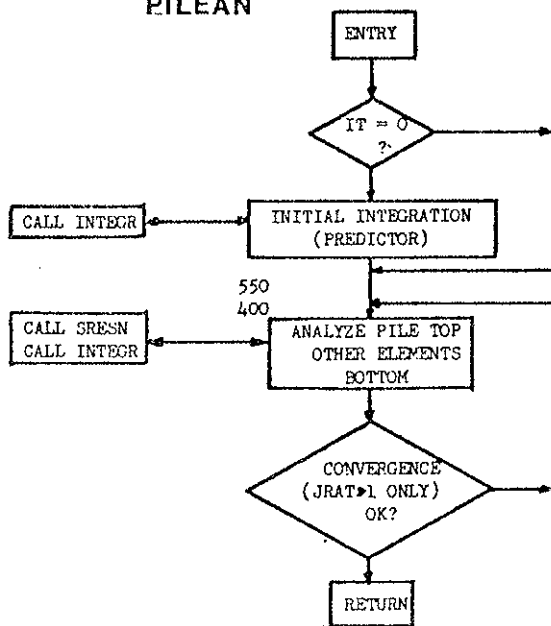
SUBROUTINE AIRSTM



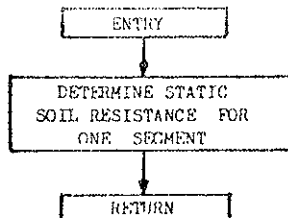
AIRSTM



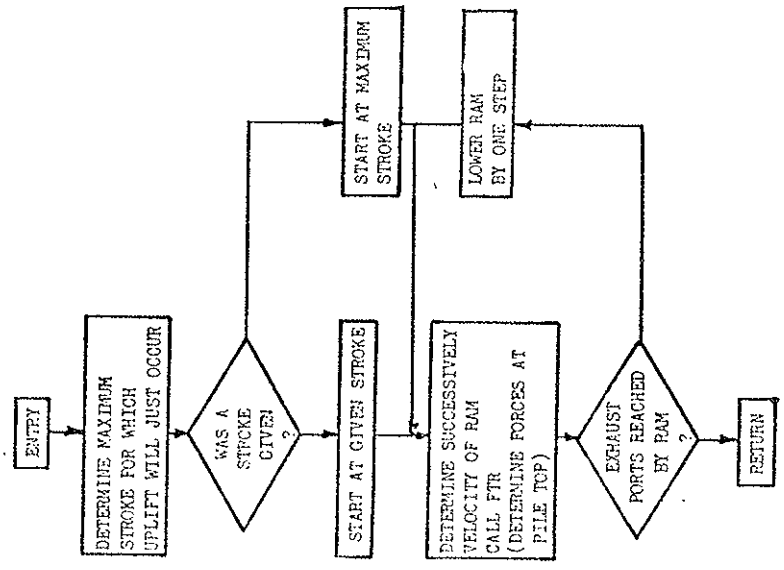
PILEAN



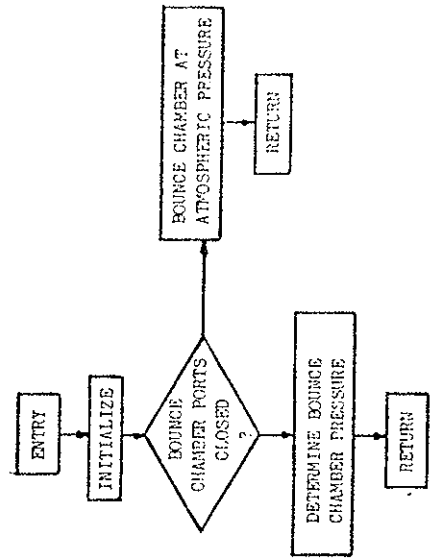
SRESN



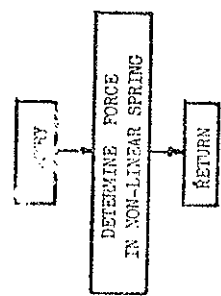
DOWN



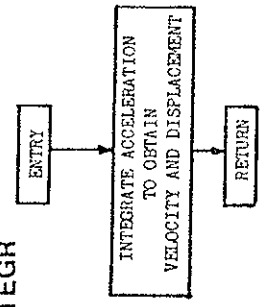
FTR



STIFF

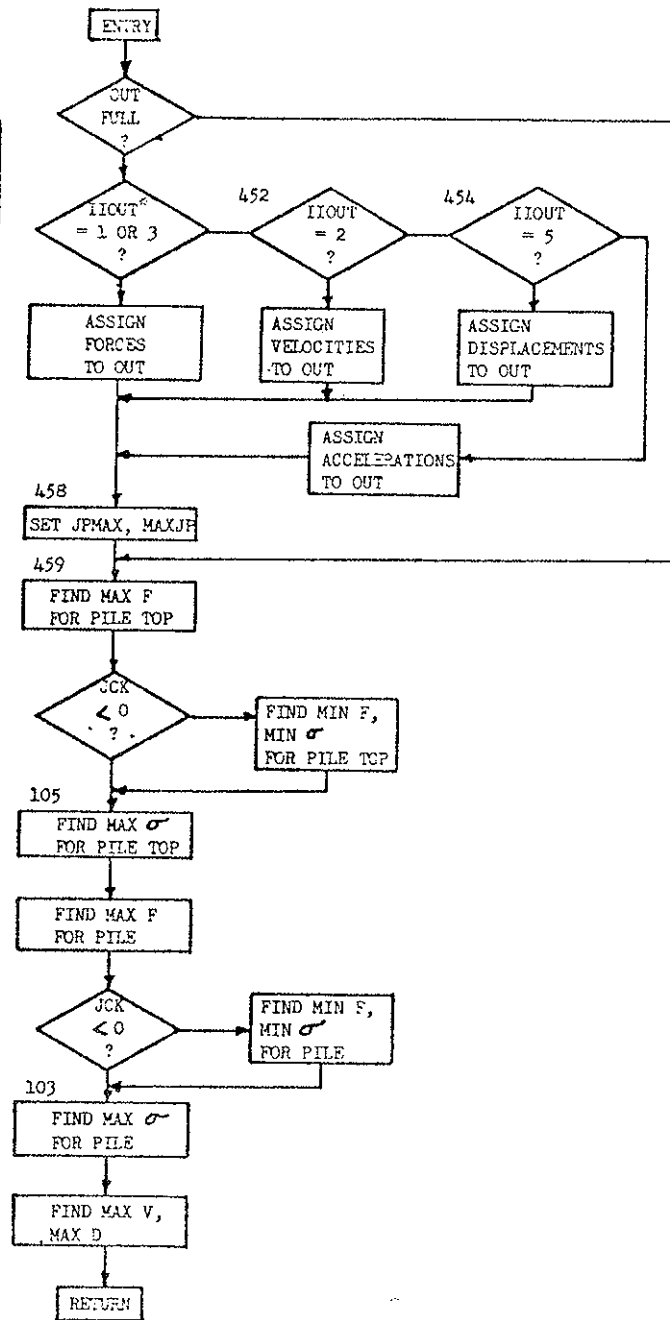


INTEGR

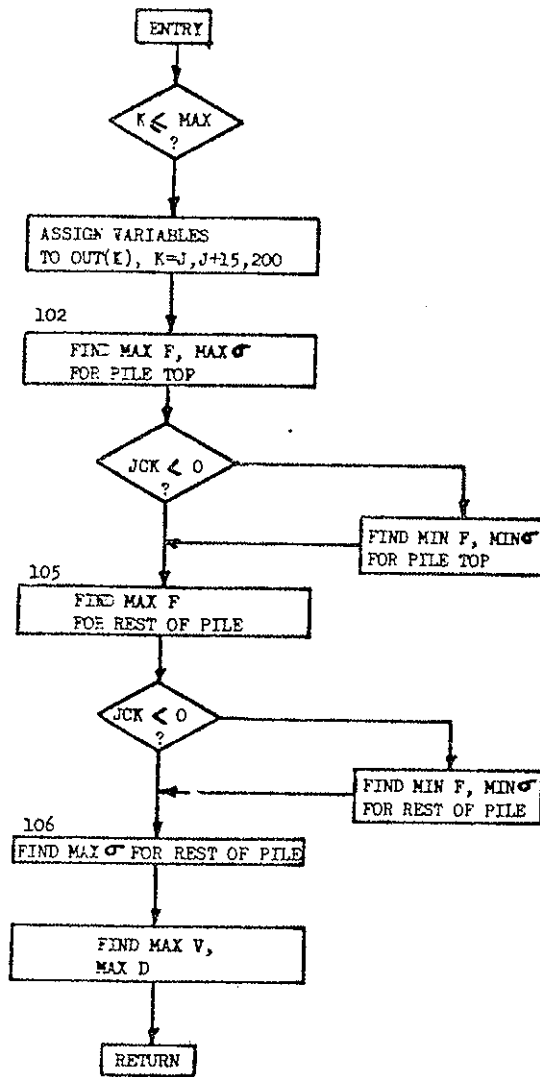


FILL

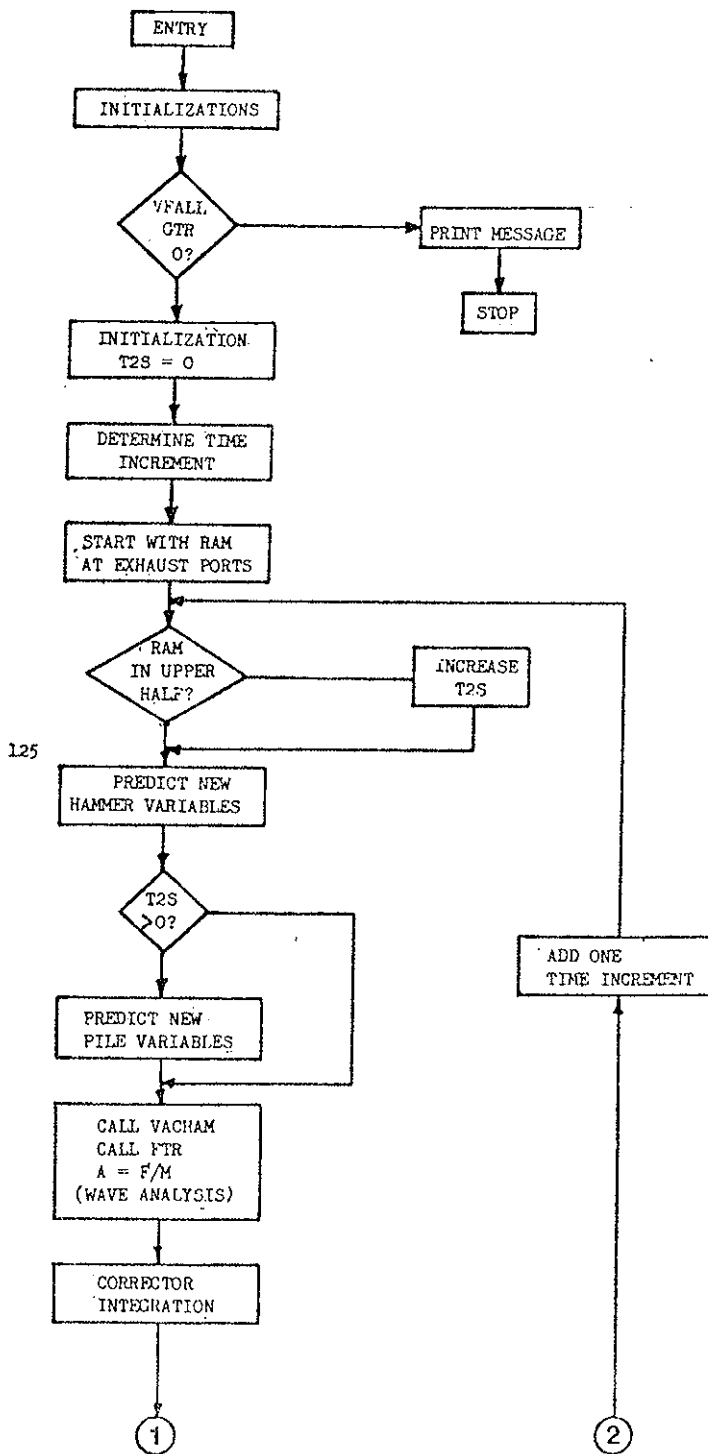
* IOUT
 = IOUT - 10 FOR 9 < IOUT < 19
 = IOUT - 20 FOR 19 < IOUT



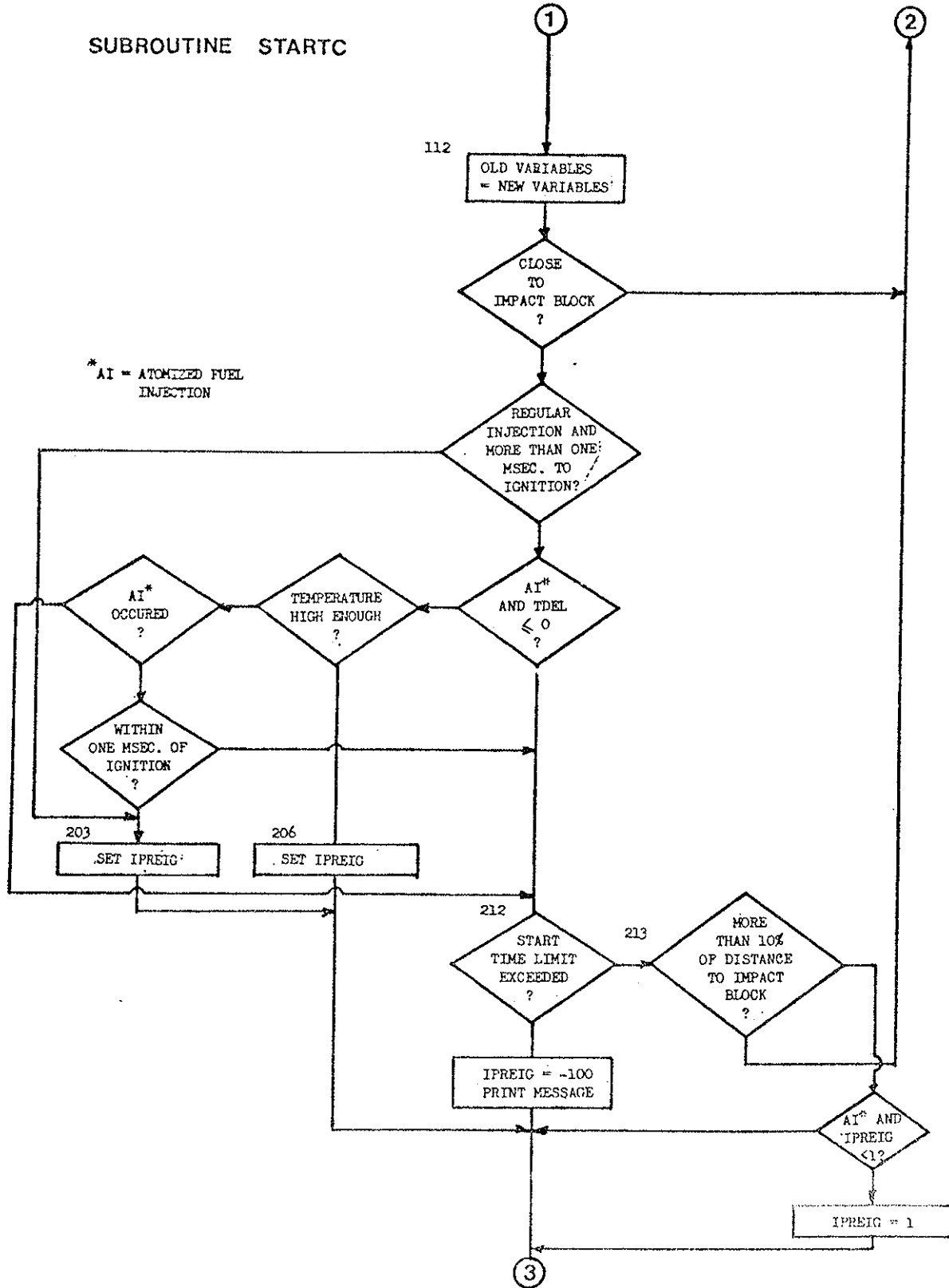
FILL2



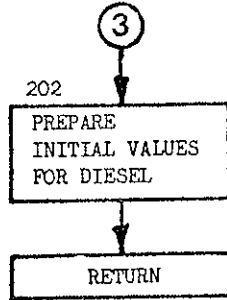
SUBROUTINE STARTC



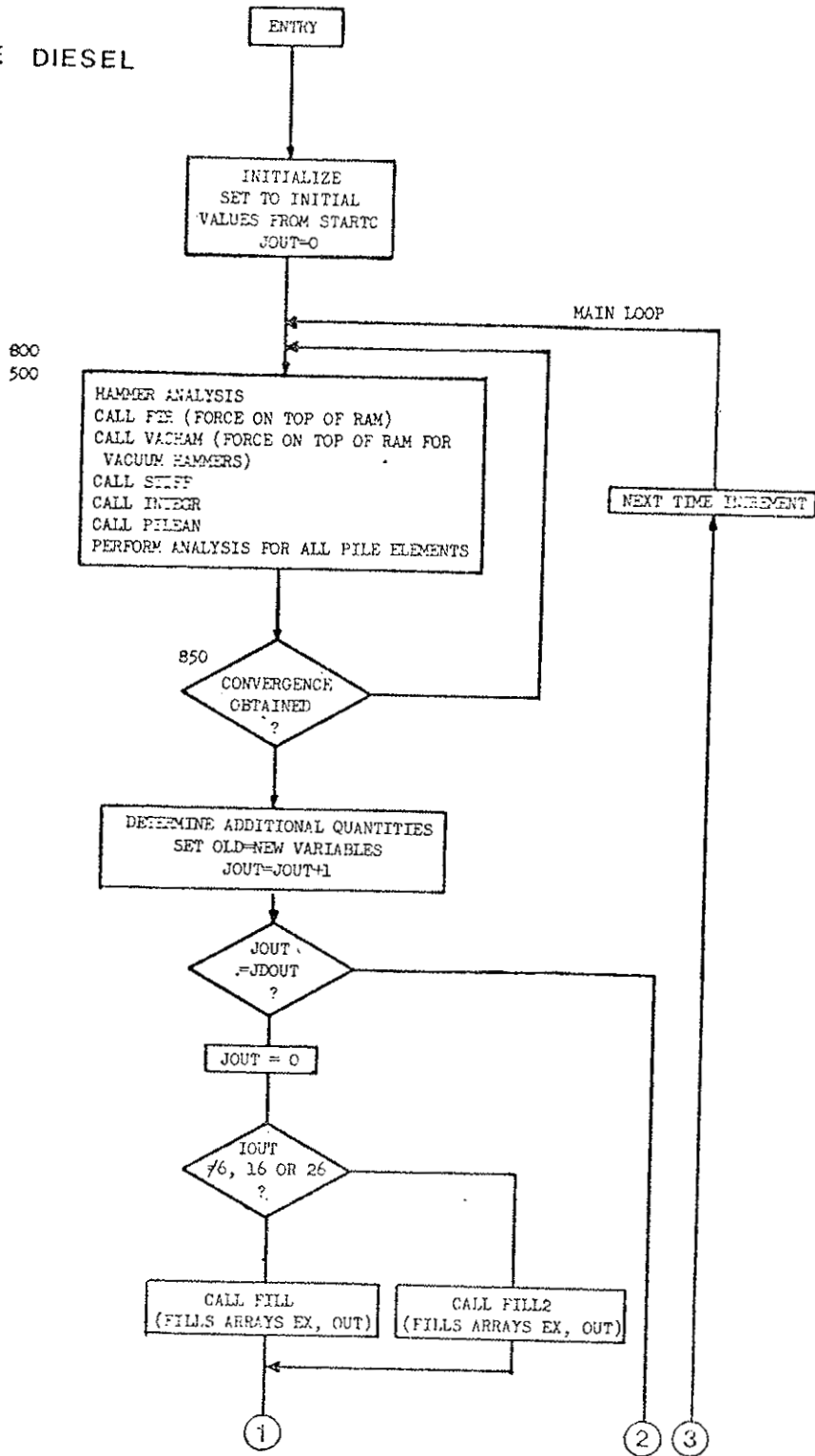
SUBROUTINE STARTC



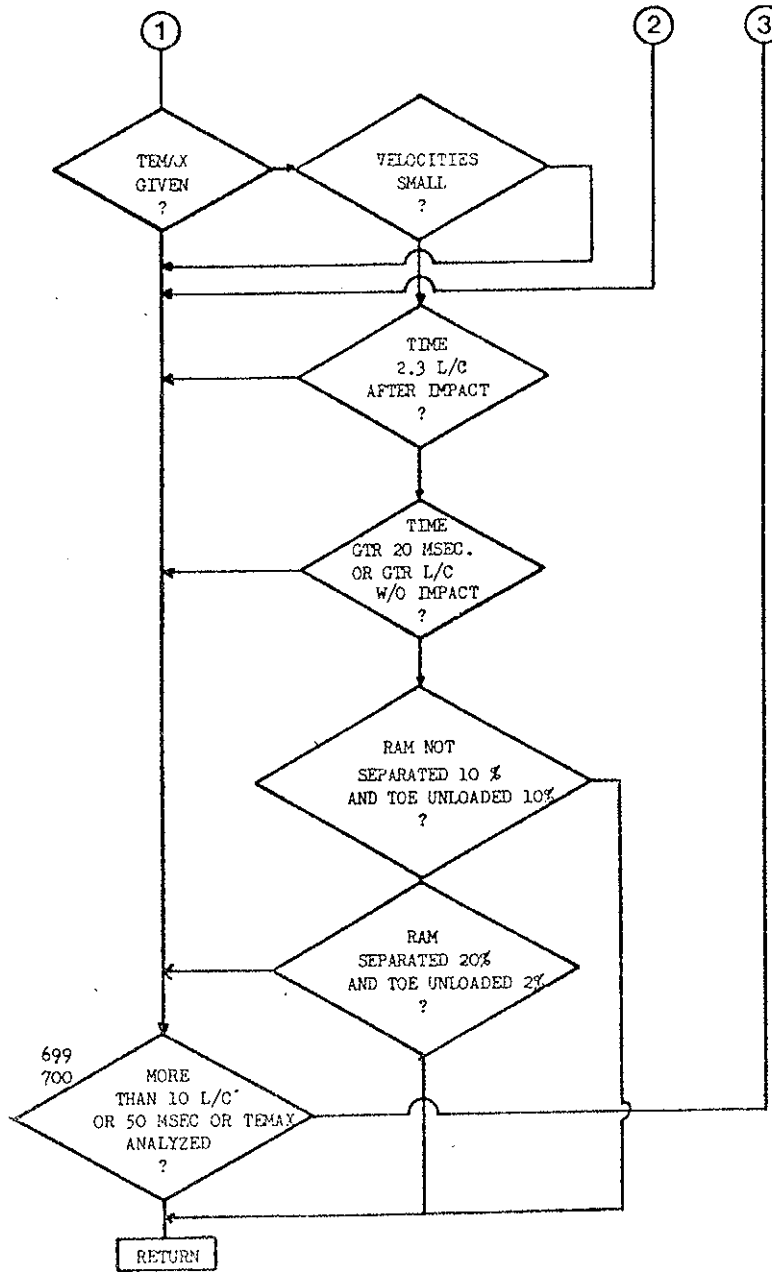
SUBROUTINE STARTC

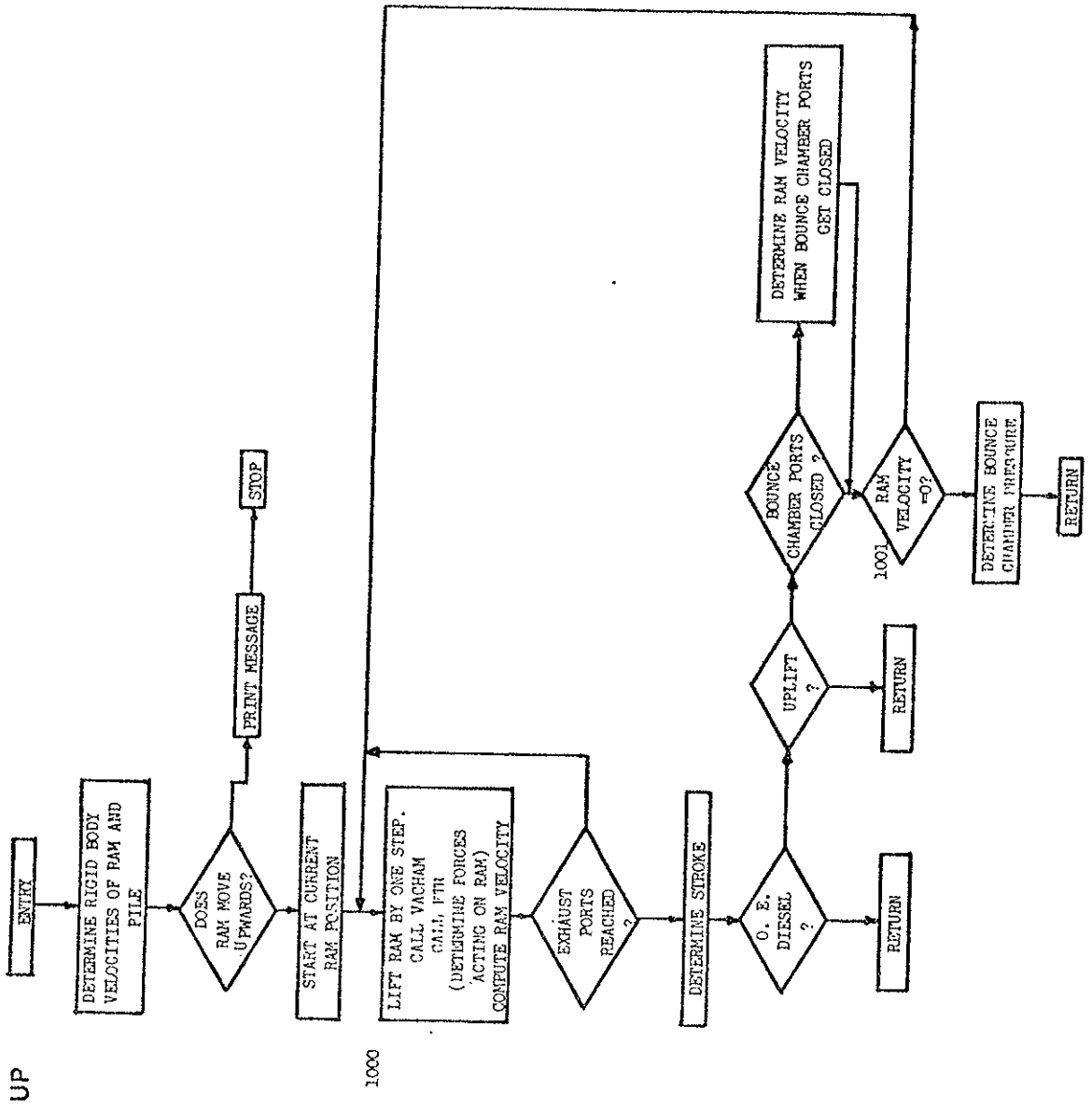


SUBROUTINE DIESEL

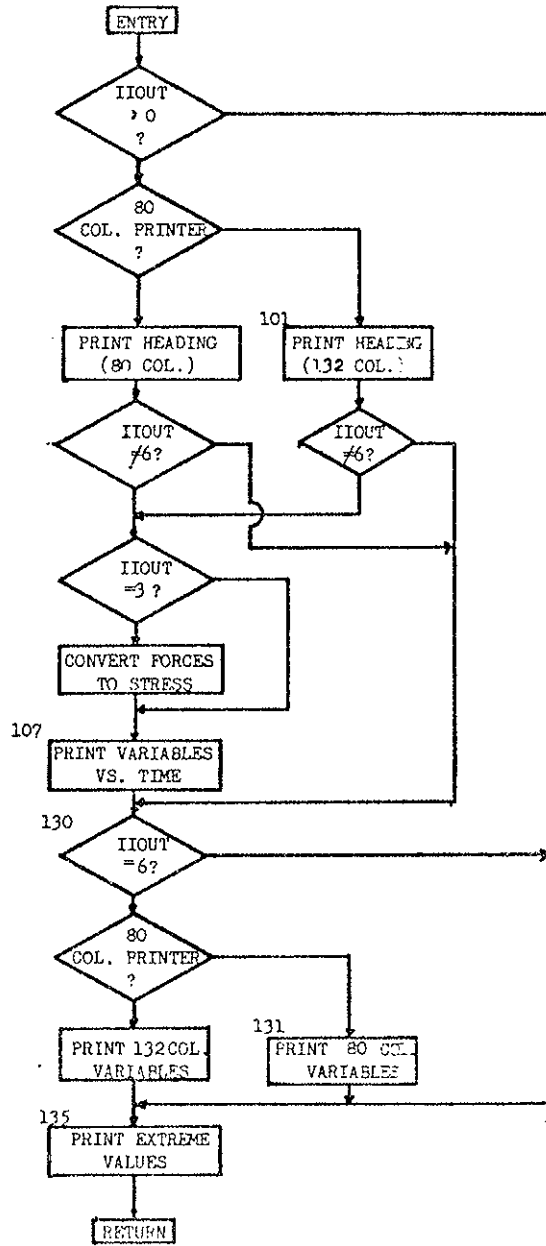


DIESEL

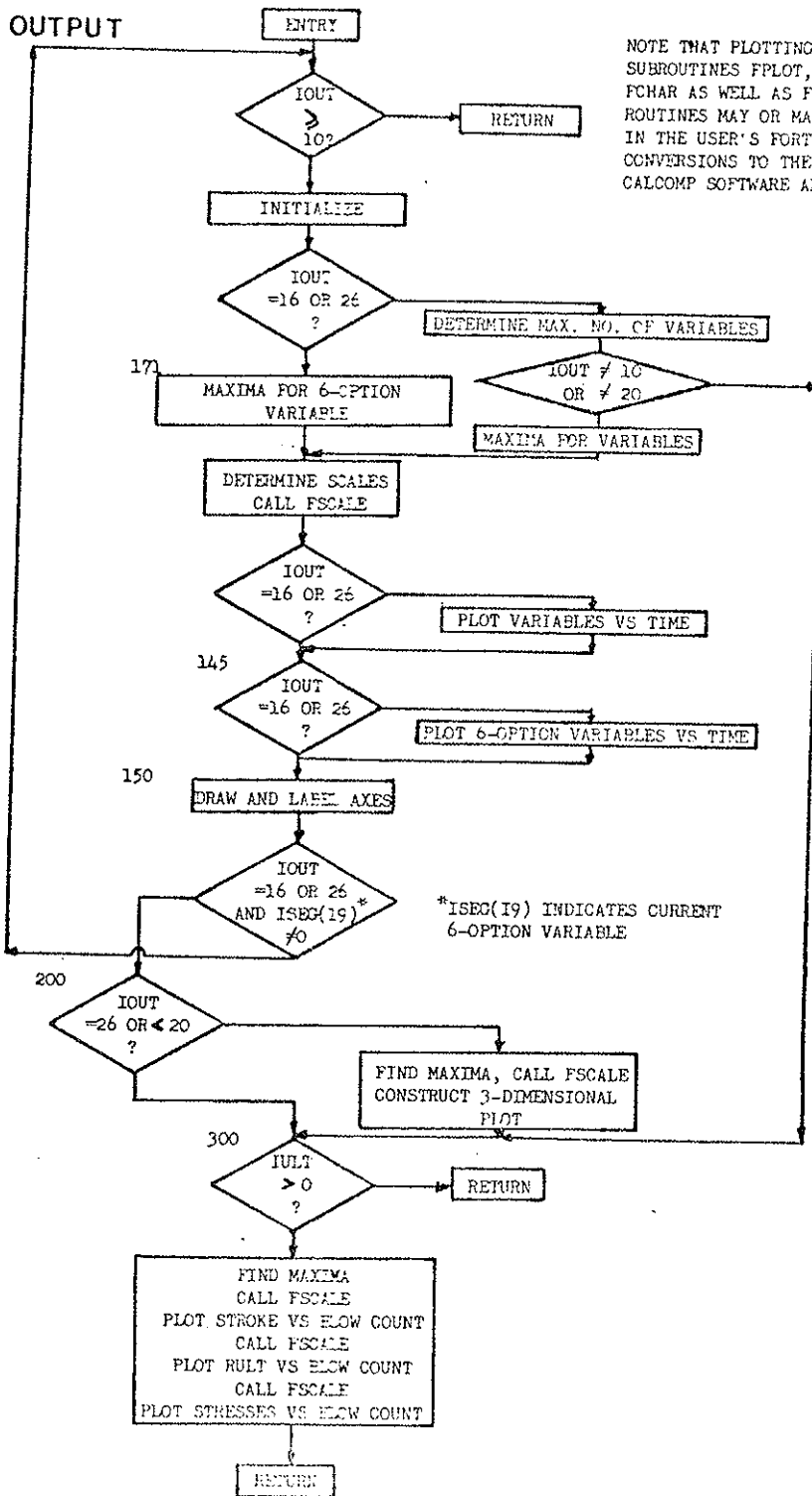




OUT2



OUTPUT



NOTE THAT PLOTTING INVOLVES CALLS TO SUBROUTINES F PLOT, FMOVE, FSCALAR, FCHAR AS WELL AS FSCALE. THESE ROUTINES MAY OR MAY NOT BE AVAILABLE IN THE USER'S FORTRAN LIBRARY. CONVERSIONS TO THE STANDARD CALCOMP SOFTWARE ARE AVAILABLE.

5. LIST OF COMMON VARIABLES

Major variables used in WEAP are stored in COMMON. The following variables appear in order:

<u>Item No.</u>	<u>Name</u>	<u>Type Dimension</u>	<u>Description</u>
1	VINH	Real 10	Initial velocity of hammer components just prior to impact; loaded in STARTC, MAIN.
2	DINH	Real 10	Initial displacement of hammer components; otherwise as in VINH.
3	AOH, VOH DOH	Real 10	Acceleration, velocity and displacement of hammer components at the end of previous time increment. Temporarily used in STARTC, DIESEL, AND AIRSTM.
4	ANH, VNH DNH	Real 10	As in 3 but for current time increment.
5	STH	Real 10	Stiffness of hammer segments; loaded in HAEL and IPTN.
6	HM	Real 10	Mass of hammer segments, otherwise as in 5.
7	AM	Real 10	Assembly masses; loaded in HAEL.
8	STA	Real 10	Assembly stiffnesses; loaded in HAEL.
9	MA	Integer	Number of assembly segments.
10	ESOIL	Real	The soil's coefficient of restitution, squared; loaded in IPTN; used in SRESN.
11	AOP, ANP	Real 99	Acceleration of pile segments at end of previous and at end of current time increment, respectively; loaded in STARTC, PILEAN, DIESEL and AIRSTM.
12	DOP, DNP	Real 99	As in 11 but for pile displacement.

<u>Item No.</u>	<u>Name</u>	<u>Type Dimension</u>	<u>Description</u>
13	VOP, VNP	Real 99	As in 11 but for pile velocities.
14	STP	Real 99	As in 5 but for pile stiffnesses.
15	PM	Real 99	As in 6 but for pile masses.
16	CDP	Real 99	Pile damping parameters; loaded in IPTN.
17	RES, RESO	Real 99	Static soil resistance for end of current and end of previous time increment, respectively. Initialized in AIRSTM, DIESEL, SRESN.
18	SJ	Real 99	Soil damping parameter.
19	SOK	Real 99	Static soil stiffness; loaded in IPTN and MAIN.
20	QS	Real 99	Soil quake; loaded in IPTN.
21	SU	Real 99	Ultimate static resistance. Initialized in SOILN; reloaded in IPTN and MAIN.
22	SPLICE	Real 99	Splice force or slack displacement. Loaded in IPTN.
23	DEPIB	Real	Distance between exhaust ports and impact block. Loaded in HAEL.
24	ARAM	Real	Cross sectional area of ram (bottom); loaded in HAEL.
25	ECUS, ECAP, EPT, EANV	Real	Coefficients of restitution (squared) of pile cushion, capblock, pile top and impact block; loaded in IPTN.
26	VFALL	Real	Ram velocity at time of port closure (diesel) or impact (air/steam). Loaded in IPTN, DOWN, VACHAM, MAIN.
27	VCHAM	Real	Volume of combustion chamber; loaded in HAEL.

<u>Item No.</u>	<u>Name</u>	<u>Type Dimension</u>	<u>Description</u>
28	XPT	Real	Total pile length; loaded in IPTN.
29	SJJJ	Real	Ignition duration; loaded in HAEL or IPTN.
30	EMP	Real	Elastic modulus at pile top; loaded in IPTN.
31	RHO	Real	Mass density at pile top; loaded in IPTN.
32	DTH, DTP	Real	Time increments for hammer and pile, respectively; loaded in IPTN.
33	N, M	Integer	Number of pile, ram segments, respectively; loaded in IPTN and HAEL, respectively.
34	JRAT	Integer	Ratio of pile to hammer time increment; loaded in IPTN.
35	IULT	Integer	RULT - analysis counter; loaded in IPTN as total number of analyses, reloaded in MAIN as counter.
36	RULT	Real	Ultimate capacity; loaded in IPTN; reloaded in MAIN.
37	SOILM	Real	Soil mass; added to pile bottom element; use restricted.
38	DSACP	Real 4	Rounding displacements for impact zones; loaded in IPTN.
39	FPTO	Real	Force at pile top at end of last time increment; loaded in STARTC, DIESEL, AIRSTM.
40	IW	Integer	Write unit number; loaded in IPTN.
41	PCOM	Real	Pressure multiplier; loaded in IPTN.
42	TDEL	Real	Combustion delay; loaded in HAEL.
43	IOUT	Integer	Output option; loaded in IPTN.

<u>Item No.</u>	<u>Name</u>	<u>Type Dimension</u>	<u>Description</u>
44	OUT	Real 3200	Output storage for variables vs. time; loaded in FILL for 1 through 5 and FILL2 for 6 option; contains variables at increments of 200 locations for same time; used in OUT2 and OUTPUT.
45	EXPP	Real	Combustion exponent; loaded in HAEL.
46	TEMI, TEMO	Real	Gas temperature; initial value and value at end of previous time increment; loaded in STARTC.
47	INP	Integer 13	Contains segment numbers of up to 13 pile segments for which output is desired; loaded in IPTN or JJNP.
48	STROKO	Real	Stroke at beginning of analysis; loaded in IPTN and MAIN.
49	STROKN	Real	Rebound stroke; loaded in UP and MAIN.
50	VO	Real	Chamber volume at end of previous time increment; loaded in STARTC and DIESEL.
51	PO	Real	As in 50 but for pressure.
52	DFIN	Real	Final set; loaded in DIESEL, AIRSTM.
53	JDOUT	Integer	Number of time increments which are skipped between output values minus 1; loaded in AIRSTM, DIESEL.
54	JPMAX	Integer	Maximum time counter of values stored in OUT; loaded in FILL or FILL2.
55	EX	Real 600	Stores extreme values obtained in an analysis. Loaded in FILL or FILL2.
56	ICOL	Integer	Printer column indicator (0 = 80 columns, > 0 = 132 columns). Loaded in IPTN.
57	FINIMX	Real	Stores both maximum interaction force and cycle time. Loaded in AIRSTM, DIESEL, STARTC.

<u>Item No.</u>	<u>Name</u>	<u>Type Dimension</u>	<u>Description</u>
58	TEMAX	Real	Maximum analysis time; loaded in IPTN.
59	RESULT	Real 100	Contains summary results from all analyses; subscript 1-10 = RULT 11-20 = Blow Count 21-30 = Stroke 31-40 = Min. Stress 41-50 = Max. Stress 51-60 = Blow Rate 61-70 = Fuel Setting 71-80 = Bounce Chamber Pressure 81-100 = Not used; loaded in MAIN.
60	ISMITH	Integer	Damping option; loaded in IPTN.
61	ITER	Integer	Iteration counter; loaded in IPTN.
62	ITYPH	Integer	Hammer type; loaded in HAEL.
63	IPREIG	Integer	Preignition indicator; loaded in STARTC.
64	VFALLM	Real	Velocity at which uplift occurs; loaded in DOWN.
65	STRMAX	Real	Stroke at which uplift occurs; loaded in DOWN.
66	DCYL	Real	Displacement of cylinder; loaded in DIESEL, UP.
67	PBAR	Real 20	Array containing the following hammer properties in order: (for variable description see the User's Manual, Chapter 4) ART, DSF, not used, EXPB, VCT, DEPBB, DBBT, RWH, DINJ, STRM, P1, P2, P3, P4, P5, EFFICY, Mass of ram (kips sec ² /ft), RWH (Air/Steam), POWSCA, RAM; loaded in HAEL and IPTN.
68	AREA	Real 99	Cross sectional areas of pile segments; loaded in PIEL.
69	JEX	Integer 600	Time counter values corresponding to EX array; loaded in FILL or FILL2.

APPENDIX A

PROGRAM LISTING

CHI/OS 22 JUN 76:

RUN 2

GOBLE H (134600,8)

14:55:26

DATE: 22 JUN 76

SEQ. NO. 3901649

AF ASG PCF=CEO,134600,8,

LENGTH 232. LAST WRITTEN 18 JUN 76 AT 14:02:24

Δ CUR

CUR OF MARCH 3, 1976

TUESDAY, JUNE 22, 1976 AT 14:55:27

1. LIST IPTN

IPTN	SYMBOLIC FORTRAN	18 JUN 76	14:01:37	72	76	5	00226255	2465			
1	SUBROUTINE IPTN (JSELEC,ISEG,YMAX,ALPH,TITLE,IOSTR,STRM)							32	76	000004	0
2	COMMON VINH,DINH,AOH,VOH,DOH,ANH,VNH,DNH,STH,HM,AM,STA,MA,ESOIL,							1	76	000016	5
3	1 AOP,ANP,DOP,DNP,VOP,VNP,STP,PM,CDP,RES,RESO,SJ,SOK,OS,SU,							1	76	000033	0
4	2 SPLICE,DEPIB,ARAM,ECUS,ECAP,EPT,EANV,VFALL,VCHAM,XPT,SJJJ,							1	76	000046	2
5	3 EMP,RHO,DTH,DTP,N,M,JRAT,IULT,RULT,SOILM,DSACP,							1	76	000061	5
6	4 FPTO,IW,PCOM,TDEL,IOUT,OUT,EXPP,TEMI,TEMO,INP,STROKO,STROKN,VO,PO							1	76	000073	2
7	5 ,DFIN,JDOUT,JPMAX,EX,ICOL,FINTMX,TEMAX,RESULT,ISMITH,ITER							1	76	000110	0
8	6 ,ITYPH,IPREIG,VFALLM,STRMAX,DCYL,PBAR,AREA,JEX							53	76	000123	3
9	COMMON/SLAK/OPEN(99)							74	76	000134	5
10	DIMENSION VINH(10),DINH(10),AOH(10),VOH(10),DOH(10),ANH(10),							1	76	000141	2
11	1 VNH(10),DNH(10),STH(10),HM(10),VNP(99),							1	76	000154	5
12	2 AOP(99),ANP(99),DOP(99),DNP(99),VOP(99),STP(99),PM(99),CDP(99),							1	76	000165	1
13	3 RES(99),RESO(99),SJ(99),SOK(99),OS(99),SU(99),SPLICE(99),							1	76	000201	3
14	4 DSACP(3),OUT(3200),INP(13),EX(600),RESULT(100),JEX(600)							53	76	000214	5
15	1 ,AP(20),XP(20),TITLE(10),EE(4),SMITH(2),VISCOU(2),DTYP(2)							1	76	000227	4
16	2 ,AM(10),STA(10),RP(20),EP(20),ALPH(99),HNAME(2),JSELEC(13),							9	76	000243	0
17	3 ISEG(15),PBAR(20),AREA(99)							52	76	000256	4
18	C							70	76	000264	4
19	C INITIALIZE							70	76	000265	5
20	C							70	76	000271	0
21	DATA SMITH,VISCOU /4H SMI,4HHTH ,4H VIS,4HCOUS /							1	76	000272	1
22	IW=6							43	76	000303	4
23	ICOL=132							46	76	000305	3
24	IR=5							43	76	000310	0
25	NHAMR = 100							60	76	000311	5
26	PELE=5.0							43	76	000314	5
27	NLIM=98							43	76	000317	2
28	MLIM=10							43	76	000321	4
29	DO 30 I=1,3							43	76	000324	0
30	30 DSACP(I)=.01							43	76	000327	0
31	CDP(1)=0.0							43	76	000332	4
32	SPLICE(1)=0.							43	76	000335	3
33	DO 40 I=2,99							43	76	000340	4
34	SPLICE(I)=-5000.0							43	76	000343	5
35	40 CDP(I)=0.0							43	76	000347	5
36	EXPP=1.3							43	76	000353	1
37	TEMI=25.0							43	76	000355	4
38	SOILM=0.0							43	76	000360	2
39	SJJJ=0.0							43	76	000363	0

41

40		DULT=0.0			43	76	000365	3
41		DO_146_I=1,13			27	76	000370	0
42		146 JSELEC(I)=0			27	76	000373	2
43	C				70	76	000377	0
44	C	START TO READ DATA			70	76	000400	1
45	C	READ AND PRINT TITLE			70	76	000404	4
46	C	IF NO MORE DATA - QUIT			70	76	000411	3
47	C				70	76	000416	4
48	1	FORMAT (20I4)			1	76	000417	5
49	2	FORMAT (10F8.2)			1	76	000423	4
50	4	FORMAT (1H1,19X,44HW E A P - WAVE EQUATION ANALYSIS FOR PILES//			72	76	000427	5
51		110X,64HTHIS PROGRAM WAS PREPARED FOR THE FEDERAL HIGHWAY ADMINISTR			69	76	000444	3
52		2ATION/22X,38HBY GOBLE & ASSOCIATES, CLEVELAND, OHIO//10X,10A4)			71	76	000461	1
53		READ (IR,5,END=9876,ERR=1020) TITLE			70	76	000475	1
54		GO TO 9877			61	76	000504	2
55	9876	IN =-1000			61	76	000507	1
56		RETURN			61	76	000512	4
57	9877	CONTINUE			61	76	000514	5
58		ALPH(1) = 1.0			67	76	000520	1
59	5	FORMAT (10A4)			70	76	000523	3
60		WRITE (IW,4) TITLE			70	76	000527	2
61	C				70	76	000533	3
62	C	READ OPTIONS AND SPECS.			70	76	000534	4
63	C				70	76	000542	0
64		READ (IR,1,END=9876,ERR=1020) IOUT,IJJ,IHAMR,IOSTR,IFUEL,IPEL,N,			70	76	000543	1
65		1 ISPL,NCROSS,IBEDAM,IPERCS,ISMITH,ITYS,MAXJRT,IPHI,ITER,IDAHA			70	76	000557	2
66	C				70	76	000573	1
67	C	CHECK AND/OR CORRECT OPTIONS			70	76	000574	2
68	C				70	76	000602	4
69		II=IABS(IOUT)			43	76	000603	5
70		PM(99) = 0.0			62	76	000607	1
71		IF (IDAHA.EQ.0) PM(99) = 2.0			69	76	000612	2
72		IF (IDAHA.GT.0) PM(99) = IDAHA			62	76	000620	2
73		CDP(99) = 0.0			58	76	000626	4
74		IF (IPERCS.LT.0.0) CDP(99) = 1.0			58	76	000632	0
75		IPERCS = IABS(IPERCS)			70	76	000640	4
76		IF (IBEDAM.EQ.0) IBEDAM = 1			64	76	000645	2
77		IF (IBEDAM.LT.0) IBEDAM = 0			64	76	000653	0
78		IF (II.GE.20) II=II-20			43	76	000660	4
79		IF (II.GE.10) II=II-10			43	76	000665	3
80		IF (II.GT.6) IOUT=0			43	76	000672	2
81		IF (IFUEL.LT.1.OR.IFUEL.GT.5) IFUEL=1			41	76	000676	4
82		IF (ITER.EQ.0) ITER=2			60	76	000706	1
83		IF (ITER.LT.0) ITER=0			60	76	000712	5
84		IF (MAXJRT.LE.0) MAXJRT=1			21	76	000717	3
85		IF (IHAMR.GT.NHAMR) WRITE (IW,3)			1	76	000724	5
86		IF (IHAMR.LT.0) IHAMR=0			60	76	000733	3
87		IF (II.EQ.6) IJJ = 0			67	76	000740	3
88	3	FORMAT (27H UNKNOWN HAMMER TYPE, SORRY)			1	76	000745	0
89		IF (IHAMR.GT.NHAMR) STOP			1	76	000755	2
90	10	FORMAT (48H ***N WAS NOT SET,EXTENDED INPUT NOT POSSIBLE***)			25	76	000762	3
91		IF (N.LE.1.AND.IPEL.GT.0) WRITE (IW,10)			67	76	000776	3
92		IF (N.LE.1.AND.IPEL.GT.0) STOP			67	76	001006	2
93	C				70	76	001014	4

94	C	EXTENDED PILE INPUT	70	76	001015	5
95	C		70	76	001022	3
96		IF (IPEL.EQ.2) READ (IR,2) (STP(I),I=1,N)	25	76	001023	4
97		IF (IPEL.EQ.2) READ (IR,2) (PM(I),I=1,N)	25	76	001033	5
98		IF (IPEL.GE.1) READ (IR,2) (ALPH(I),I=1,N)	1	76	001043	5
99		IF (IPEL.LT.2) GO TO 142	13	76	001054	1
100		9 FORMAT (50H MASSES AND STIFFNESSES FOR FIRST ELEMENT ARE NOT	21	76	001061	2
101		*9HSPECIFIED)	21	76	001075	3
102		IF (PM(1).LT.10.E-8.OR.STP(1).LT.10.E-8) WRITE (IW,9)	21	76	001100	5
103		IF (PM(1).LT.10.E-8.OR.STP(1).LT.10.E-8) STOP	21	76	001113	0
104		DO 143 I=1,N	13	76	001123	5
105		STP(I)=STP(I)*12.0	53	76	001127	0
106		PM(I)=PM(I)/32.17	53	76	001133	1
107		IF (STP(I).LT.0.0001) STP(I)=STP(I-1)	53	76	001137	1
108	143	IF (PM(I).LT.10.E-8) PM(I)=PM(I-1)	53	76	001146	4
109	142	CONTINUE	13	76	001156	2
110		DO 600 I=2,99	66	76	001161	3
111	600	IF (ALPH(I).LT.0.0001) ALPH(I)=ALPH(I-1)	21	76	001164	5
112		M1=8	1	76	001175	3
113		M2=9	1	76	001177	2
114	C		70	76	001201	1
115	C	DRIVING SYSTEM INPUT	70	76	001202	2
116	C		70	76	001207	1
117		READ (IR,2) HM(M2),STH(M1),STH(M2)	9	76	001210	2
118		HM(M2)=HM(M2)/32.17	9	76	001217	2
119		STH(M2) = STH(M2)*12.	1	76	001223	4
120		STH(M1) = STH(M1)*12.	1	76	001230	2
121		READ (IR,2) (EE(I),I=1,4),TEMAX,B,A,C	54	76	001235	0
122		IF (STH(M2).LT.1.0) EE(4) = 1.0	60	76	001244	2
123		DO 141 I=1,4	1	76	001252	5
124		IF (EE(I).LT.0.00001.OR.EE(I).GT.1.0) EE(I) = 0.85	64	76	001256	0
125	141	EE(I)=EE(I)**2	1	76	001267	4
126		EANV=EE(1)	1	76	001273	5
127		ECAP=EE(2)	1	76	001276	4
128		EPT=EE(3)	1	76	001301	3
129		ECUS=EE(4)	1	76	001304	1
130	C		1	76	001307	0
131	C	NCROSS EQL 0 MEANS UNIFORM	1	76	001310	1
132	C		70	76	001316	0
133	C	START PILE INPUT	70	76	001317	1
134	C		70	76	001323	2
135		NN=2	1	76	001324	3
136		XP(I) = 0.0	1	76	001326	2
137		READ (IR,2) XPT,AP(1),EP(1),RP(1),SOILM,TDORGA,A1,B1,C1	69	76	001331	2
138		ATP = AP(1)/144.0	38	76	001344	0
139		IF (EP(1).LT.0.1) EP(1) = 29000.	60	76	001350	0
140		IF (RP(1).LT.0.1) RP(1) = 492.	60	76	001356	4
141		EMP=EP(1)*144.	13	76	001365	0
142		RHO=RP(1)/32170.	13	76	001376	3
143		IF (NCROSS.LT.1) GO TO 221	21	76	001374	2
144		I=2	1	76	001401	5
145	C	FOR NONUNIFORM PILES	70	76	001403	3
146	220	READ (IR,2) XP(I),AP(I),EP(I),RP(I)	1	76	001410	2
147		NN=I	1	76	001420	1

148		IF (EP(I).LT.0.1) EP(I) = EP(I-1)	60	76	001422 0
149		IF (RP(I).LT.0.1) RP(I)=RP(I-1)	1	76	001430 5
150		I = I+1	1	76	001437 2
151	12	FORMAT (48H PILE PROPERTIES AT BOTTOM OF PILE NOT SPECIFIED)	37	76	001441 4
152		IF (XP(I-1).LT.XP(I-2)) WRITE (IW,12)	37	76	001455 4
153		IF (XP(I-1).LT.XP(I-2)) STOP	37	76	001465 1
154		IF (XP(I-1).LT.XPT) GO TO 220	1	76	001473 1
155		GO TO 222	1	76	001501 2
156	221	CONTINUE	1	76	001504 0
157		AP(2)=AP(1)	1	76	001507 2
158		XP(2)=XPT	16	76	001512 2
159		EP(2)=EP(1)	1	76	001515 0
160		RP(2)=RP(1)	1	76	001520 0
161	222	CONTINUE	1	76	001523 0
162	C	DETERMINE NO. OF STATIONS SPECIFIED	70	76	001526 1
163		DO 2100 I=1,20	1	76	001535 4
164		IF (XP(I).GE.XPT-10E-8) GO TO 2101	1	76	001541 1
165	2100	CONTINUE	1	76	001550 1
166	2101	NN=I	1	76	001553 3
167		IF (IPEL.NE.0) GO TO 2102	1	76	001556 1
168		N1=XPT/PELE+1	7	76	001563 3
169		IF (N.LE.1) N=N1	7	76	001566 5
170		IF (N.GT.NLIM) N=NLIM	1	76	001572 4
171	C		70	76	001577 2
172	C	DETERMINE PILE SEGMENTS	70	76	001600 3
173	C		70	76	001605 5
174	2102	CALL PIEL(NN,N,XP,AP,EP,RP,STP,PM,ALPH,IOUT,IW,ICOL,IPEL,	60	76	001607 0
175		1 AREA)	52	76	001622 5
176	C		70	76	001625 1
177	C	OBTAIN HAMMER DATA	70	76	001626 2
178	C		70	76	001632 5
179		CALL HAEL(IHAMR,DTCRH,DTCRA,HNAME,IR)	40	76	001634 0
180		IF (ABS(TDORGA).GT.10.E-8) SJJJ = TDORGA	69	76	001643 3
181		STRM = PBAR(10)	39	76	001653 3
182	C		70	76	001657 1
183	C	CHECK ON SELF-GENERATED PILE DATA	70	76	001660 2
184	C		70	76	001667 3
185		DO 150 I=1,N	41	76	001670 4
186		IF (STP(I).GT.100.0.AND.PM(I).GT.10E-8) GO TO 150	42	76	001673 5
187	7	FORMAT (54H PILE MASSES OR STIFFNESSES TOO SMALL,CHECK INPUT INFO)	41	76	001705 2
188		WRITE (IW,7)	41	76	001722 1
189		STOP	41	76	001725 2
190	150	CONTINUE	41	76	001727 1
191	C	FUEL PRESSURE AND PRESSURE MULTIPLIER FOR DIESELS	70	76	001732 2
192		PSPEC=PBAR(10+IFUEL)	41	76	001744 1
193		IF (PSPEC.LT.0.01.AND.IFUEL.GT.1) PSPEC=1000.0	41	76	001750 4
194		IF (VCHAM.GT.10.E-8) PCOM=PSPEC/(14.7*(1.+DEPIB*ARAM/VCHAM)**1.35)	57	76	001761 4
195		NM1=N-1	1	76	001776 1
196		N1=N+1	18	76	002000 3
197		IF (A.GT.0.1) PCOM = A	17	76	002002 4
198		SOILM=SOILM/32.17	41	76	002007 3
199		IF (A1.GT.10.E-8) DSACP(1)=A1	41	76	002013 3
200		IF (B1.GT.10.E-8) DSACP(2)=B1	41	76	002021 4
201		IF (C1.GT.10.E-8) DSACP(3)=C1	41	76	002027 5

202		IF (C.GT.0.1) EXPP = C	17	76	002036	0
203		IF (ABS(B).GT.10.E-8) TDEL=B	58	76	002042	5
204	C		70	76	002050	5
205	C	READ SOIL AND ADDITIONAL HAMMER DATA	70	76	002052	0
206	C		70	76	002061	4
207		READ (IR,2,END=9876,ERR=1020) QS(1),QS(N+1),SJ(1),SJ(N+1),RULT,	70	76	002062	5
208		1 ESOIL,STROKE,EFFICY,PSTEAM,RWT	70	76	002076	5
209		SJS=SJ(1)	55	76	002105	3
210		SJT=SJ(N+1)	55	76	002110	1
211		IF (EFFICY.GT.0.001) PBAR(16)=EFFICY	49	76	002113	1
212		IF (RWT.GT.0.0.AND.ITYPH.NE.3) PBAR(8)=RWT	50	76	002122	3
213		IF (ITYPH.NE.3) GO TO 160	49	76	002132	5
214	C		70	76	002140	1
215	C	FOR A/S HAMMERS OVERRIDE FILE DATA	70	76	002141	2
216	C		70	76	002150	4
217		IF (STROKE.GT.0.001) PBAR(10)=STROKE	49	76	002151	5
218		IF (STROKE.LT.0.01) STROKE=STRM	52	76	002161	1
219		IF (RWT.GT.0.0) PBAR(18)=RWT	50	76	002167	4
220		IF (PSTEAM.GT.PBAR(11)) PSTEAM=PBAR(11)	49	76	002175	4
221		IF (PSTEAM.LT.0.001) PSTEAM=PBAR(11)	49	76	002205	3
222	C	DOUBLE ACTING AIR/STEAM STROKE (EQUIVALENT)	70	76	002214	5
223		IF (PBAR(11).GT.10.0) STROKE=PBAR(10)*(1.0+(PSTEAM*PBAR(18)))/(66	76	002225	4
224		1 PBAR(11)*PBAR(17)))	66	76	002241	3
225		VFALL=SQRT(STROKE*64.34*PBAR(16))	49	76	002246	1
226		WRITE (IW,161) STROKE,PBAR(16),VFALL	49	76	002255	0
227	161	FORMAT (16H STROKE (EQUIV.) ,F4.1,4H FT, ,11H EFFICIENCY ,	49	76	002264	2
228		1 F5.2,18H , IMPACT VELOCITY ,F5.1,5H FT/S)	49	76	002300	1
229			49	76	002310	5
230		IF (PBAR(11).GT.10.0) WRITE (IW,162) PSTEAM,PBAR(11),PBAR(18),	49	76	002311	4
231		1 PBAR(17)	49	76	002325	3
232	162	FORMAT (22H ACTUAL/ MAX. PRESSURE ,F6.1,1H/,F6.1,23H PSI, REACT./R	54	76	002330	2
233		1AM WEIGHT,F6.1,1H/,F6.1,5H KIPS)	49	76	002345	3
234		DO 164 I=21,30	69	76	002354	3
235	164	RESULT(I) = STROKE	69	76	002360	0
236	160	CONTINUE	49	76	002365	0
237	C		70	76	002370	1
238	C	DETERMINE ULTIMATE CAPACITY TO BE ANALYZED	70	76	002371	2
239	C		70	76	002402	0
240		RULT=2.*RULT	25	76	002403	1
241		RMAX=RULT	60	76	002406	2
242		IF (ESOIL.LT.10.E-4.OR.ESOIL.GT.1.0) ESOIL=1.0	50	76	002411	0
243		ESOIL=ESOIL**2.	7	76	002422	0
244		DO 114 I=2,N	64	76	002425	4
245		SJ(I)=SJ(1)	1	76	002430	5
246	114	QS(I)=QS(1)	1	76	002433	5
247	C		70	76	002437	4
248	C	READ EXTENDED SOIL INPUT	70	76	002440	5
249	C		70	76	002446	2
250		IF (ITYS.LT.-1) READ(IR,2) (QS(I),I=1,NI)	58	76	002447	3
251		IF (ITYS.LT.0) READ (IR,2) (SJ(I),I=1,NI)	25	76	002457	4
252		IF (ITYS.LT.-1) READ(IR,2) (SU(I),I=1,NI)	58	76	002467	5
253		IF (ITYS.GE.0) GO TO 230	58	76	002500	0
254		DO 200 I=2,NI	25	76	002505	1
255		IF (QS(I).LT.0.0001) QS(I)=QS(I-1)	25	76	002510	3

256		IF (SJ(I).LT.0.0001) SJ(I)=SJ(I-1)	25	76	002517 3
257		IF (ITYS.LT.-1.AND.SU(I).LT.0.0001) SU(I)=SU(I-1)	64	76	002526 3
258	200	CONTINUE	26	76	002540 0
259	230	CONTINUE	58	76	002543 2
260	C		70	76	002546 4
261	C	DISTRIBUTE RESISTANCE	70	76	002547 5
262	C		70	76	002554 5
263		IF (ITYS.GE.-1) CALL SOILN (ITYS,IPERCS,N,XPT,SU,IR,ALPH)	58	76	002556 0
264		SUS=0.	21	76	002571 0
265		DO 184 I=1,N1	21	76	002573 1
266	184	SUS=SUS+SU(I)	21	76	002576 3
267		SUJ = SUS - SU(N1)	25	76	002602 3
268	C		70	76	002606 4
269	C	DISTRIBUTE CASE DAMPING	70	76	002607 5
270	C		70	76	002615 1
271		DO 185 I=1,N1	21	76	002616 2
272		IF (ISMITH.GT.0.OR.I.EQ.N1) GO TO 187	64	76	002621 4
273		IF (ITYS.LT.0) GO TO 1187	74	76	002631 1
274		IF (SUJ.LT.10.E-10) SJ(I)=SJ(I)/FLOAT(N)	64	76	002636 1
275		IF (SUJ.GE.10E-10) SJ(I)=SJ(I)*SU(I)/SUJ	64	76	002646 1
276	1187	CONTINUE	74	76	002656 1
277		SJ(I)=SJ(I)*SQRT(STP(I)*PM(I))	64	76	002661 3
278	187	CONTINUE	64	76	002667 5
279		QS(I)=QS(I)/12.	21	76	002673 1
280		SU(I)=SU(I)/SUS	21	76	002676 5
281	C		70	76	002702 3
282	C	CHECK QUAKES	70	76	002703 4
283	C		70	76	002707 1
284	185	IF (QS(I).LT.0.0001) QS(I)=.08333	21	76	002710 2
285		IF (ISMITH.LT.1) SJ(N1)=SJ(N1)*SQRT(STP(N)*PM(N))	28	76	002717 5
286	6	FORMAT(I4,2F8.2)	74	76	002735 3
287		DO 301 I=1,99	76	76	002741 4
288	301	OPEN(I)=0.0	74	76	002745 0
289		IF (ISPL.LT.1) GO TO 144	15	76	002750 4
290	C		70	76	002755 5
291	C	READ SPLICE AND PLOTTER INFO.	70	76	002757 0
292	C		70	76	002765 3
293		DO 300 I=1,ISPL	8	76	002766 4
294		READ(IR,6,END=9876,ERR=1020),J,SPLICE(J),OPEN(J)	74	76	003000 0
295	300	CONTINUE	74	76	003011 2
296		DO 302 I=2,N	75	76	003017 4
297	302	OPEN(I)=OPEN(I)+OPEN(I-1)	74	76	003022 5
298	144	CONTINUE	15	76	003030 5
299		IF (IABS(IOUT).GT.9) READ (IR,2,END=1000,ERR=1020) YMAX	70	76	003034 0
300		IF (IABS(IOUT).GT.9) READ (IR,1,END=1000,ERR=1020) (ISEG(I),I=1,15	71	76	003046 4
301	1)		36	76	003063 1
302		GO TO 1003	36	76	003064 4
303	1000	CONTINUE	36	76	003067 3
304	1001	FORMAT (51H INSUFFICIENT PLOTTER INFORMATION, THEREFORE VALUES *	37	76	003072 5
305		*11H ARE ASSUMED)	36	76	003107 4
306		YMAX = 0.0	71	76	003113 3
307		DO 1004 I=1,15	71	76	003116 2
308	1004	ISEG(I) = 0	71	76	003121 5
309		WRITE (IW,1001)	36	76	003125 4

310	1003	CONTINUE	36	76	003131	2
311	C	DETERMINE PILE DAMPING	24	76	003134	4
312		DFAC=FLOAT(IBEDAM)/50.	24	76	003141	5
313		DO 145 I=1,N	24	76	003146	4
314	145	CDP(I)=DFAC*SQRT(PM(I)*STP(I))	41	76	003151	5
315	C		70	76	003160	5
316	C	PRINT INPUT INFO	70	76	003162	0
317	C		70	76	003166	1
318		CALL OUTIN(SU,SJ,GS,STH,STP,HM,PM,IHAMR,EE,N,M,IW,XPT,ISMITH,	1	76	003167	2
319		IHNAME,ESOIL,AP(1),EMP,RHO,SPLICE,CDP,ALPH,ITYPH)	48	76	003203	0
320		DO 876 I=1,2	1	76	003214	3
321		DTYP(I) = VISCOU(I)	1	76	003217	4
322		IF (ISMITH.EQ.1) DTYP(I)=SMITH(I)	1	76	003224	0
323	876	CONTINUE	1	76	003232	5
324		IF (IPHI.LT.100.AND.ITYPH.LT.3) IPHI=140	60	76	003236	1
325		IF (IPHI.LT.100.AND.ITYPH.EQ.3) IPHI = 160	60	76	003246	1
326		PHI = FLOAT(IPHI)/100.	1	76	003256	3
327		PFAC = 1./PHI	60	76	003263	2
328	8	FORMAT (/15X,26HOPTIONS AND SPECIFICATIONS/ 1 4H PHI,F7,2,4X,12HS-DAMPING ,2A4,4X,10HRWT (KIPS),F10,2/ 2 5H IOUT,F16,4X,10HP-DAMPING ,I10,4X,14HSOIL DIST. NO.,I6/ 3 6H IFUEL,I5,4X,6HJ SKIN,F14,2,4X,11HTDEL (SEC.),F9,4/ 4 6H IOSTR,I5,4X,5HJ TOE,F15,2,4X,10HTEMAX (MS),F10,2)	64	76	003266	4
329		DO 188 I=2,N	62	76	003277	4
330		IF (SPLICE(I).GT.0.0) SPLICE(I) = -SPLICE(I)	62	76	003313	0
331		IF (ABS(SPLICE(I)).LT.10.E-8) SPLICE(I) = 10.E-8	64	76	003326	1
332		IF (SPLICE(I).LT.0.0.AND.SPLICE(I).GE.-.5)	64	76	003340	5
333		* WRITE (IW,189) I,SPLICE(I)	64	76	003353	1
334	189	FORMAT (10H AT SEGMT,I2,11H SLACK (FT),F10,5)	64	76	003356	2
335			73	76	003367	0
336	188	CONTINUE	67	76	003400	2
337	47	FORMAT (7H SOILM=,F5,2,5H KIPS,8H DS ANV=,F5,3, 1 8H DS CAP=,F5,3,9H DS PTOP=,F5,3)	64	76	003410	4
338		IS = ITYS	73	76	003430	1
339		IF (IS.EQ.11) IS = 0	65	76	003442	0
340		WRITE (IW,8) PHI,DTYP,PBAR(8),IOUT,IBEDAM,IS,IFUEL,SJS,B,IOSTR,	70	76	003445	2
341		1 SJT,TEMAX	43	76	003457	1
342			47	76	003466	2
343			47	76	003471	0
344			62	76	003475	3
345			62	76	003511	3
346	506	FORMAT (15X,10EFFICIENCY,F10,3)	76	76	003514	3
347		IF (ITYPH.LT.3.AND.PBAR(16).GT.0.001) WRITE (IW,506) PBAR(16)	76	76	003524	0
348		IF (SOILM.GT.0.0.OR.A1.GT.0.0.OR.B1.GT.0.0.OR.C1.	70	76	003537	4
349		1 GT.0.0) WRITE (IW,47) SOILM,A1,B1,C1	70	76	003551	1
350		IULT = 0	1	76	003560	5
351		IF (RULT.GT.10.E-4) GO TO 502	36	76	003563	2
352		IF (RULT.LT.0.) GO TO 191	21	76	003571	3
353	C		70	76	003576	5
354	C	GENERATE RULT VALUES	70	76	003600	0
355	C		70	76	003604	5
356		RULT = SQRT(EMP*RHO)*ATP*3.0	38	76	003606	0
357		I = RULT*0.05 + 0.5	47	76	003614	0
358		IF (I.LT.1) I = 1	47	76	003620	2
359		RULT = I * 20.0	47	76	003624	2
360		IF (EMP.LT.3000000.) RULT=RULT/2.0	1	76	003630	0
361		RMAX=RULT*5.0	60	76	003637	0
362		IULT=10	11	76	003642	2
363		GO TO 502	21	76	003644	4

364	191	CONTINUE							
365	C				1	76	003647	2	
366	C	READ RULT VALUES			70	76	003652	4	
367	C				70	76	003653	5	
368		READ (IR,2) (RESULT(I),I=1,10)			70	76	003660	0	
369		DO 500 I=1,10			21	76	003661	1	
370		IF (RESULT(I).LT.0.001) GO TO 501			21	76	003667	3	
371	500	CONTINUE			70	76	003672	5	
372	501	CONTINUE			70	76	003701	4	
373		RMAX=RESULT(I-1)*2.0			21	76	003705	0	
374		IULT = I-1			60	76	003710	1	
375	C				38	76	003714	4	
376	C	CONVERT FIRST RULT TO KIPS			70	76	003717	3	
377	C				70	76	003720	4	
378		RULT=RESULT(I)*2.0			70	76	003726	3	
379	502	CONTINUE			54	76	003727	4	
380		IF (ITYPH.NE.1) GO TO 505			21	76	003733	5	
381		DEFFALL=STROKE-DEPIB			42	76	003737	0	
382		IF (DEFFALL.GT.0.001) GO TO 106			1	76	003744	2	
383	C	STROKE FOR VACCUUM HAMMER			1	76	003750	4	
384		STROKE = 5.			70	76	003757	0	
385		IF (PBAR(20).GT.10.E-8) STROKE=3.0			1	76	003764	4	
386		DCYL =0.0			58	76	003767	4	
387		DEFFALL = STROKE-DEPIB			58	76	003776	4	
388	106	CONTINUE			1	76	004001	2	
389	C				1	76	004006	0	
390	C	DETERMINE RAM VELOCITY AT PORTS			70	76	004011	2	
391	C				70	76	004012	3	
392		IF (DEFFALL.GT.0.0) VFALL=SQRT(64.34*DEFFALL)			70	76	004021	2	
393		IF (PBAR(20).GT.10.E-8) CALL VACHAM(PBAR,ARAM,DEPIB,DCYL,STROKE,			59	76	004022	3	
394		I FSUCK,VFALL,IOUT,IW)			58	76	004033	0	
395	505	CONTINUE			58	76	004047	1	
396	C	SOIL STIFFNESS			50	76	004054	0	
397		DO 115 I=1,N1			70	76	004057	1	
398		SU(I)=SU(I)*RULT			15	76	004063	0	
399		SOK(I) = SU(I) / QS(I)			1	76	004066	2	
400	115	CONTINUE			1	76	004072	1	
401	C	CHECK TIME INCREMENT INCLUDING SOIL STIFFNESS			1	76	004077	0	
402		RRAT = RMAX/RULT			70	76	004102	2	
403		IF (CDP(99).GT.0.5) RRAT = 1			60	76	004113	3	
404		STCR = STP(N)+SOK(N)+SOK(N1)			61	76	004117	2	
405		DTCRP = PM(N)/(STCR*RRAT)			62	76	004125	2	
406		IF (CDP(99).GT.0.5) DTCRP=PM(N)/(STP(N)+SOK(N)+(RMAX-RULT)/QS(N1))			62	76	004133	2	
407		NM1=N-1			61	76	004140	4	
408		DO 1010 I=1,NM1			60	76	004155	1	
409		ST = STP(I+1)			60	76	004157	3	
410		IF (STP(I).GT.ST) ST=STP(I)			60	76	004163	1	
411		ST=ST+SOK(I)*RRAT			60	76	004166	3	
412		DTM=PM(I)/ST			60	76	004174	1	
413		IF (DTM.LT.DTCRP) DTCRP=DTM			60	76	004200	1	
414	1010	CONTINUE			60	76	004203	2	
415		DTCRP=SQRT(DTCRP)			60	76	004211	0	
416	C	CUSHION DAMPER			60	76	004214	2	
417		PM(99) = PM(99)*SQRT(STH(M)*HM(M))/50.0			70	76	004220	2	
					69	76	004224	1	

418	C	DETERMINE TIME INCREMENTS FOR HAMMER AND PILE GTR CRITICAL	1	76	004234	0
419		DTP=PFAC*DTCRP	1	76	004247	3
420		DTH=PFAC*DTCRH	1	76	004253	0
421	C	CHECK TIME INCREMENTS	70	76	004256	3
422		IF (DTH.LT.10.E-6.OR.DTP.LT.10.E-6) WRITE (IW,119)	55	76	004263	3
423		IF (DTH.LT.10.E-6.OR.DTP.LT.10.E-6) STOP	55	76	004275	1
424	119	FORMAT (47H HAMMER OR PILE T.I. EQUAL TO ZERO; CHECK INPUT)	55	76	004305	1
425		IF (DTCRA.LT.DTCRH) DTH=PFAC*DTCRA	1	76	004321	2
426		IF (DTH.GT.DTP) GO TO 33	41	76	004330	2
427		DO 118 JRAT=1,20	41	76	004335	3
428		IF (FLOAT(JRAT+1)*DTH.GT.DTP) GO TO 117	41	76	004341	2
429	118	CONTINUE	41	76	004351	1
430	C		70	76	004354	2
431	C	DETERMINE TIME INCREMENTS IF HAMMER INCREMENTS SMALLER THAN PILE	70	76	004355	3
432	C	T.I.	70	76	004372	0
433	C		70	76	004374	1
434	117	IF (MAXJRT.LT.1) MAXJRT=1	41	76	004375	2
435		IF (JRAT.GT.MAXJRT) JRAT=MAXJRT	41	76	004403	3
436		DTP=DTH*FLOAT(JRAT)	41	76	004412	0
437		GO TO 22	41	76	004416	2
438	33	DTH=DTP	41	76	004420	5
439		JRAT=1	41	76	004423	4
440	22	CONTINUE	41	76	004425	5
441	21	FORMAT (15X,15HTIME INCR. (MS),F5.3)	62	76	004430	5
442		DTPM=DTP*1000.	1	76	004440	5
443	C		70	76	004444	2
444	C	SET UP LOADING STIFFNESSES	70	76	004445	3
445	C		70	76	004453	2
446		IF (ITYPH.EQ.3) STH(M1)=STH(M1)*ECUS	54	76	004454	3
447		IF (ITYPH.LT.3) STH(M1)=STH(M1)*ECAP	54	76	004463	5
448		IF (MA.GT.0) STA(MA)=STA(MA)*ECAP	54	76	004473	1
449		STH(M2) = STH(M2)*ECUS	1	76	004502	0
450		STP(I) = STP(I)*EPT	1	76	004506	5
451		STH(M) = STH(M)*EANV	1	76	004513	1
452		WRITE (IW,21) DTPM	62	76	004517	4
453		STROKE = STROKE	1	76	004523	5
454	C	SET UP OUTPUT VARIABLES	70	76	004527	3
455		IF (IJJ.EQ.1) READ (IR,1,END=1005,ERR=1020) (INP(I),I=1,13)	70	76	004534	5
456		GO TO 1006	36	76	004550	1
457	1005	IJJ=0	36	76	004553	0
458	1006	CONTINUE	36	76	004555	5
459		IF (IJJ.NE.1) CALL JJNP (ICOL,N,INP,JMAX)	11	76	004561	1
460	C	... JMAX=NO. OF PILE VARIABLES THAT CAN BE PRINTED (JMAX.LE.N)	11	76	004571	2
461	C	NOT USED HERE; RECOMPUTED IN ANALYSIS	11	76	004605	3
462		RETURN	1	76	004615	2
463	1020	CONTINUE	70	76	004617	3
464	1021	FORMAT (11H DATA ERROR)	70	76	004622	5
465		WRITE (IW,1021)	70	76	004630	5
466		STOP	70	76	004634	3
467		END	1	76	004636	2

.9760 CRU

END OF RUN: 2 CRU ACCOUNT: 134600,8 CARDS: 4 IN 0 OUT PAGES: 10 CORE USAGE: 25.0K MAX. 20.8K AVERAGE

PIEL		SYMBOLIC FORTRAN 07 JUN 76 12:31:33 20 24 5 00132516					522			
1		SUBROUTINE PIEL (MP,N,XP,AP,EP,RP,STP,PM,ALPH,IOUT,IW,ICOL,					16	24	000004	0
2		1 IPEL,AREA)					14	24	000017	2
3		DIMENSION XP(20),AP(20),EP(20),RP(20),STP(99),PM(99),ALPH(99)					1	24	000022	3
4		1,AREA(99)					14	24	000036	1
5		SALPH=0.0					1	24	000046	2
6		DO 101 I1=1,N					1	24	000051	0
7		101 SALPH=SALPH+ALPH(I1)					1	24	000054	2
8	C						18	24	000061	3
9	C	ELEMENT COUNTER I2, CROSS SECTION INDEX I3					18	24	000062	4
10	C						18	24	000073	2
11		IF (IPEL.EQ.2) GO TO 108					24	24	000074	3
12		I2=1					1	24	000101	4
13		I3=2					1	24	000103	3
14		IJ=I3-1					1	24	000105	2
15		XT=0.0					1	24	000107	4
16		XB=(XP(MP)*ALPH(I2))/SALPH					1	24	000111	5
17		DX = XB					22	24	000117	2
18		AT=AP(1)					1	24	000121	4
19		RPT=RP(1)					7	24	000124	1
20		EPT=EP(1)					7	24	000126	5
21		GO TO 110					1	24	000131	3
22	C						18	24	000134	1
23	C	REPEAT 102 LOOP FOR EVERY SEGMENT I2					18	24	000135	2
24	C						18	24	000145	0
25		102 XT=XB					1	24	000146	1
26		DX = (XP(MP)*ALPH(I2))/SALPH					18	24	000150	5
27		XB = XT + DX					19	24	000156	5
28		110 STP(I2)=10.E12					21	24	000166	3
29		PM(I2)=0.0					1	24	000172	5
30		AREA(I2)=0.0					14	24	000175	4
31	C						18	24	000200	5
32	C	REPEAT 103 LOOP UNTIL NEXT CROSS SECTION DEPTH IS BELOW SEGMENT					18	24	000202	0
33	C						18	24	000216	2
34		103 IF (XB.LT.XP(I3)) GO TO 107					1	24	000217	3
35		X1=XP(I3)-XT					1	24	000226	0
36		XT=XP(I3)					1	24	000231	1
37		AAO=(AT+AP(I3))/2.					15	24	000233	5
38		AT=AP(I3+1)					1	24	000240	0

39		RPO=(RPT+RP(I3))/2.	7	24	000243 0
40		RPT=RP(I3+1)	7	24	000247 2
41		EPO=(EPT+EP(I3))/2.	7	24	000252 3
42		EPT=EP(I3+1)	7	24	000256 5
43		IF (X1.LT.10.E-8) GO TO 105	18	24	000262 0
44		AREA(I2) = AREA(I2)+ AAO*X1	18	24	000267 4
45		SUMM=RPO*X1*AAO/(144000.0*32,17)	7	24	000275 2
46		SUMK=AAO*EPO/X1	7	24	000304 0
47		PM(I2)=SUMM+PM(I2)	1	24	000307 4
48		STP(I2)=(SUMK*STP(I2))/(SUMK+STP(I2))	1	24	000313 5
49	105	I3=I3+1	10	24	000323 2
50		IF (I3.GT.MP) GO TO 118	10	24	000326 2
51		IJ=I3-1	10	24	000333 2
52		GO TO 103	1	24	000335 4
53	107	X1=XB-XT	1	24	000340 2
54	C		18	24	000343 3
55	C	FOR BOTTOM PORTION OF SEGMENT	18	24	000344 4
56	C		18	24	000353 1
57		XX = XP(I3) - XP(IJ)	18	24	000354 2
58		XF = 1.0	18	24	000360 5
59		IF (XX.GT.0.0) XF = (XB-XP(IJ))/XX	18	24	000363 2
60		AAU =AP(IJ)+(AP(I3)-AP(IJ))*XF	18	24	000372 2
61		AAO=(AT+AAU)/2.	15	24	000400 4
62		AT=AAU	1	24	000404 2
63		RPU = RP(IJ)+(RP(I3)-RP(IJ))*XF	18	24	000406 3
64		RPO=(RPT+RPU)/2.	8	24	000415 0
65		RPT=RPU	8	24	000420 5
66		EPU = EP(IJ)+(EP(I3)-EP(IJ))*XF	18	24	000423 1
67		EPO=(EPU+EPT)/2.	8	24	000431 4
68		EPT=EPU	8	24	000435 3
69		IF (X1.LT.10.E-8) GO TO 118	18	24	000437 5
70		AREA(I2) = AREA(I2)+AAO*X1	18	24	000445 3
71		SUMM=RPO*X1*AAO/(144000.0*32,17)	8	24	000453 0
72		SUMK=AAO*EPO/X1	8	24	000461 4
73		STP(I2)=(SUMK*STP(I2))/(SUMK+STP(I2))	1	24	000465 2
74		PM(I2)=SUMM+PM(I2)	1	24	000474 5
75	118	AREA(I2) = AREA(I2)/DX	20	24	000501 0
76		IF (I2.EQ.N) GO TO 108	20	24	000506 4
77		I2=I2+1	1	24	000513 3
78	C		18	24	000515 5
79	C	NEXT SEGMENT	18	24	000517 0
80	C		18	24	000522 3
81		GO TO 102	1	24	000523 4
82	108	CONTINUE	16	24	000526 2
83	C		18	24	000531 4
84	C	PRINT GIVEN INFORMATION	18	24	000532 5
85	C		18	24	000540 1
86	111	FORMAT (20X,16HPILE DESCRIPTION/)	7	24	000541 2
87	112	FORMAT (16H X BEL. TOP (FT),13F8.1)	1	24	000550 5
88	113	FORMAT (16H A (SQ. IN.) ,13F8.1)	1	24	000560 4
89	114	FORMAT (16H E (KSI) ,13F8.0)	1	24	000570 0
90	115	FORMAT (17H GAMMA (LB/CU FT) ,F7.1,12F8.1)	7	24	000576 4
91		WRITE (1W,111)	1	24	000607 4
92		MAX=6	1	24	000613 1

93		IF (ICOL.EQ.132) MAX=13	1	24	000615	1
94		I4=1	6	24	000622	1
95		I5=MAX	1	24	000624	0
96		IF (I5.GT.MP) I5=MP	6	24	000626	1
97	116	WRITE (IW,112) (XP(I),I=I4,I5)	1	24	000632	3
98		WRITE (IW,113) (AP(I),I=I4,I5)	1	24	000641	3
99		WRITE (IW,114) (EP(I),I=I4,I5)	1	24	000647	5
100		WRITE (IW,115) (RP(I),I=I4,I5)	1	24	000656	1
101		IF (I5.GE.MP) GO TO 117	6	24	000664	3
102		I4=I5+1	6	24	000671	3
103		I5=I4+MAX	6	24	000673	5
104		IF (I5.GI.MP) I5=MP	6	24	000676	3
105		GO TO 116	1	24	000702	5
106	117	CONTINUE	16	24	000705	3
107		IF (IPEL.NE.2) RETURN	15	24	000710	5
108	C		18	24	000715	3
109	C	DETERMINE AVERAGE AREAS FOR THE CASE WHERE MASSES AND	18	24	000716	4
110	C	STIFFNESSES WERE GIVEN	18	24	000731	1
111	C		18	24	000736	2
112		DL = 0.0	15	24	000737	3
113		DO 145 I=1,N	15	24	000742	0
114		DO 146 J=2,MP	23	24	000750	3
115		IF (XP(J).GT.DL) GO TO 147	15	24	000753	5
116	146	CONTINUE	15	24	000761	2
117	147	AREA(I)=AP(J-1)	15	24	000764	4
118	145	DL = DL + ALPH(I)*XP(MP)/SALPH	24	24	000776	3
119		RETURN	4	24	001005	4
120		END	1	24	001007	5
1 SUBROUTINE DIESEL						
2		COMMON VINH,DINH,AOH,VOH,DOH,ANH,VNH,DNH,STH,HM,AM,STA,MA,ESOIL,	1	37	000010	0
3		1 AOP,ANP,DOP,DNP,VOP,VNP,STP,PM,COP,RES,RESO,SJ,SOK,QS,SU,	1	37	000024	1
4		2 SPLICE,DEPIB,ARAM,ECUS,ECAP,EPT,EANV,VFALL,VCHAM,XPT,SJJJ,	1	37	000037	3
5		3 EMP,RHO,DTH,DTP,N,M,JRAT,IULT,RULT,SOILM,DSACP,	1	37	000053	0
6		4 PPTO,IW,PCOM,TOEL,IOUT,OUT,EXPP,TEMI,TEMO,INP,STROKO,STROKN,VO,PO	1	37	000064	3
7		5 ,DFIN,JDOUT,JPMAX,EX,ICOL,FINTMX,TEMAX,RESULT,ISMITH,ITER	1	37	000101	1
8		6 ,ITYPH,IPREIG,VFALLM,STRMAX,DCYL,PBAR,AREA,JEX	25	37	000114	4
9		DIMENSION VINH(10),DINH(10),AOH(10),VOH(10),DOH(10),ANH(10),	1	37	000126	0
10		1 VNH(10),DNH(10),STH(10),HM(10),VNP(99),	1	37	000141	3
11		2 AOP(99),ANP(99),DOP(99),DNP(99),VOP(99),STP(99),PM(99),COP(99),	1	37	000151	5
12		3 RES(99),RESO(99),SJ(99),SOK(99),QS(99),SU(99),SPLICE(99),	1	37	000166	1
13		4 DSACP(3),OUT(3200),INP(13),EX(600),RESULT(100),JEX(600)	25	37	000201	3
14		5 ,AOAH(99),VOAH(99),DOAH(99),AREA(99),DELAH(99),STA(10),AM(10)	1	37	000214	2
15		6 ,PBAR(20)	4	37	000230	2
16		KLIM = 200	1	37	000233	2
17	C	KLIM ... MAX NO. OF VALUES OF ONE VARIABLE IN OUTPUT,	1	37	000236	1
18	C	KLIM*15=DIM(OUT)	1	37	000250	4
19		EXPT = 1.-1./EXPP	1	37	000254	5
20		CDH = PM(99)	30	37	000260	5
21	C		35	37	000264	0
22	C	SET UP OUTPUT VARIABLES	35	37	000265	1
23	C		35	37	000272	3
24		IIOUT = IABS(IOUT)	36	37	000277	2
25		IF (IIOUT.GE.20) IIOUT=IIOUT-20	10	37	000303	3

26	IF (IIOUT.GE.10) IIOUT=IIOUT-10	10	37	000312	0
27	SCJ = 0.0	34	37	000320	3
28	IAI = 0.0	34	37	000323	1
29	JQOUT=0	10	37	000325	5
30	JDOUT = 6.0*XPT/(SQRT(EMP/RHO)*DTP)	1	37	000330	0
31	JQOUT = JDOUT / 100	1	37	000337	1
32	TJD = FLOAT(JDOUT)*DTP*1000.	1	37	000343	3
33	IF (TJD.GI.0.5) JDOUT = (0.0005/DTP) + 0.5	1	37	000351	3
34	IF (JDOUT.LT.1) JDOUT=1	1	37	000361	5
35	IF (IIOUT.EQ.6) JDOUT = JDOUT + 1	5	37	000366	5
36	C JMAX ... NUMBER OF OUTPUT TIME TO ANALYSIS PILE TIME INCREMENT	1	37	000375	4
37	JMAX = 6	1	37	000411	5
38	IF (ICOL.GT.0) JMAX=13	1	37	000414	2
39	IF (JMAX.GT.N) JMAX=N	1	37	000421	1
40	C JMAX ... NUMBER OF PILE VARIABLES PRINTED (DEPENDENT ON FORM	1	37	000425	5
41	C SIZE)	1	37	000441	4
42	C	35	37	000444	0
43	C INITIALIZE	35	37	000445	1
44	C	35	37	000450	2
45	FINIMX=0.5	7	37	000451	3
46	TS = 0.0005	9	37	000454	2
47	IF (PBAR(9).GT.0.0) TS = SJJJ	33	37	000457	2
48	IF (PBAR(9).GT.0.0.AND.TS.LT.10.E-8) TS = 0.01	33	37	000465	3
49	RED=0.90	8	37	000476	3
50	IF (EMP.LT.3000000.) RED = 0.0	14	37	000501	0
51	KK=0	1	37	000507	2
52	N1=N+1	1	37	000511	1
53	NM1=N-1	1	37	000513	2
54	FPT0 = FPT0	1	37	000515	4
55	FTI1 = FPT0	1	37	000520	4
56	FTPOA =FTPO	1	37	000523	4
57	TLIM = 0.0	31	37	000526	4
58	FANVOA = 0.0	1	37	000531	3
59	FCUOA = VO	20	37	000534	4
60	VNPIT = VOP(N)	1	37	000537	3
61	VNHIT = VINH(M)	1	37	000543	0
62	VO = VCHAM + (DINH(M+1)-DINH(M))*ARAM	1	37	000546	4
63	TIMP = 0.0	1	37	000556	1
64	TI=0.0	7	37	000561	0
65	TI=0.0	7	37	000563	1
66	TBLOW = -1.0	9	37	000565	2
67	RIMP = 1.	1	37	000570	3
68	RIDEL = 1.	1	37	000573	1
69	PCOMR = PCOM	1	37	000576	0
70	IF (PBAR(19).GT.0.0) GO TO 801	22	37	000601	1
71	PCOMF = 0.5*(STROKO-DEPIB)/DEPIB	1	37	000607	3
72	C REDUCTION DUE TO POOR SCAVENGING	1	37	000616	1
73	IF (PCOMF.GT.1.0) PCOMF = 1.0	1	37	000625	1
74	IF (PCOMF.LT.0.5) PCOMF = 0.5	5	37	000633	2
75	PCOM=1.0+(PCOM-1.0)*PCOMF	22	37	000641	3
76	801 CONTINUE	22	37	000646	5
77	PCOM1=PCOM	17	37	000652	0
78	IFIRST=0	1	37	000654	5
79	IREM = 0	1	37	000657	2

80		V1PMAX = 0.0	9	37	000661	5
81		DNPMAX = 0.0	1	37	000665	0
82		MM1=M-1	1	37	000670	1
83		M1=M+1	1	37	000672	3
84		M2=M+2	1	37	000674	4
85		G=32.17	1	37	000675	5
86		ENTHRU = 0.0	1	37	000701	1
87		EMAX = 0.0	1	37	000704	2
88		T2LC = 2000.*XPT*SQRT(RHO/EMP)	1	37	000707	1
89	C	LIMIT OF MAIN LOOP	35	37	000740	0
90		NT = 0.005*T2LC/DTP	1	37	000744	3
91		JCK = NT	29	37	000750	5
92		IF (FLOAT(NT)*DTP.LT.0.050) NT = 0.050/DTP	35	37	000763	4
93		IF (TEMAX.GT.1.0) NT=TEMAX/(1000.0*DTP)	8	37	000774	0
94	C		35	37	001003	5
95	C	LOADING AND REBOUND STIFFNESSES FOR PILE TOP, ANVIL, AND CAPBLOCK	35	37	001005	0
96	C		35	37	001021	3
97		STPT1 = STH(M2)*STP(1)/(STH(M2)+STP(1))	1	37	001022	4
98		STPT2 = (STH(M2)/ECUS)*(STP(1)/EPT)/((STH(M2)/ECUS)+STP(1)/EPT)	1	37	001032	3
99		STAL = STH(M)	1	37	001046	3
100		STAR = STAL/EANV	1	37	001051	5
101		STCL = STH(M1)	1	37	001055	4
102		STCR = STCL/ECAP	1	37	001061	1
103	C	HAMMER VARIABLES TO INITIAL VALUES	35	37	001065	0
104		DO 100 I=1,M2	1	37	001074	2
105		VOH(I)=VINH(I)	18	37	001077	4
106		DOH(I)=DINH(I)	18	37	001103	1
107		VOAH(I)=VINH(I)	1	37	001106	4
108		DOAH(I)=DINH(I)	1	37	001112	2
109		DELAH(I) = 0.0	1	37	001116	0
110		AOAH(I)=G	1	37	001121	3
111	100	CONTINUE	1	37	001124	1
112	C	CONVERGENCE CRITERIA	35	37	001127	3
113		EPSV = 0.02	1	37	001134	2
114		EPSF = 1.0	1	37	001137	2
115		DO 97 I=1,N1	1	37	001142	1
116	97	AOP(I) = G	1	37	001145	2
117		DO 450 I=1,3200	26	37	001150	5
118	450	OUT(I)=0.0	1	37	001154	3
119	C	EXTREMUM ARRAY	35	37	001160	0
120		DO 460 J=1,6	1	37	001163	5
121		DO 460 I=1,N	1	37	001167	0
122		IJ=N*(J-1)+I	1	37	001172	1
123		EX(IJ) = 0.0	1	37	001175	2
124		JEX(IJ) = 0	31	37	001200	3
125	460	CONTINUE	29	37	001203	3
126		ACYL = G	19	37	001206	5
127		PBR = 0.144*14.7*PBAR(1)	19	37	001211	2
128		IF (PBAR(1).LT.10.E-8) PBR = 2.117*ARAM	27	37	001216	3
129		DELP = 0.0	27	37	001226	2
130		IF (PBAR(1).GT.ARAM) DELP = 2.117*(PBAR(1)-ARAM)	27	37	001231	1
131		IF (PBAR(20).GT.0.0) PBR = 0.0	26	37	001242	3
132	C	CYLINDER VARIABLES	35	37	001250	5
133		CYLM=PBAR(8)/G	23	37	001255	2

134		DCYL = DOH(M1)	19	37	001260	5
135		VCYL = VOH(M1)	19	37	001264	2
136	C	IGNITION VARIABLES	35	37	001267	5
137		TSLOW = 1.0	16	37	001274	2
138		TSA=1.0/PCOM	17	37	001277	2
139		TSB=(1.0-TSA)/TS	17	37	001302	3
140	C	CONTACT AREA VARIABLES	35	37	001306	2
141		AFA=(3.-FINTMX)/2.	7	37	001313	3
142		AFB=(1.-FINTMX)/(2.*DSACP(1))	7	37	001317	4
143		EXP = 1.35	5	37	001325	5
144		IF (IOUT.LT.0) WRITE (IW,653)	1	37	001330	4
145	C		29	37	001336	5
146	C	START MAIN LOOP	29	37	001340	0
147	C		29	37	001344	0
148		DO 700 J=1,NT	1	37	001345	1
149		IF (J.GT.JCK) JCK=-1	26	37	001350	3
150		IF (VIPMAX.LT.VNP(1)) VIPMAX = VNP(1)	9	37	001355	0
151		IF (RED.LT.0.01) GO TO 103	9	37	001364	3
152		IF (VNP(1).GT.0.9*VIPMAX.OR.VIPMAX.LT.5.0) GO TO 103	17	37	001372	0
153		STPT1 = STPT2 * RED	14	37	001404	0
154		RED = 0.0	9	37	001410	2
155	103	CONTINUE	8	37	001413	0
156		DPOS = DOH(M)-DCYL	20	37	001416	1
157	C	FORCE ON TOP OF RAM FOR REGULAR CLOSED END OR VAC HAMMER	35	37	001422	2
158		PTR = FTR(DPOS,PBAR,IWT)	19	37	001435	3
159		DBC = -DOH(M)	26	37	001442	4
160		IF (PBAR(20).GT.0.0) CALL VACHAM(PBAR,ARAM,DEPIB,DCYL,DBC,PTR,	26	37	001446	0
161	1	VFALL,IOUT,IW)	26	37	001461	5
162		IF (PBAR(8).GT.0.0) ACYL=G+(PBR-PTR)/CYLM	23	37	001465	4
163		IF (PBAR(20).GT.0.0) PTR=PTR+ARAM*2.117	27	37	001475	5
164		IF (PBAR(20).LT.10.E-8) PTR = PTR-DELP	27	37	001505	4
165		DCYL = DCYL + VCYL * DTP	19	37	001515	2
166		VCYL = VCYL + ACYL * DTP	19	37	001522	3
167		IF (DCYL.GT.DOH(M1)) VCYL = VOH(M1)	19	37	001527	4
168		IF (DCYL.GT.DOH(M1)) DCYL = DOH(M1)	19	37	001536	5
169		TI=TI+DTP*1000.0	7	37	001546	0
170	C	ABSOLUTE TIME COUNTER (MILLISEC.)	35	37	001551	5
171	C	TIME AFTER IMPACT (TIMP)	35	37	001561	0
172	C	RIMP = 0.0 MEANS FIRST IMPACT HASS OCCURED	35	37	001566	3
173		IF (RIMP.GT.0.5) TIMP=(DOH(M)-DOH(M1))/VOH(M)	17	37	001577	1
174		IF (RIMP.LT.0.5) TIMP=TIMP+DTP	17	37	001610	0
175		TI=TI+DTP*(1.0-PCOM/PCOM1)	17	37	001616	2
176	C	IGNITION START FOR ATOMIZED FUEL	32	37	001623	5
177		IF (PBAR(9).GT.0.0.AND.RIMP.LT.0.5) RIDEL = 0.0	32	37	001632	5
178		IF (TIMP.GE.TDEL.AND.IPREIG.EQ.0) RIDEL = 0.0	16	37	001644	0
179		IF (J.GT.0.AND.J.EQ.IPREIG) RIDEL = 0.0	16	37	001654	5
180		IF (RIDEL.GT.0.5.OR.PCOM.LT.0.01) GO TO 470	7	37	001664	4
181		TEMO=TEMO*PCOM	7	37	001675	1
182		PO=PO*PCOM	7	37	001700	4
183		EXP = EXPP	5	37	001703	3
184		PCOM = 0.0	5	37	001706	2
185	470	CONTINUE	5	37	001711	1
186		IF (TT.LT.50.) GO TO 471	9	37	001714	3
187		IF (DDC.LT.12.*DEPIB) GO TO 471	9	37	001721	4

188	C	FOR RAM REACHING PORTS			
189		IF (TBLOW.LT.0.) PBLOW = (PN-2.12)*100.*GTP	35	37	001730 1
190		TBLOW = 1.0	9	37	001735 2
191		PN = PN - PBLOW	9	37	001745 5
192		IF (PN.LT.2.117) PN = 2.117	9	37	001750 5
193		PO = PN	9	37	001754 3
194	471	CONTINUE	30	37	001762 1
195		II=0	9	37	001764 3
196	C		1	37	001767 5
197	C	OVERALL LOOP STARTS	35	37	001771 4
198	C		35	37	001772 5
199	800	CONTINUE	35	37	001777 3
200		FCUO = FCUOA	1	37	002000 4
201		FTPO = FTPOA	1	37	002004 0
202		FANVO = FANVOA	1	37	002007 1
203		DO 99 I=1,M2	1	37	002012 2
204		VOH(I)=VOAH(I)	1	37	002015 5
205		AOH(I)=AOAH(I)	1	37	002021 0
206	99	DOH(I)=DOAH(I)	1	37	002024 3
207	C		1	37	002030 0
208	C	HAMMER ANALYSIS	1	37	002034 1
209	C		1	37	002035 2
210		DO 600 JH=1,JRAT	1	37	002041 2
211		IIH=0	1	37	002042 3
212	C	PREDICTION	1	37	002046 2
213		DO 585 I=1,M2	1	37	002050 2
214		IF (IT.EQ.0) ANH(I)=AOH(I)	1	37	002053 3
215		IF (II.GT.0) ANH(I)=AOH(I)+FLOAT(JH)*DELAH(I)	1	37	002056 5
216	585	CONTINUE	1	37	002064 2
217		DO 101 I=1,M2	1	37	002075 1
218	101	CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	1	37	002100 3
219	C		1	37	002103 5
220	C	HAMMER CYCLE STARTS	35	37	002115 0
221	C		35	37	002116 1
222	500	CONTINUE	35	37	002122 5
223		I=1	1	37	002124 0
224		ANH(I)=G+((DNH(2)-DNH(1))*STH(1)+PTR)/HM(1)	1	37	002127 2
225		CALL INTEGR (DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	21	37	002131 0
226		DO 102 I=2,MM1	1	37	002141 3
227		I1=I+1	1	37	002152 0
228		IM1=I-1	1	37	002155 3
229		ANH(I)=G+((DNH(I1)-DNH(I))*STH(I)+(DNH(IM1)-DNH(I))*STH(IM1))/HM(I	1	37	002157 4
230	1)		1	37	002162 0
231		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	1	37	002176 3
232	102	CONTINUE	1	37	002200 0
233	C	NOW CHECK DISPLACEMENT OF RAM AND IMPACT BLOCK	1	37	002210 2
234		V = VCHAM + ARAM*(DNH(M1)-DNH(M))	1	37	002213 4
235		IF (V.LT.VCHAM) V = VCHAM	1	37	002225 0
236		PN = PO*(VO/V)**EXP	1	37	002233 5
237		TEMN = TEMO*(PN/PO)**EXPT	5	37	002241 1
238	C	LAST RAM ELEMENT	1	37	002245 3
239		DDIS = DNH(M1)-DNH(M)	1	37	002252 5
240		IF (PCOM.LT.0.1) TSLOW=TSB+TSB*TI	1	37	002257 0
241		IF (TSLOW.GT.1.0) TSLOW = 1.0	17	37	002263 4
			5	37	002272 3

242		PE = PN*TSLOW	5	37	002300	4
243		AFAC = 1.0	5	37	002304	0
244		IF (DDIS.LT.-DSACP(1)) AFAC=AFA+AFB*DDIS	7	37	002306	5
245		IF (DDIS.LT.-3.*DSACP(1)) AFAC=FINTMX	7	37	002316	5
246		FP = PE*ARAM*AFAC	5	37	002326	2
247		FANV = 0.0	1	37	002332	2
248		IF (DDIS.GT.0.0) GO TO 811	1	37	002335	1
249		DVEL = VNH(M1) - VNH(M)	1	37	002342	4
250		OSAC = DSACP(1)	1	37	002347	4
251		DDISO = DOH(M1)-DOH(M)	1	37	002353	2
252		DDD = DDIS - DDISO	1	37	002360	1
253	C	ANVIL SPRING FORCE ALWAYS TENSION= +	1	37	002364	2
254		CALL STIFF (DDIS,DVEL,STAL,STAR,FANVO,FANV,DSAC,DDD)	1	37	002374	0
255		IF (FANV.GT.0.0) FANV = 0.0	1	37	002406	0
256	811	FINTAC = -FP+FANV	1	37	002413	4
257		ODC=DDIS*12.0	1	37	002420	3
258		IF (DDIS.GT.0.0) IREM = 0	1	37	002423	5
259		IF (DDIS.LT.0.0) RIMP = 0.0	1	37	002431	1
260		IF (DDIS.LT.0.0.AND.IREM.NE.1) IFIRST = 1	1	37	002436	5
261	C	IFIRST = 1 MEANS IMPACT HAS OCCURED THE FIRST TIME	35	37	002447	0
262		IF (IFIRST.NE.1) GO TO 675	1	37	002461	0
263		TIMP=0.0	29	37	002466	3
264		JCK=0.0015*T2LC/DTP	29	37	002471	0
265		TTI = TT	1	37	002475	2
266		IREM = 1	1	37	002477	5
267		TLIM = TT + 1.15*T2LC	1	37	002502	2
268		IF (TLIM.LT.10.) TLIM = 10.	1	37	002507	0
269	675	CONTINUE	1	37	002514	4
270		ANH(M) = G + ((DNH(MM1)-DNH(M))*STH(MM1)+FINTAC)/HM(M)	1	37	002520	0
271		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,M)	1	37	002532	2
272	C	ANVIL	1	37	002542	4
273		FINTAC = -FINTAC	1	37	002546	0
274		DHA = 0.0	30	37	002550	5
275		ODIS = DNH(M2)-DNH(M1)	1	37	002553	3
276		FCU = 0.0	1	37	002560	2
277		IF (DDIS.GT.0.0) GO TO 812	1	37	002563	0
278		DVEL = VNH(M2)-VNH(M1)	1	37	002570	3
279		DHA = DVEL*CDH	30	37	002575	2
280		DSAC = DSACP(2)	1	37	002600	5
281		DDISO = DOH(M2)-DOH(M1)	1	37	002604	3
282		DDD = DDIS - DDISO	1	37	002611	3
283		FCUO = -FCU	1	37	002615	4
284		CALL STIFF (DDIS,DVEL,STCL,STCR,FCUO,FCU,DSAC,DDD)	1	37	002620	5
285		IF (FCU.GT.0.0) FCU=0.0	1	37	002632	3
286		FCUO = -FCU	1	37	002637	3
287	812	CONTINUE	1	37	002642	4
288		ANH(M1) = G+(FINTAC+FCU+DHA)/HM(M1)	30	37	002646	0
289		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,M1)	1	37	002655	1
290	C	CAP	1	37	002665	4
291		DT1 = FLOAT(JH)*DTH	1	37	002667	4
292		I = 1	1	37	002674	0
293		DNP1 = DOP(1)+(DT1/DTP)*(DNP(1)-DOP(1))	1	37	002676	0
294		VNP1 = VOP(1) + (DT1/DTP)*(VNP(1)-VOP(1))	1	37	002705	5
295		IF (IT.GT.0) GO TO 575	1	37	002716	0

296		CALL INTEGR(DT1,AOP,AOP,VOP,DOP,VNP,DNP,I)	1	37	002722	5
297		DNP1 = DNP(1)	1	37	002733	1
298		VNP1 = VNP(1)	1	37	002736	3
299	575	CONTINUE	1	37	002741	5
300		DDIS = DNP1-DNH(M2)	1	37	002745	1
301		VCAP = 0.0	30	37	002751	3
302		PTD = 0.0	30	37	002754	2
303		OVEL = 0.0	30	37	002757	0
304		FTP = 0.0	1	37	002761	5
305		IF (DDIS.GT.0.0) GO TO 813	1	37	002764	3
306		OVEL = VNP1-VNH(M2)	1	37	002772	0
307		VCAP = VNH(M2)	30	37	002776	2
308		DSAC = DSACP(3)	1	37	003001	5
309		DDISO = DDP(1)-DOH(M2)	1	37	003005	3
310		DDD = DDIS-DDISO	1	37	003012	2
311		FTPO = -FTPO	1	37	003016	1
312		CALL STIFF (DDIS,OVEL,STPT1,STPT2,FTPO,FTP,DSAC,DDD)	1	37	003021	2
313		IF (FTP.GT.-SPLICE(1)) FTP = -SPLICE(1)	1	37	003033	2
314		FTPO = -FTPO	1	37	003043	1
315	813	FCU = -FCU	1	37	003046	2
316		ANH(M2) = G+(FCU+FTP+OVEL*CDP(1)-DHA)/HM(M2)	30	37	003052	0
317		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,M2)	1	37	003062	4
318		ITH = ITH +1	1	37	003073	1
319		FTP = -FTP	1	37	003076	2
320	C		35	37	003101	1
321	C	IF DTH EQ DTP (JRAT=1) THE FOLLOWING IS SKIPPED (ONE TIME	35	37	003102	2
322	C	INCREMENT)	35	37	003115	4
323	C		35	37	003120	5
324		IF (JRAT.EQ.1) GO TO 650	1	37	003122	0
325		IF (ITH.GT.ITER) GO TO 650	1	37	003127	1
326		ERR = ABS(VNHIT-VNH(M))	1	37	003134	4
327		VNHIT = VNH(M)	1	37	003141	4
328		IF (ERR.GT.EPSV) GO TO 500	1	37	003145	1
329	650	CONTINUE	1	37	003152	4
330		DO 501 I=1,M2	1	37	003156	0
331		DELAH(I) = AOH(I)+(ANH(I)-AOH(I))/FLOAT(JRAT)	1	37	003161	2
332		AOH(I)=ANH(I)	1	37	003172	1
333		VOH(I)=VNH(I)	1	37	003175	3
334	501	DOH(I)=DNH(I)	1	37	003200	5
335		TEMO = TEMN	1	37	003205	0
336		PO = PN	1	37	003210	0
337		VO = V	1	37	003212	2
338		FANVO = FANV	1	37	003214	3
339		FTPO = FTP	1	37	003217	4
340		FCUO = FCU	1	37	003222	3
341	600	CONTINUE	1	37	003231	5
342	C		35	37	003236	2
343	C	PILE ANALYSIS	35	37	003237	3
344	C		35	37	003243	1
345		CALL PILEAN (IPT,AOP,VOP,DOP,ANP,VNP,DNP,STP,PM,SPLICE,SJ,CDP,RES,	1	37	003246	2
346		1 RESO,SU,SOK,VNPIT,VCAP,SOILM,SCJ,DTP,N,IT,ISMITH,EPVS,IAI,ESOIL,	34	37	003277	3
347		2 FTP,JRAT,ITER)	1	37	003314	0
348	C		35	37	003317	5
349	C	CHECK FOR CONVERGENCE	35	37	003321	0

350	C			35	37	003326	0
351		IT = IT + 1		1	37	003327	1
352		IF (IT.GT.ITER) GO TO 850		1	37	003332	1
353		ERRV = ABS(VNPIT-VNP(N))		1	37	003337	3
354		IF (JRAT.EQ.1) VNPIT = VNP(N)		1	37	003344	4
355		IF (JRAT.EQ.1.AND.ERRV.GT.EPSV) GO TO 800		1	37	003352	5
356		ERR = ABS(FTIT-FTP)		1	37	003363	0
357		FIIT = FTP		1	37	003367	2
358		IF (ERR.GT.EPSF) GO TO 800		1	37	003372	1
359	850	CONTINUE		1	37	003377	4
360	C	PREPARE OUTPUT FOR CURRENT		35	37	003403	0
361	C	TIME INCREMENT AND		35	37	003410	5
362	C	SET UP STARTING VALUES		35	37	003415	2
363	C	FOR NEXT STEP		35	37	003422	3
364		RSUM = 0.0		1	37	003426	1
365		DSUM=SJ(N1)*VNP(N)		1	37	003431	0
366		IF (ISMITH.EQ.1) DSUM=RES(N1)*DSUM		1	37	003435	1
367		IF (DSUM.LT.0.0) DSUM = 0.0		1	37	003444	1
368		VMAXP = 0.0		35	37	003451	5
369		DO 401 I=1,N1		1	37	003454	5
370		RESO(I) = RES(I)		1	37	003460	1
371		AOP(I) = ANP(I)		1	37	003464	0
372		VOP(I)=VNP(I)		1	37	003467	4
373		RSUM = RSUM + RES(I)		1	37	003473	0
374		DDAM=SJ(I)*VNP(I)		1	37	003477	3
375		IF (ISMITH.EQ.1) DDAM=DDAM*RES(I)		1	37	003503	3
376		IF (I.NE.N1) DSUM = DSUM + DDAM		1	37	003512	2
377		IF (ABS(VOP(I)).GT.VMAXP) VMAXP = ABS(VOP(I))		35	37	003520	5
378	401	DOP(I)=DNP(I)		1	37	003531	4
379		IF (DNP(N).GT.DNPMAX) DNPMAX = DNP(N)		1	37	003535	5
380		FANVOA = FANVO		1	37	003545	2
381		FCUOA = FCUO		1	37	003550	5
382		FTPOA = FTPO		1	37	003554	0
383		DO 402 I=1,M2		1	37	003557	1
384		AOAH(I)=ANH(I)		1	37	003562	3
385		VOAH(I)=VNH(I)		1	37	003566	0
386	402	DOAH(I)=DNH(I)		1	37	003571	3
387		DOPINC = DOP(1)*12.		1	37	003575	5
388		DEN = VOP(1)*DTP*FTP		1	37	003602	1
389	C	TRANSFERRED ENERGY		35	37	003606	4
390		ENTHRU = ENTHRU + DEN		1	37	003613	1
391		IF (ENTHRU.GT.EMAX) EMAX = ENTHRU		1	37	003617	5
392		JOUT = JOUT + 1		1	37	003626	4
393		FTPO = FTP		37	37	003632	2
394		IF (JOUT.NE.JDOUT) GO TO 699		1	37	003635	1
395		JOUT = 0		1	37	003643	1
396		PT=PTR/PBAR(1)		22	37	003645	4
397		NQ2 = N/2		1	37	003653	4
398		NQ21 = NQ2+1		1	37	003656	2
399		PG = PE/0.144		5	37	003661	3
400	652	FORMAT (1H,14,4I2,F6.1,F8.3,2F6.1,2F8.1,2F6.1,2(F6.1,F8.1),3F8.1,		1	37	003664	5
401		1 F7.3,F6.0)		1	37	003701	5
402		DO 686 I=NQ2,NM1		31	37	003705	0
403		I1 = I+1		31	37	003710	5

404		COM = DNP(1)-DNP(I1)	31	37	003713	2
405		F = COM*STP(I1)	31	37	003717	5
406		IF (F.LT.SPLICE(I1)) F = SPLICE(I1)	31	37	003723	3
407		IF (F.GT.0.0.OR.SPLICE(I1).LT.5) GO TO 685	31	37	003732	4
408		F = 0.0	31	37	003743	2
409		IF (COM.LT.SPLICE(I1)) F = (COM-SPLICE(I1))*STP(I1)	31	37	003745	4
410	685	FOPN = F	31	37	003757	3
411		IE (I.EQ.NO2) EQP5 = F	31	37	003762	5
412	686	CONTINUE	31	37	003767	4
413	C	DEBUG_OUTPUT	35	37	003773	0
414	653	FORMAT	1	37	003776	3
415	1	(132H JP F T H P TIME DCHAM VRU VAN FINT EPT	1	37	004001	3
416	A	V1 EN V5 F5 VN FN CP RS DA	1	37	004013	4
417	B	D T)	1	37	004023	0
418		IF (IOUT.LT.0) WRITE (IW,652)	1	37	004025	5
419	A	J,IFIRST,IT,ITH,IPT,IT,DDC ,VNH(M),VNH(M1),FINTAC,FTP,VOP(1),	1	37	004034	0
420	1	ENTHRU,VOP(NO2),FOP5,VOP(N),FOPN,PQ,RSUM,DSUM,DOPIINC,PT	22	37	004050	1
421	IF	(IOUT.LT.6) CALL FILL(KK,KLIM,OUT,JMAX,IOUT,DNH,STH,	1	37	004067	3
422	1	FINTAC,FTP,INP,DNP,STP,SPLICE,VNH,VNP,ANH,ANP,JPMAX,MAXJP,EX,AREA,	25	37	004102	3
423	2	J,M,N,JEX,JCK)	26	37	004117	1
424	C	JPMAX IS THE LATEST J	1	37	004123	0
425	C	FILLS IN THE OUTPUT ARRAY OUT(KLIM*15)	1	37	004130	0
426	C	MAXJP IS THE MAXIMUM NUMBER OF VALUES STORED OF ONE VARIABLE	1	37	004140	0
427	C	MAXJP EQUALS APPROX. JPMAX/JOQUT	1	37	004153	5
428		IF (IOUT.EQ.6) CALL FILL2(N,M,OUT,PQ,DNH,FTP,VNP,DNP,FOP5,SJ,	1	37	004162	5
429	1	RSUM,DSUM,RES,FOPN,KK,ICOL,IT,ISMITH,J,MAXJP,JPMAX,STP,AREA,	1	37	004176	4
430	2	SPLICE,EX,JEX,J,JCK)	26	37	004212	2
431	C	CHECK WHETHER ANALYSIS WAS CARRIED OUT SUFFICIENTLY LONG	35	37	004217	1
432		IF (TEMX.GT.1.0) GO TO 699	31	37	004232	2
433		IF (VIPMAX*0.20.LT.VMAXP) GO TO 699	37	37	004247	1
434		IF (TT.LT.TLIM) GO TO 699	1	37	004256	2
435		IF (TI.LT.T2LC.OR.TI.LT.20.0) GO TO 699	37	37	004300	5
436		IF ((DNH(M1)-DNH(M)).GT.,1*DEPIB.AND.DNP(N).LT.DNPMAX*0.9) GO TO	5	37	004310	4
437	1	701	1	37	004324	5
438		IF ((DNH(M1)-DNH(M)).GT.,2*DEPIB.AND.DNP(N).LT.0.98*DNPMAX) GO TO	5	37	004326	5
439	1	701	5	37	004343	1
440	699	IFIRST = 0	1	37	004345	1
441	700	CONTINUE	1	37	004350	5
442	701	CONTINUE	1	37	004354	1
443		FINTMX = FLOAT(J)*DTP	28	37	004357	3
444		DFIN = (DNPMAX-QS(N))*12.	1	37	004364	1
445		VNH(10)=VCYL	19	37	004371	3
446		DNH(10) = ACYL	20	37	004374	4
447		IF (IOUT.LT.0) WRITE (IW,654) ENTHRU,EMAX	31	37	004400	1
448		ANP(99) = EMAX	31	37	004410	2
449	654	FORMAT (30H0TRANSFERRED ENERGY, FIN- MAX=,F6.1,2H -,F6.1,5H K-FT)	9	37	004413	5
450		PCOM = PCOMR	1	37	004431	0
451		TEMO = TEMN	1	37	004434	1
452		RETURN	1	37	004437	1
453		END	1	37	004441	2

1.1938 CRU

END OF RUN: 2 CRU ACCOUNT: 134600,8 CARDS: 4 IN 0 OUT PAGES: 12 CORE USAGE: 25.0K MAX. 21.3K AVERAGE

OUTIN	SYMBOLIC FORTRAN	30 APR 76	14:19:32	12 16	5	00155104	610			
1	SUBROUTINE OUTIN (SU,SJ,QS,STH,STP,HM,PM,IHAMR,EE,N,M,IW,XPT,							1	16	000004 0
2	IISMITH,HNAME,ESOIL,APO,EMP,RHO,SPLICE,CDP,ALPH,ITYPH)							13	16	000031 0
3	DIMENSION HNAME(2),SU(99),SJ(99),QS(99),STH(10),HM(10),EE(4),							6	16	000043 2
4	1STP(99),PM(99),CREE(4),SPLICE(99),CDP(99)							6	16	000057 0
5	2,ALPH(99)							11	16	000067 2
6	4 FORMAT (15X,14H HAMMER MODEL ,2A4)							1	16	000072 1
7	5 FORMAT(6X,41H ELEMENT WEIGHT STIFFNESS COEFF.)							1	16	000101 3
8	6 FORMAT(6X,43H NUMBER (KIPS) (K/IN) RESTITUTION)							1	16	000113 2
9	7 FORMAT(6X,9H ANVIL ,F10.3,F10.1,F10.3)							1	16	000125 2
10	8 FORMAT(6X,9H CAP ,F10.3,F10.1,F10.3)							1	16	000135 2
11	9 FORMAT (6X,9H CUSHION ,10X,F10.1,F10.3)							1	16	000145 1
12	10 FORMAT (6X,9HPILE TOP ,20X,F10.3)							1	16	000155 3
13	12 FORMAT (2X,51H WEIGHT STIFFN. PDAMP. SPLICE SOIL-S SOIL-D)							5	16	000165 0
14	19 FORMAT (1H+,52X,16H QUAKE L.8.T.)							8	16	000201 1
15	13 FORMAT (2X,43HNO. (KIPS) (K/IN) (KS/FT) (KIPS) (PCT.))							5	16	000210 4
16	20 FORMAT (1H+,45X,23H (IN.) (FT.))							8	16	000224 0
17	1302 FORMAT (1H+,45X,7H(KS/FT))							10	16	000233 2
18	1301 FORMAT (1H+,46X,6H(S/FT))							8	16	000241 5
19	1401 FORMAT (9X,12,4X,F10.3,F10.1,3F10.3,F10.1)							8	16	000250 1
20	14 FORMAT (15,F8.3,F8.0,F8.2,F8.0,3F8.3,F8.1)							3	16	000261 2
21	15 FORMAT (2X,3HTOE,32X,3F8.3)							3	16	000272 1
22	16 FORMAT (/2X,35HCOEFFICIENT OF RESTITUTION OF SOIL ,F8.3)							8	16	000300 3
23	11 FORMAT (26X,17H PILE PROPERTIES //)							3	16	000313 5
24	17 FORMAT (2X,12HPILE LENGTH=,F5.0,5H FT.,,14H AREA(AT TOP)=,							8	16	000323 3
25	*F6.1,5H S-IN)							10	16	000337 1
26	18 FORMAT (2X,17HE. MODUL(AT TOP)=,F6.0,6H KSI.,,							8	16	000342 4
27	*19H SPEC. WT.(AT TOP)=,F5.0,10H LBS/CU FT//)							8	16	000354 1
28	4711 FORMAT(1H)							1	16	000365 0
29	M1=M+1							1	16	000370 5
30	M2=M+2							1	16	000373 0
31	DO 4712 I=1,3							1	16	000375 1
32	C							16	16	000400 3
33	C OUTPUT HAMMER MODEL							16	16	000401 4
34	C							16	16	000406 2
35	4712 WRITE (IW,4711)							1	16	000407 3
36	DO 100 I=1,4							1	16	000414 0
37	100 CREE(I) = SQRT(EE(I))							1	16	000417 1
38	WRITE (IW,4) HNAME(1),HNAME(2)							1	16	000424 4
39	WRITE (IW,5)							1	16	000433 0
40	WRITE (IW,6)							1	16	000436 1
41	WTH=HM(1)*32.17							1	16	000441 2
42	STFH=0.0							1	16	000445 0
43	I=1							1	16	000447 3
44	WRITE (IW,1401) I,WTH							8	16	000451 1
45	IF (M.EQ.1) GO TO 104							13	16	000455 5
46	DO 101 I=2,M							1	16	000462 3
47	STFH=STH(I-1)/12.0							1	16	000465 4
48	WTH=HM(I)*32.17							1	16	000471 5
49	101 WRITE (IW,1401) I,WTH,STFH							9	16	000475 3
50	104 CONTINUE							13	16	000503 5
51	ANVWT=HM(M1)*32.17							1	16	000507 0
52	ANVSTF=STH(M)/12.0							1	16	000513 1
53	CAPWT=HM(M2)*32.17							1	16	000517 2

54	CAPSTF=STH(M1)/12.0	1	16	000523	3
55	IF (ITYPH.EQ.3) CAPWT=HM(M1)*32.17	13	16	000536	4
56	IF (ITYPH.EQ.3) CAPSTF=STH(M)/12.0	13	16	000545	4
57	IF (ITYPH.NE.3) WRITE (IW,7) ANVWT,ANVSTF,CREE(1)	13	16	000554	4
58	EREP=CREE(2)	14	16	000575	0
59	IF (ITYPH.EQ.3) EREP=CREE(1)	14	16	000600	1
60	WRITE (IW,8) CAPWT,CAPSTF,EREP	14	16	000606	1
61	CSHSTF=STH(M2)/12.0	1	16	000614	3
62	IF (ITYPH.EQ.3) CSHSTF=STH(M1)/12.0	14	16	000620	4
63	IF (CSHSTF.GT.10.**9.) CSHSTF=0.	2	16	000627	5
64	WRITE (IW,9) CSHSTF,CREE(4)	1	16	000636	3
65	IF (CSHSTF.LT.0.00001) CSHSTF=10.**12	2	16	000644	1
66	WRITE (IW,10) CREE(3)	1	16	000653	4
67	C	16	16	000670	1
68	C PRINT PILE PROPERTIES AND MODEL	16	16	000671	2
69	C	16	16	000700	1
70	4713 FORMAT (1H1)	16	16	000701	2
71	WRITE (IW,4713)	16	16	000705	2
72	WRITE (IW,11)	1	16	000711	0
73	WRITE (IW,17) XPT,AP0	3	16	000714	2
74	A1 = EMP/144.	16	16	000727	5
75	A2 = RHO*32170.	16	16	000733	1
76	WRITE (IW,18) A1,A2	16	16	000736	5
77	WRITE (IW,12)	1	16	000743	1
78	WRITE (IW,19)	4	16	000746	3
79	WRITE (IW,13)	1	16	000751	5
80	WRITE (IW,20)	4	16	000755	1
81	IF (ISMITH.EQ.1) WRITE (IW,1301)	6	16	000760	3
82	IF (ISMITH.LT.1) WRITE(IW,1302)	6	16	000767	1
83	SALPH=0.0	11	16	000775	4
84	DO 200 I=1,N	11	16	001000	2
85	200 SALPH=SALPH+ALPH(I)	11	16	001003	4
86	DD = 0.0	1	16	001010	5
87	DO 102 I=1,N	1	16	001013	2
88	PWT= PM(I)*32.17	1	16	001016	3
89	PSTF= STP(I)/12.0	1	16	001022	2
90	QSF=QS(I)*12.0	1	16	001026	2
91	DD=DD+XPT*ALPH(I)/SALPH	11	16	001031	5
92	102 WRITE (IW,14) I,PWT,PSTF,CDP(I),SPLICE(I),SU(I),SJ(I),QSF,DD	3	16	001036	5
93	A1 = QS(N+1)*12.0	16	16	001063	2
94	WRITE (IW,15) SU(N+1),SJ(N+1),A1	16	16	001067	2
95	22 FORMAT (45H SKIN FRICTION CONSTANT FOR ALL RULT VALUES)	15	16	001076	0
96	IF (CDP(99).GT.0.5) WRITE (IW,22)	15	16	001111	3
97	A1 = SQRT(ES0IL)	16	16	001125	4
98	WRITE (IW,16) A1	16	16	001131	3
99	RETURN	1	16	001135	2
100	END	1	16	001137	3

OUT2	SYMBOLIC FORTRAN 11 MAY 76 12:19:22 14 18 5 00313614	992			
1	SUBROUTINE OUT2(OUT,JNP,STH,HM,STP,PM,EMP,RHO,ICOL,M,N,DTP,JPMAX,	1	18	000004	0
2	1JDOUT,JEMAX,FINTMX,EX,JMAX,IW,IOUT,RESULT,AREA,JEX,ITYPH)	13	18	000020	2
3	DIMENSION STH(10),HM(10),STP(99),PM(99),CONV(10),STSCNV(15),	1	18	000033	3
4	1OUT(3200),JNP(13),EX(600),RESULT(99),AREA(99),JEX(600)	7	18	000047	0
5	REAL IN	15	18	000061	4

6		DATA P,PSI,SEA,IN/4H P,4HPSI),4H SEA,4H IN)/	14	18	000064	0
7		H1 = P	13	18	000074	4
8		H2 = PSI	13	18	000076	5
9		IF (ITYPH.EQ.3) H1 = SEA	13	18	000101	2
10		IF (ITYPH.EQ.3) H2 = IN	13	18	000106	3
11	1	FORMAT (25X,30H HAMMER AND PILE FORCES(KIPS))	1	18	000113	3
12	2	FORMAT (22X,36H HAMMER AND PILE VELOCITIES(FT/SEC))	1	18	000125	0
13	3	FORMAT (24X,32H HAMMER AND PILE STRESSES (KSI))	1	18	000137	3
14	4	FORMAT (22X,36H HAMMER AND PILE ACCELERATIONS(G'S))	1	18	000151	2
15	5	FORMAT (20X,40H HAMMER AND PILE DISPLACEMENTS (INCHES))	1	18	000163	5
16	6	FORMAT (51X,30H HAMMER AND PILE FORCES(KIPS))	1	18	000177	1
17	7	FORMAT (48X,36H HAMMER AND PILE VELOCITIES(FT/SEC))	1	18	000210	4
18	8	FORMAT (50X,32H HAMMER AND PILE STRESSES (KSI))	1	18	000223	1
19	9	FORMAT (48X,36H HAMMER AND PILE ACCELERATIONS(G'S))	1	18	000235	0
20	10	FORMAT (46X,40H HAMMER AND PILE DISPLACEMENTS (INCHES))	1	18	000247	3
21	11	FORMAT (2X,25H JP TIME HAMMER , 9X,15H PILE ELEMENTS)	1	18	000263	0
22	12	FORMAT(6X,29H (MS) RAM M ANVIL TOP ,16,418)	1	18	000276	2
23	13	FORMAT (2X,25H JP TIME HAMMER ,30X,15H PILE ELEMENTS)	1	18	000307	5
24	14	FORMAT (6X,29H (MS) RAM M ANVIL TOP ,16,1118)	1	18	000323	1
25	15	FORMAT (15,F6.1,15F8.1)	1	18	000335	0
26	16	FORMAT (15,F6.1,15F8.3)	1	18	000342	4
27	21	FORMAT(15,F6.1,15F8.2)	7	18	000350	2
28	31	FORMAT (15H J TIME ,A4,5X,28HD RAM ANV FTOP VTOP,	13	18	000355	4
29		*62H DTOP FMID VMID DMID SUM ST SUM DP RT TOE FTOE,	3	18	000370	4
30		*15H VTOE DTOE)	3	18	000403	2
31	32	FORMAT (15H J TIME ,A4,6X,27HFTOP VTOP DTOP FTOE,	13	18	000407	2
32		*15H VTOE DTOE)	3	18	000422	2
33	33	FORMAT (8X,9H(MS) (,A4,32H (IN) (IN) (KIPS) (FT/S) ,	13	18	000426	2
34		*61H (IN) (KIPS) (FT/S) (IN) (KIPS) (KIPS) (KIPS) (KIPS),	3	18	000441	3
35		*14H (FT/S) (IN))	3	18	000455	0
36	34	FORMAT (8X,9H(MS) (,A4,32H (KIPS) (FT/S) (IN) (KIPS),	13	18	000461	1
37		*14H (FT/S) (IN))	3	18	000474	3
38	35	FORMAT (15,F7.1,F9.1,2F8.3,2(F9.1,F6.1,F8.3),F9.1,2F7.1,F9.1,F6.1,	1	18	000500	4
39		*F8.3)	1	18	000515	4
40	36	FORMAT (15,F7.1,F9.1,2(F9.1,F6.1,F8.3))	1	18	000517	5
41		M1 = M+1	1	18	000530	1
42		KLIM = 200	1	18	000532	4
43		IMAX = (JMAX+2)*KLIM	1	18	000535	3
44		K = 1	1	18	000542	0
45		KK = 1	1	18	000544	0
46		M2 = M+2	1	18	000546	1
47		EHMR = 31000.*144.	1	18	000550	4
48		RHUMR=0.492/32.17	10	18	000554	5
49	C	OUTPUT OPTION FOR PRINTING	17	18	000565	0
50		IIOUT = IABS(IIOUT)	17	18	000572	5
51		IF (IIOUT.GE.20) IIOUT=IIOUT-20	6	18	000577	0
52		IF (IIOUT.GE.10) IIOUT=IIOUT-10	6	18	000605	3
53		IF (IIOUT.EQ.0) GO TO 135	12	18	000614	0
54	C	FOR IIOUT EQ 0 DO NOT PRINT VARIABLES VS TIME	17	18	000621	2
55		CONV(1) = 1.	1	18	000632	2
56		CONV(2) = 1.	1	18	000635	3
57		CONV(3)=1.0	7	18	000640	4
58		CONV(4) = 1./32.17	1	18	000643	4
59		CONV(5) = 12.	1	18	000647	5

60		IF (ICOL.GT.0) GO TO 101	1	18	000653	1
61	C	TITLE FOR 132 COLUMNS	17	18	000660	2
62		IF (IIOUT.EQ.1) WRITE (IW,1)	1	18	000665	2
63		IF (IIOUT.EQ.2) WRITE (IW,2)	1	18	000673	2
64		IF (IIOUT.EQ.3) WRITE (IW,3)	1	18	000701	2
65		IF (IIOUT.EQ.4) WRITE (IW,4)	1	18	000707	2
66		IF (IIOUT.EQ.5) WRITE (IW,5)	1	18	000715	2
67		IF (IIOUT.EQ.6) WRITE (IW,32) H1	13	18	000723	2
68		IF (IIOUT.EQ.6) WRITE (IW,34) H2	13	18	000732	0
69		IF (IIOUT.EQ.6) GO TO 130	1	18	000740	4
70		WRITE (IW,11)	1	18	000746	0
71		WRITE (IW,12) (JNP(I),I=2,JMAX)	1	18	000751	2
72		GO TO 102	1	18	000757	5
73	101	CONTINUE	1	18	000762	3
74	C	TITLE FOR 80 COLUMNS	17	18	000765	5
75		IF (IIOUT.EQ.1) WRITE (IW,6)	1	18	000772	4
76		IF (IIOUT.EQ.2) WRITE (IW,7)	1	18	001000	4
77		IF (IIOUT.EQ.3) WRITE (IW,8)	1	18	001006	4
78		IF (IIOUT.EQ.4) WRITE (IW,9)	1	18	001014	4
79		IF (IIOUT.EQ.5) WRITE (IW,10)	1	18	001022	4
80		IF (IIOUT.EQ.6) WRITE (IW,31) H1	13	18	001030	5
81		IF (IIOUT.EQ.6) WRITE (IW,33) H2	13	18	001037	1
82		IF (IIOUT.EQ.6) GO TO 130	1	18	001045	5
83		WRITE (IW,13)	1	18	001053	1
84		WRITE (IW,14) (JNP(I),I=2,JMAX)	1	18	001056	3
85	102	CONTINUE	1	18	001065	0
86		IF (IIOUT.NE.3) GO TO 107	1	18	001070	2
87	C		17	18	001075	4
88	C	STRESS CONVERSION FROM FORCES	17	18	001076	5
89	C		17	18	001105	2
90		CONST1 = 1./144.	16	18	001117	0
91		IF (M.GT.1) CONST1 = SQRT(STH(M-1)*HM(M)/(EHMR*RHOHMR))	16	18	001122	5
92		CONST2=SQRT(STH(M)*HM(M1)/(EHMR*RHOHMR))	1	18	001135	3
93		CONST1=1.0/(CONST1*144.0)	10	18	001145	3
94		CONST2=1.0/(CONST2*144.0)	10	18	001152	5
95		DO 103 I=1,JMAX	1	18	001160	1
96		K = JNP(I)	1	18	001163	5
97	103	STSCNV(I)=1.0/AREA(K)	7	18	001166	4
98		K = 1	1	18	001174	0
99		KK = 1	1	18	001176	0
100		DO 106 I=1,IMAX	1	18	001200	1
101		IF (I.GT.KLIM) GO TO 104	1	18	001203	5
102		OUT(I) = OUT(I)*CONST1	1	18	001211	0
103		GO TO 106	1	18	001215	5
104	104	IF (I.GT.2*KLIM) GO TO 105	1	18	001220	3
105		OUT(I) = OUT(I)*CONST2	1	18	001226	5
106		GO TO 106	1	18	001233	4
107	105	OUT(I) = OUT(I) * STSCNV(KK)	1	18	001236	2
108		K = K+1	1	18	001245	1
109		IF (K.LE.KLIM) GO TO 106	1	18	001247	3
110		K = 1	1	18	001254	4
111		KK = KK + 1	1	18	001256	4
112	106	CONTINUE	1	18	001261	4
113	C		17	18	001265	0

114	C	FOR ALL EXCEPT 6 OPTION	17	18	001266	1
115	C		17	18	001273	3
116	107	DO 108 I=1,IMAX	1	18	001274	4
117	108	OUT(I) = OUT(I) * CONV(IIOUT)	1	18	001301	1
118		JJ = 0	1	18	001310	1
119	109	DO 110 JP = JDOUT,JPMAX,JDOUT	18	18	001320	3
120		JJ = JJ+1	1	18	001327	3
121		JJMAX = (JMAX+1)*KLIM+JJ	1	18	001332	1
122		TT = FLOAT(JP)*DTP*1000.	1	18	001337	2
123		IF (IIOUT.EQ.1.OR.IIOUT.EQ.2.OR.IIOUT.EQ.4) WRITE (IW,15) JP,TT,	7	18	001344	3
124		1(OUT(I),I=JJ,JJMAX,KLIM)	7	18	001360	4
125		IF (IIOUT.EQ.3) WRITE (IW,21) JP,TT,(OUT(I),I=JJ,JJMAX,KLIM)	7	18	001366	0
126		IF (IIOUT.EQ.5) WRITE (IW,16) JP,TT,(OUT(I),I=JJ,JJMAX,KLIM)	1	18	001401	3
127	110	CONTINUE	1	18	001414	5
128	C		17	18	001420	1
129	C	FOR 6 OPTION ONLY, FIRST 132 COLUMNS	17	18	001421	2
130	C		17	18	001431	0
131	130	IF (IIOUT.NE.6) GO TO 135	1	18	001432	1
132		IF (IIOUT.EQ.6.AND.ICOL.EQ.0) GO TO 131	1	18	001440	2
133		KP=0	10	18	001450	1
134		DO 132 JP = JDOUT,JPMAX,JDOUT	18	18	001457	0
135		KP=KP+1	10	18	001465	1
136		KPMAX=15*KLIM+KP	10	18	001467	3
137	132	WRITE (IW,35) JP,(OUT(I),I=KP,KPMAX,KLIM)	10	18	001473	2
138		GO TO 135	1	18	001504	1
139	C		17	18	001506	5
140	C	6 OPTION - 80 COLUMNS	17	18	001510	0
141	C		17	18	001515	0
142	131	KP=0	11	18	001516	1
143		DO 134 JP = JDOUT,JPMAX,JDOUT	18	18	001525	4
144		KP=KP+1	10	18	001533	5
145		KPMAX1=200+KP	1	18	001536	1
146		KP2 = 4*KLIM+KP	16	18	001544	5
147		KPMAX2=9*KLIM+KP	9	18	001550	3
148	134	WRITE (IW,36) JP,(OUT(I),I=KP,KPMAX1,KLIM),(OUT(I),I=KP2,KPMAX2,	10	18	001554	2
149		1 KLIM)	7	18	001571	1
150	135	CONTINUE	1	18	001573	3
151	C		17	18	001576	4
152	C	FOR ALL OPTIONS EXTREME VALUE TABLES	17	18	001577	5
153	C		17	18	001607	3
154		17 FORMAT(13X,56H TABLE OF EXTREME VALUES FOR PILE AND TIME OF OCCURR	7	18	001610	4
155		1ENCE)	7	18	001625	4
156	18	FORMAT (4X,79H'ELEM. FMAX FMIN MINSTR MAXST	7	18	001627	5
157	1R	VELMX DISMX / 7X,76HNO. KIPS KIPS	7	18	001641	2
158	1	KSI KSI FT/S INCH)	7	18	001651	1
159	19	FORMAT (5X,15,4(F8.1,1H(,I3,1H)),F8.2,1H(,I3,1H),F8.3,1H(,I3,1H))	7	18	001656	2
160	20	FORMAT (1H)	1	18	001673	1
161		DO 115 I=1,3	1	18	001677	0
162	115	WRITE (IW,20)	1	18	001702	1
163		WRITE (IW,17)	1	18	001706	2
164		WRITE (IW,18)	1	18	001711	4
165		DO 120 I=1,N	1	18	001715	0
166		NN = 5*N + I	1	18	001720	1
167	120	WRITE (IW,19) I,(EX(J),JEX(J),J=I,NN,N)	7	18	001723	2

168 RETURN 1 18 001733 5
 169 END 1 18 001736 0

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IMPACTN      SYMBOLIC FORTRAN 18 JUN 78 14:01:51 46 50 5 00241010 2371
000100 1. C ***** W E A P WAVE EQUATION ANALYSIS FOR PILES ***** 000000
000100 2. C THIS PROGRAM WAS DEVELOPED FOR THE FEDERAL HIGHWAY ADMINISTRATION 000000
000100 3. C BY GOBLE, CONSULTING ENGINEERS 000000
000100 4. C FOR INFORMATION REGARDING THE PROGRAM CODING CONTACT 000000
000100 5. C GOBLE, CONSULTING ENGINEERS 000000
000100 6. C 12434 CEDAR ROAD #10 000000
000100 7. C CLEVELAND HEIGHTS, OHIO 44106 000000
000100 8. C (216)-721-0220 000000
000100 9. C 000000
000100 10. C 000000
000100 11. C NOTE THAT THIS PROGRAM UTILIZES A DATA FILE WHICH CONTAINS 000000
000100 12. C PROPRIETARY INFORMATION. THIS DATA, THEREFORE, MAY NOT BE 000000
000100 13. C DUPLICATED IN ANY FORM WITHOUT THE EXPRESSED WRITTEN PERMISSION 000000
000100 14. C OF EITHER THE FHWA OR GOBLE CONSULTING ENGINEERS 000000
000100 15. C 000000
000100 16. C 000000
000101 17. C COMMON VINH, DINH, AOH, VOH, DOH, ANH, VNH, DNH, STH, HM, AM, STA, MAT, ESIL, 000000
000101 18. C 1 AOP, ANP, DOP, DNP, VOP, VNP, STP, PM, CDP, RES, RES0, SJ, SOK, QS, SU, 000000
000101 19. C 2 SPLICE, DEPIB, ARAM, ECUS, ECAP, EPT, EANY, VFALL, VCHAM, XPT, SJJJ, 000000
000101 20. C 3 EMP, RHO, OTH, DTP, N, M, JRAT, IULT, RULT, SOILM, DSACP, 000000
000101 21. C 4 FPTO, IW, PCOM, TDEL, IOUT, OUT, EXPP, TEMI, TEMO, INP, STROK0, STROKN, VO, PO 000000
000101 22. C 5 ,DFIN, JDOUT, JPMAX, EX, ICOL, FINTMX, TEMAX, RESULT, ISMITH, ITER 000000
000101 23. C 6 , ITYPH, IPREIG, VFALLM, STRMAX, DCYL, PBAR, AREA, JEX 000000
000103 24. C DIMENSION VINH(10), DINH(10), AOH(10), VOH(10), DOH(10), ANH(10), 000002
000103 25. C 1 VNH(10), DNH(10), STH(10), HM(10), VNP(99), 000002
000103 26. C 2 AOP(99), ANP(99), DOP(99), DNP(99), VOP(99), STP(99), PM(99), CDP(99), 000002
000103 27. C 3 RES(99), RES0(99), SJ(99), SOK(99), QS(99), SU(99), SPLICE(99), 000002
000103 28. C 4 DSACP(3), OUT(3200), INP(13), EX(600), RESULT(100), JEX(600) 000002
000103 29. C 5, AM(10), STA(10), ISEG(15), JSELEC(13), ALPH(99), TITLE(10) 000002
000103 30. C 6 , PBAR(20), STINH(4,10), STINP(4,99), AREA(99), STRAR(10) 000002
000104 31. 9877 CONTINUE 000002
000105 32. STRMAX = 0.0 000002
000106 33. CALL IPTN (JSELEC, ISEG, YMAX, ALPH, TITLE, IOSTR, STRM) 000002
000107 34. IF (IW.GT.-100) IWR = IW 000002
000111 35. IF (IW.LT.-100) GO TO 9876 000013
000111 36. C 000022
000111 37. C IPTN READS ALL INPUT INFORMATION AND SETS UP HAMMER-PILE-SOIL- 000022
000111 38. C MODEL 000022
000111 39. C NOTE IPTN AUTOMATICALLY CALLS FOR THE RIGHT HAMMER AND SETS ITYPH 000022
  
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99

000111	40.	C	FOR AIR STEAM HAMMERS STROKE AND VFALL ARE SET	000022
000111	41.	C		000022
000111	42.	C	INITIALIZE FOR NEW PROBLEM	000022
000111	43.	C		000022
000113	44.		N1 = N+1	000030
000114	45.		PCOMR = PCOM	000033
000115	46.		IIOUT = IOUT	000035
000116	47.		IF (IOUT.GT.9) IIOUT = IOUT-10	000040
000120	48.		IF (IOUT.GT.19) IIOUT = IOUT-20	000050
000122	49.		IFUEL = 0	000062
000123	50.		IWT = 0	000063
000124	51.		DSTROK = 0.0	000064
000125	52.		INOW = 0	000065
000126	53.		EPSSTR=0.05	000066
000126	54.	C	MAXIMUM QUAKE AT BOTTOM ELEMENT	000066
000127	55.		Q = QS(N)	000070
000130	56.		IF (QS(N1).GT.Q) Q = QS(N1)	000073
000132	57.		COLI=(XPT/SQRT(EMP/RHO))*1000.	000103
000133	58.		NULT=IULT	000115
000134	59.		IF (NULT.EQ.0) NULT = 1	000117
000136	60.		IULT = 1	000123
000137	61.		DULT=RULT*1.25	000125
000140	62.		IF (PBAR(1).LT.10.E-5) PBAR(1) = ARAM	000130
000142	63.		IF (ITYPH.NE.2) GO TO 502	000137
000144	64.		EPSSTR = 0.03	000143
000144	65.	C	FOR CLOSED END HAMMERS, DETERMINES VFALL FOR UPLIFT CONDITION	000143
000145	66.		CALL DOWN(STRMAX,VFALLM,PBAR,HM,DEPIB,ARAM,IW,IOUT,DOWNT)	000145
000146	67.		IF (STROKO.GT.0.1) GO TO 503	000160
000150	68.		VFALL = VFALLM	000165
000151	69.		STROKO = STRMAX	000167
000152	70.		GO TO 502	000171
000152	71.	C	IF STROKE IS ASSIGNED FIND CORRESPONDING VFALL	000171
000153	72.	503	CALL DOWN(STROKO,VFALL,PBAR,HM,DEPIB,ARAM,IW,IOUT,DOWNT)	000173
000154	73.	502	CONTINUE	000206
000155	74.	501	ISTR=0	000206
000155	75.	C	INITIALIZE FOR NEW RULT	000206
000156	76.		IFA = 0	000206
000157	77.		RTONS = RULT/2.0	000207
000160	78.		RTOE = SU(N1)*0.5	000212
000161	79.		WRITE (IW,9) RTONS,RTOE	000216
000165	80.	9	FORMAT (1H1,30X,5HRULT=,F7.1,9H, AT TOE=,F7.1,5H TONS/31X,33(1H-)/	000225
000165	81.		1)	000225
000166	82.		NM1=N-1	000225
000167	83.		DTR = 1.0	000236
000170	84.		DTRM = 1.0	000240
000171	85.		DO 200 I=1,NM1	000241
000171	86.	C	CHECK DAMPING	000241
000174	87.		SDJ=SU(I)	000250
000175	88.		IF (ISMITH.EQ.1) SDJ=SU(I)*SJ(I)	000251
000177	89.		IF (SDJ.GT.10.E-8) DTR = PM(I)/SDJ	000256
000201	90.		IF (DTR.LT.DTRM) DTRM = DTR	000266
000203	91.	200	CONTINUE	000277
000205	92.		SDJ = SJ(N) + SJ(N+1)	000277
000205	93.		IF (ISMITH.EQ.1) SDJ=SU(N)*SU(N)+SJ(NI)*SU(NI)	000303

000210	94.		IF (SDJ.GT.10.E-8) DTR = PM(N)/SDJ	000315
000212	95.		IF (DTR.LT.DTRM) DTRM = DTR	000326
000214	96.		DTRM = DTRM*0.8	000335
000215	97.		IF (DTRM.GE.DTP) GO TO 711	000340
000217	98.		IF (DTRM.LT.0.00001) WRITE (IW,21) DTRM	000344
000223	99.		IF (DTRM.LT.0.00001) GO TO 401	000363
000225	100.		DTP = DTRM	000365
000226	101.		DTH = DTP/FLOAT(JRAT)	000367
000227	102.		DTRM = DTRM*1000	000373
000230	103.		WRITE (IW,18) DTRM	000376
000233	104.	711	CONTINUE	000405
000234	105.	21	FORMAT (51H DAMPING GTR CRITICAL, T.I. REQUIRED LSS .01 MSEC.(.E10	000405
000234	106.		1.5,1H))	000405
000235	107.	18	FORMAT (43H LARGE DAMPING REQUIRES NEW TIME INCREMENT=.F7.4,	000405
000235	108.		1 6H MSEC.)	000405
000236	109.	500	CONTINUE	000405
000237	110.		JMAX = 6	000405
000240	111.		IF (ICOL.GT.0) JMAX = 13	000406
000242	112.		IF (JMAX.GT.N) JMAX = N	000414
000244	113.		IF (ITYPH.LT.3) GO TO 650	000423
000244	114.	C	PERFORM AIR/STM HAMMER ANALYSIS AND GO TO OUTPUT (OUT2)	000423
000246	115.		CALL AIRSTM	000430
000247	116.		IBLOW=-1	000432
000250	117.		DFIN = ((DFIN/12.0)+05(N)-Q)*12.0	000434
000251	118.		IF (ABS(DFIN).GT.0.0) IBLOW=12.0/DFIN+0.5	000443
000253	119.		GO TO 900	000456
000254	120.	650	CONTINUE	000460
000254	121.	C	BEGIN ANALYSIS FOR ALL DIESEL HAMMERS	000460
000255	122.		VFR = VFALL	000460
000256	123.		CALL STARTC	000461
000257	124.		TSTART = FINTMX	000463
000260	125.		M2=M+2	000465
000260	126.	C	STORE INITIAL VALUES FROM START ANALYSIS (MAY BE REUSED)	000465
000261	127.		STINH(1,10)=VO	000470
000262	128.		STINH(2,10)=FP10	000472
000263	129.		STINH(3,10)=FLOAT(IPREIG)+0.1	000474
000264	130.		STINH(4,10)=PO	000501
000265	131.		DO 660 I=1,M2	000503
000270	132.		STINH(1,I)=AOH(I)	000513
000271	133.		STINH(2,I)=VINH(I)	000514
000272	134.	660	STINH(3,I)=QINH(I)	000516
000274	135.		DO 661 I=1,N1	000525
000277	136.		STINP(1,I)=AOP(I)	000525
000300	137.		STINP(2,I)=VOP(I)	000526
000301	138.		STINP(3,I)=DOP(I)	000530
000302	139.	661	STINP(4,I)=RESO(I)	000532
000304	140.	662	IF (IPREIG.LT.--99) IBLOW=0	000536
000306	141.		IF (IPREIG.LT.--99) GO TO 700	000547
000310	142.		CALL DIESEL	000551
000311	143.		TDIES = FINTMX	000553
000311	144.	C	DETERMINE STROKE BY ANALYZING UPWARD MOTION	000553
000312	145.		CALL UP(PBAR,DMH,VNH,DNP,VNP,HM,PM,M,N,DCYL,DFIN,VFALL,VO,PO,	000555
000312	146.		1 VFALLM,STROKN,DEPIB,ARAM,EXPP,IWT,IW,IOUT,PSI,TUP)	000555
000312	147.	C	NO PENETRATION AND NO INCREASE OF STROKE MEANS QUIT (300)	000555

000313	148.		IF (DFIN.LT.0.0.AND.STROKN.LT.STROKO*1.025) GO TO 300	000607
000315	149.		IF (IOSTR.LT.0) GO TO 750	000625
000315	150.	C	REDUCE FUEL FOR UPLIFT (901)	000625
000317	151.		IF (IWT.EQ.1.AND.ITYPH.EQ.2.AND.IOSTR.NE.1) GO TO 901	000631
000317	152.	C	NO UPLIFT BUT PREVIOUS UPLIFT MEANS FUEL SUFFICIENTLY REDUCED(300)	000631
000321	153.		IF (IFUEL.GT.0.AND.IOSTR.NE.1) GO TO 300	000653
000323	154.		IF (ITYPH.EQ.1.AND.STROKN.GT.STRM) WRITE (IW,20)	000667
000326	155.	20	FORMAT (35H *** CAUTION RAM MIGHT BLOW OUT ***)	000711
000327	156.		IF (ITYPH.EQ.1.AND.STROKN.GT.STRM) STROKN=STRM	000711
000327	157.	C	DETERMINE STROKE DIFFERENCE,STORE NEW STROKE	000711
000331	158.		DOSTR = DSTROK	000730
000332	159.		DSTROK = ABS(STROKN-STROKO)/STROKO	000732
000333	160.		ISTR = ISTR + 1	000737
000334	161.		STRAR(ISTR) = STROKO	000742
000335	162.		IF (IOSTR.EQ.1.OR.DSTROK.LT.EPSSTR) GO TO 300	000745
000337	163.		STROKO = STROKN	000761
000340	164.		IF (ISTR.GT.4.AND.DSTROK.GT.(FLOAT(20-ISTR))*0.05*DOSTR) GO TO 301	000763
000342	165.		IF (ISTR.GT.6) GO TO 301	001006
000344	166.		IF (ITYPH.GT.1.OR.ISTR.LT.2.OR.PBAR(20).GT.0.0) GO TO 675	001013
000346	167.		IF (STRAR(ISTR).GT.STRAR(ISTR-1).AND.STROKN.GT.STRAR(ISTR)) GO TO	001040
000346	168.		* 675	001040
000350	169.		IF (STRAR(ISTR).LT.STRAR(ISTR-1).AND.STROKN.LT.STRAR(ISTR)) GO TO	001054
000350	170.		* 675	001054
000350	171.	C	FOR A NEW STROKE WHICH IS BETWEEN PREVIOUS VALUES	001054
000352	172.		STROKN = 0.5*(STRAR(ISTR)+STROKN)	001072
000353	173.		VFALL = SQRT((STROKN-DEPIB)*64.34)	001076
000354	174.		STROKO = STROKN	001105
000354	175.	C	AT 650 A NEW COMPLETE ANALYSIS STARTS	001105
000355	176.		GO TO 650	001107
000355	177.	C	CHANGE FUEL SETTING	001107
000356	178.	750	ISTR = ISTR + 1	001111
000357	179.		IF (IWT.EQ.1) STROKN = STRMAX	001123
000361	180.		DSTRF = (STROKN-STROKO)/STROKO	001127
000362	181.		IF (IWT.EQ.1) DSTRF = DSTRF + (VFALL-VFALLM)/VFALLM	001133
000364	182.		IF (ABS(DSTRF).GT..2) DSTRF=.2*DSTRF/ABS(DSTRF)	001142
000366	183.		STRAR(ISTR) = STROKN	001154
000367	184.		STROKN = STROKO	001157
000370	185.		IF (ABS(DSTRF).LT.EPSSTR) GO TO 300	001161
000372	186.		IF (ISTR.GT.4) GO TO 663	001166
000374	187.	76	FORMAT (21H FUEL SETTING CHANGED,F6.3,7H PERCT.)	001173
000375	188.		IF (IWT.EQ.1.AND.DSTRF.LT.0.1.AND.ITYPH.EQ.2) DSTRF=DSTRF + 0.1	001173
000377	189.		PCOM = PCOM*(1.0-DSTRF)	001222
000400	190.		IF (IOUT.LT.0) WRITE(IW,76) DSTRF	001226
000404	191.		VFALL = VFR	001240
000405	192.		GO TO 761	001242
000406	193.	675	CONTINUE	001244
000407	194.		IF (DSTROK.GT.2.0*EPSSTR) GO TO 650	001244
000407	195.	C	INITIAL VALUES ARE REUSED (AT 662)	001251
000411	196.		DO 666 I=1,M2	001257
000414	197.		VINH(I)=STINH(2,I)	001261
000415	198.		IF (I.LE.M) VINH(I)=(VFALL/VFR)*VINH(I)	001272
000417	199.		AOH(I)=STINH(1,I)	001274
000420	200.	666	DINH(I)=STINH(3,I)	001300
000422	201.		GO TO 667	

000423	202.	901	IFUEL = IFUEL + 1	001302
000424	203.		VFALL = VFALLM	001304
000425	204.		PCOM = PCOMR*(1.0-FLOAT(IFUEL)/10.)	001306
000426	205.	11	FORMAT (36H FUEL SETTING REDUCED BY TEN PERCENT)	001315
000427	206.		IF (IFUEL.GT.4) GO TO 663	001315
000431	207.		IF (IOUT.LT.0) WRITE (IW,11)	001321
000431	208.	C	REINITIALIZE	001321
000434	209.	761	CONTINUE	001333
000435	210.		DO 664 I=1,M2	001333
000440	211.		AOF(I)=STINH(1,I)	001340
000441	212.		VINH(I)=STINH(2,I)	001341
000442	213.	664	DINH(I)=STINH(3,I)	001343
000444	214.	667	CONTINUE	001347
000445	215.		DO 665 I=1,N1	001347
000450	216.		AOP(I)=STINP(1,I)	001354
000451	217.		VOP(I)=STINP(2,I)	001355
000452	218.		DOP(I)=STINP(3,I)	001357
000453	219.	665	RESO(I)=STINP(4,I)	001361
000455	220.		VO=STINH(1,10)	001364
000456	221.		FPTO=STINH(2,10)	001366
000457	222.		IPREIG=STINH(3,10)	001370
000460	223.		PO=STINH(4,10)	001376
000461	224.		GO TO 662	001400
000462	225.	663	WRITE (IW,12)	001402
000464	226.		GO TO 300	001406
000465	227.	12	FORMAT (20H NO FUEL CONVERSION)	001410
000466	228.	301	WRITE (IW,8)	001410
000470	229.	8	FORMAT (25H STROKE DOES NOT CONVERGE)	001415
000471	230.	300	CONTINUE	001415
000471	231.	C		001415
000471	232.	C	ONE RULT HAS BEEN ANALYZED	001415
000471	233.	C		001415
000472	234.		IBLOW=-1	001415
000473	235.		DFIN = ((DFIN/12.0)+OS(N)-Q)*12.0	001416
000474	236.		IF (ABS(DFIN).GT.0.0) IBLOW=12./DFIN + 0.5	001425
000476	237.		IF (IFUEL.EQ.5) IFUEL = 4	001440
000500	238.	900	CONTINUE	001447
000500	239.	C	OUTPUT FOR ALL HAMMERS	001447
000501	240.		CALL OUT2(OUT,INP,STH,HM,STP,PM,EMP,RHO,ICOL,M,N,DTP,JPMAX,JDOUT,	001447
000501	241.		1 TEMAX,FINTMX,EX,JMAX,IW,IOUT,RESULT,AREA,JEX,ITYPH)	001447
000502	242.		WRITE (IW,51) ANP(99)	001500
000505	243.	51	FORMAT (1H0,13X,43HTHE MAXIMUM TRANSFERRED ENERGY (ENTHRU) WAS,	001506
000505	244.		1 F6.1,5H K-FT)	001506
000506	245.	1	FORMAT (1H0,13X,37HSTROKES ANALYZED AND LAST RETURN (FT),7F6.1)	001506
000507	246.		IF (IOSTR.LT.0) GO TO 753	001506
000511	247.		IF (IFUEL.LT.1.AND.ITYPH.LT.3) WRITE (IW,1) (STRAR(I),I=1,ISTR),	001512
000511	248.		* STROKN	001512
000522	249.	77	FORMAT (1H0,13X,42HRETURNED STROKES AND STROKE ANALYZED, (FT),	001547
000522	250.		1 7F6.1)	001547
000523	251.		GO TO 754	001551
000524	252.	753	WRITE (IW,77) (STRAR(I),I=1,ISTR),STROKO	001566
000534	253.		PCOMC = (PCOM-PCOMR)/(PCOMR*0.01)	001574
000535	254.	70	FORMAT (14X,22H TOTAL PRESSURE CHANGE,F6.1,2H %)	001574
000536	255.		WRITE (IW,70) PCOMC	001574

000541	256.	754	CONTINUE	001603
000542	257.		I1 = -IULT	001603
000542	258.	C	JDOUT NO. OF TIME INCREMENTS BETWEEN OUTPUT VALUES + 1	001603
000542	259.	C	JPMAX MAXIMUM J IN OUTPUT	001603
000542	260.	C	MAXJP MAXIMUM NO. OF OUTPUT VALUES	001603
000543	261.		MAXJP=(JPMAX/JDOUT)-1	001604
000544	262.		STP1=STP(1)/EPT	001611
000544	263.	C		001611
000544	264.	C	PLOT	001611
000544	265.	C		001611
000545	266.		IF (IULT.LT.NULT) CALL OUTPUT(OUT,RESULT,INP,IOUT,DTP,JPMAX,JDOUT,	001616
000545	267.		1 I1,I,MAXJP,ALPH,ISEG,JSELEC,YMAX,COL1,TITLE,STP1,PM(I))	001616
000545	268.	C		001616
000545	269.	C	DETERMINE OVERALL EXTREMA FOR SUMMARY	001616
000545	270.	C		001616
000547	271.		SEXMIN = 10.**6	001647
000550	272.		SEXMAX = 0.0	001651
000551	273.		IF (CDP(99).LT.0.5) SU(N1)=SU(N1)/RULT	001652
000553	274.		DO 600 I=1,N	001711
000556	275.		IF (CDP(99).LT.0.5) SU(I)=SU(I)/RULT	001711
000560	276.		IF (SEXMAX.LT.EX(I+3*N)) SEXMAX=EX(I+3*N)	001715
000562	277.	600	IF (SEXMIN.GT.EX(I+2*N)) SEXMIN=EX(I+2*N)	001724
000565	278.		RESULT(IULT)=RULT*.5	001737
000566	279.		RESULT(IULT+10) = IBLOW	001743
000567	280.		IF (RESULT(IULT+10).LT.0.0) RESULT(IULT+10) = 9999999.	001746
000571	281.		RESULT(IULT+20) = STROKO	001755
000572	282.		RESULT(IULT+30) = SEXMIN	001760
000573	283.		RESULT(IULT + 40) = SEXMAX	001762
000574	284.		RESULT(IULT+60) = IFUEL	001764
000575	285.		IF (IOSTR.LT.0) RESULT(IULT+60) = ISTR-1	001767
000577	286.		IF (ITYPH.EQ.3) GO TO 700	002001
000577	287.	C		002001
000577	288.	C	DETERMINE BLOWS/MINUTE FOR DIESELS	002001
000577	289.	C		002001
000601	290.		IF (ITYPH.EQ.1.AND.PBAR(20).GT.0.1) GO TO 680	002005
000603	291.		IF (ITYPH.EQ.2) GO TO 681	002022
000605	292.		TDOWN = SQRT((STROKO-DEPIB)*0.06217)*2.0	002026
000606	293.		GO TO 682	002037
000606	294.	C	FOR CLOSED END HAMMERS FIND CYCLE TIME	002037
000607	295.	681	CALL DOWN(STROKO,VFALL,PBAR,HM,DEPIB,ARAM,IW,INOW,TDOWN)	002041
000610	296.		TDOWN=TDOWN*2.0	002053
000611	297.		GO TO 682	002056
000612	298.	680	DCY = 0.0	002060
000612	299.	C	FOR VACUUM HAMMERS ONLY - DETERMINE TIME OF CYCLE	002060
000613	300.		CALL VACHAM(PBAR,ARAM,DEPIB,DCY,STROKO,FSTR,VFT,INOW,IW)	002060
000614	301.		DD = DEPIB-0.001	002073
000615	302.		CALL VACHAM(PBAR,ARAM,DEPIB,DCY,DD,FSBO,VFB,IOUT,IW)	002076
000616	303.		AC = 32.17*(1.0+0.5*(FSTR+FSBO)/PBAR(20))	002111
000617	304.		TDOWN = SQRT((STROKO-DEPIB)*2.0/AC)*2.0	002120
000620	305.	682	CONTINUE	002133
000620	306.	C	TOTAL CYCLE TIME	002133
000621	307.		TT = TDOWN + TSTART+TDIES+TOP	002133
000622	308.		RESULT(IULT+50) = 60./TT	002137
000622	309.	C	DETERMINE BOUNCE CHAMBER PRESSURE	002137

000623	310.		IF (PBAR(1).GT.0.0) RESULT(IULT+70)=(PBAR(8)/PBAR(1))/0.144	002144
000625	311.		IF (IFUEL.EQ.0) RESULT(IULT+70)=PSI	002155
000627	312.	700	IF (IULT.GE.NULT.OR.IBLOW.GT.1200.OR.IBLOW.LT.0) GO TO 402	002163
000627	313.	C		002163
000627	314.	C		002163
000627	315.	C	SET UP FOR NEXT RULT ANALYSIS	002163
000627	316.	C		002163
000631	317.		I=IULT	002210
000632	318.		IF (ITYPH.EQ.3.OR.IOSTR.NE.0) GO TO 904	002212
000632	319.	C	NEW STROKE (ASSUMED) FOR DIESELS	002212
000634	320.		STROKN = 1.2*STROKO	002224
000635	321.		IF (IULT.GT.1) STROKN = RESULT(IULT+20)*1.5-RESULT(IULT+19)*0.5	002230
000637	322.		IF (STROKN.GT.STRM) STROKN=STRM	002244
000637	323.	C	VELOCITY AT PORTS	002244
000641	324.		IF (STROKN.GT.DEPIB) VFALL=SQRT(64.34*(STROKN-DEPIB))	002253
000643	325.		DCYL = 0.0	002270
000644	326.		IF (PBAR(20).GT.0.0) CALL VACHAM(PBAR,ARAM,DEPIB,DCYL,STROKN,FSUCK	002271
000644	327.		I, VFALL, IOUT, IW)	002271
000646	328.		IF (ITYPH.EQ.1) GO TO 904	002310
000650	329.		VFALL = VFALLM	002314
000651	330.		IF (STROKN.LT.STRMAX)	002316
000651	331.		ICALL DOWN(STROKN,VFALL,PBAR,ARM,DEPIB,ARAM,IW,IOUT,TDOWN)	002316
000653	332.		IF (STROKN.GT.STRMAX) STROKN = STRMAX	002336
000655	333.		IF (IFUEL.GT.0) IFUEL = IFUEL - 1	002345
000657	334.		PCOM = PCOMR*(1.0-FLOAT(IFUEL)*0.1)	002354
000660	335.	904	CONTINUE	002364
000661	336.		IF (IOSTR.EQ.0) STROKO = STROKN	002364
000663	337.		IF (IOSTR.NE.0) VFALL = VFR	002367
72 000663	338.	C		002367
000663	339.	C	NEW STATIC SOIL RESISTANCE	002367
000663	340.	C		002367
000665	341.		IF (RESULT(I+I).GT.0.001) RULT=RESULT(I+I)*2.	002404
000667	342.		IF (RESULT(I+1).GT.0.001) GO TO 399	002412
000671	343.		RULT = RULT + DULT	002414
000672	344.	399	IULT=IULT+1	002420
000673	345.		IF (CDP(99).GT.0.5) GO TO 395	002422
000675	346.		DO 398 I=1,N1	002427
000700	347.		SU(I) = SU(I)*RULT	002434
000701	348.	398	SOK(I) = SU(I)/QS(I)	002436
000703	349.		GO TO 501	002441
000704	350.	395	CONTINUE	002443
000704	351.	C	FOR CONSTANT SKIN FRICTION	002443
000705	352.		SU(N1)=SU(N1)+RULT-RESULT(IULT-1)*2.0	002443
000706	353.		SOK(N1) = SU(N1)/QS(N1)	002452
000707	354.		GO TO 501	002454
000710	355.	401	IULT=IULT-1	002456
000710	356.	C		002456
000710	357.	C	NOW ONE PROBLEM IS DONE	002456
000710	358.	C	OUTPUT SUMMARY AND	002456
000710	359.	C	FINAL PLOT	002456
000710	360.	C		002456
000711	361.	402	IF (DFIN.LE.0.0) WRITE (IW,23)	002461
000714	362.	4	FORMAT (1H1,12X,10A4/1H0,28X,7HSUMMARY)	002471
000715	363.		WRITE (IW,4) TITLE	002471

000724	364.	23	FORMAT (53H ANALYSIS WAS TERMINATED AS NO PERMANENT SET RESULTED)	002523
000725	365.		WRITE (IW,5)	002523
000727	366.		IF (ITYPH.LT.3) WRITE (IW,40)	002530
000732	367.	40	FORMAT(1H+,57X,8H RLOWS?)	002537
000733	368.		IF (ITYPH.EQ.2) WRITE (IW,30)	002537
000736	369.		WRITE (IW,6)	002546
000740	370.		IF (ITYPH.EQ.2) WRITE (IW,31)	002553
000743	371.		IHP=50	002562
000744	372.		IF (ITYPH.LT.3) IHP=60	002564
000746	373.		IF (ITYPH.LT.3) WRITE (IW,41)	002570
000751	374.	41	FORMAT (1H+,57X,8H MINUTE)	002602
000752	375.		DO 800 I=1,IULT	002602
000755	376.		WRITE (IW,7) I,(RESULT(J),J=I,IHP,10)	002607
000765	377.		IFF = RESULT(60+I)	002623
000766	378.		IF (RESULT(I+10).LT.0.0) RESULT(I+10)=99999999.	002630
000770	379.	800	IF (ITYPH.EQ.2) WRITE (IW,14) IFF,RESULT(I+70)	002636
000776	380.	14	FORMAT (1H+,35X,1H(,I1,1H),27X,F10.1)	002653
000777	381.	30	FORMAT (1H+,67X,8HB.C. PR.)	002653
001000	382.	31	FORMAT (1H+,72X,3HPSI)	002653
001001	383.	5	FORMAT (1H0,55H NO R ULT BLOW CT STROKE MIN STR MAX	002653
001001	384.		1 STR)	002653
001002	385.	6	FORMAT (11X,44HTONS 1/FT FT KSI KSI)	002653
001003	386.	7	FORMAT (16,F10.1,F10.0,3F10.2,F10.1)	002653
001004	387.		IF (MULT.EQ.1) IULT = 0	002653
001006	388.		CALL OUTPUT (OUT,RESULT,INP,IOUT,DTP,JPMAX,JDOUT,IULT,N,MAXJP,	002660
001006	389.		1 ALPH,ISEG,JSELEC,YMAX,COLI,TITLE,STP1,PM(I))	002660
001007	390.	400	CONTINUE	002704
001010	391.	10	FORMAT (IH1)	002704
001010	392.	C		002704
001010	393.	C	GO TO READ NEW PROBLEM	002704
001010	394.	C		002704
001011	395.		GO TO 9877	002704
001012	396.	9876	CONTINUE	002706
001013	397.		WRITE (IWR,10)	002706
001015	398.		END	002714

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1			SUBROUTINE FILL(KK,KLIM,OUT,JMAX,IIOUT,DNH,STH,FINTAC,FTP,INP,	2	8	000004	0
2			1 DNP,STP,SPLICE,VNH,VNP,ANH,ANP,JPMAX,MAXJP,EX,AREA,J,M,N,JEX,JCK)	4	8	000017	5
3			DIMENSION OUT(1),DNH(1),STH(1),INP(1),DNP(1),STP(1),	1	8	000034	3
4			1SPLICE(1),VNH(1),VNP(1),ANH(1),ANP(1),EX(1),AREA(1),JEX(1)	3	8	000046	3
5			KK = KK + 1	1	8	000061	5
6			M1 = M+1	2	8	000064	5
7			KKMAX = (JMAX+1)*KLIM + KK	1	8	000067	2
8	C			7	8	000074	5
9	C		CHECK IF OUT IS FULL	7	8	000076	0
10	C			7	8	000102	5
11			IF (KK.GT.KLIM) GO TO 459	1	8	000104	0
12			K2 = KK + KLIM	1	8	000111	2
13			K3 = K2 + KLIM	1	8	000114	5
14			K4 = K3 + KLIM	1	8	000120	2
15			L = 1	1	8	000123	5
16			IF (IIOUT.NE.1.AND.IIOUT.NE.3) GO TO 452	1	8	000125	5
17	C			7	8	000135	5
18	C		FORCE ASSIGNMENT	7	8	000137	0
19	C			7	8	000143	1

20		OUT(KK) = 0.0		8	8	000153	5
21		IF (M.GT.1) OUT(KK)=(DNH(M-1)-DNH(M))*STH(M-1)		8	8	000157	1
22		OUT(K2) = FINTAC		1	8	000170	1
23		OUT(K3) = FTP		1	8	000174	0
24		L = 2		1	8	000177	2
25		KK1 = INP(2)		1	8	000201	2
26		DO 451 I = K4, KKMAX, KLIM		1	8	000204	3
27		KM1 = KK1-1		1	8	000211	4
28		COM = DNP(KM1)-DNP(KK1)		6	8	000224	1
29		OUT(I) = COM*STP(KK1)		6	8	000231	1
30		IF (OUT(I).LT.SPLICE(KK1)) OUT(I) = SPLICE(KK1)		1	8	000235	5
31		IF (OUT(I).GT.0.0.OR.SPLICE(KK1).LT.--.5) GO TO 686		6	8	000247	0
32		OUT(I) = 0.0		6	8	000260	4
33		IF (COM.LT.SPLICE(KK1)) OUT(I) = (COM-SPLICE(KK1))*STP(KK1)		6	8	000263	5
34		IF (OUT(I).LT.0.0.AND.OUT(I).GT.--.5) OUT(I) = 0.0		6	8	000277	1
35	686	CONTINUE		6	8	000310	4
36		L = L+1		1	8	000314	0
37	451	KK1 = INP(L)		1	8	000316	2
38		GO TO 458		1	8	000322	2
39	452	IF (IIOUT.NE.2) GO TO 454		1	8	000325	0
40	C			7	8	000333	1
41	C	VELOCITY ASSIGNMENT		7	8	000334	2
42	C			7	8	000341	0
43		OUT(KK) = VNH(M)		1	8	000342	1
44		OUT(K2) = VNH(M1)		1	8	000346	0
45		DO 453 I=K3, KKMAX, KLIM		1	8	000352	0
46		KK1 = INP(L)		1	8	000356	5
47		OUT(I) = VNP(KK1)		1	8	000362	0
48	453	L = L + 1		1	8	000366	0
49		GO TO 458		1	8	000371	3
50	454	IF (IIOUT.NE.5) GO TO 456		1	8	000374	1
51	C			7	8	000402	2
52	C	DISPLACEMENT ASSIGNMENT		7	8	000403	3
53	C			7	8	000410	5
54		OUT(KK) = DNH(M)		1	8	000412	0
55		OUT(K2) = DNH(M1)		1	8	000415	5
56		DO 455 I=K3, KKMAX, KLIM		1	8	000421	5
57		KK1 = INP(L)		1	8	000426	4
58		OUT(I) = DNP(KK1)		1	8	000431	5
59	455	L = L+1		1	8	000435	5
60		GO TO 458		1	8	000441	0
61	C			7	8	000443	4
62	C	ACCELERATION ASSIGNMENT		7	8	000444	5
63	C			7	8	000452	1
64	456	OUT(KK) = ANH(M)		1	8	000453	2
65		OUT(K2) = ANH(M1)		1	8	000460	0
66		DO 457 I=K3, KKMAX, KLIM		1	8	000464	0
67		KK1 = INP(L)		1	8	000470	5
68		OUT(I) = ANP(KK1)		1	8	000474	0
69	457	L = L+1		1	8	000500	0
70	458	JPMAX = J		1	8	000503	1
71		MAXJP = KK		1	8	000506	4
72	459	CONTINUE		1	8	000511	3
73	C			1	8	000514	5

74	C	CALCULATE THE EXTREMUM ARRAY EX(N*6)	1	8	000516	0
75	C	FIRST FORCES AND STRESSES	7	8	000526	5
76	C		7	8	000534	3
77		SEX1 = FTP/AREA(1)	4	8	000535	4
78		IF (FTP.GT.EX(1)) JEX(1)=J	3	8	000541	5
79		IF (FTP.GT.EX(1)) EX(1) = FTP	1	8	000547	2
80	C		7	8	000555	3
81	C	JCK LESS 0 MEANS NO MINIMUM FORCE OR MINIMUM STRESS CHECK	7	8	000556	4
82	C		7	8	000572	0
83		IF (JCK.LT.0) GO TO 105	4	8	000573	1
84		IF (FTP.LT.EX(N+1)) JEX(N+1)=J	3	8	000600	1
85		IF (FTP.LT.EX(N+1)) EX(N+1) = FTP	1	8	000606	3
86		IF (SEX1.LT.EX(2*N+1)) JEX(2*N+1)=J	5	8	000634	3
87		IF (SEX1.LT.EX(2*N+1)) EX(2*N+1)=SEX1	5	8	000643	4
88	105	CONTINUE	4	8	000653	1
89		IF (SEX1.GT.EX(3*N+1)) JEX(3*N+1)=J	5	8	000675	3
90		IF (SEX1.GT.EX(3*N+1)) EX(N+3+1)=SEX1	5	8	000704	4
91		DO 461 I=2,N	1	8	000714	1
92		COM = DNP(I-1)-DNP(I)	6	8	000726	1
93		FELP = COM*STP(I)	6	8	000732	5
94		IF (FELP.LT.SPLICE(I)) FELP = SPLICE(I)	1	8	000736	5
95		IF (FELP.GT.0.0.OR.SPLICE(I).LT.-.5) GO TO 685	6	8	000746	4
96		FELP = 0.0	6	8	000757	4
97		IF (COM.LT.SPLICE(I)) FELP = (COM-SPLICE(I))*STP(I)	6	8	000762	3
98		IF (FELP.LT.0.0.AND.FELP.GT.-.5) FELP = 0.0	6	8	000774	2
99	685	CONTINUE	6	8	001004	5
100		SELP = FELP/AREA(I)	1	8	001010	1
101		IF (FELP.GT.EX(I)) JEX(I)=J	3	8	001014	3
102		IF (FELP.GT.EX(I)) EX(I) = FELP	1	8	001022	1
103		IF (JCK.LT.0) GO TO 103	4	8	001030	4
104		IF (FELP.LT.EX(N+1)) JEX(N+1)=J	3	8	001035	4
105		IF (FELP.LT.EX(N+1)) EX(N+1) = FELP	1	8	001044	1
106		IF (SELP.LT.EX(2*N+1)) JEX(2*N+1)=J	3	8	001053	2
107		IF (SELP.LT.EX(2*N+1)) EX(2*N+1) = SELP	1	8	001062	3
108	103	CONTINUE	4	8	001072	2
109		IF (SELP.GT.EX(3*N+1)) JEX(3*N+1)=J	3	8	001075	4
110	461	IF (SELP.GT.EX(3*N+1)) EX(3*N+1) = SELP	1	8	001104	5
111	C		7	8	001115	3
112	C	VELOCITY AND DISPLACEMENT EXTREMA	7	8	001116	4
113	C		7	8	001125	5
114		DO 462 I=1,N	1	8	001127	0
115		IF (VNP(I).GT.EX(4*N+1)) JEX(4*N+1)=J	3	8	001132	1
116		IF (VNP(I).GT.EX(4*N+1)) EX(4*N+1) = VNP(I)	1	8	001141	4
117		DISI = DNP(I) * 12.	1	8	001152	1
118		IF (DISI.GT.EX(5*N+1)) JEX(5*N+1)=J	3	8	001156	3
119	462	IF (DISI.GT.EX(5*N+1)) EX(5*N+1) = DISI	1	8	001165	4
120		RETURN	1	8	001176	1
121		END	1	8	001200	2
1		SUBROUTINE PILEAN (ITP,AOP,VOP,DOP,ANP,VNP,DNP,STP,PM,SPLICE,SJ,	1	10	000004	0
2		1 CDP,RES,RESO,SU,SOK,VNPI,VCAP,SOILM,SUJJ,OTP,N,IT,ISMITH,EPV,	1	10	000020	1
3		2 IAI,ESOIL,FTP,JRAT,ITER)	2	10	000034	4
4		DIMENSION AOP(99),VOP(99),DOP(99),ANP(99),DNP(99),PM(99),STP(99),	3	10	000042	1
5		1 SPLICE(99),SJ(99),CDP(99),RES(99),RESO(99),SU(99),SOK(99),VNP(99)	1	10	000056	3

6	C				8	10	000073	1
7	C	THIS SUBPROGRAM PERFORMS A WAVE ANALYSIS FOR A PILE SUBJECT			8	10	000074	2
8	C	TO EITHER OF THE FOLLOWING THREE PILE TOP VARIABLES			8	10	000110	0
9	C	IAI EQUAL TO 0, HAMMER FORCE USED IN PILEAN			1	10	000122	1
10	C	IAI EQUAL TO 1, MEASURED FORCE USED IN PILEAN			1	10	000133	1
11	C	IAI EQUAL TO 2, MEASURED ACCELERATION USED IN PILEAN			1	10	000144	2
12	C	START TO COMPUTE THE PILE VARIABLES			1	10	000156	4
13	C				1	10	000166	1
14	C	SPLICE IS USUALLY AT THE STRUCTURAL LIMIT OF THE PILE MATERIAL (TENSION, T			1	10	000167	2
15	C	HUS NEGATIV) EXCEPT WHERE SPLICES GIVE LOWER VALUE			1	10	000205	3
16	C	IF SPLICE IS BETWEEN 0 AND -0.5 FT IT IS A SLACK			8	10	000217	3
17	C	IT IS THE ITERATION COUNTER (ASSIGNED OUTSIDE)			8	10	000231	1
18	C	FU AND FO ARE FORCES AT AN ELEMENT DOWNWARDS POSITIVELY			8	10	000242	3
19	C				8	10	000255	3
20		NM1=N-1			1	10	000256	4
21		I TP=0			1	10	000261	0
22		IF (IT.GT.0) GO TO 550			1	10	000263	0
23		I1=1			1	10	000267	5
24	C	PREDICTION			8	10	000271	4
25		IF (IAI.LT.2) CALL INTEGR (DTP,AOP,AOP,VOP,DOP,VNP,DNP,I1)			1	10	000274	5
26		IF (IAI.EQ.2) CALL INTEGR (DTP,AOP,ANP,VOP,DOP,VNP,DNP,I1)			1	10	000310	0
27		G=32.170			1	10	000323	1
28		DO 300 I=2,N			1	10	000325	4
29		CALL INTEGR(DTP,AOP,AOP,VOP,DOP,VNP,DNP,I)			1	10	000330	5
30	300	CONTINUE			1	10	000341	1
31	550	CONTINUE			1	10	000344	3
32	C				8	10	000347	5
33	C	WAVE ANALYSIS			8	10	000351	0
34	C	FIRST PILE TOP			8	10	000354	4
35	C				8	10	000360	3
36	400	FU = (DNP(2)-DNP(1))*STP(2)+(VNP(2)-VNP(I))*CDP(2)			4	10	000361	4
37		FOD = (VCAP-VNP(1))*CDP(1)			4	10	000374	1
38		IF (FOD+FTP.LT.SPLICE(1)) FOD = 0.0			4	10	000401	4
39		IF (IAI.EQ.1) FOD = 0.0			4	10	000410	5
40		IF (FU.GT.-SPLICE(2)) FU = -SPLICE(2)			1	10	000415	5
41		IF (SPLICE(2).LT.-0.5.OR.FU.LT.0.0) GO TO 175			10	10	000444	5
42		EXT = DNP(2)-DNP(1)			5	10	000455	4
43		IF (EXT.LT.0.0) GO TO 175			5	10	000462	0
44		FU = 0.0			5	10	000467	2
45		IF (EXT.LT.-SPLICE(2)) GO TO 175			5	10	000471	5
46		FU = (EXT+SPLICE(2))*STP(2)+(VNP(2)-VNP(I))*CDP(2)			5	10	000500	3
47	175	CONTINUE			5	10	000512	1
48		I = I			4	10	000515	3
49		CALL SRESN (DNP,DOP,SU,SOK,I,RESO,RES,ESOIL)			1	10	000517	3
50		DA = VNP(1)*SJ(I)			1	10	000530	1
51		IF (ISMITH.EQ.1) DA = DA * RES(1)			1	10	000534	1
52		IF (IAI.NE.2) GO TO 200			1	10	000543	0
53		FTP = (ANP(1)-G)*PM(1)-FU-FOD+RES(1) + DA			4	10	000550	0
54		GO TO 201			1	10	000550	1
55	200	CONTINUE			4	10	000562	5
56		ANP(1) = G + (FTP+FOD+FU-RES(1)-DA)/PM(1)			4	10	000566	1
57		CALL INTEGR(DTP,AOP,ANP,VOP,DOP,VNP,DNP,I)			1	10	000576	2
58	C				8	10	000606	4
59	C	ELEMENTS BETWEEN TOP AND TOE			8	10	000607	5

60	C			8	10	000616	1
61	201	DO 301 I=2,NM1		1	10	000617	2
62		I1=I+1		1	10	000623	3
63		IM1=I-1		1	10	000625	4
64		FO = -FU		4	10	000630	0
65		FU = (ONP(I1)-DNP(I1))*STP(I1) + (VNP(I1)-VNP(I))*CDP(I1)		4	10	000632	3
66		IF (FU.GT.-SPLICE(I1)) FU = -SPLICE(I1)		1	10	000645	2
67		IF (SPLICE(I1).LT.-0.5.OR.FU.LT.0.0) GO TO 176		9	10	000672	4
68		EXT = DNP(I1)-DNP(I)		5	10	000703	4
69		IF (EXT.LT.0.0) GO TO 176		5	10	000710	1
70		FU = 0.0		5	10	000715	3
71		IF (EXT.LT.-SPLICE(I1)) GO TO 176		7	10	000740	5
72		FU=(EXT+SPLICE(I1))*STP(I1)+(VNP(I1)-VNP(I))*CDP(I1)		7	10	000747	4
73	176	CONTINUE		6	10	000761	4
74		CALL SRESN (DNP,DOP,SU,SOK,I,RESO,RES,ESOIL)		1	10	000765	0
75		DA = VNP(I) * SJ(I)		1	10	000775	4
76		IF (ISMITH.EQ.1) DA = DA * RES(I)		1	10	001002	0
77		ANP(I) = G + (FO+FU-RES(I)-DA)/PM(I)		4	10	001010	5
78		CALL INTEGR(DTP,AOP,ANP,VOP,DOP,VNP,DNP,I)		1	10	001020	1
79	301	CONTINUE		1	10	001030	3
80	C			8	10	001033	5
81	C	START PILE TOE ANALYSIS		8	10	001035	0
82	C			8	10	001042	2
83		N1=N+1		1	10	001043	3
84		DNP(N1)=DNP(N)		1	10	001045	4
85		CALL SRESN (DNP,DOP,SU,SOK,N1,RESO,RES,ESOIL)		1	10	001051	1
86		IF (RES(N1).LT.0.0) RES(N1)=0.0		1	10	001062	0
87		DAB=VNP(N)*SJ(N1)		1	10	001070	3
88		IF (ISMITH.EQ.1) DAB=DAB*RES(N1)		1	10	001074	3
89		IF (DAB.LT.0.0) DAB=0.0		1	10	001103	1
90		DCUBE = 0.		4	10	001110	1
91		IF (VNP(N).GT.0.0.AND.SJJJ.GT.0.) DCUBE = VNP(N)**3*SJJJ		4	10	001113	0
92		CALL SRESN (DNP,DOP,SU,SOK,N,RESO,RES,ESOIL)		1	10	001125	5
93		DA = VNP(N) * SJ(N)		1	10	001136	3
94		IF (ISMITH.EQ.1) DA = DA*RES(N)		1	10	001142	5
95		FO = -FU		4	10	001151	2
96		ANP(N) = G + (FO-RES(N)-DA-RES(N1)-DAB-DCUBE)/(PM(N)+SOILM)		4	10	001153	5
97		CALL INTEGR(DTP,AOP,ANP,VOP,DOP,VNP,DNP,N)		1	10	001167	1
98		ITP=ITP+1		1	10	001177	3
99		IF (JRAT.EQ.1) GO TO 750		1	10	001202	1
100		IF (ITP.GT.ITER) GO TO 750		1	10	001207	2
101	C	CHECK CONVERGENCE (IF DTH NOT EQ DTP)		8	10	001214	5
102		ERR = ABS(VNPIP-VNP(N))		1	10	001224	4
103		VNPIP = VNP(N)		1	10	001231	4
104	C	FOR NO CONVERGENCE REPEAT ANALYSIS		8	10	001235	1
105		IF (ERR.GT.EPSV) GO TO 400		1	10	001244	3
106	750	CONTINUE		1	10	001252	0
107		RETURN		1	10	001255	2
108		END		1	10	001257	3
1		SUBROUTINE STARTC		1	43	000004	0
2		COMMON VINH,DINH,AOH,VOH,DOH,ANH,VNH,DNH,STH,HH,AM,STA,MA,ESOIL,		1	43	000010	0
3		1 AOP,ANP,DOP,DNP,VOP,VNP,STP,PM,CDP,RES,RESO,SJ,SOK,OS,SO,		1	43	000024	1
4		2 SPLICE,DEPIB,ARAM,ECUS,ECAP,EPT,EANV,VFALL,VCHAM,XPT,SJJJ,		1	43	000037	3

5	3	EMP,RHO,OTH,DTP,N,M,JRAT,IULT,RULT,SOILM,DSACP,	1	43	000053	0
6	4	FPTO,IW,PCOM,TDEL,IOUT,OUT,EXPP,TEMI,TEMO,INP,STROKO,STROKN,VO,PO	1	43	000064	3
7	5	OFIN,JDOUT,JPMAX,EX,ICOL,FINTMX,TEMAX,RESULT,ISMITH,ITER	1	43	000101	1
8	6	ITYPH,IPREIG,VFALLM,STRMAX,DCYL,PBAR,AREA,JEX	27	43	000114	4
9		COMMON/SLAK/OPEN(99)	42	43	000132	3
10		DIMENSION VINH(10),DINH(10),AOH(10),VOH(10),DOH(10),ANH(10),	1	43	000137	0
11	1	VNH(10),DNH(10),STH(10),HM(10),VNP(99),	1	43	000152	3
12	2	AOP(99),ANP(99),DOP(99),DNP(99),VOP(99),STP(99),PM(99),CDP(99),	1	43	000162	5
13	3	RES(99),RESO(99),SJ(99),SOK(99),QS(99),SU(99),SPLICE(99),	1	43	000177	1
14	4	DSACP(3),OUT(3200),INP(13),EX(600),RESULT(100),JEX(600)	27	43	000212	3
15	5	AM(10),STA(10),PBAR(20),AREA(99)	26	43	000225	3
16		IPREIG=0	10	43	000234	4
17		TCC=0.0	10	43	000237	1
18		TEMIK=272.0+TEMI	10	43	000241	3
19		EXP=1.35	10	43	000245	2
20		STTIME = 0.0	31	43	000247	5
21		EXPT=1.0-1.0/EXP	10	43	000253	0
22		PATM=14.7*.144	10	43	000256	5
23		TEMPC=0.0	10	43	000262	2
24		IF (TDEL.GT.1.0) TEMPC=TDEL	10	43	000265	0
25		IF (VFALL.LT.0.0) WRITE (IW,5)	10	43	000272	4
26	5	FORMAT (27H NEGATIVE VELOCITY IN START)	10	43	000301	0
27		IF (PBAR(16).GT.0.5) VFALL=VFALL*SQRT(PBAR(16))	10	43	000311	2
28		IF (VFALL.LT.0.0) STOP	10	43	000322	3
29		M1=M+1	10	43	000327	2
30		M2=M+2	10	43	000331	3
31		N1=N+1	10	43	000333	4
32	C		36	43	000335	5
33	C	INITIALIZE HAMMER AND PILE VARIABLES	36	43	000337	0
34	C		36	43	000346	4
35		DCYL = 0.0	29	43	000347	5
36		DELP = 0.0	30	43	000352	4
37		IF (PBAR(I).GT.ARAM) DELP=(PBAR(I)-ARAM)*0.144*14.7	30	43	000355	3
38		FSUCK=0.0	30	43	000367	2
39		G=32.17	10	43	000372	0
40		RAM=0.0	10	43	000374	2
41		DO 100 I=1,M2	10	43	000376	4
42		IF (I.LT.M1) RAM = RAM + HM(I)	11	43	000402	0
43		VOH(I)=0.0	10	43	000410	2
44		DOH(I)=0.0	10	43	000413	1
45		ANH(I) = 0.0	25	43	000416	0
46	100	AOH(I)=0.0	10	43	000421	1
47		DO 101 I=1,N1	10	43	000424	4
48		RES(I) = 0.0	32	43	000430	0
49		RESO(I)=0.0	10	43	000433	1
50		ANP(I) = 0.0	25	43	000436	1
51		VNP(I) = 0.0	25	43	000441	2
52		DNP(I) = 0.0	25	43	000444	3
53		VOP(I)=0.0	10	43	000447	4
54		DOP(I)=OPEN(I)	42	43	000460	5
55	101	AOP(I)=0.0	10	43	000464	2
56	C		36	43	000467	5
57	C	DETERMINE TIME INCREMENT	36	43	000471	0
58	C		36	43	000476	3

59	STCL = STH(M1)*0.5	14	43	000477	4
60	STP1=STH(M2)*STP(1)/(STH(M2)+STP(1))	10	43	000503	5
61	STP1 = STP1 * 0.5	14	43	000513	1
62	DT1=HM(M1)/STCL	10	43	000517	1
63	DT=HM(M2)/STP1	10	43	000522	5
64	IF (DT.GT.DT1) DT=DT1	10	43	000526	2
65	DT1=HM(M2)/STCL	10	43	000533	0
66	IF (DT.GT.DT1) DT=DT1	10	43	000536	4
67	DT1=PM(1)/STP1	10	43	000543	2
68	IF (DT.GT.DT1) DT=DT1	10	43	000546	5
69	DO 102 I=2,N	10	43	000553	3
70	DT1=PM(I)/(STP(I)+SOK(I))	31	43	000556	4
71	IF (I.EQ.N) DT1=PM(N)/(STP(N)+SOK(N)+SOK(N+1))	31	43	000564	0
72	102 IF (DT.GT.DT1) DT=DT1	10	43	000575	0
73	NM1 = N-1	33	43	000602	2
74	DAM = SJ(N) + SJ(N+1)	33	43	000605	0
75	IF (ISMITH.EQ.1) DAM = SJ(N)*SU(N) + SJ(N+1)*SU(N+1)	33	43	000611	4
76	DAM = DAM/SQRT(PM(N)*STP(N))	33	43	000623	2
77	DO 141 I = 1,N	33	43	000631	2
78	DA = SJ(I)/SQRT(PM(I)*STP(I))	33	43	000634	5
79	IF (ISMITH.EQ.1) DA = DA*SU(I)	33	43	000643	0
80	IF (DA.GT.DAM) DAM = DA	33	43	000651	2
81	141 CONTINUE	33	43	000656	2
82	DAM = DAM*1.2	40	43	000661	4
83	IF (DAM.LT.2.) DAM = 2.	39	43	000665	0
84	DT = SQRT(DT)/DAM	33	43	000672	0
85	IF (DT.LT.0.00001) DT = 0.00001	33	43	000676	0
86	TLIM = 2.5*DEPIB/VFALL	16	43	000704	3
87	DLIM = 0.25*DEPIB	16	43	000711	2
88	IF (DLIM.LT.0.2) DLIM = 0.2	16	43	000713	2
89	C PLACE RAM AT EXHAUST PORTS	36	43	000723	0
90	DTOP=-DEPIB	11	43	000730	5
91	DOH(M)=-DEPIB	10	43	000733	5
92	VOH(M)=VFALL	10	43	000737	1
93	AOH(M)=6	10	43	000742	2
94	DO = VCHAM/ARAM	16	43	000744	5
95	DCU = PM(99)	35	43	000750	3
96	DCT = 0.05*SQRT(STP1*HM(M2))	16	43	000753	4
97	DIN=DEPIB+VCHAM/ARAM	10	43	000761	4
98	N5=N/2	10	43	000766	1
99	IN=0	10	43	000770	2
100	J=0	10	43	000772	1
101	T25 = 0.0	12	43	000773	5
102	FSLW = 0.0	12	43	000776	3
103	D3LC = (3.0*XPT/SQRT(EMP/RHO))*VFALL	36	43	001001	3
104	DACRT = 0.5*DEPIB	35	43	001010	5
105	IF (D3LC.GT.DACRT) DACRT = D3LC	36	43	001014	5
106	C DACRT IS DISTANCE AT WHICH PILE ANALYSIS STARTS	36	43	001023	2
107	IF (YOUT.LT.0) WRITE (IW,1) DT,VFALL,DEPIB	10	43	001034	5
108	1 FORMAT (14H DT, V, DEPIB,F8.5,2F8.2)	10	43	001045	1
109	2 FORMAT(13X,102HFP FC FT V RAM V ANV V TO	32	43	001054	5
110	1P V MID V TOE D CH D TOP TOE RES)	32	43	001066	0
111	IF (IOUT.LT.0) WRITE (IW,2)	12	43	001075	5
112	109 CONTINUE	10	43	001103	3

113		STIME = STIME+DT	31	43	001106 7
114		IN=IN+1	10	43	001112 5
115		J=J+1	10	43	001115 1
116		FSLOW = FSLOW + 0.025	33	43	001117 1
117		IF (FSLOW.GT.1.) FSLOW = 1.0	12	43	001123 5
118		IF (DOH(M).GT.-DACRT) T2S = T2S + 0.025	33	43	001131 5
119		IF (T2S.GT.1.0) T2S = 1.0	12	43	001141 4
120		IT = 1	18	43	001147 0
121	125	CONTINUE	18	43	001151 1
122	C	PREDICTOR INTEGRATION	36	43	001154 2
123		DO 103 I=M,M2	10	43	001161 2
124		VNH(I)=AOH(I)*DT+VOH(I)	10	43	001164 4
125	103	DNH(I)=VOH(I)*DT+DOH(I)	16	43	001171 4
126		IF (T2S.LT.10.E-8) GO TO 114	16	43	001177 2
127		DO 104 I=1,N	10	43	001205 2
128		VNP(I)=AOP(I)*DT+VOP(I)	10	43	001210 3
129	104	DNP(I)=VOP(I)*DT+DOP(I)	16	43	001215 3
130	114	CONTINUE	16	43	001223 1
131		IF (IT.EQ.3) GO TO 112	38	43	001316 0
132	C	START WAVE ANALYSIS	36	43	001322 5
133		DBC = -DNH(M)	29	43	001327 3
134		IF (PBAR(20).GT.0.0) CALL VACHAM(PBAR,ARAM,DEPIB,DCYL,DBC,FSUCK,	30	43	001332 5
135		I VFALL,IOUT,IW)	30	43	001347 0
136		FP=ARAM*PATM*((DIN/(DO+DNH(M1)-DNH(M)))*EXP)	16	43	001352 5
137		FP = FP-FSUCK*DELP	29	43	001363 7
138		FCU = (DNH(M2)-DNH(M1))*STCL+(VNH(M2)-VNH(M1))*DCU	16	43	001367 5
139		FPT = (DNP(I)-DNH(M2))*STP1+(VNP(I)-VNH(M2))*DCT	16	43	001401 3
140		DTOP=DNH(M)-DNH(M1)	11	43	001413 1
141		ANH(M) = G+(-FP+FTR(DTOP,PBAR,IWT))*FSLOW/ARAM	12	43	001417 3
142		ANH(M1) = G + (FP*FSLOW+FCU)/HM(M1)	12	43	001430 2
143		ANH(M2)=G+(-FCU+FPT)/HM(M2)	10	43	001437 3
144		DO 305 I=M,M2	39	43	001445 1
145		VNH(I) = VOH(I) + ANH(I)*DT	39	43	001450 3
146	305	DNH(I) = DOH(I) + (VOH(I) + VNH(I)) * 0.5 * DT	39	43	001456 1
147		IF (T2S.LT.10.E-8) GO TO 112	12	43	001470 0
148		FO = -FPT*T2S	12	43	001476 0
149		DO 105 I=1,N	10	43	001501 2
150		FU = (DNP(I+1)-DNP(I))*STP(I+1)+(VNP(I+1)-VNP(I))*CDP(I+1)	16	43	001504 3
151		RES(I)=RES(I)+(DNP(I)-DOP(I))*SOK(I)	10	43	001517 4
152		IF (RES(I).GT.SU(I)) RES(I)=SU(I)	10	43	001527 1
153		DAM=SJ(I)*VNP(I)	10	43	001536 0
154		IF (ISMITH.EQ.1) DAM=DAM*RES(I)	10	43	001541 5
155		IF (I.EQ.N) GO TO 106	10	43	001550 2
156		ANP(I)=G+(FO+FU-RES(I)-DAM)/PM(I)	10	43	001555 0
157		VNP(I) = VOP(I) + ANP(I) * DT	39	43	001563 5
158		DNP(I) = DOP(I) + (VNP(I) + VOP(I))*DT*0.5	39	43	001572 0
159	105	FO=-FU	10	43	001602 2
160	106	RES(N1)=RES(N1)+(DNP(N)-DOP(N))*SOK(N1)	12	43	001605 1
161		IF (RES(N1).GT.SU(N1)) RES(N1)=SU(N1)	10	43	001615 5
162		DAT = SJ(N1)*VNP(N)	12	43	001625 2
163		IF (ISMITH.EQ.1) DAT=DAT*RES(N1)	10	43	001631 4
164		ANP(N) = G+(FO-RES(N1)-RES(N)-DAT-DAM)/(PM(N)+SOILM)	12	43	001640 2
165		VNP(I) = VOP(I)	39	43	001652 2
166		VNP(N) = VOP(N) + ANP(N)*DT	39	43	001656 0

167		DNP(N) = DOP(N) + (VNP(N) + VOP(N))*0.5*DT	39	43	001663	4
168	112	CONTINUE	12	43	001674	0
169		VNP(N1) = VNP(N)	35	43	001702	2
170		DNP(N1) = DNP(N)	35	43	001706	1
171		DO 107 I=M,M2	10	43	001721	0
172		AQH(I)=ANH(I)	10	43	001724	2
173		VOH(I)=VNH(I)	10	43	001727	4
174	107	DOH(I)=DNH(I)	10	43	001733	0
175		IF (T2S.LT.10.E-8) GO TO 113	12	43	001737	0
176		DO 108 I=1,N1	10	43	001745	0
177		RESO(I)=RES(I)	10	43	001750	2
178		AOP(I)=ANP(I)	10	43	001753	5
179		VOP(I)=VNP(I)	10	43	001757	1
180	108	DOP(I)=DNP(I)	10	43	001762	3
181	113	CONTINUE	12	43	001766	3
182		D1=DOP(1)*12.	10	43	001771	5
183		DCHAM=(DOH(M1)-DOH(M))*12.	10	43	001775	1
184		IF ((IQUI.LT.0.AND.IN.EQ.10) WRITE (IW,6) J,FP,FCU,FPT,VOH(M),	10	43	002002	4
185		1VOH(M1),VOP(1),VOP(N5),VOP(N),DCHAM,D1	10	43	002016	2
186		2,RESO(N1)	32	43	002026	1
187		IF (IN.EQ.10) IN=0	10	43	002031	1
188	6	FORMAT (IS,11F10.3)	32	43	002035	2
189	C	RAM CLOSE ENOUGH AT IMPACT BLOCK FOR IGNITION CHECK	36	43	002042	1
190		IF (-DOH(M).GT.DLIM) GO TO 109	16	43	002054	2
191		PO = FP/ARAM	16	43	002062	4
192		TEMO = TEMIK*(PO/PATM)**EXPT	16	43	002065	5
193		TIMP=1.0	10	43	002073	5
194		IF (VOH(M).GT.VOH(M1)) TIMP=(DOH(M1)-DOH(M))/(VOH(M)-VOH(M1))	10	43	002076	2
195	C		36	43	002112	0
196	C	CHECK WHETHER IGNITION OR IMPACT IS ABOUT TO OCCUR	36	43	002113	1
197	C		36	43	002125	1
198		IF (-TDEL.GT.TIMP-0.001) GO TO 203	10	43	002126	2
199		IF (PBAR(9).GT.0.0.AND.TDEL.LT.10.E-8) GO TO 212	24	43	002135	2
200		IF (TEMPC.GT.1.0.AND.TEMO.GT.TEMPC) GO TO 206	10	43	002146	4
201		IF (-DOH(M).LT.PBAR(9).AND.PBAR(9).GT.0.0) TCC = TCC + DT	24	43	002157	3
202		IF (TCC.GT.TDEL-0.001.AND.PBAR(9).GT.0.0) GO TO 203	24	43	002172	3
203		GO TO 212	10	43	002204	2
204	203	IPREIG=0.001/DTP+0.5	10	43	002207	0
205	C	IGNITION WILL OCCUR AFTER 1 MILLISEC.	36	43	002214	1
206		GO TO 202	10	43	002224	0
207	206	IPREIG = 0.00025/DTP + 0.5	13	43	002226	4
208	C	TEMPERATURE IGNITION AFTER 1/4 MSEC.	36	43	002235	0
209		GO TO 202	10	43	002244	4
210	212	CONTINUE	16	43	002247	2
211		IF (FLOAT(J)*DT.LT.TLIM) GO TO 213	16	43	002252	3
212		WRITE (IW,9)	16	43	002261	3
213		IPREIG = -100	16	43	002264	4
214		GO TO 202	16	43	002270	0
215	213	CONTINUE	16	43	002272	4
216	9	FORMAT (28H *** HAMMER WILL NOT RUN ***)	10	43	002275	5
217		IF (DNH(M1)-DNH(M).GT.0.01*DEPI8) GO TO 109	13	43	002306	1
218	C	FOR 0 TDEL AND ATOMIZED FUEL INJECTION	36	43	002316	4
219	C	(ASSUME AT IMPACT)	36	43	002326	4
220		IF (TCC.GT.10.E-8) IPREIG = (TDEL-TCC)/DTP	24	43	002333	1

221		IF (TCC.GT.10.E-8.AND.IPREIG .LT.1) IPREIG = 1	24	43	002343	3
222	202	MM1=M-1	10	43	002354	3
223		FPT0 = -FPT	24	43	002357	3
224		VO = -FCU	24	43	002362	3
225		DO 111 I=M,M2	10	43	002365	1
226		DINH(I)=DOH(I)	10	43	002370	3
227	111	VINH(I)=VOH(I)	10	43	002374	0
228		DO 110 I1=1,MM1	10	43	002400	1
229		I=M-I1	10	43	002403	5
230		VINH(I)=VOH(I)	10	43	002406	0
231	110	DINH(I) = DINH(I+1) + FP/STH(I)	13	43	002411	3
232		FINMX = STTIME	32	43	002420	4
233		RETURN	10	43	002424	2
234		END	10	43	002426	3

FILL2		SYMBOLIC FORTRAN 28 APR 76 15:09:57 9 13 5 00122703				519
1		SUBROUTINE FILL2 (N,M,OUT,PQ,DNH,FTP,VNP,DNP,FN05,SJ,RSUM,DSUM,	1	13	000004	0
2		1 RES,FTN,KK,ICOL,TT,ISMITH,J,MAXJP,JPMAX,STP,AREA,SPLICE,EX,JEX,	8	13	000020	0
3		2 J1,JCK)	8	13	000034	2
4		DIMENSION OUT(3200),DNH(10),VNP(99),DNP(99),RES(99),SJ(99),	6	13	000037	0
5		1 SPLICE(99),STP(99),AREA(99),EX(600),JEX(600)	6	13	000052	2
6		N1=N+1	2	13	000063	2
7		M1=M+1	1	13	000065	3
8		N05=N/2	1	13	000067	4
9		KK=KK+1	3	13	000072	0
10		KLIM=200	6	13	000074	2
11	C		13	13	000076	5
12	C	CHECK IF OUT IS FULL	13	13	000100	0
13	C		13	13	000104	5
14		IF (KK.GT.KLIM) GO TO 102	3	13	000106	0
15		K1=KK+KLIM	3	13	000113	2
16		K2=K1+KLIM	3	13	000116	1
17		K3=K2+KLIM	3	13	000121	0
18		K4=K3+KLIM	3	13	000123	5
19		K5=K4+KLIM	3	13	000126	4
20		K6=K5+KLIM	3	13	000131	3
21		K7=K6+KLIM	3	13	000134	2
22		K8=K7+KLIM	3	13	000137	1
23		K9=K8+KLIM	3	13	000142	0
24		K10=K9+KLIM	3	13	000144	5
25		K11=K10+KLIM	3	13	000147	5

26		K12=K11+KLIM	3	13	000153	0
27		K13=K12+KLIM	3	13	000156	1
28		K14=K13+KLIM	3	13	000161	2
29		K15=K14+KLIM	3	13	000164	3
30	C		13	13	000167	4
31	C	ASSIGN VARIABLES TO OUT	13	13	000170	5
32	C		13	13	000176	1
33		OUT(KK)=TT	3	13	000177	2
34		OUT(K1)=PQ	3	13	000202	1
35		OUT(K2)=DNH(M)*12.	3	13	000205	0
36		OUT(K3)=DNH(M1)*12.	3	13	000211	1
37		OUT(K4)=FTP	3	13	000215	3
38		OUT(K5)=VNP(1)	3	13	000220	3
39		OUT(K6)=DNP(1)*12.	3	13	000224	0
40		OUT(K7)=FN05	3	13	000230	1
41		OUT(K8)=VNP(N05)	3	13	000233	2
42		OUT(K9)=DNP(N05)*12.	3	13	000237	1
43		OUT(K10)=RSUM	3	13	000243	4
44		OUT(K11)=DSUM	3	13	000247	0
45		RTOE = SJ(N1)*VNP(N)	13	13	000256	4
46		IF (RTOE.LT.0.0001) RTOE=0.	1	13	000263	1
47		IF (ISMITH.EQ.1) RTOE=RTOE*RES(N1)	1	13	000270	5
48		OUT(K12)=RES(N1)+RTOE	3	13	000277	5
49		OUT(K13)=FTN	3	13	000304	
50		OUT(K14)=VNP(N)	3	13	000307	4
51		OUT(K15)=DNP(N)*12.	3	13	000313	2
52		IF (ICDL.NE.0) GO TO 101	3	13	000317	4
53		OUT(K7)=OUT(K13)	3	13	000324	5
54		OUT(K8)=OUT(K14)	3	13	000330	
55		OUT(K9)=OUT(K15)	3	13	000330	
56	101	JPMAX=J	3	13	000340	
57		MAXJP=KK	3	13	000343	2
58	C	CALCULATE THE EXTREMUM ARRAY EX(N*6)	4	13	000345	5
59	102	CONTINUE	6	13	000355	3
60		SEX1=FTP/AREA(1)	8	13	000360	4
61		IF (FTP.GT.EX(1)) JEX(1)=J1	6	13	000364	3
62		IF (FTP.GT.EX(1)) EX(1)=FTP	6	13	000372	1
63	C		13	13	000377	5
64	C	JCK LESS 0 MEANS NO MINIMUM FORCE AND STRESS CHECK	13	13	000401	0
65	C		13	13	000413	0
66		IF (JCK.LT.0) GO TO 105	8	13	000414	1
67		IF (FTP.LT.EX(N+1)) JEX(N+1)=J1	6	13	000421	1
68		IF (FTP.LT.EX(N+1)) EX(N+1)=FTP	4	13	000427	4
69		IF (SEX1.LT.EX(2*N+1)) JEX(2*N+1)=J1	6	13	000436	1
70		IF (SEX1.LT.EX(2*N+1)) EX(2*N+1)=SEX1	4	13	000445	3
71	105	CONTINUE	9	13	000455	0
72		IF (SEX1.GT.EX(3*N+1)) JEX(3*N+1) = J1	10	13	000477	1
73		IF (SEX1.GT.EX(3*N+1)) EX(3*N+1) = SEX1	10	13	000506	5
74		DO 103 I=2,N	4	13	000516	4
75		COM = DNP(I-1)-DNP(I)	11	13	000530	0
76		FELP = COM*STP(I)	11	13	000534	4
77		IF (FELP.LT.SPLICE(I)) FELP=SPLICE(I)	4	13	000540	4
78		IF (FELP.GT.0.0.OR.SPLICE(I).LT.--.5) GO TO 685	11	13	000550	1
79		FELP = 0.0	11	13	000561	1

80		IF (COM.LT.SPLICE(I)) FELP = (COM-SPLICE(I))*STP(I)	11	13	000564	0
81		IF (FELP.LT.0.0.AND.FELP.GT.-0.5) FELP=0.0	11	13	000575	5
82	685	CONTINUE	11	13	000606	1
83		SELP=FELP/AREA(I)	4	13	000630	1
84		IF (FELP.GT.EX(I)) JEX(I)=J1	6	13	000634	1
85		IF (FELP.GT.EX(I)) EX(I)=FELP	4	13	000642	1
86		IF (JCK.LT.0) GO TO 106	8	13	000650	2
87		IF (FELP.LT.EX(N+I)) JEX(N+I)=J1	6	13	000655	2
88		IF (FELP.LT.EX(N+I)) EX(N+I)=FELP	4	13	000664	0
89		IF (SELP.LT.EX(2*N+I)) JEX(2*N+I)=J1	6	13	000672	4
90		IF (SELP.LT.EX(2*N+I)) EX(2*N+I)=SELP	4	13	000702	0
91	106	CONTINUE	8	13	000711	3
92		IF (SELP.GT.EX(3*N+I)) JEX(3*N+I)=J1	6	13	000714	5
93	103	IF (SELP.GT.EX(3*N+I)) EX(3*N+I)=SELP	4	13	000724	1
94		DO 104 I=1,N	4	13	000734	2
95		IF (VNP(I).GT.EX(4*N+I)) JEX(4*N+I)=J1	6	13	000737	3
96		IF (VNP(I).GT.EX(4*N+I)) EX(4*N+I)=VNP(I)	4	13	000747	1
97		DISI=DNP(I)*12.	4	13	000757	2
98		IF (DISI.GT.EX(5*N+I)) JEX(5*N+I)=J1	6	13	000763	0
99	104	IF (DISI.GT.EX(5*N+I)) EX(5*N+I)=DISI	4	13	000772	2
100		RETURN	4	13	001002	3
101		END	1	13	001004	4

84

000101	1.	SUBROUTINE AIRSTM			000000	
000103	2.	COMMON VINH,DINH,AOH,VOH,DOH,ANH,VNH,DNH,STH,HM,AM,STA,MA,ESOIL,			000000	
000103	3.	1 AOP,ANP,DOP,DNP,VOP,VNP,STP,PM,CDP,RES,RESO,SJ,SOK,GS,SU,			000000	
000103	4.	2 SPLICE,DEPIB,ARAM,ECUS,EASS,EPT,EANV,VFALL,VCHAM,XPT,SJJJ,			000000	
000103	5.	3 EMP,RHO,DTH,DTP,N,M,JRAT,IULT,RULT,SOILM,DSACP,			000000	
000103	6.	4 FTPO,IW,PCOM,IOEL,IOUT,OUT,EXPP,TEMI,TEMO,INP,STROK0,STROKN,VO,PO			000000	
000103	7.	5 ,DFIN,JDOUT,JPMAX,EX,ICOL,FINTMX, TEMAX,RESULT,ISMITH,ITER			000000	
000103	8.	6 ,ITYPH,IPREIG,VFALLM,STRMAX,DCYL,PBAR,AREA,JEX			000000	
000104	9.	DIMENSION VINH(10),DINH(10),AOH(10),VOH(10),DOH(10),ANH(10),			000000	
000104	10.	1 VNH(10),DNH(10),STH(10),HM(10),VNP(99),AM(10),STA(10),VOA(10),			000000	
000104	11.	2 DOA(10),AOA(10),VNA(10),DNA(10),ANA(10),DELA(10),			000000	
000104	12.	3 AOP(99),ANP(99),DOP(99),DNP(99),VOP(99),STP(99),PM(99),CDP(99),			000000	
000104	13.	4 RES(99),RESO(99),SJ(99),SOK(99),GS(99),SU(99),SPLICE(99),			000000	
000104	14.	4DSACP(3),OUT(3200),INP(13),EX(600),RESULT(100),JEX(600)			000000	
000104	15.	6 ,AOAH(99),VOAH(99),DOAH(99),AOAA(99),AREA(99),DELAH(99)			000000	
000104	16.	7 ,VOAA(10),AOAA(10),DOAA(10),PBAR(20)			000000	
000105	17.	KLIM = 200			000000	
000106	18.	N1=N+1			000001	
000107	19.	NM1 = N-1			000004	
000110	20.	M1 = M+1			000007	
000111	21.	MAR = MA			000012	
000112	22.	IF (MAR.LT.1) MAR = 1			000014	

000114	23.	IAI = 0.0	000023
000115	24.	ML = MA	000024
000116	25.	CDH = PM(99)	000026
000117	26.	IF (M1.GT.ML) ML = M1	000030
000121	27.	TLIM = 0.0	000101
000122	28.	IIOUT = IABS(IOUT)	000102
000123	29.	TWOLOC=2.0*XPT/SQRT(EMP/RHO)	000104
000124	30.	TZLC = TWOLOC*1000.	000116
000125	31.	IF (IIOUT.GT.19) IIOUT = IIOUT - 20	000120
000127	32.	IF (IIOUT.GT. 9) IIOUT = IIOUT - 10	000130
000131	33.	JDOUT = 3.0*TWOLOC/DTP	000140
000132	34.	JDOUT = JDOUT / 100	000147
000133	35.	TJD = FLOAT(JDOUT)*DTP*1000.	000153
000133	36.	C TJD IS TIME INCREMENT OF OUTPUT IN MILLISECONDS	000153
000134	37.	IF (TJD.GT.0.5) JDOUT = (0.0005/DTP) + 0.5	000161
000136	38.	IF (JDOUT.LT.1) JDOUT=1	000175
000140	39.	JMAX = 6	000204
000140	40.	C 6 COLUMNS OF PILE OUTPUT IF ICOL = 0 13 OTHERWISE IF NOT GTR N	000204
000141	41.	IF (ICOL.GT.0) JMAX=13	000206
000143	42.	IF (JMAX.GT.N) JMAX=N	000214
000145	43.	KK=0	000223
000146	44.	RED = 0.9	000224
000147	45.	IF (EMP.LT.3000000.) RED = 0.0	000226
000151	46.	SCJ = 0.0	000234
000152	47.	WRAM = 0.0	000235
000153	48.	G = 32.170	000236
000154	49.	DO 902 I=1,M	000243
000157	50.	902 WRAM = WRAM + HM(I)*G	000243
000161	51.	WASS=0.0	000247
000162	52.	DO 905 I=1,MAR	000253
000165	53.	905 WASSE=WASS+AM(I)*G	000253
000167	54.	FAOA=WASS	000257
000170	55.	GAS = G*1.5	000260
000171	56.	IF (WASS.GT.0.0) GAS = (1.-SJJJ)*G*(1.+WRAM/WASS)	000263
000173	57.	GR = G	000277
000174	58.	IF (MA.LT.1) FAOA = 0.0	000301
000176	59.	FATE = 0.0	000307
000177	60.	FAO = 0.0	000310
000200	61.	FTPO=WASS+HM(M1)*G	000311
000201	62.	FTIT = FTPO	000316
000202	63.	FIPOA = FTPO	000317
000203	64.	FANVOA = 0.0	000320
000204	65.	VNPIT = VOP(N)	000321
000205	66.	VNHIT = VINH(M)	000324
000206	67.	TBEF=0.002	000327
000207	68.	VFALLB=VFALL-G*TBEF	000331
000210	69.	VINH(M1) = 0.0	000334
000211	70.	DO 903 I=1,MAR	000340
000214	71.	DECLAAT(I) = 0.0	000340
000215	72.	VOAA(I) = 0.0	000340
000216	73.	903 A0AA(I) = 0.0	000341
000220	74.	DNPMAX = 0.0	000343
000221	75.	MMI=M-1	000344
000222	76.	MAM1 = MA-1	000347

000223	77.	ENTHRU = 0.0	000352
000224	78.	EMAX = 0.0	000353
000225	79.	NT = 5.0*TWOLOC/DTP	000354
000226	80.	JCK = 1.5*TWOLOC/DTP+0.003/DTP	000364
000227	81.	IF (FLOAT(NT)*OTP.LT.0.050) NT = 0.050/DTP	000377
000231	82.	IF (TEMAX.GT.1.0) NT = TEMAX*0.001/DTP	000415
000233	83.	STPT1 = STH(M1)*STP(1)/(STH(M1)+STP(1))	000431
000234	84.	STPT2 = (STH(M1)/ECUS)*(STP(1)/EPT)/((STH(M1)/ECUS)+STP(1)/EPT)	000440
000235	85.	STAL = STH(M)	000446
000236	86.	STAR = STAL/EANV	000451
000237	87.	STASL = STA(MAR)	000453
000240	88.	STASR = STASL/EASS	000456
000241	89.	WP=0.0	000460
000242	90.	DO 51 I=1,NM1	000464
000245	91.	51 WP=WP+PM(I)*G	000464
000247	92.	DO 98 I=N,N1	000474
000252	93.	DOP(I)=0.0	000474
000253	94.	VOP(I)=0.0	000474
000254	95.	AOP(I)=G	000475
000255	96.	98 RESO(I)=0.0	000477
000257	97.	DO 52 I=2,N	000504
000262	98.	I1=N1-I	000504
000263	99.	DOP(I1)=DOP(I1+1)+((FTPO+WP)/STP(I1+1))*FLOAT(I)/FLOAT(N)	000507
000264	100.	VOP(I1)=0.0	000522
000265	101.	AOP(I1)=G	000523
000266	102.	RESO(I1)=0.0	000525
000267	103.	52 WP=WP-PM(I1)*G	000526
000271	104.	DINH(M1)=DOP(1)+SQRT(FTPO*DSACP(2)/STPT1)	000533
000272	105.	MAI=MA+1	000545
000273	106.	IF (MA.LT.1) GO TO 1906	000550
000275	107.	DOAA(MA)=DINH(M1)+SQRT(WASS*DSACP(3)/STA(MA))	000555
000276	108.	W=WASS-AM(MA)*G	000567
000277	109.	IF (MA.LT.2) GO TO 1906	000573
000301	110.	DO 906 I=2,MAM1	000600
000304	111.	I1=MA1-I	000605
000305	112.	DOAA(I1)=DOAA(I1+1)+W/STA(I1)	000611
000306	113.	906 W=W-AM(I1)*G	000615
000310	114.	1906 CONTINUE	000624
000311	115.	DO 901 I=1,M	000624
000314	116.	VINH(I)=VFALLB	000657
000315	117.	901 DINH(I)=-0.5*(VFALLB+VFALL)*TBEP+DINH(M1)	000660
000317	118.	DO 100 I=1,M1	000667
000322	119.	VOAH(I)=VINH(I)	000667
000323	120.	DOAH(I)=DINH(I)	000670
000324	121.	DELAH(I) = 0.0	000672
000325	122.	AQAH(I)=G	000673
000326	123.	100 CONTINUE	000676
000330	124.	EPSV = 0.1	000676
000331	125.	EPSF = 1.0	000700
000332	126.	LIMOUT=KLIM*16	000702
000333	127.	DO 450 I=1,LIMOUT	000705
000336	128.	450 OUT(I)=0.0	000711
000340	129.	DO 460 J=1,6	000715
000343	130.	DO 460 I=1,N	000724

000346	131.		IJ=N*(J-1)+I	000724
000347	132.		EX(IJ) = 0.0	000727
000350	133.		JEX(IJ) = 0	000731
000351	134.	460	CONTINUE	000734
000354	135.		FINTMX=0.0	000737
000355	136.		IF (IOUT.LT.0) WRITE (IW,653)	000750
000360	137.		VIPMAX = 0.0	000751
000361	138.		VINC=(TDEF-0.001)/DTP	000751
000361	139.	C		000751
000361	140.	C	START MAIN LOOP	000751
000361	141.	C		000751
000362	142.		DO 700 J=1,NT	000767
000365	143.		IF (J.GT.JCK) JCK=-1	000767
000367	144.		TT = FLOAT(J)*DTP*1000.0	000776
000370	145.		IT=0	001004
000371	146.		IF (VIPMAX.LT.VNP(1)) VIPMAX = VNP(1)	001005
000373	147.		IF (RED.LT.0.01) GO TO 103	001014
000375	148.		IF (VNP(1).GT.0.9*VIPMAX.OR.VIPMAX.LT.5.0) GO TO 103	001021
000377	149.		STPT1 = STPT2 * RED	001040
000400	150.		RED = 0.0	001043
000401	151.	103	CONTINUE	001045
000402	152.	800	CONTINUE	001045
000403	153.		FA0 = FAOA	001053
000404	154.		FTPO = FTPOA	001055
000405	155.		FANVO = FANVOA	001057
000406	156.		DO 99 I=1,ML	001067
000411	157.		VOA(I) = VOAA(I)	001067
000412	158.		AOA(I) = AOAA(I)	001070
000413	159.		DOA(I) = DOAA(I)	001072
000414	160.		VOH(I)=VOAH(I)	001074
000415	161.		AOH(I)=AOAH(I)	001076
000416	162.	99	DOH(I)=DOAH(I)	001100
000416	163.	C		001100
000416	164.	C	HAMMER ANALYSIS	001100
000416	165.	C		001100
000420	166.		DO 600 JH=1,JRAT	001106
000423	167.		YTH=0	001106
000423	168.	C	PREDICTION	001112
000424	169.		DO 585 I=1,ML	001112
000427	170.		IF (IT.EQ.0) GO TO 909	001112
000431	171.		ANH(I) = AOH(I)+FLOAT(JH)*DELAH(I)	001113
000432	172.		ANA(I) = AOA(I)+FLOAT(JH)*DELA(A I)	001122
000433	173.		GO TO 585	001126
000434	174.	909	CONTINUE	001130
000435	175.		ANH(I) = ANH(I)	001130
000436	176.		ANA(I) = ANA(I)	001131
000437	177.	585	CONTINUE	001135
000441	178.		DO 586 I=1,MAR	001135
000444	179.	586	CALL INTEGR(DTH,AOA,ANA,VOA,DOA,VNA,DNA,I)	001145
000446	180.		DO 101 I=1,M1	001164
000451	181.	101	CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	001164
000453	182.	500	CONTINUE	001201
000454	183.		I=1	001206
000455	184.		IF (VIPMAX.GT.1.5) G=6*(-SJJJ)	001211

000457	185.		IF (M.EQ,1) GO TO 904	
000461	186.		ANH(1)=G+(DNH(2)-DNH(1))*STH(1)/HM(1)	001216
000462	187.		CALL INTEGR (DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	001222
000463	188.		IF (M.EQ,2) GO TO 904	001230
000465	189.		DO 102 I=2,MM1	001242
000470	190.		I1=I+1	001246
000471	191.		IM1=I-1	001253
000472	192.		ANH(I)=G+((DNH(I1)-DNH(I))*STH(I)+(DNH(IM1)-DNH(I))*STH(IM1))/HM(I	001257
000472	193.		1)	001263
000473	194.		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,I)	001263
000474	195.	102	CONTINUE	001275
000476	196.	904	CONTINUE	001312
000476	197.	C	LAST RAM ELEMENT	001312
000477	198.		FANV = 0.0	001312
000500	199.		DANV = 0.0	001313
000501	200.		DDIS = DNH(M1)-DNH(M)	001314
000502	201.		IF (DDIS.GT.0.0) GO TO 811	001315
000504	202.		DVEL = VNH(M1) - VNH(M)	001320
000505	203.		DANV = COM*DVEL	001323
000506	204.		DSAC = DSACP(1)	001326
000507	205.		DDISO = DOH(M1)-DOH(M)	001330
000510	206.		DDD = DDIS - DDISO	001332
000510	207.	C	ANVIL SPRING FORCE ALWAYS TENSION= +	001335
000511	208.		CALL STIFF (DDIS,DVEL,STAL,STAR,FANVO,FANV,DSAC,DDD)	001335
000512	209.		IF (FANV.GT.0.0) FANV = 0.0	001337
000514	210.	811	CONTINUE	001351
000515	211.		FINTAC = FANV	001357
000516	212.		IF (TLIM.LT.0.0001.AND.FINTAC.LT.10.0) TLIM = TT+T2LC*1.15	001357
000520	213.		FTR = 0.0	001362
000521	214.		IF (M.GT.1) FTR = (DNH(MM1)-DNH(M))*STH(MM1)	001404
000523	215.		ANH(M) = G + (FTR+FINTAC+DANV)/HM(M)	001405
000524	216.		G = GR	001417
000525	217.		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,M)	001426
000526	218.		FINTAC = -FINTAC	001430
000526	219.	C	NOW CHECK TOTAL ASSEMBLY. THERE ARE AT LEAST TWO ASSEMBLY ELEMENTS	001442
000527	220.		IF (MA.LT.1) GO TO 911	001442
000531	221.		IF (VIPMAX.GT.1.5) G = GAS	001444
000533	222.		I = 1	001451
000534	223.		ANA(I)=G+(DNA(2)-DNA(1))*STA(1)/AM(1)	001464
000535	224.		CALL INTEGR (DTH,AOA,ANA,VOA,DOA,VNA,DNA,I)	001466
000536	225.		IF (MA.LT.3) GO TO 912	001474
000540	226.		DO 907 I=2,MAM1	001506
000543	227.		I1 = I+1	001513
000544	228.		IM1 = I-1	001520
000545	229.		ANA(I)=G+((DNA(I1)-DNA(I))*STA(I)+(DNA(IM1)-DNA(I))*STA(IM1))/	001524
000545	230.		1 AM(I)	001530
000546	231.		CALL INTEGR(DTH,AOA,ANA,VOA,DOA,VNA,DNA,I)	001530
000547	232.	907	CONTINUE	001542
000551	233.	912	CONTINUE	001557
000552	234.		FA = 0.0	001557
000552	235.	C	LAST ASSEMBLY ELEMENT	001560
000553	236.		DDIS = DNH(M1)-DNA(MA)	001560
000554	237.		IF (DDIS.GT.0.0) GO TO 908	001561
000556	238.		DVEL=VNH(M1)-VNA(MA)	001564
				001567

000557	239.		DSAC = DSACP(3)	001572
000560	240.		DDISO = DOH(M1)-DOA(MA)	001574
000561	241.		DDD = DDIS-DDISO	001577
000562	242.		FAO = -FAO	001601
000563	243.		CALL STIFF(DDIS,DVEL,STASL,STASR,FAO,FA,DSAC,DDD)	001603
000564	244.		IF (FA.GT.0.0) FA = 0.0	001615
000566	245.		FAO = -FAO	001622
000567	246.	908	CONTINUE	001625
000570	247.		IF (MA.EQ.1) ANA(1)=G+FA/AM(1)	001626
000572	248.		IF (MA.GT.1)	001636
000572	249.		1 ANA(MA)=G+((DNA(MAM1)-DNA(MA))*STA(MAM1)+FA)/AM(MA)	001636
000574	250.		CALL INTEGR(DTH,AOA,ANA,VOA,DOA,VNA,DNA,MA)	001653
000575	251.		FA = -FA	001665
000576	252.		G = GR	001667
000577	253.	911	CONTINUE	001672
000577	254.	C	NOW ANALYZE CAP	001672
000600	255.		DTI=FLOAT(JH)*DTH	001676
000601	256.		I=1	001703
000602	257.		DNPI=DOP(1)+(DTI/DTP)*(DNP(1)-DOP(1))	001705
000603	258.		VNPI=VOP(1)+(DTI/DTP)*(VNP(1)-VOP(1))	001713
000604	259.		IF (IT.GT.0) GO TO 575	001722
000606	260.		CALL INTEGR(DTI,AOP,AOP,VOP,DOP,VNP,DNP,I)	001723
000607	261.		DNPI = DNP(1)	001735
000610	262.		VNPI = VNP(1)	001737
000611	263.	575	CONTINUE	001742
000612	264.		FTP = 0.0	001742
000613	265.		DDIS = DNPI-DNH(M1)	001743
000614	266.		IF (DDIS.GT.0.0) GO TO 813	001746
000616	267.		DVEL = VNPI-VNH(M1)	001751
000617	268.		DSAC = DSACP(2)	001754
000620	269.		DDISO = DOP(1)-DOH(M1)	001756
000621	270.		DDD = DDIS-DDISO	001761
000622	271.		FTPO = -FTPO	001763
000623	272.		CALL STIFF(DDIS,DVEL,STPT1,STPT2,FTPO,FTP,DSAC,DDD)	001765
000624	273.		IF (FTP.GT.-SPLICE(I)) FTP = -SPLICE(I)	001777
000626	274.		FTPO = -FTPO	002006
000627	275.	813	CONTINUE	002011
000630	276.		ANH(M1) = G+(FINTAC+FA+FTP-DANV)/HM(M1)	002011
000631	277.		CALL INTEGR(DTH,AOH,ANH,VOH,DOH,VNH,DNH,M1)	002020
000632	278.		ITH = ITH +1	002033
000633	279.		FTP = -FTP	002036
000633	280.	C	CHECK TO SEE IF HAMMER HAS CONVERGED	002036
000634	281.		IF (JRAT.EQ.1) GO TO 650	002040
000636	282.		IF (ITH.GT.ITER) GO TO 650	002044
000640	283.		ERR = ABS(VNHIT-VNH(M))	002050
000641	284.		VNHIT = VNH(M)	002054
000642	285.		IF (ERR.GT.EPSV) GO TO 500	002056
000644	286.	650	CONTINUE	002063
000645	287.		DO 501 I=1,ML	002063
000650	288.		DELAA(I) = (ANA(I)-AOA(I))/FLOAT(JRAT)	002074
000651	289.		AOA(I) = ANA(I)	002077
000652	290.		VOA(I) = VNA(I)	002101
000653	291.		DOA(I) = DNA(I)	002103
000654	292.		AOH(I)=ANH(I)	002105

000655	293.		V0H(I)=VNH(I)	002107
000656	294.	501	D0H(I)=DNH(I)	002111
000660	295.		VCAP = VNH(M1)	002114
000661	296.		FANVO = FANV	002117
000662	297.		FTPO = FTP	002121
000663	298.		FAO = FA	002123
000664	299.	600	CONTINUE	002127
000664	300.	C		002127
000664	301.	C	PILE ANALYSIS	002127
000664	302.	C		002127
000666	303.		CALL PILEAN (IPT,AOP,VOP,DOP,ANP,VNP,DNP,STP,PM,SPLICE,SJ,CDP,RES,	002127
000666	304.		1 RESO,SU,SOK,VNPIT,VCAP,SOILM,SCJ,DTP,N,IT,ISMITH,EPSV,IAI,ESOIL,	002127
000666	305.		2 FTP,JRAT,ITER)	002127
000666	306.	C	CHECK TO SEE IF OVERALL CYCLE CONVERGED	002127
000667	307.		IT = IT + 1	002170
000670	308.		IF (IT.GT.ITER) GO TO 850	002173
000672	309.		ERRV = ABS(VNPIT - VNP(N))	002177
000673	310.		IF (JRAT.EQ.1) VNPIT = VNP(N)	002204
000675	311.		IF (JRAT.EQ.1.AND.ERRV.GT.EPSV) GO TO 800	002213
000677	312.		ERR = ABS(FTIT - FTP)	002230
000700	313.		FTIT = FTP	002234
000701	314.		IF (ERR.GT.EPSF) GO TO 800	002236
000703	315.	850	CONTINUE	002243
000703	316.	C		002243
000703	317.	C	GO TO 800 MEANS REPEAT	002243
000703	318.	C	GO TO 850 MEANS SUFFICIENTLY MANY ITERATIONS WERE PERFORMED	002243
000703	319.	C		002243
000704	320.		RSUM = 0.0	002243
000705	321.		DSUM = SJ(N1)*VNP(N)	002243
000706	322.		IF (ISMITH.EQ.1) DSUM = RES(N1)*DSUM	002247
000710	323.		IF (DSUM.LT.0.0) DSUM = 0.0	002263
000712	324.		VNPMAX = 0.0	002302
000713	325.		DO 401 I=1,N1	002306
000716	326.		RESO(I)=RES(I)	002306
000717	327.		AOP(I)=ANP(I)	002310
000720	328.		VOP(I)=VNP(I)	002312
000721	329.		IF (ABS(VOP(I)).GT.VNPMAX) VNPMAX = ABS(VOP(I))	002314
000723	330.		RSUM = RSUM + RES(I)	002323
000724	331.		DDAM = SJ(I)*VNP(I)	002326
000725	332.		IF (ISMITH.EQ.1) DDAM = DDAM*RES(I)	002331
000727	333.		IF (I.NE.N1) DSUM = DSUM + DDAM	002336
000731	334.	401	DOP(I)=DNP(I)	002345
000733	335.		IF (DNP(N).GT.DNPMAX) DNPMAX = DNP(N)	002351
000735	336.		FANVOA = FANVO	002361
000736	337.		FAOA = FAO	002363
000737	338.		FTPOA = FTPO	002365
000737	339.	C	SET OLD EQUAL NEW	002365
000740	340.		DO 402 I=1,ML	002372
000743	341.		AOAA(I) = ANA(I)	002372
000744	342.		VOAA(I) = VNA(I)	002373
000745	343.		DOAA(I) = DNA(I)	002375
000746	344.		AOAH(I)=ANH(I)	002377
000747	345.		VOAH(I)=VNH(I)	002401
000750	346.	402	DOAH(I)=DNH(I)	002403

000752	347.		DOPINC = DOP(1)*12.	002406
000753	348.		DRA = (DNH(M1)-DNH(M))*12.0	002411
000754	349.		SEA = (DNH(M1)-DNA(MAR))*12.0	002417
000755	350.		DBO = DNP(N)	002423
000756	351.		DEN = VOP(1)*DTP*FTP	002426
000757	352.		ENTHRU = ENTHRU + DEN	002432
000760	353.		IF (ENTHRU.GT.EMAX) EMAX = ENTHRU	002434
000762	354.		JOUT = JOUT + 1	002443
000763	355.		IF (JOUT.NE.JDOUT) GO TO 699	002446
000765	356.		JOUT = 0	002451
000766	357.		FTP0=FTP	002452
000766	358.	C	OUTPUT FOR NEG. OPTION ONLY	002452
000767	359.		NO2 = N/2	002458
000770	360.		NO21 = NO2+1	002461
000771	361.	652	FORMAT (1H,14,12,3I1,3F6.1,F6.2,2F7.0,2F6.1,2(F6.1,F6.1),3F8.1,	002470
000771	362.		1 2F6.1,F7.2)	002470
000772	363.		DO 686 I=NO2,NM1	002470
000775	364.		I1 = I+1	002470
000776	365.		COM = DNP(I)-DNP(I1)	002474
000777	366.		F = COM*STP(I1)	002477
001000	367.		IF (F.LT.SPLICE(I1)) F = SPLICE(I1)	002502
001002	368.		IF (F.GT.0.0.OR.SPLICE(I1).LT.-.5) GO TO 685	002512
001004	369.		F = 0.0	002530
001005	370.		IF (COM.LT.SPLICE(I1)) F = (COM-SPLICE(I1))*STP(I1)	002531
001007	371.	685	FOPN = F	002544
001010	372.		IF (I.EQ.NO2) FOP5 = F	002545
001012	373.	686	CONTINUE	002555
001014	374.	653	FORMAT	002555
001014	375.		1 (132H JP FIPH TIME VRU VAN SEP FINT FTP VI	002555
001014	376.		A EN V5 F5 VN FN FA RS DA D	002555
001014	377.		BDB SEA)	002555
001015	378.		IF (IOUT.LT.0) WRITE (IW,652)	002555
001015	379.		A J,IFIRST,IT,ITH,IPT,TT,VNH(M),VNH(M1),DRA,FINTAC,FTP,VOP(1	002555
001015	380.		1),ENTHRU,VOP(NO2),FOP5,VOP(N),FOPN,FA,RSUM,DSUM,DOPINC,DBO,SEA	002555
001047	381.	24	FORMAT (16F8.4)	002625
001047	382.	C	STORE OUTPUT	002625
001050	383.		IF (IOUT.LT.6) CALL FILL(KK,KLIM,OUT,JMAX,IOUT,DNH,STH,FINTAC,	002625
001050	384.		1 FTP,INP,DNP,STP,SPLICE,VNH,VNP,ANH,ANP,JPMAX,MAXJP,EX,AREA,J,M,N,	002625
001050	385.		2 JEX,JCK)	002625
001050	386.	C		002625
001050	387.	C	FILLS IN THE OUTPUT ARRAY OUT(KLIM*15)	002625
001050	388.	C		002625
001052	389.		IF (IOUT.EQ.6) J1 = J	002666
001054	390.		IF (IOUT.EQ.6) CALL FILL2 (N,M,OUT,SEA,DNH,FTP,VNP,	002674
001054	391.		IDNP,FOP5,SJ,RSUM,DSUM,RES,FOPN,KK,ICOL,TT,ISMITH,J1,MAXJP,JPMAX,	002674
001054	392.		2STP,AREA,SPLICE,EX,JEX,J,JCK)	002674
001056	393.		IF (IOUT.GE.0) GO TO 463	002736
001060	394.		IF (FINTAC.GT.FINTMX) FINTMX=FINTAC	002742
001062	395.	463	CONTINUE	002752
001063	396.		IF (EMAX.GT.1.0) GO TO 699	002752
001065	397.		IF (TT.LT.TLIM) GO TO 699	002757
001067	398.		IF (TT.LT.0.5*T2LC) GO TO 700	002764
001071	399.		IF (DNP(N).LT.DNPMAX*0.8) GO TO 701	002772
001073	400.	699	IFIRST = 0	003001

001073	401.	C			003001
001073	402.	C	END OF MAIN LOOP		003001
001073	403.	C			003001
001074	404.	700	CONTINUE		003004
001076	405.	701	CONTINUE		003004
001077	406.		DFIN = (DNPMAX-QS(N))*12.		003004
001100	407.		EFIN = ENTHRU		003010
001101	408.		IF (DFIN.GT.0.0) IBLCT =12./DFIN		003012
001103	409.	654	FORMAT (1H0,10H FIN SET= ,F8.4,21H INCHES FIN ENTHRU= ,F6.1,		003024
001103	410.		1 19H K-FT MAX ENTHRU= ,F6.1,19H K-FT BLOW COUNT= ,IS)		003024
001104	411.		IF (IOUT.LT.0) WRITE (IW,654) DFIN,EFIN,EMAX,IBLCT		003024
001113	412.		ANP(99) = EMAX		003041
001114	413.		RETURN		003043
001115	414.		END		003112

000101	1.		SUBROUTINE VACHAM(PBAR,ARAM,DEPIB,DCYL,DBC,FSUCK,VFALL,IOUT,IW)		000000
000103	2.		DIMENSION PBAR(I)		000000
000104	3.		ART = PBAR(1)		000000
000105	4.		EXPB = PBAR(4)		000002
000106	5.		DSTART = PBAR(2)		000004
000107	6.		EXP=EXPB		000006
000110	7.		VIN = PBAR(5)		000007
000110	8.	C	DSTART IS THE DISTANCE THAT THE RAM MOVES BEFORE IT CLOSSES THE VAC		000007
000110	9.	C	CHAMBER OFF.		000007
000111	10.		DELA = ART-ARAM		000011
000112	11.		DIN = VIN/DELA		000013
000113	12.		PATM = 0.144*14.7		000015
000114	13.		IF (DBC.LT.DEPIB) GO TO 100		000017
000114	14.	C	DBC IS THE DISTANCE OF THE RAM FROM THE ANVIL ZERO POSITION OR IT		000017
000114	15.	C	IS THE STROKE,I.E DBC = DOH(M) FOR -DOH(M) LT DEPIB DBC=STROKE		000017
000114	16.	C	DBC LESS THAN DEPIB MEANS DBS IS NOT STROKE		000017
000116	17.		FAC = 1.0/(EXPB-1.0)		000024
000117	18.		RAMW = PBAR(20)		000030
000117	19.	C	PBAR(20) IS THE RAM WEIGHT WHICH IS NOT NEEDED FOR OTHER THAN VAC		000030
000117	20.	C	HAMMERS		000030
000120	21.		RMS = RAMW/32.17		000032
000121	22.		DB = DBC + DIN - DSTART		000034
000122	23.		DE = DEPIB + DIN - DSTART		000037
000123	24.		VFALL = DELA*PATM*(DIN**EXP)*FAC/RAMW		000043
000124	25.		VFALL = VFALL*(DB**(1.-EXP)-DE**(1.-EXP))		000055
000125	26.		VFALL = VFALL + (1.0+PATM*DELA/RAMW)*(DBC-DEPIB)*2.0*32.17		000074
000126	27.		IF (VFALL.LT.0.0) WRITE (IW,2) VFALL		000106
000132	28.		IF (VFALL.LT.0.0) STOP		000123
000134	29.	2	FORMAT (70H IMPROPER CONDITION MET WHEN DETERMINING RAM VELOCITY A		000127
000134	30.		1T PORTS, VFALL=, E10,4)		000127
000135	31.		VFALL = SQRT(VFALL)		000127
000136	32.		RETURN		000133
000137	33.	100	DVN = DCYL+DBC-DSTART+DIN		000136
000140	34.		IF (QVN.GT.0.0) GO TO 101		000142

000142	35.	WRITE(IW,1) DCYL,DBC,DSTART,DIN	000145
000150	36.	STOP	000156
000151	37.	101 CONTINUE	000161
000152	38.	FSUCK = (DIN/DVN)**EXPB	000161
000153	39.	IF (FSUCK.GT.1.0) FSUCK = 1.0	000170
000155	40.	FSUCK = DELA*PATM*(1.0-FSUCK)	000177
000156	41.	1 FORMAT (31H NEGATIVE VOLUME IN VAC CHAMBER,4F8.2)	000205
000157	42.	RETURN	000205
000160	43.	END	000254

HAEL		SYMBOLIC FORTRAN 28 APR 76 15:10:07 48 52 5 00131054				947		
1		SUBROUTINE HAEI (IHAMR,DTCRH,DTCRA,HNAME,IR)	18	52	000004	0		
2		COMMON VINH,DINH,AOH,VOH,DOH,ANH,VNH,DNH,STH,HM,AM,STA,MA,ES0IL,	1	52	000014	4		
3		1 AOP,ANP,DOP,DNP,VOP,VNP,STP,PM,CDP,RES,RESO,SJ,SOK,QS,SU,	1	52	000030	5		
4		2 SPLICE,DEPIB,ARAM,ECUS,ECAP,EPT,EANV,VFALL,VCHAM,XPT,SJJJ,	1	52	000044	1		
5		3 EMP,RHO,DTH,DTP,N,M,JRAT,IULT,RULT,SOILM,DSACP,	1	52	000057	4		
6		4 FPTO,IW,PCOM,TDDEL,IOUT,OUT,EXPP,TEMI,TEMO,INP,STROKO,STROKN,VO,PO	1	52	000071	1		
7		5 ,DFIN,JDOUT,JPMAX,EX,ICOL,FINTMX,TEMAX,RESULT,ISMITH,ITER	1	52	000105	5		
8		6 ,ITYPH,IPREIG,VFALLM,STRMAX,DCYL,PBAR,AREA,JEX	36	52	000121	2		
9		DIMENSION VINH(10),DINH(10),AOH(10),VOH(10),DOH(10),ANH(10),	4	52	000132	4		
10		1 VNH(10),DNH(10),STH(10),HM(10),VNP(99),	4	52	000146	1		
11		2 AOP(99),ANP(99),DOP(99),DNP(99),VOP(99),STP(99),PM(99),CDP(99),	4	52	000156	2		
12		3 RES(99),RESO(99),SJ(99),SOK(99),QS(99),SU(99),SPLICE(99),	4	52	000172	4		
13		4DSACP(3),OUT(3200),INP(13),EX(600),RESULT(100),JEX(600),	37	52	000206	0		
14		ISTORE(1,26,2),STA(10),AM(10),HNAME(2),PBAR(20)	45	52	000221	0		
15	12	FORMAT (2A4,9F8.2)	32	52	000232	1		
16	2	FORMAT (10F8.2)	1	52	000236	5		
17		IHAMRR = IHAMR	43	52	000242	5		
18		DO 107 I=1,20	49	52	000246	2		
19		AM(I) = 0.0	49	52	000251	4		
20	107	PBAR(I)=0.0	49	52	000254	4		
21		IF (IF.LT.1) IF = 7	43	52	000260	3		
22		CONVS = 1.0	43	52	000264	5		
23		IF (IHAMR.EQ.0) GO TO 50	43	52	000267	5		
24	C	FOR IHAMR = 0 READ FROM CARDS, OTHERWISE READ FROM FILE	52	52	000275	0		
25		DO 52 I=1,IHAMR	43	52	000310	0		
26	52	READ (IF,54) ((STORE(1,J,K),K=1,2),J=1,26)	43	52	000313	4		
27	54	FORMAT (2A4/6(8E10.5/),2E10.5)	43	52	000324	3		
28		REWIND IF	47	52	000333	2		
29		GO TO 51	43	52	000336	0		
30	50	CONTINUE	43	52	000340	3		
31		READ (IR,12) ((STORE(1,I,J),J=1,2),I=1,5),	43	52	000343	3		
32		1 STORE (1,6,1)	48	52	000353	5		
33		READ (IR,2) STORE(1,6,2), ((STORE(1,I,J),J=1,2),	48	52	000357	3		
34		1 I=7,25),STORE(1,26,1)	43	52	000370	5		
35		CONVS = 12.	43	52	000375	5		
36	51	CONTINUE	43	52	000400	5		
37		IHAMRR = IHAMR	48	52	000403	5		
38		IHAMR = 1	43	52	000407	2		
39		ITYPH = STORE(1,26,1)	43	52	000412	0		
40		MA = STORE(IHAMR,25,2)	19	52	000416	4		
41		M=STORE(IHAMR,8,2)	32	52	000423	3		
42		IF (M.LT.1) WRITE(IW,10) IHAMRR	48	52	000427	4		
43	10	FORMAT (33H INSUFFICIENT HAMMER INFO. IHAMR=,I3)	18	52	000436	1		

44	IF (M.LT.1) STOP	18	52	000450	1
45	M1=M+1	1	52	000454	0
46	M2=M+2	1	52	000456	1
47	STH(M1)=STH(8)	1	52	000460	2
48	STH(M2)=STH(9)	1	52	000463	5
49	HM(M2)=HM(9)	1	52	000467	2
50	DO 100 I1=1,2	1	52	000472	3
51	100 HNAME(I1)=STORE(IHAMR,1,I1)	1	52	000475	5
52	HM(M1)=STORE(IHAMR,9,1)/32.17	6	52	000504	2
53	STH(M)=STORE(IHAMR,9,2)*CONVS	38	52	000512	3
54	IF (STH(M1).LT.0.1) STH(M)=2.0*STH(M)	1	52	000520	5
55	IF (STH(M1).LT.0.1) STH(M1)=STH(M)	1	52	000530	2
56	IF (STH(M2).LT..1) STH(M2)=10.0**12	1	52	000537	2
57	STOST=STH(M2)	12	52	000546	3
58	STH(M2)=STOST*STP(1)/(STOST+STP(1))	12	52	000551	5
59	IF (ITYPH.EQ.3) HM(M1)=HM(M2)	27	52	000561	0
60	IF (ITYPH.EQ.3) STH(M)=STH(M1)	27	52	000567	1
61	IF (ITYPH.EQ.3) STH(M1)=STH(M2)	27	52	000575	3
62	IF (ITYPH.EQ.3) M2=M1	27	52	000604	0
63	DO 102 I1=1,M2	1	52	000610	4
64	I2=I1-1	1	52	000614	1
65	IF (I1.EQ.M) HM(I1)=STORE(IHAMR,I1+1,1)/32.17	9	52	000616	3
66	IF (I1.GE.M) GO TO 101	1	52	000627	2
67	HM(I1)=STORE(IHAMR,I1+1,1)/32.17	6	52	000634	1
68	STH(I1)=STORE(IHAMR,I1+1,2)*CONVS	38	52	000642	5
69	101 IF (I1.EQ.1) I2=1	1	52	000651	5
70	DO 102 I3=I2,I1	1	52	000656	3
71	TCRH=HM(I1)/STH(I3)	1	52	000662	1
72	IF (I3.EQ.1) DTCRH=TCRH	1	52	000666	3
73	IF (TCRH.LT.DTCRH) DTCRH=TCRH	1	52	000673	3
74	102 CONTINUE	1	52	000701	4
75	DTCRH=SQRT(DTCRH)	1	52	000704	5
76	DTCRA=1000.0	1	52	000710	5
77	STH(M2)=STOST	12	52	000714	0
78	IF (MA.EQ.0) GO TO 105	1	52	000717	2
79	DO 103 J1=1,MA	2	52	000724	1
80	J2=J1-1	1	52	000727	4
81	IF (J1.EQ.1) J2=1	1	52	000732	0
82	J3 = J1 + 21	18	52	000736	0
83	AM(J1)=STORE(IHAMR,J3,1)/32.17	6	52	000741	1
84	STA(J1)=STORE(IHAMR,J3,2)	38	52	000747	3
85	DO 103 J4=J2,J1	1	52	000754	5
86	TCHA=AM(J1)/STA(J4)	1	52	000760	3
87	IF (TCRA.LT.DTCRA) DTCRA=TCRA	1	52	000764	5
88	103 CONTINUE	1	52	000773	0
89	DTCRA=SQRT(DTCRA)	1	52	000776	1
90	105 TDEL=STORE(IHAMR,10,1)	6	52	001002	1
91	VIN=STORE(IHAMR,11,1)/1728.0	20	52	001007	4
92	VCHAM=STORE(IHAMR,11,2)/1728.0	20	52	001015	4
93	DEPIB=STORE(IHAMR,12,1)/12.	20	52	001024	0
94	ARAM=STORE(IHAMR,12,2)/144.0	20	52	001031	4
95	SJJJ = STORE(IHAMR,10,2)	51	52	001037	4
96	DO 106 I=17,21	18	52	001044	5
97	PBAR(I-16)=STORE(IHAMR,I,1)	18	52	001050	2

98	106	PBAR(I-11)=STORE(IHAMR,I,2)	18	52	001056	0
99		PBAR(11)=STORE(IHAMR,13,1)	20	52	001064	3
100		PBAR(12)=STORE(IHAMR,13,2)	20	52	001072	0
101		PBAR(13)=STORE(IHAMR,14,1)	20	52	001077	3
102		PBAR(14)=STORE(IHAMR,14,2)	20	52	001105	0
103		PBAR(15)=STORE(IHAMR,15,1)	20	52	001112	3
104		PBAR(19)=STORE(IHAMR,19,1)	33	52	001120	0
105	C	PBAR(20) GTR 0 IDENTIFIES VACUUM CHAMBER HAMMER	52	52	001125	3
106		PBAR(20)=STORE(IHAMR,21,2)	46	52	001137	0
107		STRM=STORE(IHAMR,16,1)/12.	16	52	001144	3
108		PBAR(10)=STRM	23	52	001152	0
109		STRMAX = 0.0	51	52	001155	2
110	C	PCOM IS PRESSURE FOR CORRESPONDING FUEL SETTING	23	52	001160	3
111	C	PBAR(11) IS THE MAXIMUM DOWN PRESSURE ON A DOUBLE ACTING A/S	29	52	001172	0
112	C	HAMMER IDENTIFIER FOR SUCH HAMMERS	29	52	001205	5
113		EXPP=STORE(IHAMR,16,2)	13	52	001215	1
114		IF (EXPP.LT.0.1) EXPP=1.3	17	52	001222	0
115		IF (PBAR(1).LT.10.0E-8) PBAR(1) = ARAM	49	52	001227	2
116		EFFICY=STORE(IHAMR,15,2)	29	52	001237	0
117		IF (EFFICY.LT.0.01) EFFICY=0.95	29	52	001244	1
118		RMWT=0.0	20	52	001252	4
119		DO 220 I=1,M	20	52	001255	1
120	220	RMWT=RMWT+HM(I)	20	52	001260	2
121		PBAR(16)=EFFICY	29	52	001272	1
122		PBAR(17)=RMWT*32.17	50	52	001335	1
123		PBAR(18)=STORE(IHAMR,19,2)	50	52	001341	3
124		RETURN	1	52	001656	2
125		END	1	52	001660	3
000101	1.	SUBROUTINE UP(PBAR,DNH,VNH,DNP,VNP,HM,PM,M,N,DCYL,DFIN,VFALL,VO,			000023	
000101	2.	1 PO,VFALLM,STROKE,DEPIB,ARAM,EXPP,IWT,IW,IOUT,RSI,T)			000023	
000103	3.	DIMENSION PBAR(1),DNH(1),VNH(1),DNP(1),VNP(1),HM(1),PM(1)			000023	
000103	4.	C			000023	
000103	5.	C THIS PROGRAM IS USED TO DETERMINE THE VELOCITY OF THE RAM OF			000023	
000103	6.	C CED'S AT THE EXHAUST PORTS. THE ANALYSIS USES A RIGID BODY			000023	

000103	7.	C	PILE AND RAM ASSUMPTION FOR CLOSED END HAMMERS WHOSE UPWARD	000023
000103	8.	C	VELOCITY WILL NOT CAUSE UPLIFT THE TOTAL STROKE IS DETERMINED	000023
000103	9.	C		000023
000104	10.		PATM = 14.7*0.144	000023
000105	11.		DFI = DFIN/12.	000025
000106	12.		T = 0.0	000030
000107	13.		VR = 0.0	000031
000110	14.		M1 = M+1	000032
000111	15.		M2=M+2	000035
000112	16.		VP=VNH(M1)*HM(M1)+VNH(M2)*HM(M2)	000040
000113	17.		VCHAM = VO	000056
000114	18.		IF (DNH(M1).GT.DNH(M)) VCHAM = VO-(DNH(M1)-DNH(M))*ARAM	000060
000116	19.		G = 32.17	000101
000117	20.		CYLM=PBAR(8)/G	000103
000120	21.		FATM = PATM*ARAM	000106
000121	22.		DELP = 0.0	000111
000122	23.		IF (PBAR(1).GT.0.0) DELP=(PBAR(1)-ARAM)*2.117	000112
000124	24.		IF (PBAR(1).GT.ARAM) FATM = PATM*PBAR(1)	000122
000126	25.		IF (PBAR(20).GT.0.0) FATM=0.0	000132
000130	26.		ACYL=G	000137
000131	27.		IF (PBAR(8).GT.0.0) ACYL=DNH(10)	000141
000133	28.		VCYL = VNH(10)	000147
000134	29.		PAM = 0.0	000151
000135	30.		IBLOW = 0	000152
000136	31.		RAM = 0.0	000153
000137	32.		IWT = 0	000154
000140	33.		TT=0.0	000155
000141	34.		IN=1	000156
000142	35.		1- FORMAT (7X,83HTT FP FB R VEL R POS V CYL	000160
000142	36.		1 D CYL V TOP D TOP)	000160
000143	37.		IF (IOUT.LT.0) WRITE (IW,1)	000160
000143	38.	C		000160
000143	39.	C	DETERMINE RIGID RAM VELOCITY	000160
000143	40.	C		000160
000146	41.		DO 100 I=1,M	000177
000151	42.		RAM = RAM + HM(I)	000177
000152	43.	100	VR = VR + VNH(I)*HM(I)	000201
000152	44.	C		000201
000152	45.	C	DETERMINE RIGID PILE VELOCITY	000201
000152	46.	C		000201
000154	47.		DO 101 I=1,N	000214
000157	48.		PAM = PAM + PM(I)	000214
000160	49.	101	VP = VP + VNP(I)*PM(I)	000216
000162	50.		VDR = VR/RAM	000223
000163	51.		VDP=VP/(PAM+HM(M1)+HM(M2))	000226
000164	52.		IF (VDR.GT.-0.05) WRITE (IW,3)	000234
000167	53.	3	FORMAT (41H RAM STILL MOVING DOWNWARD AT END OF BLOW)	000251
000170	54.		IF (VDR.GT.-0.05) STOP	000251
000172	55.		DDP=DNH(M)	000255
000173	56.		DDP=DNH(M1)	000257
000174	57.		DCDT=DCYL-DNH(M1)	000262
000175	58.		VCVT=VCYL-VNH(M1)	000264
000176	59.		VCYL=VDP+VCVT	000270
000177	60.		DCYL=DDP+DCDT	000272

000200	61.		IF (DCYL.GT.DDP) VCYL=VDP	000274
000202	62.		IF (DCYL.GT.DDP) DCYL=DDP	000303
000204	63.		IUP = 0	000312
000205	64.		IF (VDP.LT.0.0) IUP =1	000313
000207	65.		DPOS = DDR	000321
000210	66.		DRAIS = 0.01*DEPIB	000323
000210	67.	C		000323
000210	68.	C	START LOOP TO DETERMINE INCREMENTALLY THE RAM SPEED DURING	000323
000210	69.	C	RAM UPWARDS MOTION	000323
000210	70.	C		000323
000211	71.	1000	DELT = ABS(DRAIS/VDR)	000327
000212	72.		T=T+DELT*FLOAT(1-IBLOW)	000332
000213	73.		TT=TT+DELT	000341
000214	74.		IN=IN+1	000344
000215	75.		IF (IOUT.LT.0.AND.IN.EQ.5) WRITE (IW,2) TT,FP,FB,VDR,DPOS,VCYL,	000347
000215	76.		1 DCYL,VDP,DDP	000347
000231	77.		IF (VCYL.LT.0.0.AND.VCYL.LT.VDP) VCYL=VDP	000401
000233	78.		2 FORMAT (9F10.4)	000420
000234	79.		IF (IN.EQ.5) IN=0	000420
000236	80.		IF (IUP.EQ.1.AND.DDP.LT.DFI) VDP = 0.0	000425
000240	81.		IF (IUP.NE.1.AND.DDP.GT.DFI) VDP = 0.0	000443
000242	82.		DDP = DDP + VDP*DELT	000461
000243	83.		VCYL = VCYL + ACYL * DELT	000465
000244	84.		DCYL = DCYL + VCYL * DELT	000471
000245	85.		IF (DCYL.GE.DDP) VCYL=VDP	000474
000247	86.		IF (DCYL.GT.DDP) DCYL = DDP	000503
97 000251	87.		DPOS = DPOS - DRAIS	000512
000252	88.		V = VCHAM + (DDP-DPOS)*ARAM	000515
000253	89.		FP = 2.117*ARAM	000521
000254	90.		IF (IBLOW.EQ.0) FP =(PO*(VO/V)**EXPP)*ARAM	000524
000256	91.		DCYRA = DPOS-DCYL	000540
000257	92.		DCB = -DPOS	000543
000260	93.		FB = FTR(DCYRA,PBAR,IWT)	000545
000261	94.		IF (PBAR(20).GT.0.0) CALL VACHAM(PBAR,ARAM,DEPIB,DCYL,DCB,FB,VFALL	000553
000261	95.		1,IOUT,IW)	000553
000263	96.		IF (PBAR(8).GT.0.0) ACYL=G+(FATM-FB)/CYLM	000572
000265	97.		IF (PBAR(20).GT.0.0) FB=FB+PATM*ARAM	000603
000267	98.		IF (PBAR(20).LT.10.E-8) FB=FB-DELP	000613
000271	99.		AC = G-(FP-FB)/RAM	000623
000272	100.		VE = VDR + AC*DELT	000630
000273	101.		VDR = VE	000633
000274	102.		VO = V	000634
000275	103.		PO = FP/ARAM	000636
000276	104.		IF (DPOS.GT.-DEPIB+DCYL) GO TO 1000	000641
000300	105.		IF (IBLOW.EQ.1) GO TO 1001	000647
000300	106.	C		000647
000300	107.	C	IBLOW = 1 MEANS RAM HAS PASSED THE EXHAUST PORTS	000647
000300	108.	C		000647
000302	109.		IBLOW = 1	000653
000303	110.		VFALL = -VDR	000655
000303	111.	C	FOR OPEN END HAMMERS	000655
000304	112.		STROKE = VDR*VDR/64.34 + DEPIB	000656
000305	113.		DRAIS = 0.05	000662
000306	114.		IF (PBAR(20).GT.0.0) GO TO 1001	000664

UP

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000310	115.		IF (PBAR(6).LT.0.001) RETURN	000670
000312	116.		IF (VFALL.GT.VFALLM) IWT = 1	000113
000314	117.		IF (VFALL.GT.VFALLM) PSI=PBAR(8)/(PBAR(1)*0.144)	000711
000314	118.	C		000711
000314	119.	C	VFALL GT VFALLM MEANS UPLIFT	000711
000314	120.	C		000711
000316	121.		IF (VFALL.GT.VFALLM) RETURN	000720
000316	122.	C		000720
000316	123.	C	FOR HAMMERS WHOSE BOUNCE CHAMBER PORTS HAVE NOT YET BEEN CLOSED	000720
000316	124.	C		000720
000320	125.		DEPS = PBAR(7)-PBAR(6)-DEPIB	000115
000321	126.		IF (DEPS.LT.0.0) GO TO 1001	000730
000323	127.		IF (VDR*VDR.LT.64.34*DEPS) GO TO 1001	000733
000325	128.		VDR = SQRT(VDR*VDR-64.34*DEPS)	000743
000326	129.		DPOS = DPOS-DEPS	000755
000327	130.		VDR = -VDR	000760
000330	131.	1001	IF (VDR.LT.0.0.AND.IWT.NE.1) GO TO 1000	000762
000330	132.	C		000762
000330	133.	C	RAM REACHED UPMOST POINT	000762
000330	134.	C		000762
000332	135.		STROKE = -DPOS-DCYL	000775
000332	136.	C		000775
000332	137.	C	BOUNCE CHAMBER PRESSURE IN PSI (FOR VFALL LT VFALLM)	000775
000332	138.	C		000775
000333	139.		DPOS = -STROKE	001000
000334	140.		FB = FTR(DPOS,PBAR,IWT)	001001
000335	141.		PSI=FB/(PBAR(1)*0.144)-14.7	001007
000336	142.		RETURN	001014
000337	143.		END	001203

NO DIAGNOSTICS.

3.9710 SECONDS.

DOWN		SYMBOLIC FORTRAN	26 MAR 76	09:55:03	11	15	5	00061214	407		
1		SUBROUTINE DOWN(STRMAX,VFALLM,PBAR,HM,DEPIB,ARAM,IW,IOUT,T)							12	15	000114 3
2		DIMENSION PBAR(1),HM(1)							2	15	000127 5
3		T=0.0							12	15	000134 5
4		ART=PBAR(1)							11	15	000136 5
5		STR = STRMAX							7	15	000141 5
6		PATM=14.7*0.144							8	15	000145 0
7		PMAX=PATM+PBAR(8)/PBAR(1)							8	15	000150 4
8		PRAT=(PATM/PMAX)**(1.0/PBAR(4))							8	15	000156 0
9		VBIN = PBAR(6)*PBAR(1) + PBAR(5)							1	15	000164 3
10		DUP=(VBIN/PBAR(1))*(1.0-PRAT)							8	15	000173 1
11		DMAX=PBAR(6)-DUP							8	15	000201 2
12		DSF = PBAR(2)							15	15	000207 3
13	C	DUP = DISTANCE FROM BOUNCE CHAMBER VENTS							15	15	000212 5
14	C	DMAX = DISTANCE FROM TOP							15	15	000223 1
15	C	DPOS = DISTANCE FROM ANVIL (NEGATIVE)							15	15	000230 4
16		IF (DMAX.GT.DSF) GO TO 202							15	15	000240 3
17	C	MAX STROKE PENETRATES INTO SAFETY CHAMBER							15	15	000246 0
18		PRAT = (VBIN/(PBAR(5)+DSF*ART))**PBAR(4)							15	15	000256 3
19		PRAT = (PRAT*PATM/PMAX)**(1./PBAR(4))							15	15	000266 3
20		DMAX = DSF*PRAT							15	15	000276 0
21	202	CONTINUE							15	15	000301 4
22		DPOS = DMAX - PBAR(7)							1	15	000305 0
23		RAM = 0.0							2	15	000311 4
24		DO 200 I=1,10							3	15	000314 2
25		IF (HM(I).LT.10E-8) GO TO 201							3	15	000317 4
26	200	RAM = RAM + HM(I)							3	15	000325 5
27	201	CONTINUE							3	15	000332 4
28		STRMAX = -DPOS							1	15	000336 0
29		IF (STR.GT.STRMAX) STR = STRMAX							7	15	000341 3
30	C	START WITH RAM AT POSITION WHERE UPLIFT IS IMMINENT							1	15	000350 0
31		TOTD = -DPOS-DEPIB							1	15	000362 1
32		IF (STR.LT.0.1) GO TO 300							12	15	000402 4
33		TOTD = STR-DEPIB							12	15	000410 0
34		DPOS = -STR							12	15	000413 5
35		STRMAX = STR							12	15	000416 5
36	300	CONTINUE							12	15	000422 0
37		NST = TOTD/0.05							1	15	000425 2
38		DFALL = TOTD/FL0AT(NST)							1	15	000431 0
39		G = 32.17							1	15	000442 0
40		AO = G							1	15	000444 4
41		VO = 0.0							1	15	000446 5
42		DO 100 I=1,NST							1	15	000451 2
43		A = G*(FTR(DPOS,PBAR,IWT)-PATM*ART)/RAM							10	15	000454 5

44	VA = VO/A	4	15	000464	4
45	DELT = SQRT(VA*VA+2.*DFALL/A)-VA	4	15	000467	2
46	T = T+DELT	12	15	000476	0
47	V = VO + 0.5*(A+AO)*DELT	1	15	000500	5
48	DPOS = DPOS + DFALL	1	15	000506	0
49	VO = V	1	15	000512	2
50	100 AO = A	1	15	000514	3
51	VFALLM = V	6	15	000517	3
52	IF (STR.GT.0.1.AND.IOUT.LT.0) WRITE (IW,2) STR,V	13	15	000531	2
53	2 FORMAT (20H STROKE (LESS MAX.)=,F6.1,23H FT, VELOCITY AT PORTS=,	7	15	000543	0
54	1F6.1,5H FT/S)	7	15	000557	4
55	IF (STR.GT.0.1) RETURN	7	15	000563	2
56	IF (IOUT.LT.0) WRITE (IW,1) STRMAX,VFALLM	4	15	000570	1
57	1 FORMAT(28H MAXIMUM STROKE (NO UPLIFT)=,F6.2,30H FT MAXIMUM VELOCIT	5	15	000600	2
58	1Y AT PORTS=,F6.2,5H FT/S)	4	15	000615	2
59	RETURN	1	15	000622	5
60	END	1	15	000625	0

OUTPUT						SYMBOLIC FORTRAN 17 FEB 76 13:38:39 24 28 5 00042762 2237					
1		SUBROUTINE OUTPUT (OUT,RES,JNP,IOUT,DT,JPMAX,JDOUT,IULT,N,JDLIM,	1	28	000004	0					
2	1	ALPH,ISEG,JSELEC,YMAX,COLI,TITLE,STP1,PM1)	12	28	000020	1					
3	C	PLOT OUTPUT	1	28	000030	5					
4		DIMENSION JSELEC(13),BIG(5),PLLY(5),PRJ(5),PUNIT(5)	1	28	000034	0					
5		2 ,OUT(3200),JNP(13),RES(100),ISEG(15),ALPH(99) ,	23	28	000045	5					
6		1 JDUM(15),PYS(5),IUNIT(5),SALPH(99),TITLE(10)	6	28	000057	2					
7		3 ,RCODE(2)	27	28	000070	2					
8		I9=1	13	28	000073	2					
9	112	JPM=JDLIM	1	28	000075	1					
10	1	FORMAT (20I4)	1	28	000100	3					
11	2	FORMAT (10F8.2)	1	28	000104	2					
12		YMAXR=YMAX	10	28	000110	3					
13		IF (IABS(IOUT).LT.10) RETURN	1	28	000113	2					
14		DTP=DT*FLOAT(JDOUT)*1000.0	1	28	000121	2					
15		DTPM=DTP*FLOAT(JDLIM)	1	28	000126	5					
16	6	FORMAT (6H(RULT=,I4,1H))	27	28	000133	3					
17		IRULT = RES(1)	27	28	000141	1					
18		IU = ABS(IULT)	28	28	000153	0					
19		IF (IULT.LT.0) IULT=0	28	28	000156	3					
20		IF (IU.GT.1) IRULT = RES(IU)	28	28	000163	1					
21		ENCODE(6,RCODE) IRULT	27	28	000171	1					
22		YMIN=0.	8	28	000175	5					
23		PLENY=4.	8	28	000200	1					
24		PLENX=6.	8	28	000202	4					
25		YMAX=YMAX*.999	8	28	000205	1					
26		YMAXC=0.	8	28	000210	4					
27		IF (IABS(IOUT).EQ.16.OR.IABS(IOUT).EQ.26) GO TO 171	11	28	000213	1					
28		DO 120 J=1,13	1	28	000225	0					
29		IF (JNP(J).EQ.0) GO TO 121	1	28	000230	2					
30	120	JDUM(J)=J+2	1	28	000235	5					
31	121	JJNP=13	1	28	000241	4					
32		IF (J.NE.13) JJNP=J-1	1	28	000244	5					
33		IF (J.EQ.13.AND.JNP(13).EQ.0) JJNP=12	1	28	000251	3					
34		JJNP2=JJNP+2	1	28	000261	0					
35		IF (IABS(IOUT).EQ.10.OR.IABS(IOUT).EQ.20) GO TO 300	22	28	000264	1					

36	DO 100 ISEGNO = 3,JJNP2	7	28	000276 0
37	DO 100 J=1,JPM	7	28	000303 0
38	I = (ISEGNO-1)*200+J	7	28	000306 3
39	IF (OUT(I).LT.YMIN) YMIN = OUT(I)	7	28	000313 0
40	100 IF (OUT(I).GT.YMAXC) YMAXC = OUT(I)	7	28	000321 5
41	GO TO 170	8	28	000331 5
42	171 I1=1	8	28	000334 3
43	EAOC=1.	8	28	000337 0
44	IF (ISEG(I9).EQ.0) KK=1	13	28	000341 2
45	IF (ISEG(I9).EQ.1.OR.ISEG(I9).EQ.3) KK=4	13	28	000346 2
46	IF (ISEG(I9).EQ.2) KK=5	15	28	000356 2
47	173 DO 172 I=1,JPM	8	28	000363 2
48	IJ=KK*200+I	26	28	000372 3
49	IF (KK.EQ.5.AND.I1.EQ.2) EAOC=SQRT(STP1*PM1)	8	28	000375 3
50	IF (OUT(IJ)*EAOC.LT.YMIN) YMIN=OUT(IJ)*EAOC	13	28	000406 1
51	172 IF (OUT(IJ)*EAOC.GT.YMAXC) YMAXC=OUT(IJ)*EAOC	14	28	000416 4
52	I1=I1+1	8	28	000430 1
53	KK=KK+9	8	28	000432 3
54	IF (ISEG(I9).EQ.3) KK=5	13	28	000434 5
55	IF (KK.EQ.10) GO TO 170	15	28	000441 5
56	IF (I1.LT.3) GO TO 173	15	28	000446 5
57	170 IF (YMAXC.GT.10.*YMAX) YMAX=YMAXC	18	28	000453 4
58	DEL=(YMAX-YMAXR)/YMAX	19	28	000463 1
59	105 YM = YMAX	7	28	000467 5
60	DO 101 J=1,15	7	28	000473 2
61	IF (YM.LT.1.) GO TO 102	7	28	000476 4
62	101 YM = YM/10.	7	28	000503 4
63	102 RJ = 10.**((J-2)*2.	7	28	000507 3
64	I = YMAX/RJ+1.0	7	28	000514 3
65	YMAX = RJ*FLOAT(I)	7	28	000520 1
66	IF (ABS(DEL).LT.0.01) YMAX=YMAXR	16	28	000524 2
67	YMAXR=YMAX	18	28	000533 0
68	IF (YMIN.LT.-YMAX*4.) YMAX = -YMIN	22	28	000535 5
69	YMI = -YMAX*3.	22	28	000544 5
70	IF (YMIN.GT.-YMAX*0.75) YMI = -YMAX	22	28	000550 2
71	IF (YMIN.GT.-YMAX*0.5) YMI = -YMAX*1./3.	22	28	000557 3
72	IF (YMIN.GT.-YMAX*0.1667) YMI = 0.0	22	28	000567 3
73	YMIN = YMI	22	28	000576 4
74	YS = (YMAX-YMIN) / PLENY	7	28	000601 3
75	XS = DTPM/PLENX	7	28	000606 4
76	K = DTPM/12.	7	28	000612 2
77	XUNIT = K+1	7	28	000615 3
78	CALL FSCALE(-3.0*XS,YMIN,XS,YS,YMAX,1H,0.,0.)	7	28	000620 3
79	IF (IABS(IOUT).EQ.16.OR.IABS(IOUT).EQ.26) GO TO 145	7	28	000631 3
80	DO 103 I=1,15	7	28	000643 2
81	IF (ISEG(I).EQ.0.AND.I.GT.1) GO TO 103	7	28	000646 4
82	ISEGNO = ISEG(I)	7	28	000656 2
83	IF (ISEGNO.EQ.0) ISEGNO = 3	7	28	000662 1
84	DO 106 J = 1,JPM	7	28	000667 5
85	I1 = (ISEGNO-1)*200 + J	7	28	000673 4
86	106 CALL FPLLOT(FLOAT(J)*DTP,OUT(I1))	7	28	000700 4
87	CALL FMOVE(0.,0.)	7	28	000710 1
88	103 CONTINUE	7	28	000714 1
89	145 IF (IABS(IOUT).NE.16.AND.IABS(IOUT).NE.26) GO TO 150	10	28	000717 3

90	IF (ISEG(I9).EQ.0) KK=1	13	28	000732	2
91	IF (ISEG(I9).EQ.1.OR.ISEG(I9).EQ.3) KK=4	13	28	000737	2
92	IF (ISEG(I9).EQ.2) KK=5	13	28	000747	2
93	DO 140 J=1,JPM	6	28	000754	2
94	II=KK*200+J	26	28	000762	5
95	140 CALL FPLOT (FLOAT(J)*DTP,OUT(II))	6	28	000765	5
96	IF (KK.EQ.1) GO TO 150	6	28	000775	2
97	KK=KK+9	6	28	001002	1
98	IF (ISEG(I9).EQ.3) KK=5	13	28	001004	3
99	EAOC=1.	6	28	001011	3
100	IF (KK.EQ.5) EAOC=SQRT(STP1*PM1)	6	28	001013	5
101	CALL FMOVE (0.,0.)	6	28	001022	3
102	DO 141 J=1,JPM	6	28	001026	4
103	II=KK*200+J	26	28	001035	1
104	141 CALL FPLOT (FLOAT(J)*DTP,OUT(II)*EAOC,4HDASH)	6	28	001040	1
105	CALL FMOVE (4.*XS,3.5*YS)	6	28	001051	4
106	IF (KK.EQ.13.OR.KK.EQ.5) CALL FCHAR (0.,.12,9,9HTOP FORCE)	6	28	001057	0
107	IF (KK.EQ.14) CALL FCHAR (0.,.12,8,8HTOP VEL.)	6	28	001072	1
108	CALL FMOVE (5.*XS,3.56*YS)	6	28	001103	1
109	CALL FMOVE (5.4*XS,3.56*YS)	6	28	001110	4
110	CALL FMOVE (4.*XS,3.65*YS)	6	28	001116	2
111	IF (KK.EQ.13) CALL FCHAR (0.,.12,9,9HTOE FORCE)	6	28	001123	5
112	IF (KK.EQ.14) CALL FCHAR (0.,.12,8,8HTOE VEL.)	6	28	001135	0
113	IF (KK.EQ.5) CALL FCHAR (0.,.12,9,9HVEL.*EA/C)	6	28	001146	0
114	CALL FMOVE (5.*XS,3.71*YS)	6	28	001157	0
115	CALL FMOVE (5.4*XS,3.71*YS,4HDASH)	6	28	001164	3
116	150 CALL FMOVE (DTPM+.3*XS,-.12*YS)	6	28	001173	3
117	CALL FCHAR (0.,.12,4,4HMSEC)	8	28	001202	4
118	CALL FMOVE (DTPM+.3*XS,.05*YS)	6	28	001210	2
119	CALL FCHAR (0.,.12,3,3HL/C)	6	28	001216	4
120	CALL FMOVE (DTPM,0.)	1	28	001224	2
121	L=DTPM/XUNIT	1	28	001230	5
122	DO 130 I=1,L	1	28	001234	0
123	XUSE=XUNIT*(L-I+1)	1	28	001237	1
124	CALL FMOVE (XUSE,0.)	1	28	001243	2
125	CALL FMOVE (XUSE,-.05*YS)	1	28	001247	5
126	CALL FMOVE (XUSE-.25*XS,-.2*YS)	12	28	001255	1
127	3 FORMAT (F4.0)	1	28	001263	4
128	ENCODE (3,PL) XUSE	1	28	001267	3
129	CALL FCHAR (0.,.12,4,PL)	1	28	001273	4
130	CALL FMOVE (XUSE,0.)	1	28	001300	5
131	130 CONTINUE	1	28	001305	2
132	CALL FMOVE (0.,0.)	1	28	001310	3
133	CALL FMOVE (2.8*XS,-.4*YS)	6	28	001314	4
134	CALL FCHAR (0.,.12,4,4HTIME)	6	28	001322	1
135	CALL FMOVE (DTPM,0.)	6	28	001330	1
136	L=DTPM/COLI	10	28	001334	4
137	DO 146 I=1,L	6	28	001337	4
138	LI=L-I+1	6	28	001342	5
139	XUSE=COLI*LI	6	28	001345	2
140	CALL FMOVE (XUSE,0.)	6	28	001350	3
141	CALL FMOVE (XUSE,0.05*YS)	6	28	001355	0
142	CALL FMOVE (XUSE-.35*XS,.1*YS)	13	28	001362	2
143	ENCODE (1,LC) LI	6	28	001370	4

144	CALL FCHAR (0.,.12,4,LC)	6	28	001374	3
145	CALL FMOVE (XUSE,0.)	9	28	001401	4
146	146 CONTINUE	6	28	001406	1
147	CALL FMOVE (0.,YMIN)	1	28	001411	2
148	CALL FPLOT (-.05*XS,YMIN)	20	28	001415	5
149	CALL FMOVE (-.9*XS,YMIN-.06*YS)	20	28	001423	1
150	ENCODE (2,PLY) YMIN	20	28	001431	4
151	CALL FCHAR (0.,.12,8,PLY)	20	28	001436	0
152	CALL FMOVE(0.,YMIN)	22	28	001443	2
153	M=PLENY+.001	1	28	001447	4
154	DO 131 I=1,M	1	28	001452	5
155	YUSE=YMIN+I*YS	1	28	001456	0
156	CALL FPLOT (0.,YUSE)	1	28	001461	3
157	CALL FPLOT (-.05*XS,YUSE)	1	28	001466	0
158	CALL FMOVE (-.9*XS,YUSE-.06*YS)	1	28	001473	2
159	ENCODE (2,PLY) YUSE	1	28	001501	5
160	CALL FCHAR (0.,.12,8,PLY)	1	28	001506	1
161	CALL FMOVE (0.,YUSE)	1	28	001513	3
162	131 CONTINUE	1	28	001520	0
163	CALL FMOVE (-XS,YMAX-3.0*YS)	1	28	001523	1
164	IOUTS=IOUT	1	28	001531	1
165	IF (IABS(IOUT).GT.20) IOUT=IABS(IOUT)-10	1	28	001534	0
166	IF (ISEG(I9).EQ.1.AND.IABS(IOUT).EQ.16) IOUT=11	16	28	001544	0
167	IF (ISEG(I9).EQ.3.AND.IABS(IOUT).EQ.16) IOUT=11	16	28	001555	1
168	IF (ISEG(I9).EQ.2.AND.IABS(IOUT).EQ.16) IOUT=12	16	28	001566	2
169	IF (IABS(IOUT).EQ.11)CALL FCHAR (90.,.12,13,13HFORCE IN KIPS)	1	28	001577	3
170	IF (IABS(IOUT).EQ.12)CALL FCHAR (90.,.12,18,18HVELOCITY IN FT/SEC)	1	28	001613	1
171	IF (IABS(IOUT).EQ.13)CALL FCHAR (90.,.12,13,13HSTRESS IN KSI)	1	28	001627	4
172	IF (IABS(IOUT).EQ.14)CALL FCHAR (90.,.12,17,17HACCELERATION IN G)	1	28	001643	2
173	IF (IABS(IOUT).EQ.15) CALL FCHAR (90.,.12,15,15HDISPL IN INCHES)	1	28	001657	4
174	IF (IABS(IOUT).EQ.16) CALL FCHAR (90.,.12,15,15HPRESSURE IN PSI)	6	28	001673	5
175	IOUT=IOUTS	16	28	001710	0
176	CALL FMOVE (0.0,4.1*YS)	27	28	001720	2
177	DO 160 I=1,10	10	28	001725	2
178	160 CALL FCHAR (0.,.15,4,TITLE(I))	10	28	001730	4
179	CALL FMOVE (5.2*XS,4.1*YS)	27	28	001737	4
180	DO 161 I=1,2	27	28	001745	1
181	161 CALL FCHAR (0.,0.12,6,RCODE(I))	27	28	001750	2
182	CALL FMOVE (0.,YMAX+3.5*YS)	6	28	001757	4
183	IF (IABS(IOUT).EQ.16.OR.IABS(IOUT).EQ.26) YMAX=0.0	19	28	001765	2
184	I9=I9+1	13	28	001777	0
185	IF (IABS(IOUT).EQ.16.AND.ISEG(I9).NE.0) GO TO 112	16	28	002001	2
186	IF (IABS(IOUT).EQ.26.AND.ISEG(I9).NE.0) GO TO 112	16	28	002012	5
187	200 IF (IABS(IOUT).LT.20) RETURN	1	28	002024	2
188	IF (IABS(IOUT).EQ.26) GO TO 300	6	28	002033	0
189	PLENX=4.	1	28	002041	3
190	PLENY=1.5	1	28	002044	0
191	PLENMX=1.	1	28	002046	4
192	PLENMY=2.5	1	28	002051	2
193	YMAX=0.	1	28	002054	1
194	DO 201 J=1,JPM	1	28	002056	3
195	DO 201 I=1,JJNP	1	28	002062	0
196	K=JSELEC(I)	1	28	002065	4
197	IF (JSELEC(1).EQ.0) K=I+2	1	28	002070	4

198		IF (K.LE.2.0R.K.GT.15) GO TO 201	1	28	002076	0
199		K=(K-1)*200+J	1	28	002104	4
200		IF (OUT(K).GT.YMAX) YMAX=OUT(K)	1	28	002110	0
201	201	CONTINUE	1	28	002116	3
202		YM=YMAX	1	28	002121	5
203		DO 202 J=1,15	1	28	002124	1
204		IF (YMAX.LT.1.) GO TO 203	1	28	002127	3
205	202	YMAX=YMAX/10.	1	28	002134	5
206	203	RJ=10.** (J-2)*2.0	1	28	002141	0
207		I=YM/RJ+1.	1	28	002145	5
208		YMAX=RJ*FLOAT(I)	1	28	002150	4
209		XS=DTPM/PLENX	1	28	002154	3
210		YS=YMAX/PLENY	1	28	002157	5
211		CALL FSCALE (-3.*XS,-2.5*YS,XS,YS,YMAX,1H,0.,0.)	1	28	002163	1
212		NN=N	1	28	002174	4
213		IF (NN.GT.JNP(JJNP)) NN=JNP(JJNP)	1	28	002176	3
214		SALPH(1)=0.	1	28	002205	2
215		DO 206 I=2,JJNP	3	28	002210	2
216		JEND=JNP(I)	24	28	002214	0
217		JBEG=JNP(I-1)+1	20	28	002217	0
218		SALPH(I)=SALPH(I-1)	20	28	002222	4
219		DO 206 J=JBEG,JEND	20	28	002227	0
220	206	SALPH(I)=SALPH(I)+ALPH(J)	20	28	002233	1
221		DO 204 J=1,JJNP	21	28	002241	2
222		I=JSELEC(J)	1	28	002245	0
223		IF (JSELEC(1).EQ.0) I=J+2	1	28	002250	0
224		IF (I.LE.2.0R.I.GT.15) GO TO 204	11	28	002255	2
225		PMY=PLENMY*SALPH(J)/SALPH(JJNP)	1	28	002272	1
226		PMX=PMY*PLENMX/PLENMY	25	28	002300	4
227		PMY=-PMY*YS	1	28	002305	2
228		PMX=-PMX*XS	1	28	002310	2
229		CALL FMOVE (PMX,PMY)	1	28	002313	2
230		DO 205 K=1,JPM	1	28	002317	5
231		KK=(I-1)*200+K	1	28	002323	2
232	205	CALL FPLLOT (PMX+FLOAT(K)*DTP,PMY+OUT(KK))	1	28	002326	5
233		CALL FPLLOT (PMX+FLOAT(K)*DTP,PMY)	4	28	002337	4
234	204	CONTINUE	4	28	002346	3
235		CALL FPLLOT (PMX,PMY)	3	28	002351	5
236		CALL FMOVE (PMX+FLOAT(K)*DTP,PMY)	4	28	002356	2
237		CALL FPLLOT (DTPM,0.)	4	28	002365	1
238		CALL FMOVE (DTPM+XS*.3,0.)	3	28	002371	4
239		CALL FCHAR (0.,.12,12,12H TIME IN MSEC)	3	28	002377	1
240		CALL FMOVE (DTPM,0.)	3	28	002406	5
241		CALL FPLLOT (DTPM,-.05*YS)	3	28	002413	2
242		CALL FMOVE (DTPM,-.12*XS,-.2*YS)	3	28	002420	4
243		ENCODE (5,PLY) DTPM	3	28	002427	1
244		CALL FCHAR (0.,.12,6,PLY)	3	28	002433	3
245		CALL FMOVE (DTPM,0.)	3	28	002440	5
246		CALL FPLLOT (0.,0.,4HDASH)	3	28	002445	2
247	5	FORMAT (F6.2)	6	28	002452	4
248		CALL FPLLOT (PMX,PMY)	3	28	002456	3
249		CALL FMOVE (.55*PMX-.3*XS,.55*PMY+.1*YS)	3	28	002463	0
250		CALL FCHAR (67.5,.12,6,6HLENGTH)	3	28	002473	0
251		CALL FMOVE (-.3*XS,.25*YMAX)	3	28	002501	4

252		IF (IABS(IOUT).EQ.21) CALL FCHAR (90.,.12,5,5HFORCE)	3	28	002507	4
253		IF (IABS(IOUT).EQ.22) CALL FCHAR (90.,.12,4,4HVEL.)	3	28	002521	4
254		IF (IABS(IOUT).EQ.23) CALL FCHAR (90.,.12,6,6HSTRESS)	3	28	002533	3
255		IF (IABS(IOUT).EQ.24) CALL FCHAR (90.,.12,4,4HACC.)	3	28	002545	4
256		IF (IABS(IOUT).EQ.25) CALL FCHAR (90.,.12,6,6HDISPL.)	3	28	002557	3
257		CALL FMOVE (-.9*XS,YMAX-.05*YS)	3	28	002571	4
258		ENCODE (2,PLY) YMAX	3	28	002600	1
259		CALL FCHAR (0.,.12,8,PLY)	3	28	002604	3
260		CALL FMOVE (0.,YMAX)	3	28	002611	5
261		CALL FPLOT (-.05*XS,YMAX)	3	28	002616	2
262		CALL FMOVE (0.,YMAX)	4	28	002623	4
263		CALL FPLOT (0.,0.)	4	28	002630	1
264		CALL FMOVE (0.0,1.6*YS)	27	28	002641	5
265		DO 210 I=1,10	10	28	002646	5
266	210	CALL FCHAR (0.,.15,4,TITLE(I))	10	28	002652	1
267		CALL FMOVE (5.2*XS,1.6*YS)	27	28	002661	1
268		DO 162 I=1,2	27	28	002666	4
269	162	CALL FCHAR (0.0,.12,6,RCODE(I))	27	28	002671	5
270		CALL FMOVE (0.,YMAX+3.*YS)	6	28	002701	1
271	300	IF (IULT.EQ.0) RETURN	1	28	002706	4
272		DO 301 J=1,5	1	28	002714	0
273		BIG(J)=0.	1	28	002717	1
274		DO 302 I=1,IULT	1	28	002721	5
275		K=(J-1)*10+I	1	28	002725	3
276	302	IF (ABS(RES(K)).GT.BIG(J)) BIG(J)=ABS(RES(K))	1	28	002730	4
277		BIG(J)=BIG(J)+1.1	6	28	002742	1
278		YM=BIG(J)	1	28	002746	1
279		DO 303 I=1,15	3	28	002750	5
280		IF (YM.LT.1.) GO TO 304	1	28	002754	1
281	303	YM=YM/10.	1	28	002761	1
282	304	PRJ(J)=10.**I-2	1	28	002764	3
283		I=BIG(J)/PRJ(J)+1.	1	28	002771	1
284	301	BIG(J)=PRJ(J)*FLOAT(I)	6	28	002775	2
285		PLLY(1)=4.	1	28	003002	5
286		PLLY(2)=6.	1	28	003005	4
287		PLLY(3)=5.	1	28	003010	3
288		PLLY(4)=3.	3	28	003013	2
289		PLLY(5)=3.	3	28	003016	1
290		IF (BIG(5).GT.BIG(4)) PRJ(4)=PRJ(5)	5	28	003021	0
291		IF (BIG(4).GT.BIG(5)) PRJ(5)=PRJ(4)	5	28	003030	1
292		IF (BIG(5).GT.BIG(4)) BIG(4)=BIG(5)	1	28	003037	2
293		IF (BIG(4).GT.BIG(5)) BIG(5)=BIG(4)	1	28	003046	3
294		DO 305 J=1,5	1	28	003055	4
295		PYS(J)=BIG(J)/PLLY(J)	1	28	003060	5
296		PUNIT(J)=PRJ(J)/5.	1	28	003065	3
297	305	IUNIT(J)=BIG(J)/PRJ(J)+1.001	1	28	003071	4
298		BIG(1)=BIG(1)+1.25	1	28	003100	2
299		CALL FSCALE (-3.*PYS(2),0.,PYS(2),PYS(3),BIG(3),1H,0.,0.)	1	28	003104	3
300		I3=BIG(2)/PUNIT(2)	1	28	003117	4
301		I2W=1	1	28	003123	5
302		DO 309 I=1,I3	1	28	003125	5
303		XUSE=PUNIT(2)*I	1	28	003131	1
304		CALL FPLOT (XUSE,0.)	1	28	003134	5
305		CALL FPLOT (XUSE,-.05*PYS(3))	1	28	003141	2

306	IF (I.NE.I2W*5) GO TO 309	1	28	003147	3
307	I2W=I2W+1	1	28	003154	5
308	CALL FPLLOT (XUSE,.05*PYS(3))	3	28	003157	3
309	CALL FMOVE (XUSE-.25*PYS(2),-.2*PYS(3))	1	28	003165	3
310	ENCODE (3,PLX) XUSE	1	28	003175	2
311	CALL FCHAR (0.,.12,4,PLX)	1	28	003201	4
312	309 CALL FMOVE (XUSE,0.)	1	28	003207	0
313	CALL FPLLOT (BIG(2),0.)	1	28	003214	1
314	IF (IUNIT(3).GT.BIG(3)) IUNIT(3)=IUNIT(3)/10.+1	5	28	003221	0
315	I2=BIG(3)/IUNIT(3)	1	28	003232	1
316	DO 308 I=1,I2	1	28	003236	2
317	YUSE=IUNIT(3)*I	1	28	003241	4
318	CALL FPLLOT (BIG(2),YUSE)	1	28	003245	2
319	CALL FPLLOT (BIG(2)+.05*PYS(2),YUSE)	1	28	003252	3
320	CALL FMOVE (BIG(2)+.1*PYS(2),YUSE-.06*PYS(3))	1	28	003261	4
321	ENCODE (3,PLY) YUSE	1	28	003272	3
322	CALL FCHAR (0.,.12,4,PLY)	1	28	003276	5
323	308 CALL FMOVE (BIG(2),YUSE)	1	28	003304	1
324	CALL FPLLOT (BIG(2),BIG(3))	6	28	003312	1
325	CALL FMOVE (BIG(2)+.3*PYS(2),YUSE-.25*PYS(3))	12	28	003317	4
326	CALL FCHAR (0.,.12,3,3HFT.)	6	28	003330	3
327	DO 310 J=IULT,1,-1	1	28	003336	1
328	CALL FMOVE (RES(J+10)-.06*PYS(2),RES(J+20)-.06*PYS(3))	14	28	003342	2
329	310 CALL FSCHAR (0.,.12,62)	1	28	003354	5
330	CALL FMOVE (0.,0.)	1	28	003362	3
331	CALL FSCALE (-3.*PYS(2),0.,PYS(2),PYS(1),BIG(1),1H,0.,0.)	1	28	003366	4
332	I1=BIG(1)/PUNIT(1)	1	28	003401	5
333	I1W=1	6	28	003406	0
334	DO 306 I=1,I1	1	28	003410	0
335	YUSE=PUNIT(1)*I	10	28	003413	2
336	CALL FPLLOT (0.,YUSE)	1	28	003417	0
337	CALL FPLLOT (-.05*PYS(2),YUSE)	1	28	003423	3
338	IF (I.NE.I1W*5) GO TO 306	11	28	003431	4
339	I1W=I1W+1	6	28	003437	0
340	CALL FPLLOT (.05*PYS(2),YUSE)	3	28	003441	4
341	CALL FMOVE (-.9*PYS(2),YUSE-.05*PYS(1))	1	28	003447	4
342	ENCODE (2,PLY) YUSE	1	28	003457	3
343	CALL FCHAR (0.,.12,8,PLY)	1	28	003463	5
344	306 CALL FMOVE (0.,YUSE)	1	28	003471	1
345	CALL FPLLOT (0.,BIG(1))	10	28	003476	2
346	CALL FMOVE (-.5*PYS(2),(I1W-1)*PRJ(1)-.2*PYS(1))	13	28	003503	1
347	CALL FCHAR (0.,.12,4,4HTONS)	6	28	003514	3
348	CALL FMOVE(1.3*PYS(2),4.6*PYS(1))	10	28	003522	3
349	CALL FCHAR (0.,.12,12,12HRESISTANCE=X)	6	28	003531	2
350	CALL FMOVE (3.4*PYS(2),4.6*PYS(1))	10	28	003541	0
351	CALL FCHAR (0.,.12,7,7HSTROKE=)	6	28	003550	0
352	CALL FSCHAR (0.,.12,62)	6	28	003556	3
353	DO 307 J=IULT,1,-1	1	28	003563	3
354	CALL FMOVE (RES(J+10)-.06*PYS(2),RES(J)-.06*PYS(1))	13	28	003567	4
355	307 CALL FSCHAR (0.,.12,93)	1	28	003601	3
356	CALL FMOVE (2.3*PYS(2),-.4*PYS(1))	10	28	003607	1
357	CALL FCHAR (0.,.12,14,14HBLOWS PER FOOT)	13	28	003616	1
358	CALL FMOVE (0.,5.*PYS(1))	3	28	003626	1
359	CALL FSCALE (-3.*PYS(2),0.,PYS(2),PYS(4),BIG(4),1H,0.,0.)	3	28	003633	3

360		I4=BIG(4)/PUNIT(4)	3	28	003646	4
361		DO 311 I=1,I4	3	28	003652	5
362		YUSE=PUNIT(4)*I	6	28	003656	1
363		CALL FPLLOT (0.,YUSE)	3	28	003661	5
364		CALL FPLLOT (-.05*PYS(2),YUSE)	3	28	003666	2
365		DO 316 J=1,10	3	28	003674	3
366		XUSE=J*PRJ(4)	3	28	003677	5
367		IF (ABS(XUSE-YUSE).LT.0.001) GO TO 317	3	28	003703	1
368	316	CONTINUE	3	28	003712	5
369		GO TO 311	3	28	003716	0
370	317	CALL FPLLOT (.05*PYS(2),YUSE)	3	28	003720	4
371		CALL FMOVE (-.9*PYS(2),YUSE-.05*PYS(4))	3	28	003727	2
372		ENCODE (2,PLY) YUSE	3	28	003737	1
373		CALL FCHAR (0.,.12,8,PLY)	3	28	003743	3
374		FY=YUSE	10	28	003750	5
375	311	CALL FMOVE (0.,YUSE)	3	28	003753	1
376		CALL FPLLOT (0.,BIG(4))	6	28	003760	2
377		CALL FMOVE (-.4*PYS(2),FY-.2*PYS(4))	11	28	003765	1
378		CALL FCHAR (0.,.12,3,3HKSI)	6	28	003774	3
379		CALL FMOVE (1.2*PYS(2),2.8*PYS(4))	6	28	004002	1
380		CALL FMOVE (.9*PYS(2),2.8*PYS(4))	10	28	004011	1
381		CALL FCHAR (0.,.12,19,19HCOMPRESSIVE STRESS=)	6	28	004020	0
382		CALL FSCHAR (0.,.12,4)	6	28	004030	5
383		CALL FMOVE (3.4*PYS(2),2.8*PYS(4))	10	28	004035	4
384		CALL FCHAR (0.,.12,16,16HTENSILE STRESS=)	6	28	004044	4
385		DO 313 J=IULT,1,-1	6	28	004055	0
386		CALL FMOVE (RES(J+10)-.06*PYS(2),ABS(RES(J+30))-0.06*PYS(4))	13	28	004061	1
387	313	CALL FSCHAR (0.,.12,40)	6	28	004074	3
388		CALL FMOVE (0.,0.)	10	28	004102	1
389		I2W=1	6	28	004106	2
390		DO 314 I=1,I3	6	28	004110	2
391		XUSE=PUNIT(2)*I	6	28	004113	4
392		CALL FPLLOT (XUSE,0.)	6	28	004117	2
393		CALL FPLLOT (XUSE,-.05*PYS(4))	6	28	004123	5
394		IF (I.NE.I2W*5) GO TO 314	6	28	004132	0
395		I2W=I2W+1	6	28	004137	2
396		CALL FPLLOT (XUSE,.05*PYS(4))	6	28	004142	0
397		CALL FMOVE (XUSE-.25*PYS(2),-.2*PYS(4))	6	28	004150	0
398		ENCODE (3,PLX) XUSE	6	28	004157	5
399		CALL FCHAR (0.,.12,4,PLX)	6	28	004164	1
400	314	CALL FMOVE (XUSE,0.)	6	28	004171	3
401		CALL FPLLOT (BIG(2),0.)	6	28	004176	4
402		CALL FPLLOT (BIG(2),BIG(4))	6	28	004203	3
403		CALL FPLLOT (0.,BIG(4))	10	28	004211	0
404		DO 315 J=1,IULT	6	28	004215	5
405		CALL FMOVE (RES(J+10)-.06*PYS(2),RES(J+40)-.06*PYS(4))	13	28	004221	3
406	315	CALL FSCHAR (0.,.12,4)	6	28	004233	5
407		CALL FMOVE (.75*PYS(2),3.2*PYS(4))	10	28	004241	2
408		DO 320 I=1,10	10	28	004250	2
409	320	CALL FCHAR (0.,.15,4,TITLE(I))	10	28	004253	4
410		CALL FMOVE (0.,4,5*PYS(4))	6	28	004262	4
411		RETURN	6	28	004270	1
412		END	6	28	004272	2

SOILN		SYMBOLIC FORTRAN 30 APR 76 14:19:06 8 12 5 00147213 663			
1		SUBROUTINE SOILN (ITYS,IPERCS,N,XPT,SU,IR,ALPH)	2	12	000004 0
2			1	12	000015 1
3	C	THIS PROGRAM FINDS THE PERCENTAGE OF TOTAL PILE RESISTANCE THAT ACTS	1	12	000016 0
4	C	AT INDIVIDUAL ELEMENTS	1	12	000033 1
5	C	NOTE THAT THIS PROGRAM ASSUMES THAT PILE ELEMENTS ARE OF EQUAL LENGTH	1	12	000040 2
6	C	EXCEPT FOR TOP AND BOTTOM. THESE HAVE 1/2 THE LENGTH OF THE OTHERS.	1	12	000055 4
7	C	DIS IS AN ARRAY THAT CONTAINS DISTRIBUTIONS OF STATIC SOIL RESISTANCE	1	12	000072 4
8	C	FIRST INDEX = TYPE, SECOND INDEX 1=LOCATION 2=PERCENTAGE	1	12	000110 0
9	C	THIRD INDEX (MAX=6) FOR VALUES	1	12	000123 1
10	C	BETWEEN GIVEN POINTS OF CHANGE OF DISTRIBUTION THE RESISTANCE IS	1	12	000131 5
11	C	ASSUMED TO VARY LINEARLY	1	12	000146 2
12	C	XPT IS THE TOTAL PILE LENGTH	1	12	000153 5
13	C	PERC* STANDS FOR PERCENTAGE OF TOTAL SKIN RESISTANCE AT *	1	12	000162 1
14		DIMENSION SU(20),DIS(11,2,20),ALPH(99)	1	12	000175 3
15		DO 201 I=1,11	6	12	000205 1

16		DO 201 J=1,2	1	12	000210	3
17		DO 201 K=1,20	1	12	000213	4
18		DIS(I,J,K)=0.0	1	12	000217	0
19	201	CONTINUE	1	12	000222	3
20		IF (ITYS.LT.1) GO TO 300	11	12	000233	0
21		DIS(1,2,1)=1.	1	12	000240	1
22		DIS(5,2,1)=0.5	1	12	000243	3
23		DIS(6,2,1)=1./3.	1	12	000247	0
24		DIS(7,2,1)=1.	1	12	000252	5
25		DIS(8,2,1)=1.	1	12	000256	1
26		DIS(9,2,1)=0.5	1	12	000261	3
27		DIS(10,2,1)=1.	1	12	000265	0
28		DO 202 I=1,10	1	12	000270	3
29		DO 202 J=1,2	1	12	000273	5
30		DIS(I,J,2)=1.	1	12	000277	0
31	202	CONTINUE	1	12	000302	2
32		DO 203 I=3,5	1	12	000305	4
33		DO 203 J=1,2	1	12	000310	5
34		DIS(I,J,2) = 0.5	1	12	000314	0
35	203	CONTINUE	1	12	000317	5
36		DIS(2,1,2)=0.25	1	12	000323	1
37		DIS(4,1,2)=0.75	1	12	000326	5
38		DO 213 I=2,4	1	12	000332	3
39	213	DIS(I,2,2) = 0.0	1	12	000335	4
40		DIS(6,2,2)=1./3.	1	12	000342	2
41		DIS(9,2,2)=0.5	1	12	000346	1
42		DO 204 I=6,10	1	12	000351	4
43		DIS(I,1,2)=1./3.	1	12	000355	0
44	204	CONTINUE	1	12	000360	5
45		DIS(8,1,2)=2./3.	1	12	000364	1
46		DO 205 I=2,5	1	12	000370	0
47		DIS(I,2,3)=1.	1	12	000373	1
48	205	CONTINUE	1	12	000376	3
49		DO 206 I=6,10	1	12	000401	5
50	206	DIS(I,1,3)=1./3.	1	12	000405	1
51		DO 207 I=7,10	1	12	000411	5
52	207	DIS(I,2,3)=0.5	1	12	000415	1
53		DIS(2,1,3)=0.25	1	12	000421	3
54		DIS(3,1,3)=0.5	1	12	000425	1
55		DIS(4,1,3)=0.75	1	12	000430	4
56		DIS(5,1,3)=0.5	1	12	000434	2
57		DIS(6,2,3)=2./3.	1	12	000437	5
58		DIS(8,1,3)=2./3.	1	12	000443	4
59		DIS(9,2,3)=1.	1	12	000447	3
60		DO 208 I=1,9	1	12	000452	5
61		DO 208 J=1,2	1	12	000456	0
62	208	DIS(I,J,4)=1.	1	12	000461	1
63		DIS(6,1,4)=2./3.	1	12	000465	2
64		DIS(6,2,4)=2./3.	1	12	000471	1
65		DIS(7,2,4)=0.5	1	12	000475	0
66		DIS(8,2,4)=0.5	1	12	000500	3
67		DIS(9,1,4)=2./3.	1	12	000504	0
68		DIS(10,1,4) = 2./3.	1	12	000507	5
69		DIS(10,2,4)=0.5	1	12	000514	1

70		DO 209 I=6,10							
71	209	DIS(I,1,5)=2./3.				1	12	000517	5
72		DO 210 I=6,10				1	12	000523	1
73		DIS(I,2,5)=1.				1	12	000527	5
74	210	CONTINUE				1	12	000533	1
75		DIS(9,2,5)=0.5				1	12	000536	3
76		DO 211 I=6,10				1	12	000541	5
77		DO 211 J=1,2				1	12	000545	2
78		DIS(I,J,6)=1.				1	12	000550	4
79	211	CONTINUE				1	12	000553	5
80		DIS(9,2,6)=0.5				1	12	000557	1
81		GO TO 400				1	12	000562	3
82	500	CONTINUE				1	12	000566	0
83	C					1	12	000570	4
84	C	FOR INPUT OF SOIL DISTRIBUTION				12	12	000574	0
85	C					12	12	000575	1
86	300	ITYS=11				12	12	000603	4
87	2	FORMAT (2F8.3)				1	12	000604	5
88		DO 306 I=1,20				1	12	000607	5
89		READ (IR,2) DIS(11,1,1),DIS(11,2,1)				1	12	000613	5
90		I1=1				1	12	000617	1
91		IF (DIS(11,1,1).GT.0.0) I1=I1+1				6	12	000626	2
92		DISX=DIS(11,1,1)				6	12	000630	1
93		DIS(11,1,I1)=DIS(11,1,1)/XPT				6	12	000636	4
94		DIS(11,2,I1)=DIS(11,2,1)/100.				6	12	000642	3
95		IF (DISX.GE.XPT-0.00001) GO TO 400				8	12	000650	3
96	306	CONTINUE				1	12	000656	4
97	400	XU=0.0				1	12	000665	4
98		IF (DIS(11,1,1).GT.0.0) DIS(11,1,2)=0.0				1	12	000670	5
99		IF (DIS(11,1,1).GT.0.0) DIS(11,1,1)=0.0				9	12	000713	2
100		PERCS=IPERCS				9	12	000723	1
101		PERCS = PERCS*0.01				1	12	000733	0
102		IF (PERCS.LT.10.0E-8) PERCS = 10.0E-8				1	12	000736	1
103		SUMT=0.0				10	12	000742	2
104		SALPH=0.0				1	12	000751	5
105		DO 110 I=1,N				3	12	000754	2
106	110	SALPH=SALPH+ALPH(I)				3	12	000757	0
107		DO 100 I=1,N				3	12	000762	1
108		XO=XU				1	12	000767	1
109		DL=ALPH(I)/SALPH				1	12	000772	2
110		XU=XO+DL				3	12	000774	2
111		SUM=0.0				1	12	001000	1
112		DO 101 J=2,20				1	12	001002	4
113		XXO = DIS(ITYS,1,J)				5	12	001005	0
114		IF (XXO.GT.XO) GO TO 102				1	12	001010	2
115	101	CONTINUE				1	12	001014	4
116	102	CONTINUE				1	12	001021	5
117		J1=J-1				1	12	001025	1
118		XXO=XO				1	12	001030	3
119		PERCO=DIS(ITYS,2,J1)+ ((DIS(ITYS,2,J)-DIS(ITYS,2,J1))/(DIS(ITYS,				1	12	001032	4
120		1 1,J) - DIS(ITYS,1,J1))* (XO-DIS(ITYS,1,J1))				1	12	001034	5
121		DO 103 IJ = J,20				1	12	001051	0
122		XXU=DIS(ITYS,1,IJ)				5	12	001061	5
123		IF (XXU.GT.XU-10E-6) GO TO 104				1	12	001065	4
						5	12	001071	5

124		PERCU=DIS(ITYS,2,IJ)	1	12	001100	1
125		SUM = SUM + (PERCO+PERCU)*0.5*(XXU-XXO)	1	12	001104	4
126		XXO=XXU	1	12	001114	3
127		PERCO=PERCU	1	12	001116	5
128	103	CONTINUE	1	12	001121	5
129	104	CONTINUE	1	12	001125	1
130		XXU=XXU	1	12	001130	3
131		J=IJ	1	12	001132	4
132		J1=IJ-1	1	12	001134	3
133		PERCU=DIS(ITYS,2,J1)+ ((DIS(ITYS,2,J)-DIS(ITYS,2,J1))/(DIS(ITYS,	1	12	001136	5
134		1 1,J) - DIS(ITYS,1,J1)))*(XU-DIS(ITYS,1,J1))	1	12	001153	0
135		SUM = SUM + (PERCO+PERCU)*0.5*(XXU-XXO)	1	12	001163	5
136		SU(I) = SUM	1	12	001173	4
137		SUMT = SUMT + IM	1	12	001176	4
138	100	CONTINUE	1	12	001202	4
139		DO 105 I=1,N	1	12	001206	0
140	105	SU(I)=SU(I)*PERCS/SUMT	1	12	001211	1
141		SU(N+1)=1.-PERCS	4	12	001216	5
142		RETURN	1	12	001222	4
143		END	1	12	001224	5

SRESN							SYMBOLIC FORTRAN 02 JUL 75 15:54:44 1 1 5 00035127 60		
1		SUBROUTINE SRESN (DNP,DOP,SU,SOK,I,RESO,RESN,ESOIL)	1	1	000000	0			
2		DIMENSION DNP(20),DOP(20),SU(20),SOK(20),RESO(20),RESN(20)	1	1	000011	5			
3		SKE=SOK(I).	1	1	000025	0			
4		IF (DNP(I).LT.DOP(I)) SKE=SKE/ESOIL	1	1	000027	5			
5		DRES=(DNP(I)-DOP(I))*SKE	1	1	000037	0			
6		RESN(I)=RESO(I)+DRES	1	1	000044	1			
7		IF (RESN(I).LT.-SU(I)) RESN(I) = -SU(I)	1	1	000050	4			
8		IF (RESN(I).GT.SU(I)) RESN(I)=SU(I)	1	1	000060	3			
9		RETURN	1	1	000067	4			
10		END	1	1	000071	5			

JJNP							SYMBOLIC FORTRAN 02 MAR 76 16:50:40 1 4 5 00057663 283		
1		SUBROUTINE JJNP(ICOL,N,JNP,JMAX)	2	4	000011	5			
2		DIMENSION JNP(13)	1	4	000020	3			
3		DO 100 I=1,13	1	4	000024	3			
4	100	JNP(I)=0	1	4	000027	5			
5		JNP(1)=1	1	4	000033	0			
6		JNP(2)=2	1	4	000035	3			
7		IF (ICOL.GT.0) GO TO 200	1	4	000040	0			
8		IF (N.GT.6) GO TO 107	1	4	000045	1			
9		DO 105 I=3,N	1	4	000051	5			
10	105	JNP(I)=I	1	4	000055	0			
11		JMAX=N	1	4	000060	1			
12		GO TO 400	1	4	000062	2			
13	107	CONTINUE	2	4	000067	3			
14		JNP(6) = N	2	4	000072	5			
15		JMAX = 6	2	4	000075	4			
16		IF (N.GT.20) GO TO 301	2	4	000100	1			
17		II = 1	2	4	000105	0			
18		DO 110 I=6,18,3	1	4	000107	1			
19		IF (N.GT.I.AND.N.LE.I+2) GO TO 120	1	4	000112	5			
20	110	II=II+1	1	4	000121	5			

21	120	NCOUNT=I1	1	4	000124	5
22		DO 130 I=3,5	1	4	000130	1
23	130	JNP(I)=JNP(I-1) + NCOUNT	1	4	000133	2
24		GO TO 400	1	4	000146	4
25	301	JNP(3) = N/4	2	4	000151	2
26		JNP(4) = N/2	2	4	000155	2
27		JNP(5) = N*3/4	2	4	000160	3
28		GO TO 400	2	4	000164	0
29	C	FILLS IN THE INP ARRAY ONLY IF 132 COLUMNS IN PRINTOUT	1	4	000166	4
30	200	IF (N.GT.13) GO TO 230	1	4	000201	2
31		DO 220 I=1,N	1	4	000206	5
32	220	JNP(I)=I	1	4	000212	0
33		JMAX=N	1	4	000215	1
34		GO TO 400	1	4	000217	2
35	230	JMAX = 13	2	4	000225	5
36		JNP(13) = N	3	4	000231	2
37		IF (N.GT.20) GO TO 302	2	4	000234	2
38		DO 235 I=1,4	2	4	000241	1
39		JNP(I) = I	1	4	000244	2
40		J1 = 14 - I	1	4	000247	1
41		J2 = (N+1) - I	1	4	000252	1
42	235	JNP(J1)=J2	1	4	000255	4
43		IF (N.EQ.17) JNP(5)=5	1	4	000261	1
44		DO 240 I=5,9	1	4	000265	5
45		IF (N.GE.14.AND.N.LE.16) JNP(I) = I+(I+2)/5	4	4	000300	1
46		IF (N.EQ.17.AND.I.NE.5) JNP(I) = JNP(I-1) + 2	1	4	000310	4
47		IF (N.GT.17) JNP(I) = JNP(I-1) + 2	1	4	000321	3
48	240	CONTINUE	1	4	000330	3
49		GO TO 400	2	4	000336	1
50	302	CONTINUE	2	4	000340	5
51		NDEL = 0.1*FLOAT(N) + 0.5	2	4	000344	1
52		JNP(8) = N/2	2	4	000351	3
53		JNP(7) = JNP(8)-1	2	4	000354	4
54		DO 303 I=3,5	2	4	000360	4
55	303	JNP(I) = 2 + (I-2)*NDEL	2	4	000363	5
56		I2 = 0	2	4	000371	4
57		DO 304 I=10,I2	2	4	000373	5
58		I1 = 22-I	2	4	000377	2
59		I2 = I2 + 1	2	4	000402	0
60	304	JNP(I1) = N-I2*NDEL	2	4	000405	0
61		JNP(9) = JNP(8) + NDEL	2	4	000412	1
62		JNP(6) = JNP(7) - NDEL	2	4	000417	0
63	400	CONTINUE	1	4	000423	5
64		RETURN	1	4	000427	0
65		END	1	4	000431	1

STIFF		SYMBOLIC FORTRAN 23 SEP 75 10:02:18 3 7 5 00035417				114
1		SUBROUTINE STIFF (DDIS,DVEL,STL,STR,FO,FN,DSAC,DDD)	1	7	000004	0
2		DLST = STR - STL	1	7	000015	5
3		STI=STL+2.*DVEL*DLST	7	7	000027	5
4		IF (STI.LT.STL) STI = STL	1	7	000034	2
5		IF (STI.GT.STR) STI = STR	1	7	000041	4
6		ST1=STI*DDD/(2.0*DSAC)	2	7	000056	1
7		STQUAD=ST1*ST1-FO*STI/DSAC	2	7	000063	0

8	IF (STQUAD.LT.0) STQUAD=0	2	7	000070	3
9	STIS=ABS(ST1)+SQRT(STQUAD)	4	7	000107	3
10	IF (STIS.GT.ST1) STIS = ST1	1	7	000115	0
11	IF (STIS.LT.0.0) STIS = 0.0	1	7	000122	4
12	FN = FO + DDD*STIS	1	7	000130	2
13	1 FORMAT (2F10.2,3F10.4,3F10.0,F10.4)	5	7	000134	3
14	RETURN	1	7	000155	1
15	END	1	7	000157	2

INTEGR						SYMBOLIC FORTRAN 04 SEP 74 11:58:39 1 1 5 00035056						41		
1	SUBROUTINE INTEGR(DT,XO,XN,XIO,XIIO,XIN,XIIN,I)										1	1	000000	0
2	DIMENSION XO(10),XN(10),XIO(10),XIIO(10),XIN(10),XIIN(10)										1	1	000011	1
3	XIN(I)=XIO(I)+0.5*(XO(I)+XN(I))*DT										1	1	000024	1
4	XIIN(I)=XIIO(I)+XIO(I)*DT+(XO(I)*2.+XN(I))*DT*DT/6.										1	1	000033	1
5	RETURN										1	1	000045	0
6	END										1	1	000047	1

FTR						SYMBOLIC FORTRAN 19 JAN 76 13:39:42 1 4 5 00036033						198		
1	FUNCTION FTR (DPOS,PBAR,IWT)										1	4	000003	0
2	DIMENSION PBAR (20)										1	4	000011	0
3	IWT=0										1	4	000015	2
4	PATM=14.7*0.144										1	4	000017	2
5	ART=PBAR(1)										1	4	000023	0
6	DSF=PBAR(2)										1	4	000034	1
7	EXPB=PBAR(4)										1	4	000037	1
8	VCT=PBAR(5)										1	4	000042	2
9	DEPBB=PBAR(6)										1	4	000045	2
10	DBBT=PBAR(7)										1	4	000050	4
11	RWH=PBAR(8)+PATM*ART										2	4	000056	5
12	DBC=PBAR(7)+DPOS										1	4	000063	2
13	VBIN = DEPBB*ART + VCT										1	4	000067	1
14	IF (DBC.GT.DEPBB.OR.DBBT.LT.0.1) FTR=PATM*ART										1	4	000074	0
15	IF (DBC.GT.DEPBB.OR.DBBT.LT.0.1) RETURN										1	4	000104	5
16	VBFIN=DBC*ART+VCT										1	4	000114	4
17	VSF=DSF*ART										1	4	000120	4
18	IF (DBC.LT.DSF) VBFIN=VSF+VCT										1	4	000123	4
19	FTR=PATM*ART*(VBIN/VBFINT)**EXPB										1	4	000131	5
20	IF (DBC.LT.DSF) FTR=FTR*(VSF/(ART*DBC))**EXPB										1	4	000140	2
21	IF (FTR.GT.RWH) IWT=1										1	4	000151	1
22	IF (FTR.GT.RWH) FTR=RWH										1	4	000155	5
23	RETURN										1	4	000162	5
24	C	ART ... AREA AT TOP OF RAM								1	4	000165	0	
25	C	DSF ... DISTANCE OF SAFETY CHAMBER FROM TOP								1	4	000172	5	
26	C	EXPB ... EXPONENT								1	4	000203	4	
27	C	VCT ... VOLUME OF COMPRESSION TANK								1	4	000210	0	
28	C	DEPBB ... DISTANCE OF EXHAUST PORTS FROM TOP								1	4	000217	2	
29	C	RWH ... WEIGHT OF CYLINDER ETC.								1	4	000230	2	
30	C	DBBT ... DISTANCE BETWEEN TOP AND BOTTOM - RAM LENGTH								1	4	000237	1	
31	C	VBIN ... INITIAL VOLUME OF BOUNCE CHAMBER								1	4	000251	4	
32	C	VBFIN ... FINAL VOLUME OF BOUNCE CHAMBER								1	4	000262	1	
33	C	DBC ... CURRENT POSITION ... DISTANCE FROM TOP								1	4	000272	3	
34	END										1	4	000303	5

Δ FOR WINIT,WINIT
FOR OF JANUARY 20, 1976
HIGHEST S.E. ADDRESS: 03110

THURSDAY, FEBRUARY 12, 1976 AT 12:26:48

MAIN PROGRAM

STORAGE USED (BLOCK, NAME, LENGTH)

0001 CODE* 000053
0000 DATA* 012135

EXTERNAL REFERENCES (BLOCK, NAME)

0002 NINTR\$
0003 NOSBFU
0004 NIO1\$
0005 NIO2\$
0006 NSTOP\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000007 110G 0001 000010 113G 0001 000013 117G 0001 000032 126G 0001 000037 132G
0001 000040 135G 0000 012125 5F 0000 R 012120 BLANK 0000 I 012122 I. 0000 I 012121 IF
0000 I 012123 IJ 0000 I 012124 KJ 0000 R 000000 STORE

114

000101	1.		DIMENSION STORE(100,2,26)	000000
000103	2.		DATA BLANK/4H /	000001
000105	3.		IF = 7	000001
000106	4.	5	FORMAT (2A4/6(8E10.5/),2E10.5)	000010
000107	5.		DO 200 I=1,100	000010
000112	6.		DO 200 IJ=1,2	000010
000115	7.		STORE (I,IJ,1) = BLANK	000010
000116	8.		DO 200 KJ=2,26	000013
000121	9.	200	STORE (I,IJ,KJ) = 0.0	000013
000125	10.		DO 100 I=1,100	000032
000130	11.	100	WRITE (IF,5) ((STORE(I,IJ,KJ),IJ=1,2),KJ=1,26)	000032
000144	12.		END	000052

NO DIAGNOSTICS.

1.8402 SECONDS.

```

000101 1. DIMENSION STORE(100,2,26)
000103 2. IF = 7
000104 3. IW = 6
000105 4. IR = 5
000106 5. MAX = 100
000107 6. 3 FORMAT (20I4)
000110 7. READ (IR,3) IH,IOPT
000114 8. 5 FORMAT (2A4/6(8E10.5/),2E10.5)
000115 9. 8 FORMAT (2A4,9F8.2)
000116 10. 9 FORMAT (10F8.2)
000117 11. 10 FORMAT (1H,2A4,4(F8.3,F8.0),F8.3)
000120 12. 11 FORMAT (1H,2(F8.0,F8.3),2F8.3,F8.0,F8.4,2F8.2)
000121 13. 12 FORMAT (1H,F8.1,2F8.2,5F8.1,F8.3,F8.2)
000122 14. 13 FORMAT (1H,10F8.3)
000123 15. 14 FORMAT (1H,F8.3,3(F8.3,F8.0),F8.2,2F8.3)
000124 16. IF (NH.LT.1) GO TO 500
000126 17. IHO = 1
000127 18. DO 100 I = 1,NH
000132 19. READ (IR,3) IH
000135 20. DO 101 J = IHO,IH
000140 21. 101 READ (IF,5) ((STORE(J,IJ,KJ),IJ=1,2),KJ=1,26)
000154 22. READ (IP,6) ((STORE(IH,IJ,KJ),IJ=1,2),KJ=1,5),STORE(IH,1,6)
000170 23. READ (IR,9) STORE(IH,2,6),((STORE(IH,IJ,KJ),IJ=1,2),KJ=7,25),STORE
000170 24. 1 (IH,1,26)
000205 25. WRITE (IW,3) IH
000210 26. WRITE (IW,10) ((STORE(IH,IJ,KJ),IJ=1,2),KJ=1,5),STORE(IH,1,6)
000224 27. WRITE (IW,11) STORE(IH,2,6),((STORE(IH,IJ,KJ),IJ=1,2),KJ=7,10),
000224 28. * STORE(IH,1,11)
000241 29. WRITE (IW,12) STORE(IH,2,11),((STORE(IH,IJ,KJ),IJ=1,2),KJ=12,15),
000241 30. * STORE(IH,1,16)
000256 31. WRITE (IW,13) STORE(IH,2,16),((STORE(IH,IJ,KJ),IJ=1,2),KJ=17,20),
000256 32. * STORE(IH,1,21)
000273 33. WRITE (IW,14) STORE(IH,2,21),((STORE(IH,IJ,KJ),IJ=1,2),KJ=22,25),
000273 34. * STORE(IH,1,26)
000310 35. DO 103 IJ=2,7
000313 36. 103 STORE(IH,2,IJ) = STORE(IH,2,IJ)*12.0
000315 37. STORE(IH,2,9) = STORE(IH,2,9)*12.0
000315 38. DO 104 IJ=22,24
000321 39. 104 STORE(IH,2,IJ) = STORE(IH,2,IJ)*12.0
000323 40. IHO = IH + 1
000324 41. 100 CONTINUE
000326 42. DO 400 I = IHO,MAX
000331 43. 400 READ (IF,5) ((STORE(I,IJ,KJ),IJ=1,2),KJ=1,26)
000345 44. REWIND IF
000346 45. DO 200 I = 1,MAX
000351 46. 200 WRITE (IF,5) ((STORE(I,IJ,KJ),IJ=1,2),KJ=1,26)
000365 47. 500 CONTINUE
000365 48. C CHECK IT
000366 49. REWIND IF
000367 50. DO 300 I = 1,MAX
000372 51. READ (IF,5) ((STORE(I,IJ,KJ),IJ=1,2),KJ=1,26)
000405 52. IF (STORE(I,2,8).LT.1.0) GO TO 300

```

115

*NEW 000020
*NEW 000020
*NEW 000020
*NEW 000020
*NEW 000020
**=2 000020
000025
000027
000040
000046
000103
000121
000141
000141
000162
000170
*NEW 000210
*NEW 000231
*NEW 000231
*NEW 000252
*NEW 000252
*NEW 000273
*NEW 000273
**=2 000320
000320
000323
000333
000333
000336
000342
000342
000351
000367
000375
000375
000414
000414
000414
000416
000424
000437

000407	53.	WRITE (IW,6) I, (STORE(I,IJ,I), IJ=1,2), STOPF(I,2,8), STORE(I,2,25),			000404
000407	54.	* STORE(I,1,26)			000404
000422	55.	IF (IOPT.GT.0) WRITE (IW,10) ((STORE(I,IJ,KJ), IJ=1,2), KJ=1,5), STORE			000467
000422	56.	I (I,1,6)			000467
000437	57.	IF (IOPT.GT.0) WRITE (IW,11) STORE(I,2,6), ((STORE(I,IJ,KJ), IJ=1,2),		*NEW	000526
000437	58.	* KJ=7,10), STORE(I,1,11)		*NEW	000526
000455	59.	IF (IOPT.GT.0) WRITE (IW,12) STORE(I,2,11), ((STORE(I,IJ,KJ), IJ=1,2)		*NEW	000566
000455	60.	* KJ=12,15), STORE(I,1,16)		*NEW	000566
000473	61.	IF (IOPT.GT.0) WRITE (IW,13) STORE(I,2,16), ((STORE(I,IJ,KJ), IJ=1,2)		*NEW	000626
000473	62.	* KJ=17,20), STORE(I,1,21)		*NEW	000626
000511	63.	IF (IOPT.GT.0) WRITE (IW,14) STORE(I,2,21), ((STORE(I,IJ,KJ), IJ=1,2)		*NEW	000670
000511	64.	* KJ=22,25), STORE(I,1,26)		*NEW	000670
000527	65.	6 FORMAT (I5,4X,2A4,3F5.1)		**2	000737
000530	66.	300 CONTINUE			000737
000532	67.	REWIND IF			000737
000533	68.	END			000744

NO DIAGNOSTICS.

3.4896 SECONDS.

DELETED:	WDATA	SYMBOLIC FORTRAN	19 JAN 76	16:57:00	1	1	5	00165622	290		
DELETED:	WDATA	RELOCATABLE	19 JAN 76	16:57:00				00166264	518	00167272	20

APPENDIX B

INPUT DATA FORMS

HAMMER DATA - FILE LOADING FORM

(Include as first card of run (214); Number of hammers to be loaded; print option: =0 minimum, =1 maximum)

Hammer I.D.

(Hammer I.D.=Integer; all other data real numbers)

Name	HM(1) (kips)	STH(1) (k/in)	HM(2) (kips)	STH(2) (k/in)	HM(3) (kips)	STH(3) (k/in)	HM(4) (kips)	STH(4) (k/in)	HM(5) (kips)
							OED, CED, VCD	OED, CED, VCD	
STH(5) (k/in)	HM(6) (kips)	STH(6) (k/in)	HM(7) (kips)	M	HM(M+1)(kips)	STH(M) (k/in)	TDEL (sec)	TGI (sec)	
OED, CED, VCD			OED CED VCD DAS	OED, CED, VCD					
VFIW (in ³)	DEPIB (in)	ARAM (in ²)	PI (psi)	P2 (psi)	P3 (psi)	P4 (psi)	P5 (psi)	EFFICY	SIRM (in)
OED, CED, VCD	CED	CED	CED	CED	CED	CED	CED	CED	CED
EXPP	ART (ft ²)	DEPBB (ft)	DSF (ft)	DBBT (ft)	POWSCA (l)	RWH (kips)	EX PB	DINJ (ft)	VCT (ft ³)
VCD only	A/S, DAS								
RAM (kips)	AM(1) (kips)	STA(1) (k/in)	AM(2) (kips)	STA(2) (k/in)	AM(3) (kips)	STA(3) (k/in)		MA	ITYPH

Hammer Types: OED Open End Diesel ITYPH=1.001
 CED Closed End Diesel =2.001
 VCD Vacuum Chamber Diesel =1.001
 A/S Single Acting Air/Steam =3.001
 DAS Double Acting Air/Steam =3.001

If not otherwise noted information to be provided for all hammer types.

ELECTRONIC COMPUTER PROGRAM ABSTRACT							
TITLE OF PROGRAM		PROGRAM NO.					
WESWEAP -- Wave Equation Analysis for Piles		741-F3-R0010					
PREPARING AGENCY U. S. Army Engineer Waterways Experiment Station, Geotechnical Laboratory, P. O. Box 631, Vicksburg, MS 39180							
AUTHOR(S)		DATE PROGRAM COMPLETED	STATUS OF PROGRAM				
G. G. Goble and Frank Rausche WES Contact: Hugh M. Taylor, Jr.		July 1976	<table border="1"> <thead> <tr> <th>PHASE</th> <th>STAGE</th> </tr> </thead> <tbody> <tr> <td>INIT</td> <td>OP</td> </tr> </tbody> </table>	PHASE	STAGE	INIT	OP
PHASE	STAGE						
INIT	OP						
A. PURPOSE OF PROGRAM							
<p>The program performs wave equation analysis of piles driven by a single blow of any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesels and refined mechanical hammer models. The program can be used to predict impact stresses in piles during driving and to estimate static soil resistance on piles at the time of driving.</p>							
B. PROGRAM SPECIFICATIONS							
<p>The program development was aimed at providing a simple input and both a flexible and extensive output that include automatic plotting capabilities. The computer language is FORTRAN IV.</p>							
C. METHODS							
<p>The pile and driving systems are represented by a series of discrete masses and springs. The soil is modeled by a spring and a dashpot attached to each mass. The soil resistance so represented are linear elastic plastic. The elastic resistances are linearly proportional to the element velocity for the velocity. By using Newton's Second Law, accelerations and displacements are calculated and the computation proceeds to the next time increment.</p>							
D. EQUIPMENT DETAILS							
E. INPUT-OUTPUT							
<p>A short input and long or complete input forms are available. Common hammer property data are stored in a file. Input data is reprinted, options of printed and plotted parameters are available, and time plots are optional.</p>							
F. ADDITIONAL REMARKS							
<p>Manuals by the Federal Highway Administration that describe this program and its use are: Vol. I, Background, Report No. FHWA-IP-76-14.1; Vol. II, Users Manual, Report No. FHWA-IP-76-14.2, Vol. III, Program Documentation, Report No. FHWA-IP-76-14.3; and Vol. IV, Narrative Presentation, Report No. FHWA-IP-76-14.4.</p>							

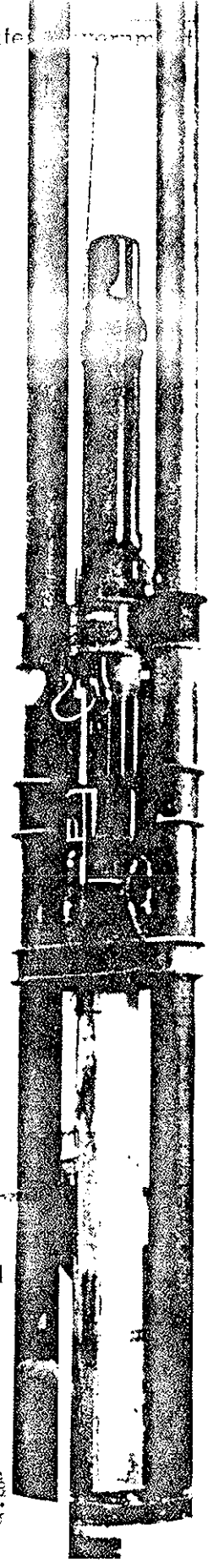
TE7
743
o.76-14.4

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76-14.4 Implementation Package

WAVE EQUATION ANALYSIS OF PILE DRIVING



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VICKSBURG, MISSISSIPPI

WEAP PROGRAM

Volume IV – Narrative Presentation



U.S. DEPARTMENT OF TRANSPORTATION
Federal Highway Administration
Offices of Research and Development
Implementation Division
Washington, D.C. 20590

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1. Report No. FHWA -IP-76-14.4		2. Government Accession No.		3. Recipient's Catalog No.																	
4. Title and Subtitle Wave Equation Analysis of Pile Driving WEAP Program				5. Report Date July, 1976																	
				6. Performing Organization Code																	
7. Author(s) Goble, G. G., and Rausche, Frank				8. Performing Organization Report No.																	
9. Performing Organization Name and Address Goble & Associates 12434 Cedar Road Cleveland Heights, Ohio 44106				10. Work Unit No. (TRAIS)																	
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15. Supplementary Notes FHWA Contract Manager: Chien-Tan Chang (HDV-22)																					
16. Abstract <p>A computer program was written and tested that performs a realistic Wave Equation Analysis of Piles driven by any type of impact hammer. Conventional pile and soil models were used in addition to both a thermodynamic model for diesels and refined mechanical hammer models.</p> <p>The program development was aimed at providing a simple input and both a flexible and extensive output that includes automatic plotting capabilities. Pile Driving Hammer data were prepared and stored in a file for most of the commonly encountered models. The computer language is FORTRAN IV.</p> <p>The program was extensively tested against measured pile top force and velocity data and against measured diesel combustion pressure and stroke. This volume is the fourth in a series. The others in the series are:</p> <table border="1"> <thead> <tr> <th><u>Vol No.</u></th> <th><u>FHWA No.</u></th> <th><u>Short Title</u></th> <th><u>NTIS (PB) No.</u></th> </tr> </thead> <tbody> <tr> <td>1</td> <td>IP-76-14.1</td> <td>Background</td> <td></td> </tr> <tr> <td>2</td> <td>IP-76-14.2</td> <td>User's Manual</td> <td></td> </tr> <tr> <td>3</td> <td>IP-76-14.3</td> <td>Program Documentation</td> <td></td> </tr> </tbody> </table>						<u>Vol No.</u>	<u>FHWA No.</u>	<u>Short Title</u>	<u>NTIS (PB) No.</u>	1	IP-76-14.1	Background		2	IP-76-14.2	User's Manual		3	IP-76-14.3	Program Documentation	
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17. Key Words COMBUSTION, COMPUTERS, DESIGN, DIESEL, DYNAMICS, FOUNDATIONS, IMPACT, PILE DRIVING, SOIL MECHANICS, WAVE EQUATION			18. Distribution Statement No restrictions. Copies of this volume are available from: National Technical Information Service Springfield, Virginia 22161																		
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WAVE EQUATION ANALYSIS OF
PILE DRIVING

Narrative Presentation

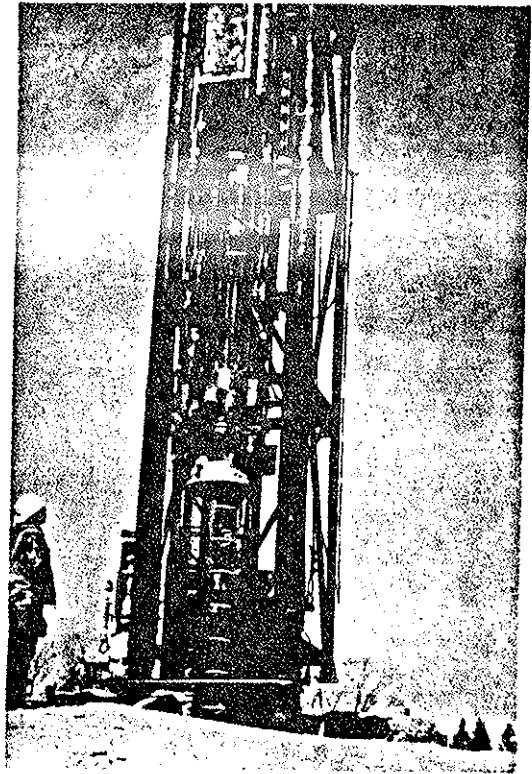
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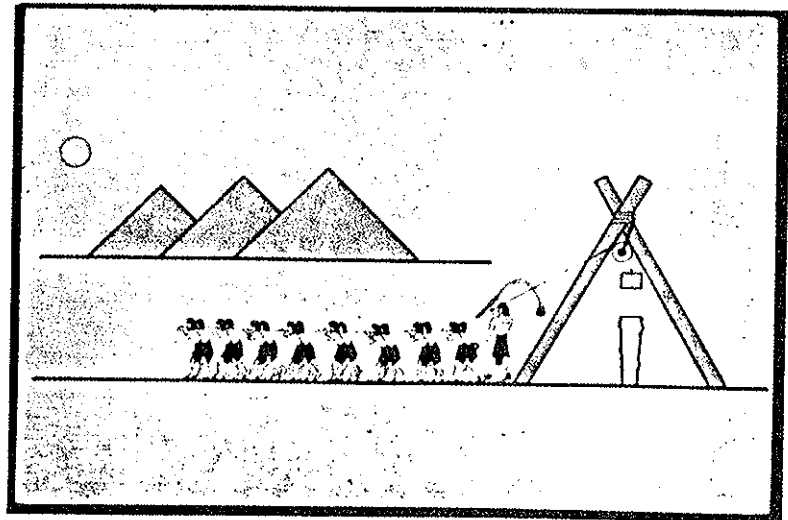
Part I

Background

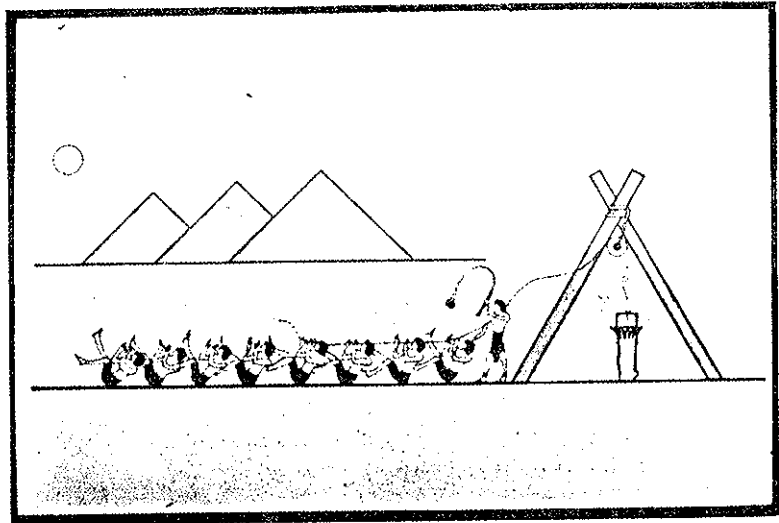
A1. When foundation engineers view pile driving and reflect on the fact that the driving operation induces failure in the pile-soil system, it is natural that they should attempt to use measurements made during driving to determine bearing capacity.



A2. Piles have been used as a foundation element since antiquity. They were driven with primitive equipment and were designed by a completely intuitive approach.



A3. Probably, failure of the pile or the driving operation was a common occurrence. During the last century pile usage increased with the development of the single acting steam hammer.



A4. Methods for predicting pile strength were proposed using dynamic data obtained during driving. Since the only realistic measurements at that time were ram stroke and pile set (or blow count)



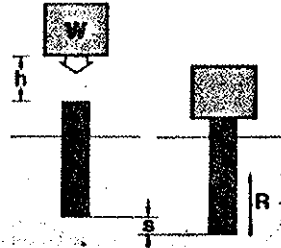
A5. it was natural that the engineer would turn to energy concepts, equating ram energy, Wh , to work done on the soil, Rs . Of course, rather gross assumptions were implied. The resistance of force, R , was assumed constant and the delivered energy was assumed to be the potential energy of the ram at the top of the stroke. The effects of cushions and helmets were neglected.

THE FUNDAMENTAL PILE DRIVING FORMULA

HAMMER ENERGY = WORK OF SOIL RESISTANCE

$$Wh = Rs$$

$$R = \frac{Wh}{s}$$



A6. Since poor results were achieved, a large factor of safety was introduced in order to insure against failure and the "losses" were estimated in terms of "lost" set.

ENGINEERING NEWS RECORD FORMULA

$$R = \frac{12 W \text{ (kips)} h \text{ (feet)}}{\text{S.F.} (s \text{ (inch)} + \text{"lost" set (inch)})}$$

S.F. = 6 = SAFETY FACTOR

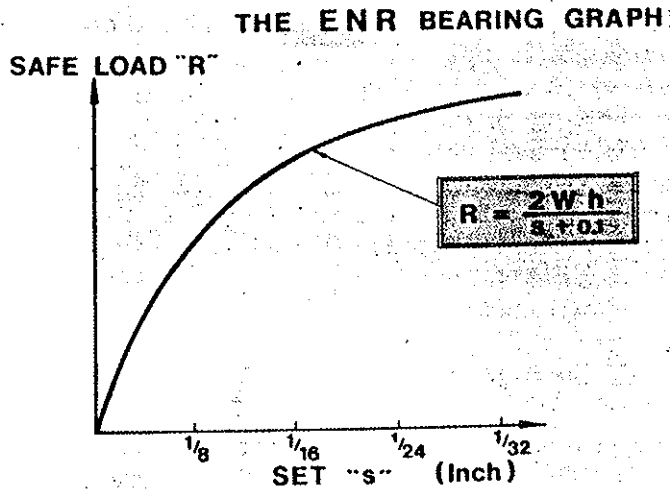
lost set = 0.1 inch

$$R = \frac{2Wh}{s + 0.1}$$

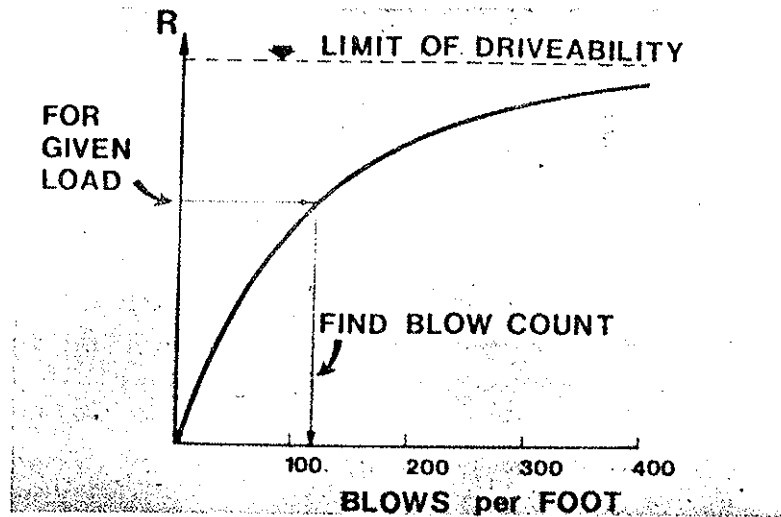
A7. A large number of formulas appeared. While dynamic formulas have been generally discredited, they are still widely applied today because of their simplicity and the lack of a better, well-recognized approach.



A8. It is interesting and instructive to examine a graphical representation of the ENR formula. Here pile capacity is plotted versus permanent set in a representation known as a bearing graph. Other formulas will have a similar shape. In application, a particular set or blow count (hammer blows per foot) is related to the pile design load.



A9. Alternatively, for a specified capacity the required blow count can be determined. Also, an equipment selection can be made. For example, a particular system can only overcome a certain maximum resistance. At the upper range of the curve, a large change in blow count does not indicate a comparable change in capacity.



A10. Now consider the causes for problems with the dynamic formulas. First, the derivation is not based on a realistic treatment of the driving system nor does it recognize the variability of equipment performance. Typical driving systems can include many elements in addition to the ram such as helmet, capblock, cushion, and anvil. Second, the soil resistance is very crudely treated. The assumption is made that the soil resistance is a constant force and this assumption neglect even the most obvious characteristics of real soil performance. Third, the pile is assumed to be rigid. This neglects completely the real flexibility and the wide variations it can have.

PROBLEMS WITH FORMULAS

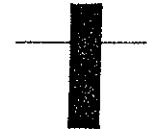
(1) SIMPLIFIED DRIVING SYSTEM



(2) CRUDE SOIL MODEL

R constant

(3) RIGID PILE



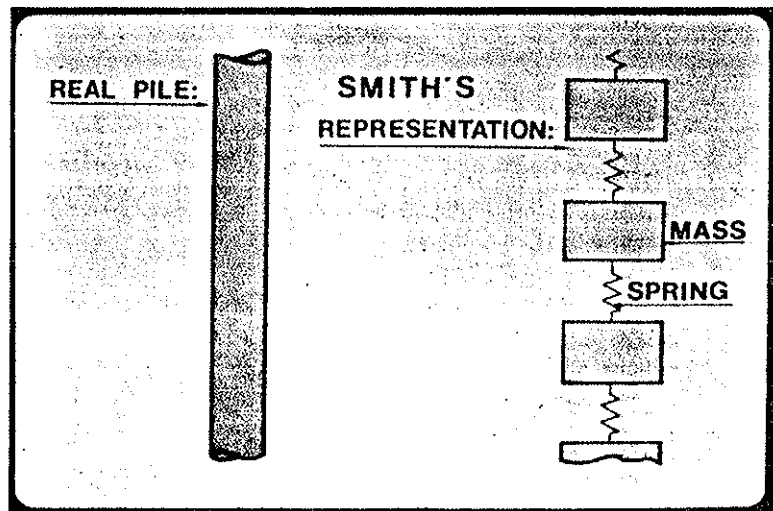
A11. As an alternative, the Wave Equation Analysis was proposed in the last century. The Wave Equation is actually a linear differential equation which was derived from Newton's Second Law. Since real pile driving problems are not easily solved by the Wave Equation.

THE WAVE EQUATION

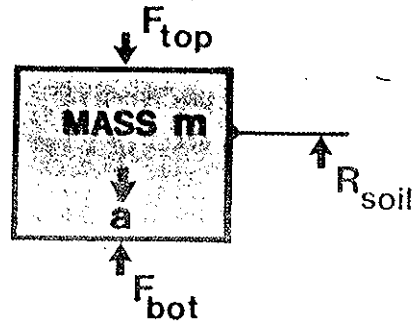
$$\frac{\partial^2 u}{\partial t^2} = c^2 \frac{\partial^2 u}{\partial x^2}$$

- u** Displacement
- c** Wave Speed
- t, x** Time, Length Coordinates

A12. E.A.L. Smith proposed, in the 1950's, to divide the pile into relatively large segments (say a few feet each) and represent each of them by a discrete mass and a spring. The total pile and driving system is so represented and then the motion of the pile is analyzed by applying Newton's Second Law successively over very short time increments.



A13. The forces acting on a typical pile element are those due to the strain at its top and bottom and to the soil resistance. Thus, from Newton's Second Law, the elements' acceleration at a particular time is equal to the sum of F_{top} , F_{bottom} , and R_{soil} , divided by the mass of the element.



ACCELERATION:

$$a = \frac{F_{top} - F_{bot} - R_{soil}}{m}$$

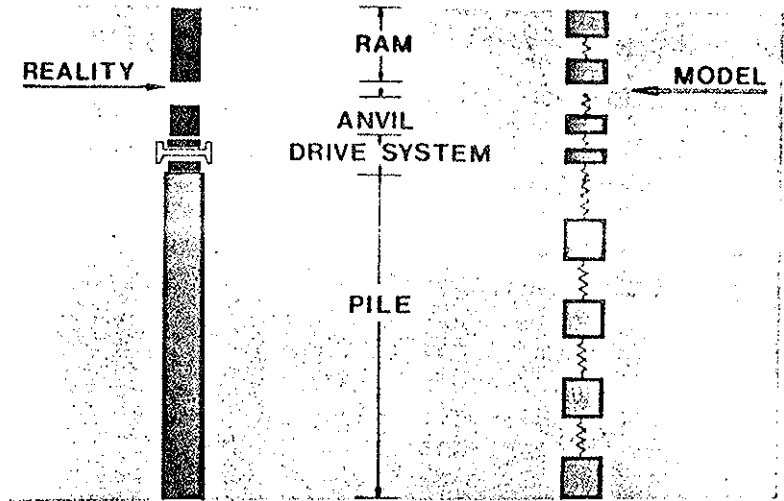
A14. Having determined the acceleration of the element, its velocity and displacement can be determined by integration. For this integration a small time increment is used during which the acceleration is assumed constant. The results of the integration becomes more accurate as the time increment is made shorter. Of course, as the time increment is made shorter the

VELOCITY = ACCELERATION x TIME INCREMENT

DISPLACEMENT = VELOCITY x TIME INCREMENT

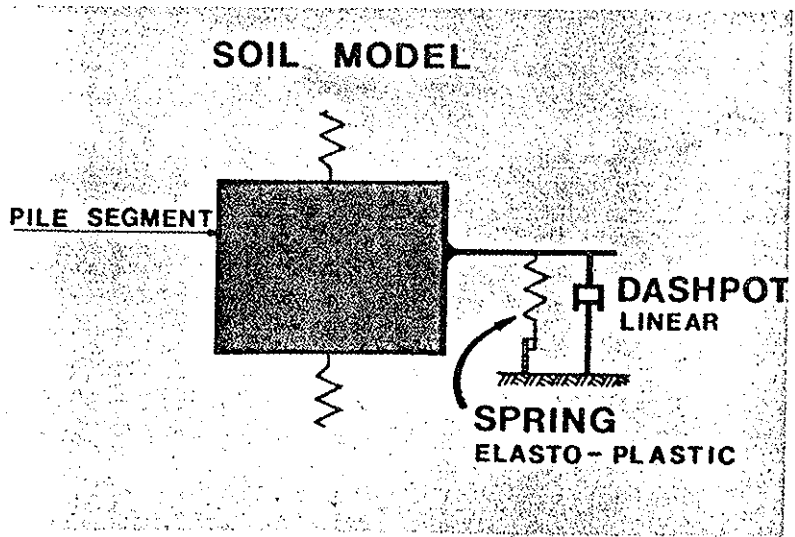
computational effort is increased. The cycle is completed by the determination of the compression in the spring between neighboring elements from the calculated element displacements. The compression then can be converted to forces acting on the elements and the computation repeated for the next time increment.

A15. In summary, the total pile driving system is represented by a series of masses and springs. It is analyzed by giving the ram an initial velocity determined by the manufacturer's stated characteristics and then analyzing the system using the force-inertia balance over the required time period consisting of a large number of time increments. The

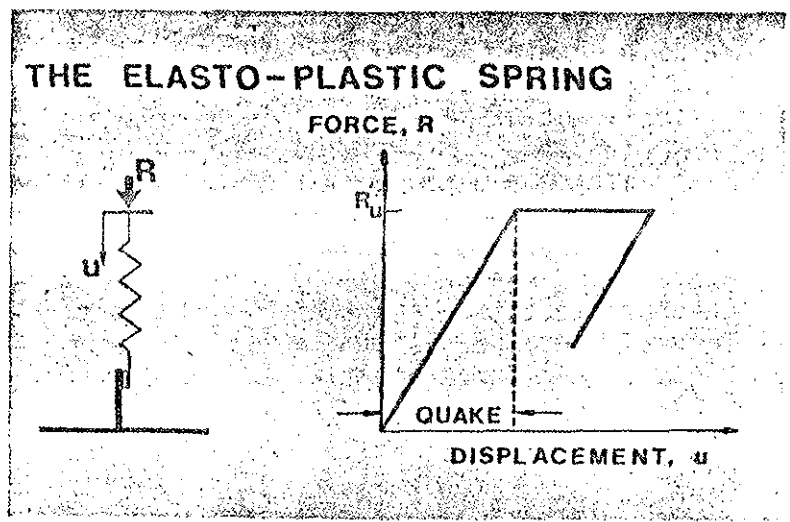


mass and spring characteristics can be calculated from the physical characteristics of the pile and the driving system.

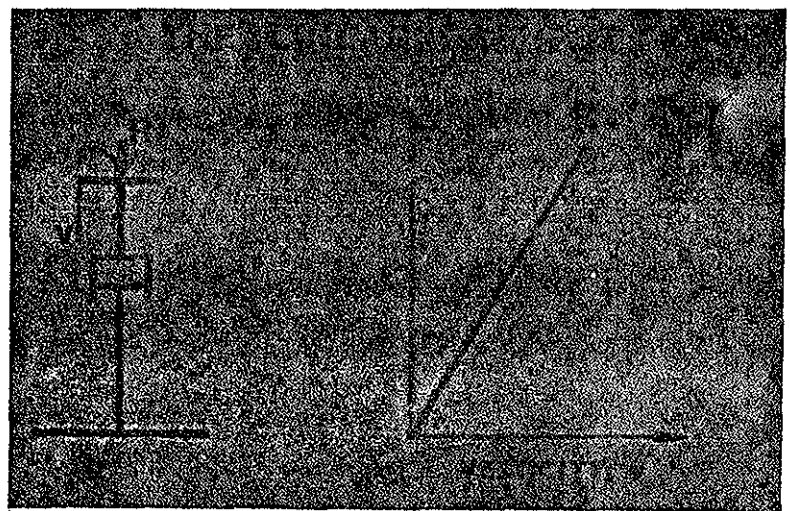
A16. But pile driving analysis must include more than just the pile and the driving system. A soil resistance force has been mentioned. Now consider some of the details of the description of this force. It is assumed to be composed of a static and a dynamic portion and it can be represented by a spring and a dashpot.



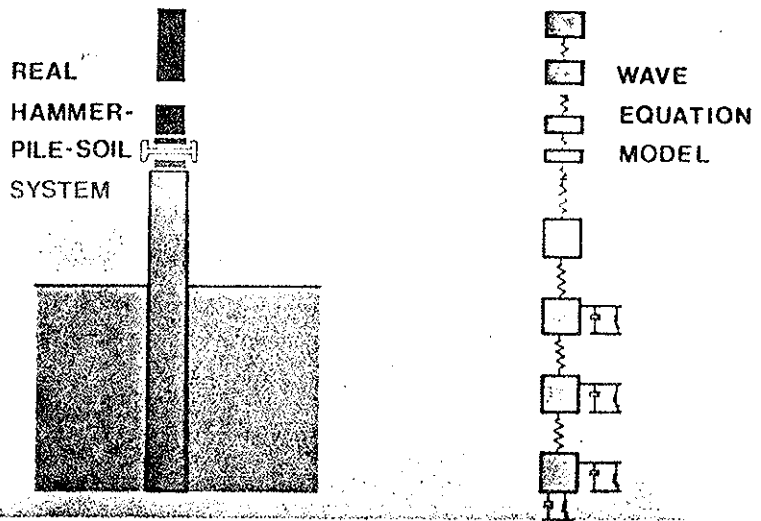
A17. The spring or static portion of the resistance relates the soil resistance to the pile element displacement. The soil resistance force is assumed to increase linearly with the displacement until a displacement called the quake is reached. With further displacement the force remains constant. Unloading occurs along a line parallel to the loading line.



A18. The dynamic portion of the resistance is represented by a dashpot that assumes a linear relationship between the resistance force and the element velocity.

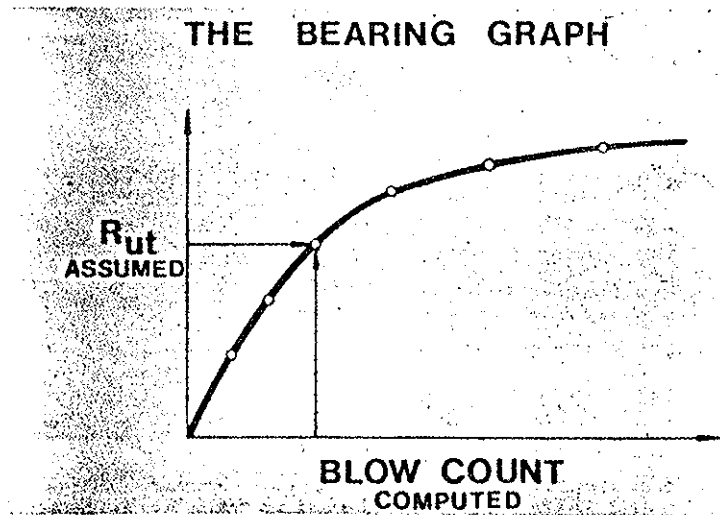


A19. So the total pile driving analysis problem is represented by the series of masses and springs representing the pile and the driving system with the soil resistance on the embedded portion modeled by elastic plastic springs and linear dashpots attached to the appropriate elements.



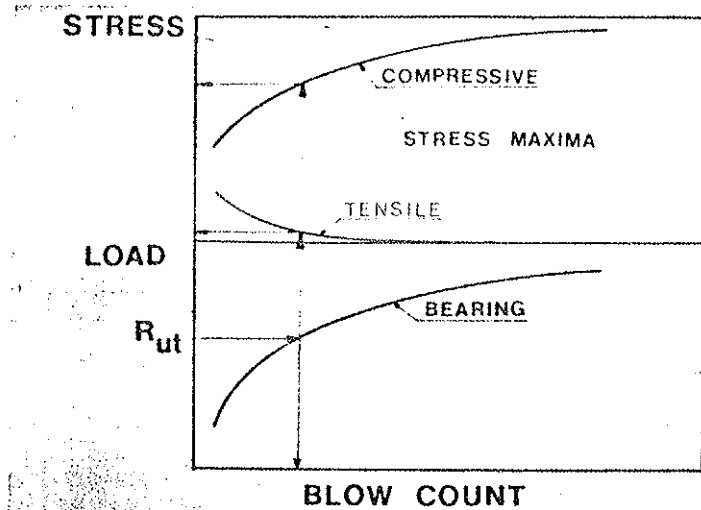
The engineer must supply all of the physical characteristics of the system including the soil. Computer programs have been prepared to perform the analysis and these programs have come to be known as Wave Equation programs. In this presentation the name wave equation will refer to computer programs of this general type.

A20. The result of the wave equation analysis is a bearing graph of appearance similar to that produced by a dynamic formula. It is produced by assuming an R_u value and a damping constant to represent the soil acting on each element. The sum of all the R_u 's is the ultimate static capacity of the pile, R_{UT} . For a particular R_{UT} the wave

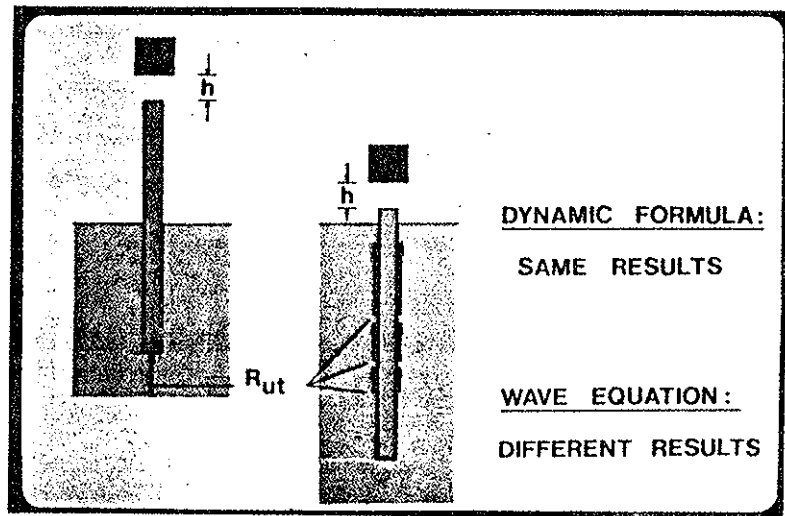


equation analysis is performed and the permanent pile displacement is calculated. The bearing graph is constructed from a set (say 5 or 6) of these points by interpolation.

A20a. In addition to the bearing graph the critical stresses in the pile can also be determined since the forces in the pile at each element for every time increment were found during the analysis.



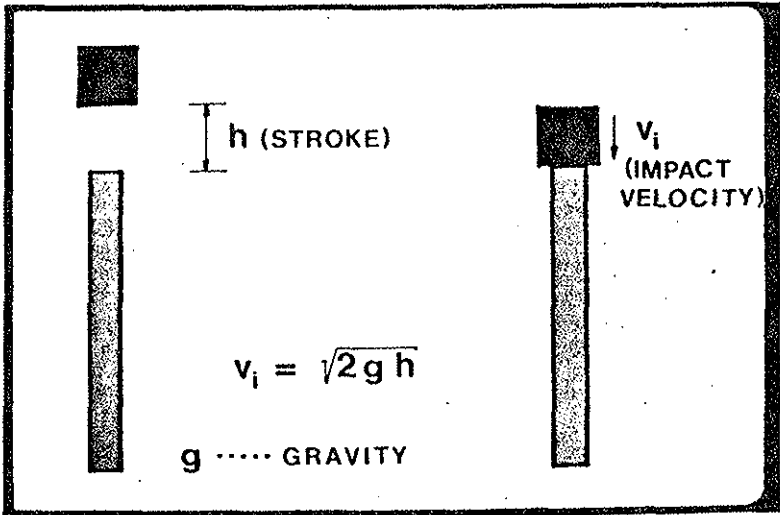
A21. It should be noted that a bearing graph is associated with a particular pile penetration. While it is similar in appearance to the graph generated by a dynamic formula, it represents something quite different. The dynamic formula has as its only variable, hammer energy with, perhaps, an estimate of losses. The wave equation includes pile flexibility, an estimate of soil performance, and driving system characteristics. In the case shown here the dynamic formula would give the same bearing graph in both cases while the wave equation bearing graph would be different.



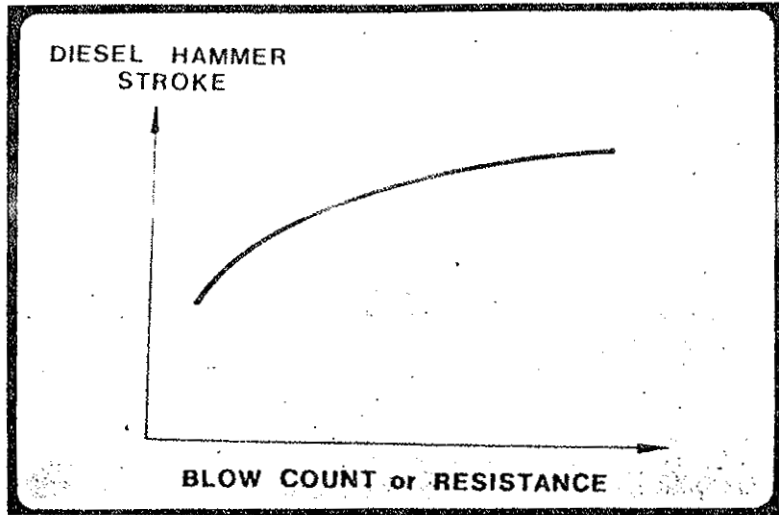
A22. Consider now a similar evaluation of the wave equation to that made previously for the dynamic formulas. First, the driving system treatment is improved. Accurate information on the dynamic performance of the driving system is required. Of course, if hammer performance is not up to its rated characteristics, that cannot be predicted by the wave equation. Second, the pile is accurately modeled. Third, the soil model, while crude, is clearly improved over that used in the dynamic formula. More accurate representations of the soil could be created but they are not justified due to the difficulty in determining the required parameters.

- (1) DRIVING SYSTEM TREATMENT IMPROVED
- (2) PILE MODELED ACCURATELY
- (3) SOIL MODEL CRUDE BUT IMPROVED

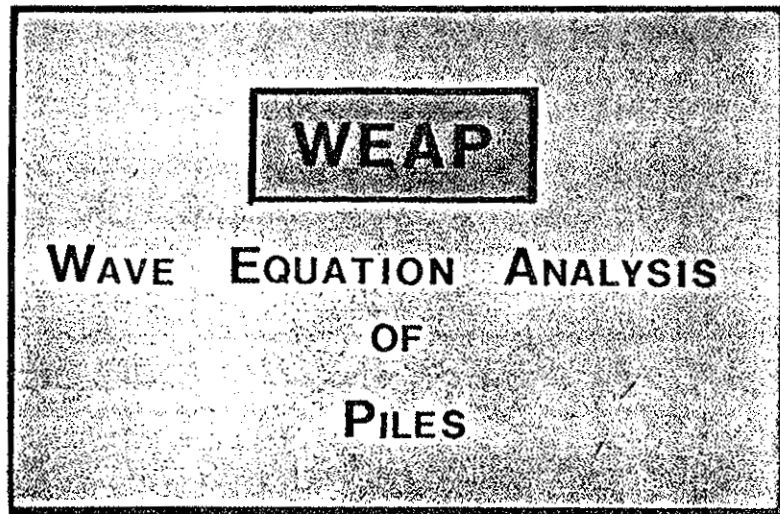
A23. The computer programs which have previously been available have used as basic input the ram impact velocity as computed from the stroke. This characteristic is particularly undesirable for application to diesel hammers



A24. where the stroke is dependent on driving resistance and is not known in advance. Difficulties have been encountered, particularly with the stress predictions obtained from these programs.

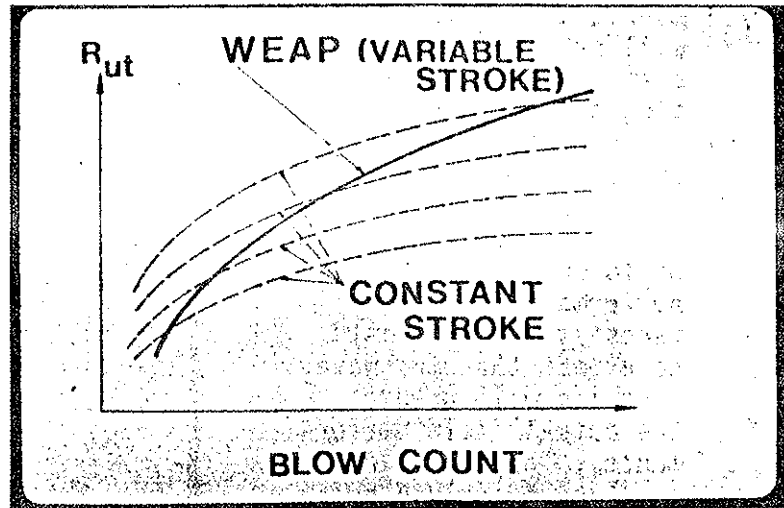


A24a. In order to find a solution to this problem, the Federal Highway Administration contracted with Goble & Associates to develop a wave equation program based on a more realistic model for pile driving hammers, particularly diesel hammers. Furthermore, the requirement was included to test the program against the large volume of force and velocity measurements made during the Case



Western Reserve University Pile Driving Research Program. The result of this contract is a computer program called WEAP.

A25. The basic program is illustrated by the bearing graph shown here. If a separate bearing graph were obtained for each possible stroke, the family of curves shown dashed would be obtained. Now if the stroke were measured in the field together with the blow count, the appropriate curve could be used to determine the capacity. A series of strokes and related blow counts would produce the solid curve which is the actual bearing graph. Unfortunately, a driving system could not be evaluated prior to going to the field.



A26. The WEAP Program provides an improved treatment of the combustion forces and a more accurate representation of air/steam hammers in addition to analyzing the total thermo-mechanical cycle of the diesel hammer and determining stroke (or bounce chamber pressure for closed end diesels) as an output quantity.

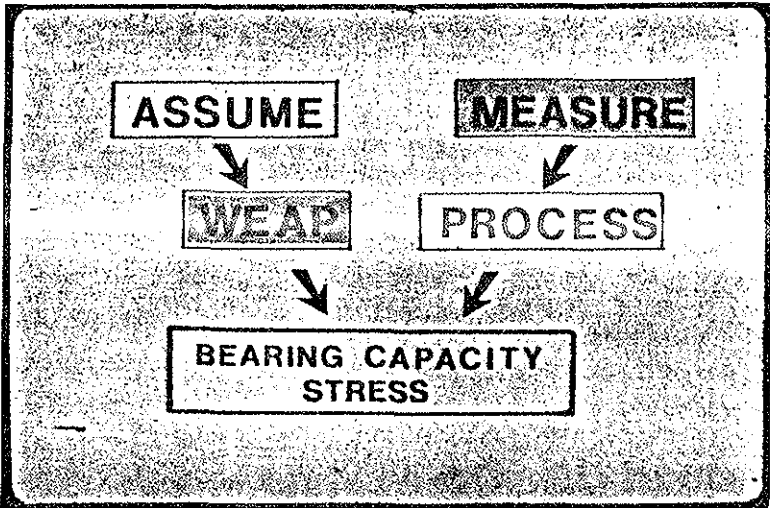
A good wave equation program which realistically

WEAP

IMPROVEMENTS

1. METAL TO METAL IMPACT SIMULATION
2. AIR/STEAM MODEL MORE DETAILED
3. THERMODYNAMIC CYCLE INCLUDED
4. STROKE AN OUTPUT FOR DIESELS

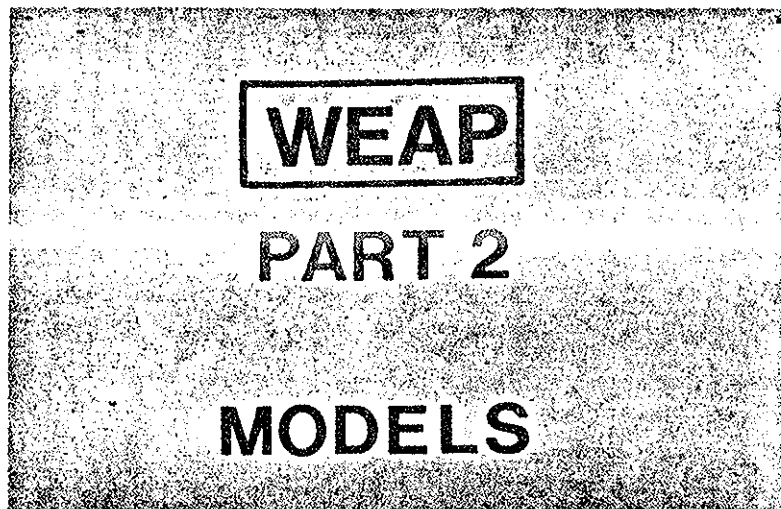
A27. models the total system will provide the best possible means of evaluating pile driving short of going to the field. In many cases it is impossible to provide the proper soil constants or to anticipate hammer performance. In such cases it is unrealistic to expect that the wave equation will predict the actual field performance. These problems can best be solved by field measurements using modern electronic devices which are now available.



Part II

Models

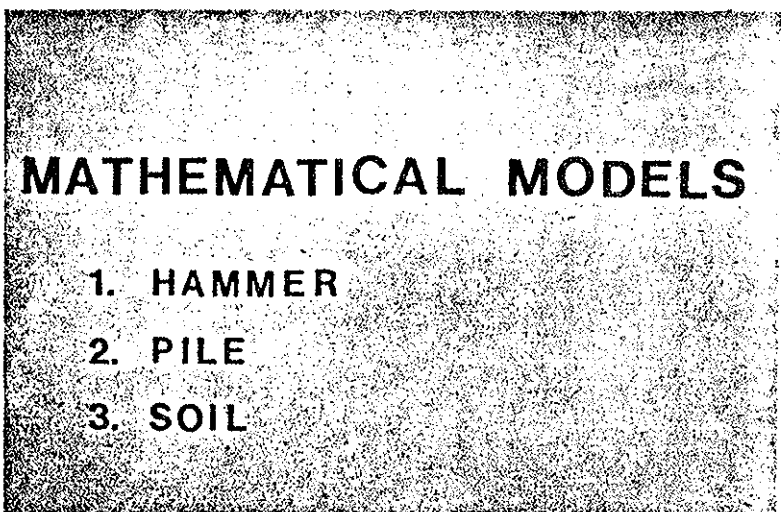
B1. In the previous section of this presentation the use of the wave equation program in the selection of equipment and driving criteria for pile foundations was discussed. Only very general considerations were covered. In this section a greater level of detail will be presented.



WEAP
PART 2
MODELS

B2. The operation of pile driving hammers and equipment will be discussed and the model used to represent the equipment, the pile and the soil will be described in detail.

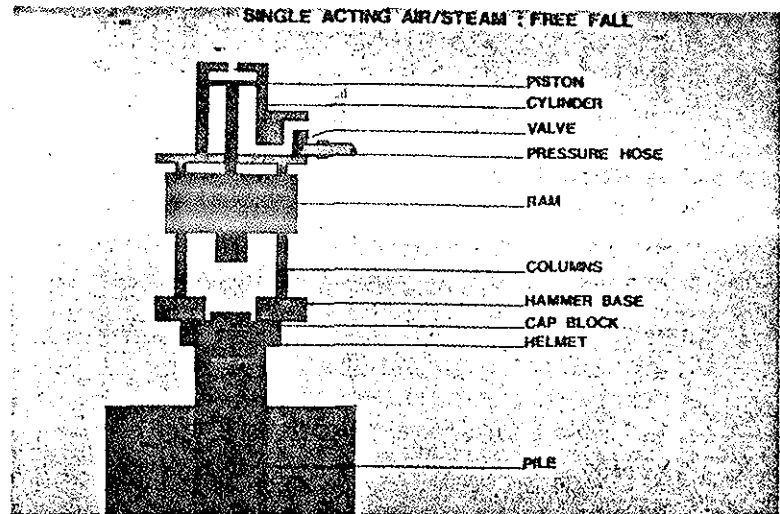
When creating a mathematical model of a pile driving hammer, it is necessary that the hammer operation be clearly understood. First, single acting air/steam hammers will be treated.



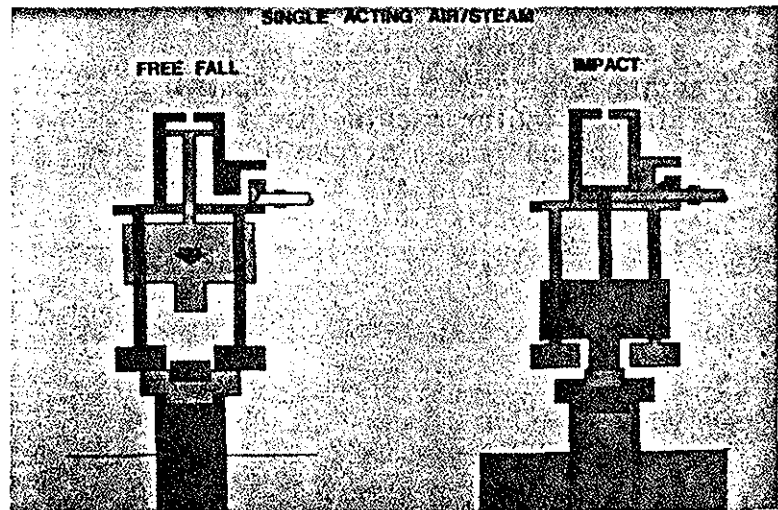
MATHEMATICAL MODELS

- 1. HAMMER**
- 2. PILE**
- 3. SOIL**

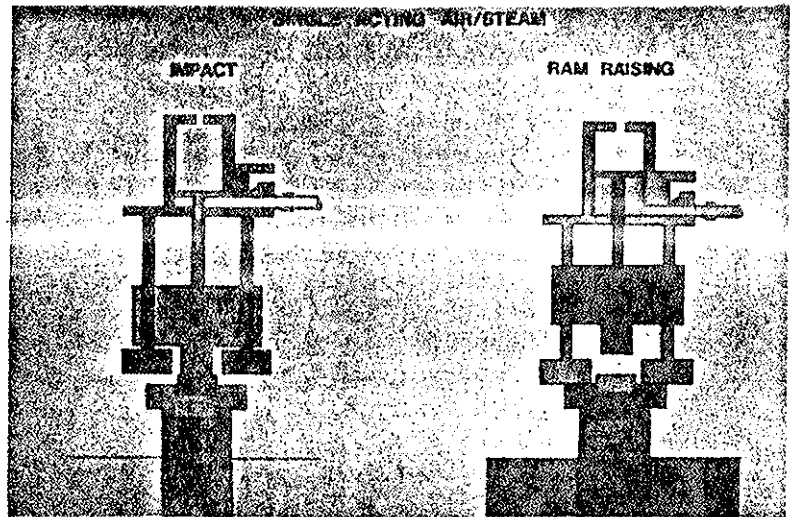
B3. The driving system consists of the hammer and a helmet holding the top of the pile in position and containing a capblock on which the ram impacts. For concrete piles a cushion is inserted between the pile top and the helmet. Assume that the cycle begins with the ram in the free fall down stroke sliding on the columns. The exhaust valve is open allowing the cylinder under the piston to exhaust directly to the atmosphere. At this time the hammer assembly consisting of the hammer base, columns, and cylinder are all resting on the helmet (and, hence, on the pile top).



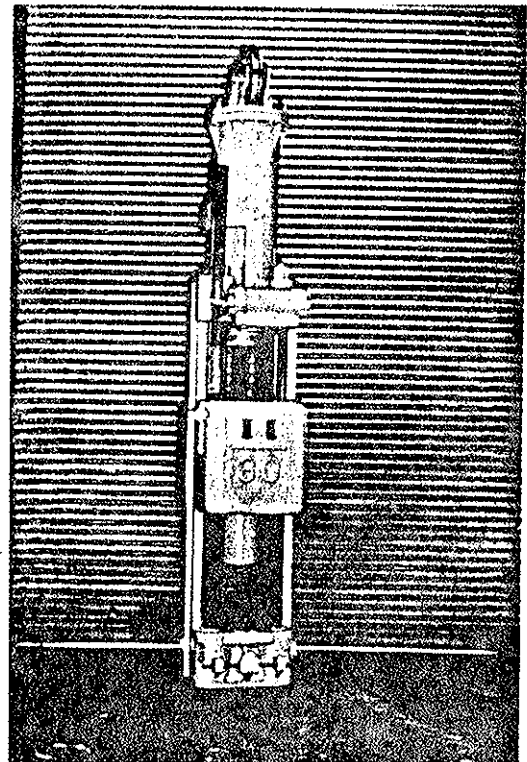
B4. At the bottom of the down stroke the ram impacts the capblock and drives it rapidly down together with the helmet and the pile top. The assembly is left up in the air falling under the action of gravity. Slightly before reaching the bottom of the stroke the intake valve is opened allowing the active gas (either compressed air or steam) to enter the cylinder. Therefore, since the gas pressure acts on the bottom of the cylinder it provides a force in addition to gravity which drives the assembly down.



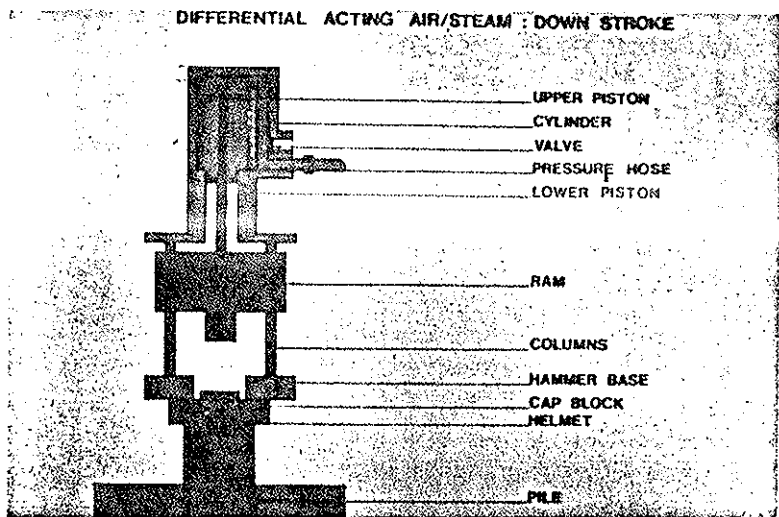
B5. The assembly impact with the helmet can be substantial since its weight is as great as the ram and the pile rebound velocity is often comparable to the ram impact velocity. The ram gains its upward velocity due to the pile rebound and the active gas pressure in the cylinder. At some point during the upstroke the exhaust valve opens and the ram "coasts" on up to the top of its stroke. Therefore, the stroke of an air/steam hammer is not necessarily constant.



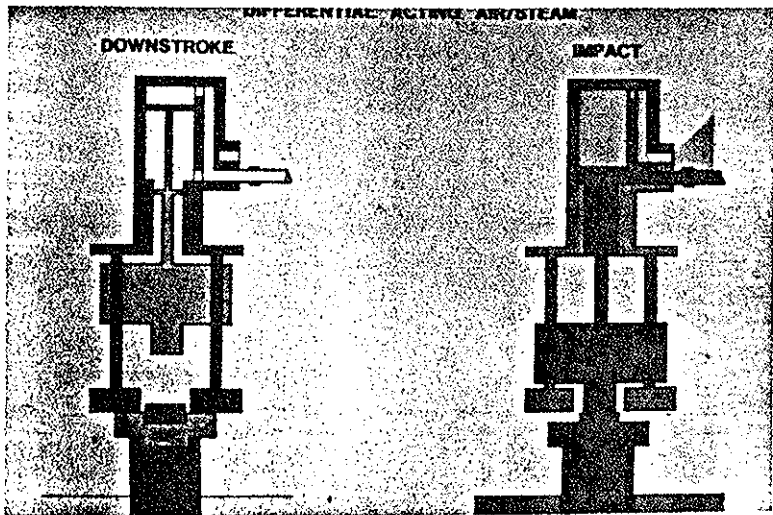
B6. In this slide a typical single acting air/steam hammer is shown. This particular machine has a 6,000 pound ram, a stroke of about three feet, and an operating speed of about 60 blows per minute.



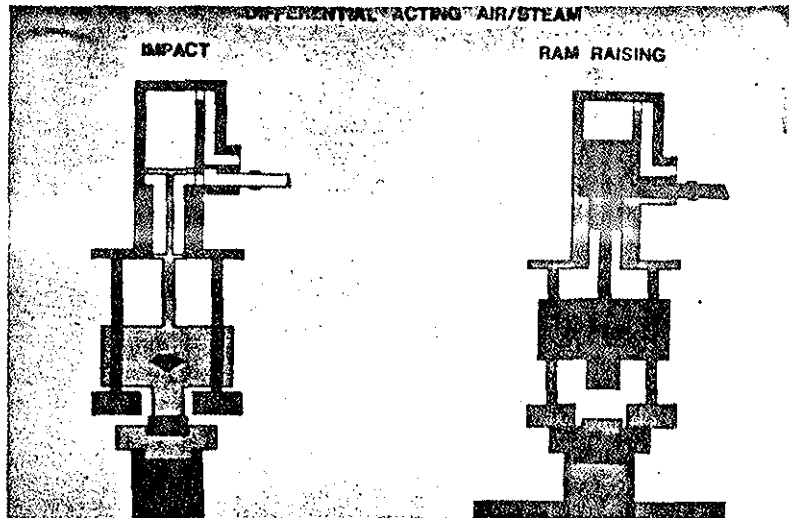
B7. The double acting air/steam hammer has a similar appearance but some important differences in operating characteristics. Of the various types of double acting air/steam hammers, the differential acting type will be described. The main components of this hammer are the same as the single acting machine. The differences are in the configuration of the cylinder. The piston has three surfaces which are subjected to gas pressure, the upper and lower surfaces of the upper piston and the upper surface of the much smaller, lower piston. Thus, since the same pressure acts on all surfaces there is a net down force on the piston which is reacted by the weight of the hammer assembly. The ram is acted on by both gravity and the applied force and is accelerated more quickly than for the single acting hammer. The upward acting force on the assembly must be limited so as not to lift the assembly off the helmet. This provides a limitation on the force which can be applied to the ram.



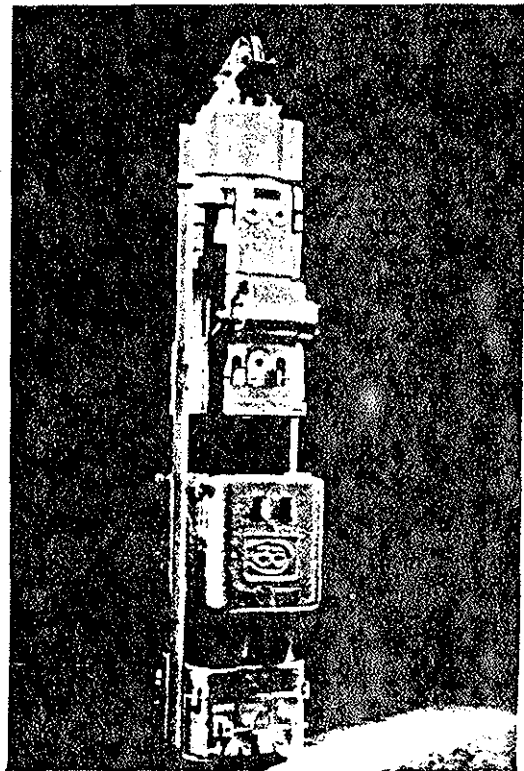
B8. When the ram impacts on the capblock the helmet is driven out from under the hammer assembly. The assembly then falls to the helmet for a second impact similar to the single acting hammer. Shortly before reaching the bottom of the stroke the valve changes so as to release all gas pressure from the upper chamber. Now with the upper chamber at ambient pressure the net force on the ram is up.



B9. The cycle is completed at the end of the up stroke when gas pressure is again applied in both the upper and lower chambers. The purpose of the double acting concept is to shorten the stroke and speed up the operation while maintaining about the same ram impact velocity. Typically, double acting hammers operate at about twice the speed of single acting hammers. As mentioned above, the differential acting hammer has been described. There are, of course, other types of double acting hammers.



B10. The machine shown here is double acting and runs at about 110 blows per minute.



B11. This slide shows another example of the double acting hammer. This particular machine operates on hydraulic fluid supplied by a pump. The system is a closed one since the spent fluid is returned to the pump. The pump is equipped to allow the control of the fluid pressure.



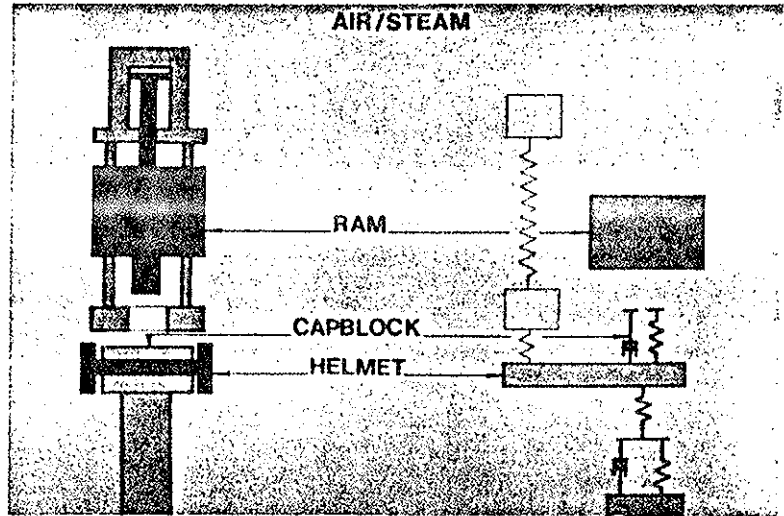
B12. The rating of both the single and double acting hammer assumes a predictable and constant energy. To include any energy losses the impact velocity is usually calculated using a reduced energy obtained by multiplication with an efficiency factor which is less than one. From the reduced energy an impact velocity can be calculated for input to the Wave Equation. Both hammer types can be handled in the same computer model.

$$\text{ENERGY } E = W h e$$

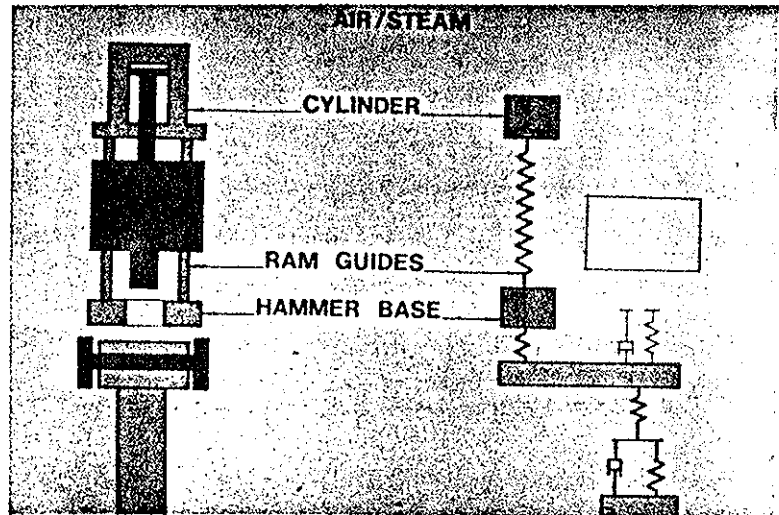
W... WEIGHT OF RAM
h ... HEIGHT OF FALL
e ... EFFICIENCY

$$\text{IMPACT VELOCITY} = \sqrt{2 g h e}$$

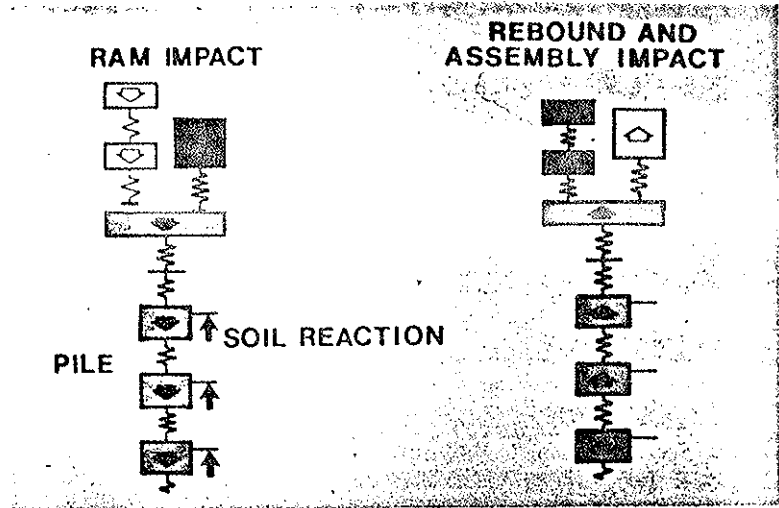
13a. The mathematical model used in the WEAP program is more detailed than that which has been used previously. The ram can be represented by a series of masses and springs. The spring shown connecting the bottom ram element and the helmet is capable of carrying compression only. For currently available air/steam hammers the rams are so stocky that they can be represented by a single mass element. The details of the modeling of the helmet-capblock assembly will be discussed later.



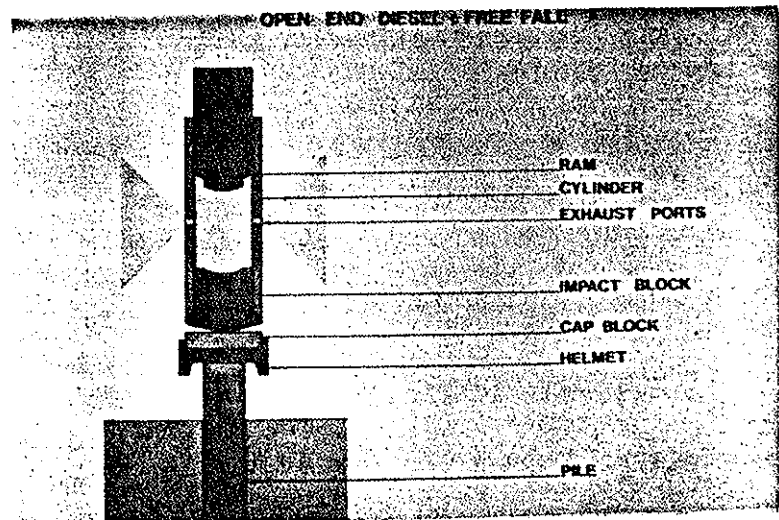
13b. The hammer assembly is also modeled by a series of masses and springs with the bottom spring at the interface of the assembly and the helmet also a compression-only spring. For the majority of currently available hammers, two masses are required; one representing the cylinder and the other, the hammer base. The connecting spring gets its flexibility primarily from the hammer columns.



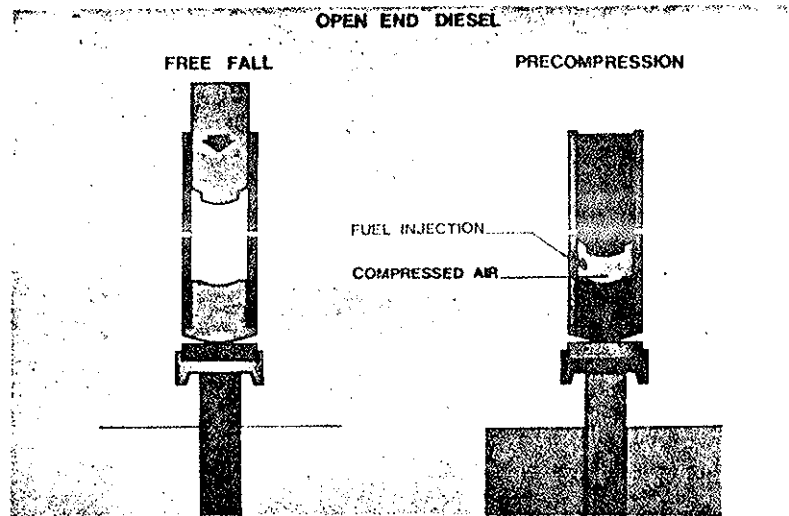
B14. The wave equation program begins by giving the ram element (or elements) the rated impact velocity. As impact occurs all segments of the wave equation model obtain a velocity and, therefore, move downward. The portions of the assembly segments are also continuously computed in their free fall. The motion of the pile segments causes the soil resistance forces to be activated. Once the pile has rebounded and all soil springs are elastic, computations are terminated.



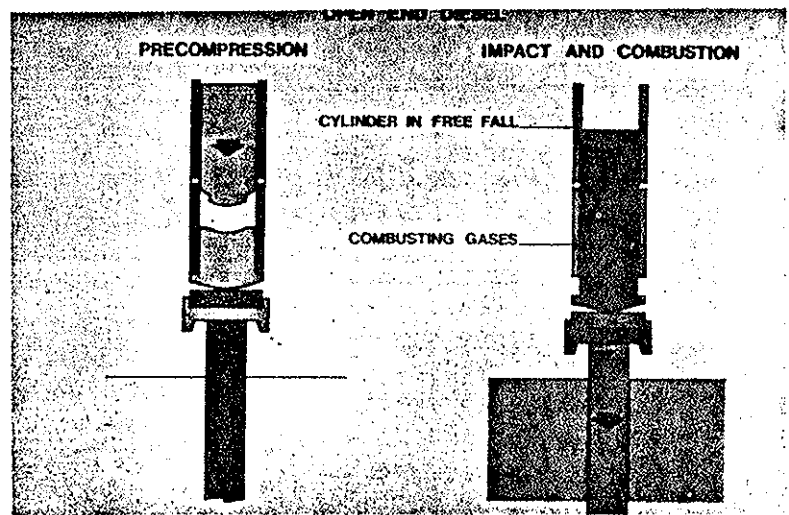
B15. Consider now the single acting or open end diesel hammer. It consists of three important elements: a ram, an impact block or anvil, and a cylinder all resting on a capblock and helmet similar to that used for air/steam hammers. To start the hammer the ram is lifted by the crane and dropped. Excess air is blown out of the exhaust ports or if this is a downstroke during operation combustion products and air are blown out.



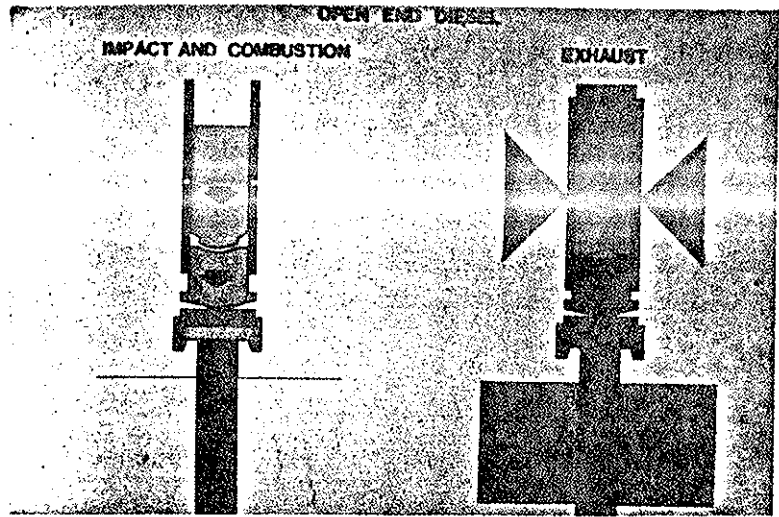
B16. When the ram falls past the exhaust ports they are blocked and the precompression phase begins. The falling ram activates a cam to inject fuel into the cylinder. This may be in liquid form falling onto the impact block or it may be atomized. In the latter case it is injected only shortly before impact.



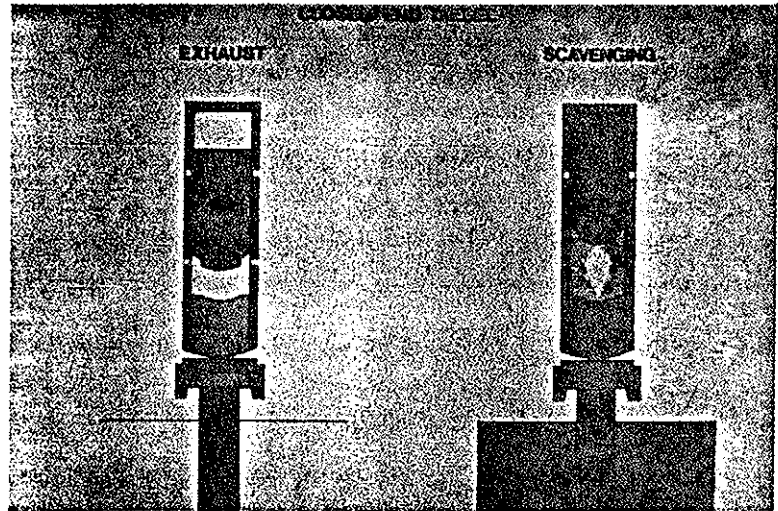
B17. The ram is shaped so that at impact a small combustion chamber remains with a high compression ratio. If the fuel is injected as a liquid it is atomized on impact. Combustion occurs causing a very rapid increase in pressure.



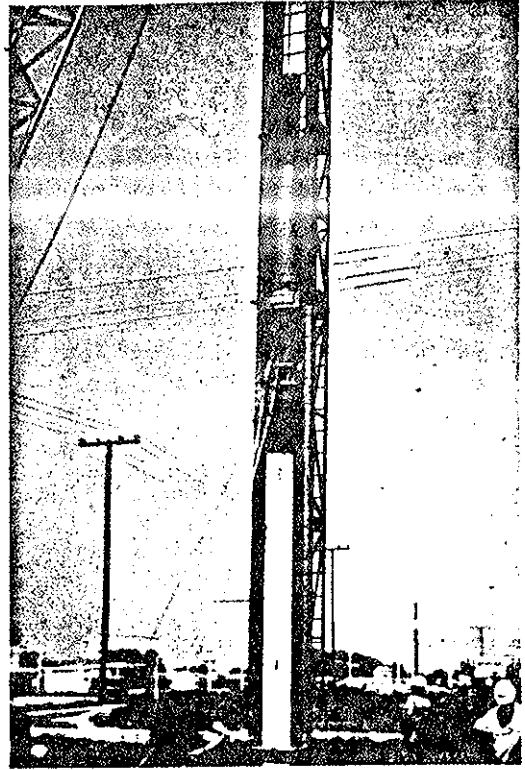
B18. As a result of the combined effect of the combustion chamber pressure and the pile rebound, the ram moves upward allowing an expansion of the combustion products. When it passes the exhaust ports the excess pressure is blown off.



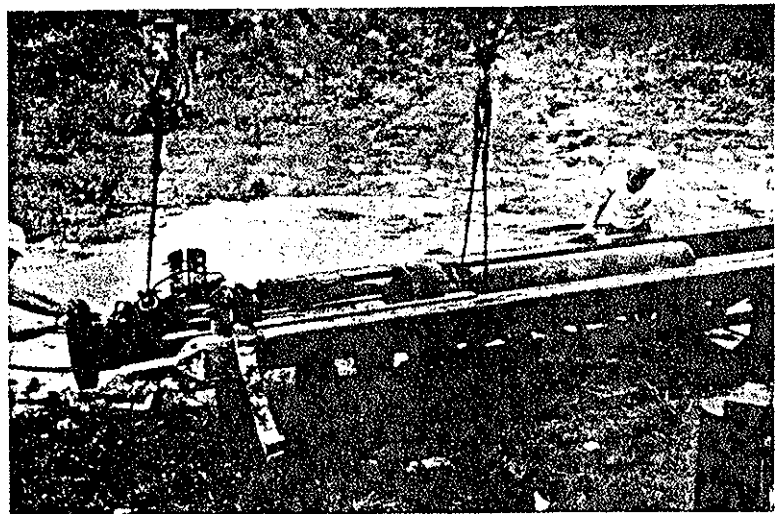
B19. However, the ram has an upward velocity so it continues upward in a free fall condition, in the process drawing in fresh air in a scavenging phase. The maximum height that the ram achieves depends on the pile rebound as well as the combustion effect. In normal operation the stroke can vary over a wide range depending on the pile rebound and stiffness. Strokes ranging between four feet and nine feet are not abnormal for various hammer types.



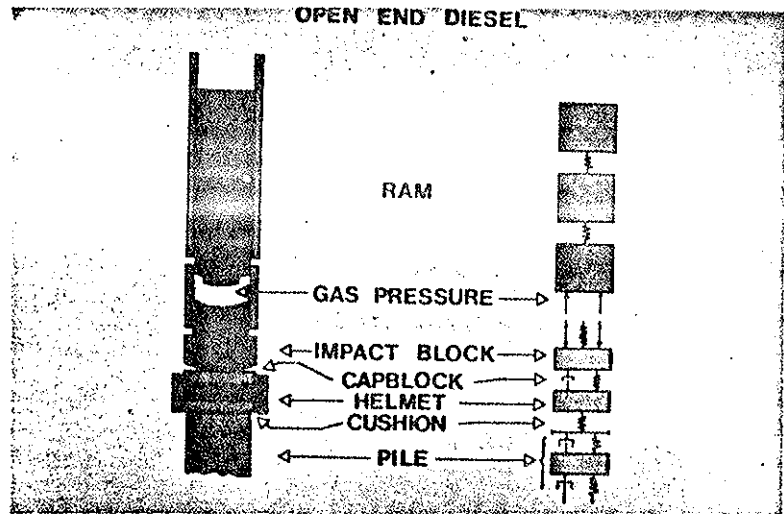
B20. An open end diesel hammer is shown here. The photograph was taken when the ram was near the top of the stroke and a large part of the ram extends beyond the top of the cylinder.



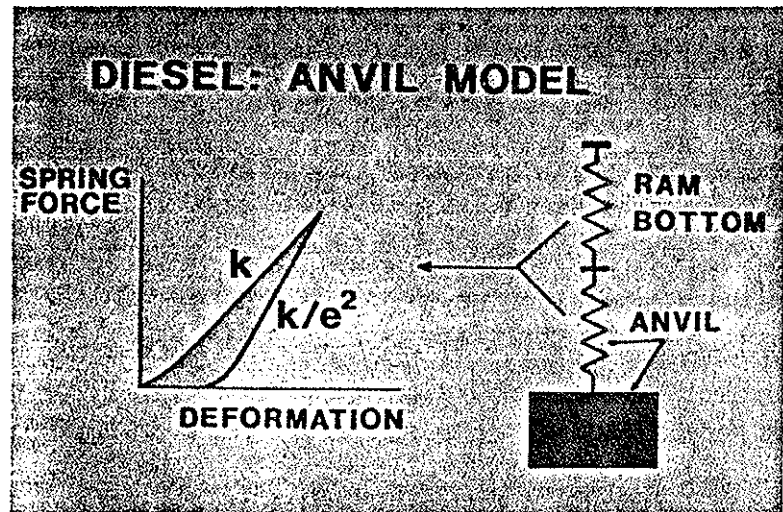
B21. In this photograph, another open end diesel hammer is shown. The impact block is partially extended and clearly visible at the bottom of the hammer.



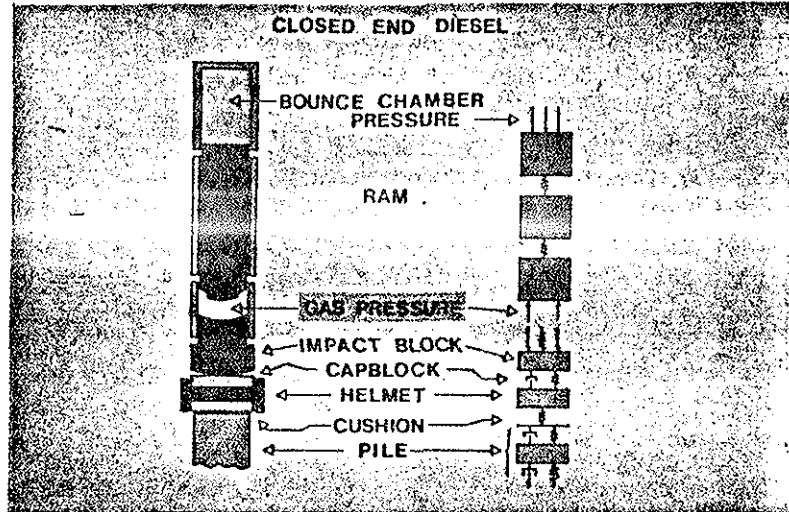
B22. The hammer is modeled in the WEAP program as shown here. The ram consists of a series of masses and springs. Since rams of diesel hammers are typically longer and more slender than is common for air/steam hammers, several elements are used. The impact block is represented by a single element.



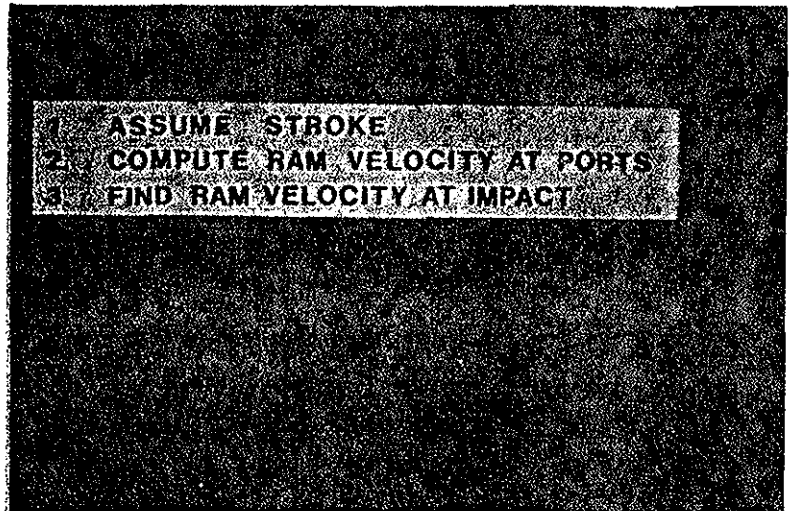
B23. The steel to steel impact of the ram on the impact block represents a particular problem and one which must be handled carefully. The research conducted during the development of the WEAP program leads to the use of an impact block spring which represents simply the combined stiffness of the last ram segment and the anvil. The spring is modeled with non-linear properties as are the springs of all cushion materials. The change of slope is identified by a coefficient of restitution, e . The model below the impact block is identical with that used for the air/steam hammer.



B24. The combustion chamber pressure must also be active between the ram and the impact block. The resulting forces can best be described by discussing the operation of the program as far as the hammer is concerned.



B25. The program begins with either an assumed or a user specified ram stroke. With a knowledge of the stroke the velocity of the ram as it falls past the exhaust ports can be calculated from elementary kinematics. When the exhaust ports are closed the combustion chamber pressure and ram velocity are determined using a rigid body assumption and step-wise calculating ram velocity resulting from the action of gravitational and combustion forces.



B26. As the ram moves down the combustion chamber pressure is calculated from the gas law. When the ram reaches the impact block the velocity is known and the wave analysis can be performed in a fashion similar to that described earlier.

GAS LAW:

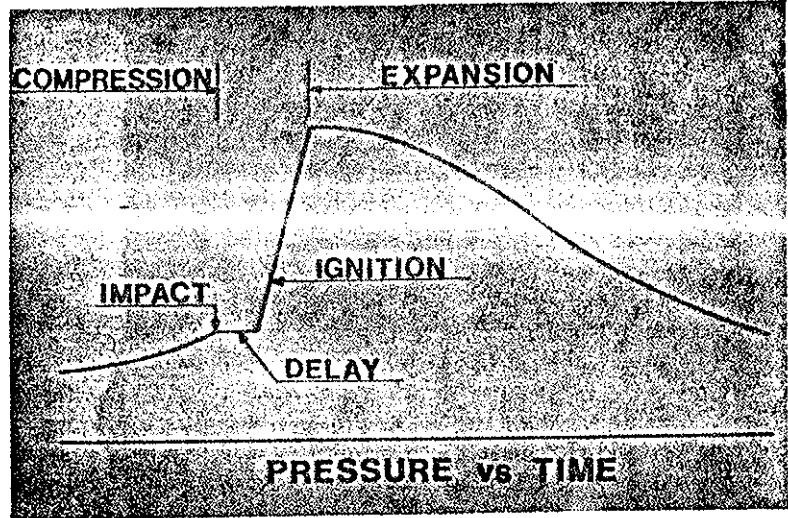
PRESSURE:
$$p = p_{atm} \left(\frac{V_i}{V} \right)^{exp}$$

p_{atm} Atmospheric Pressure
 V_i, V Initial, Current Volume
exp Gas Property

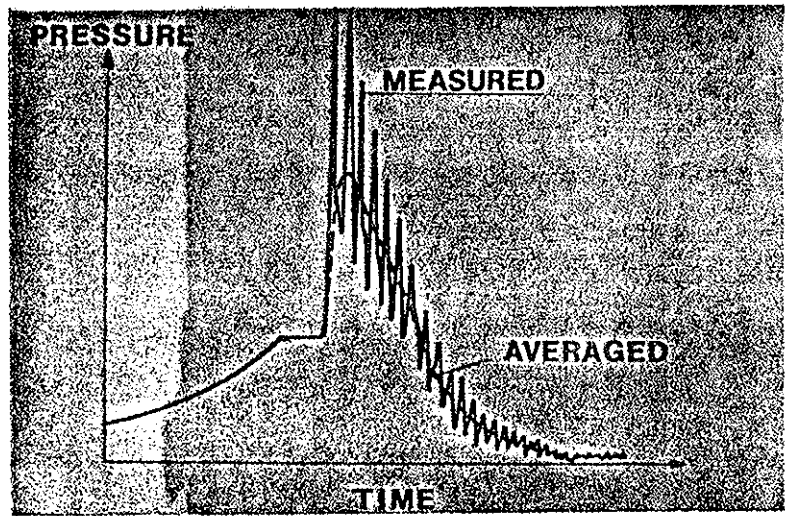
B27. Measurements of combustion chamber pressure have shown that there is normally a delay between ram impact and ignition. This delay time can vary depending on job conditions. It is usually about one or two milliseconds. During this time the impact induced wave is traveling down the pile and the combustion chamber pressure is about constant.

1. ASSUME STROKE
2. COMPUTE RAM VELOCITY AT PORTS
3. FIND RAM VELOCITY AT IMPACT
4. IMPACT AND COMBUSTION DELAY
5. IGNITION; GAS FORCE TO MAXIMUM
6. EXPANSION

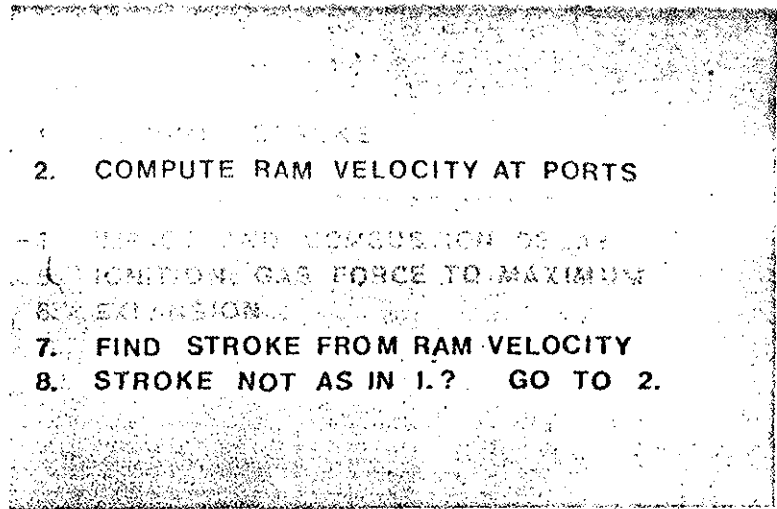
B28. When ignition occurs the pressure increases very rapidly to some maximum value. This pressure can be calculated from combustion laws. However, due to the imperfect nature of diesel hammer combustion the computed pressure is not reliable. Therefore, measured maximum pressures are used in the WEAP program. From this time on the combustion chamber pressure used in WEAP is determined by the volume of the chamber using the gas law.



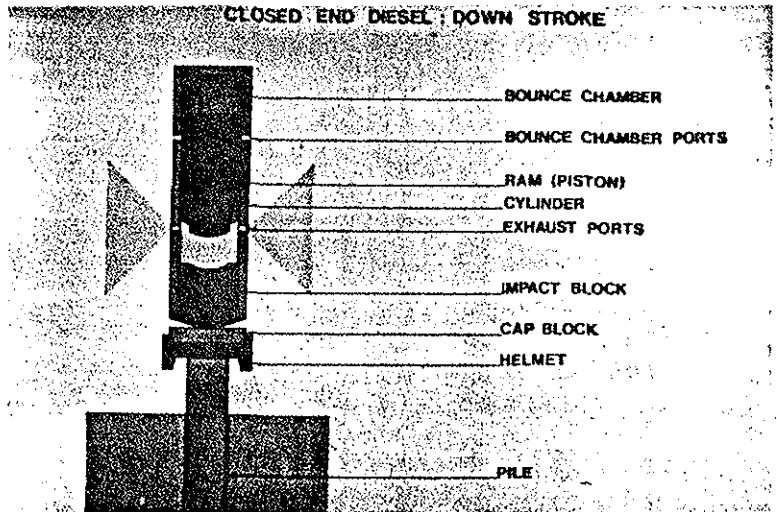
B29. Actually, the pressure will have a high frequency fluctuation which is induced by ignition. However, this phenomenon cannot be realistically modeled and it has no significant effect on the pile dynamic performance.



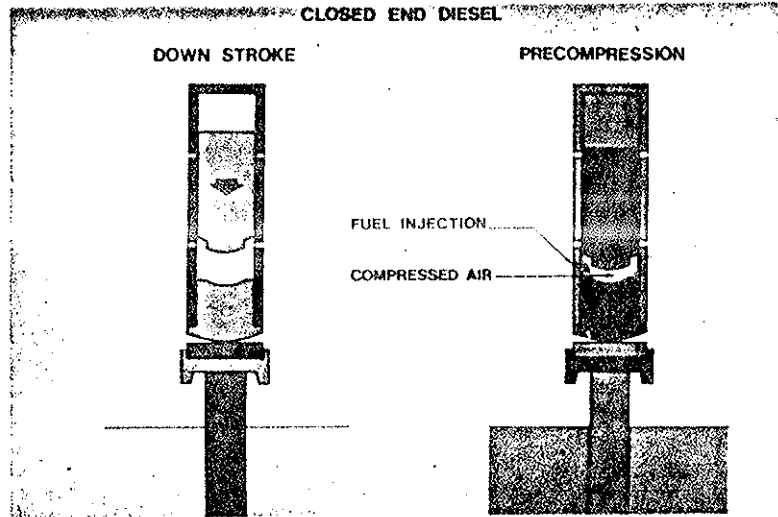
B30. When the ram clears the exhaust ports the pressure is dropped to zero and the rigid body velocity calculated. The stroke can then be determined. If it agrees with the starting stroke, computation is stopped. Otherwise, a new calculation cycle is started using the rebound stroke as the beginning stroke. Convergence is usually rapid.



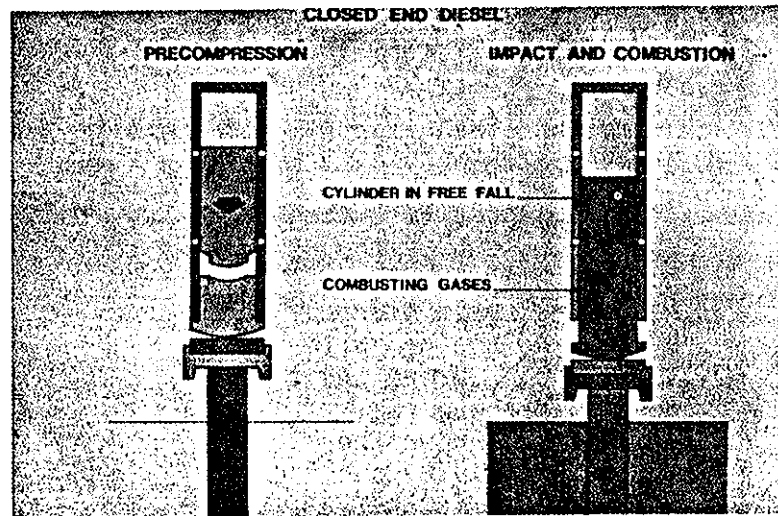
B31. The closed end diesel hammer is in some important ways different from the open end hammer. The top of the cylinder in this case is closed so that on the up stroke air is trapped in the resulting chamber, known as the bounce chamber. The hammer is again started by lifting the ram with the crane. During the down stroke the ram accelerates under the action of gravity and the air pressure in the bounce chamber.



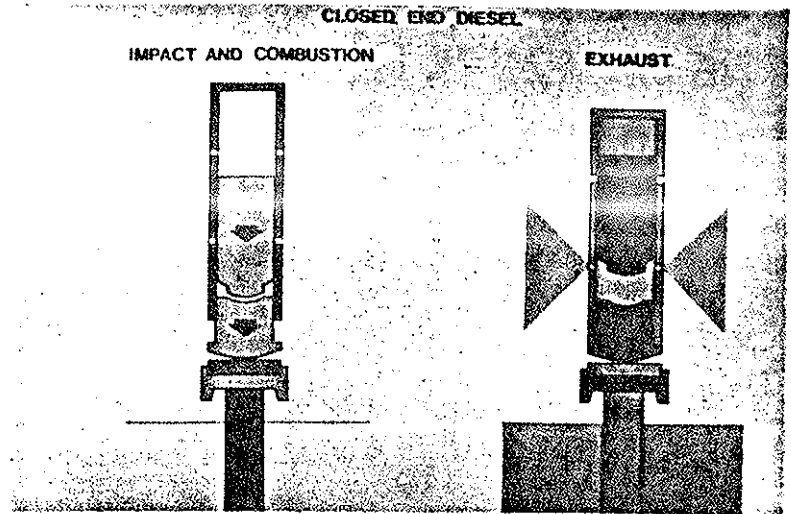
B32. Again when the ram passes the exhaust ports the gas in the combustion chamber is compressed.



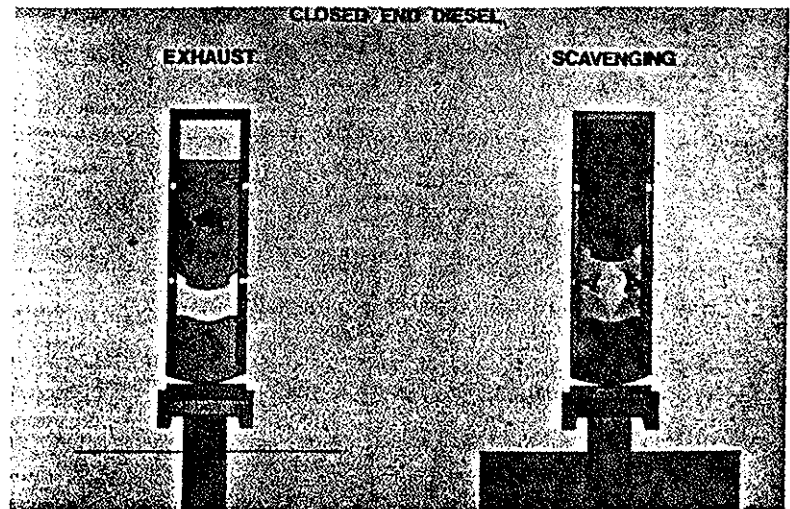
B33. At the bottom of the stroke the ram impacts with the anvil driving is and the pile top down. The bounce chamber ports are cleared and the bounce chamber pressure comes to atmospheric. Ignition occurs at about the same time. The ram then moves upward due to the action of both the pile rebound and the combustion chamber pressure.



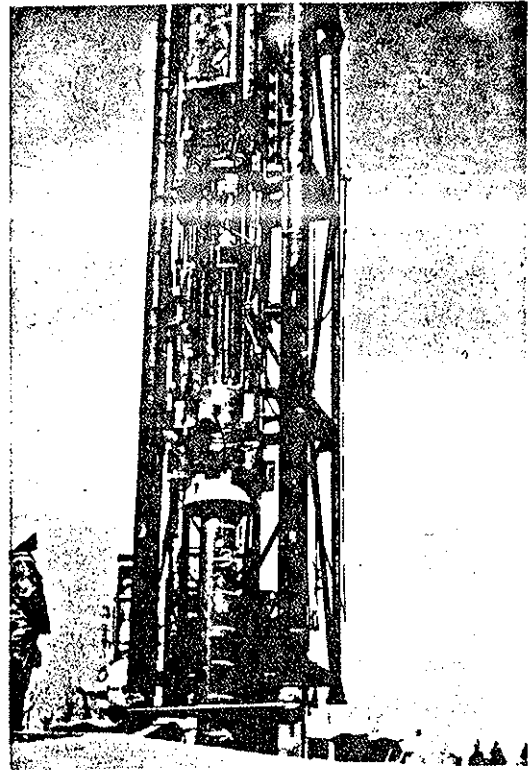
B34. When the exhaust ports are cleared the combustion chamber goes down to the ambient atmospheric pressure. Also, a pressure is built up in the bounce chamber slowing the ram down more quickly than would gravity alone.



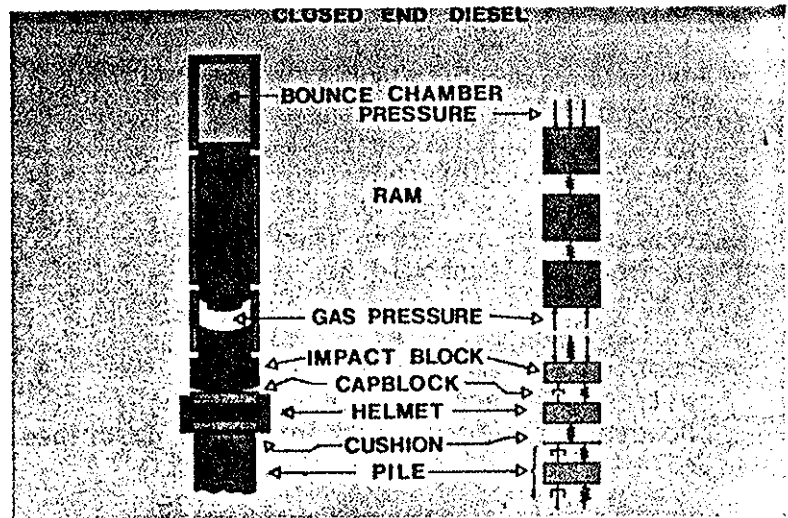
B35. As the ram continues on up fresh air is drawn into the combustion chamber and scavenging occurs. If the upward velocity of the ram is too great the bounce chamber pressure will become so large that the cylinder is lifted off the helmet. This action is undesirable and it is prevented by reducing the throttle. It should be understood that some hammers have a more complex system associated with the bounce chamber. However, the basic concept is as described.



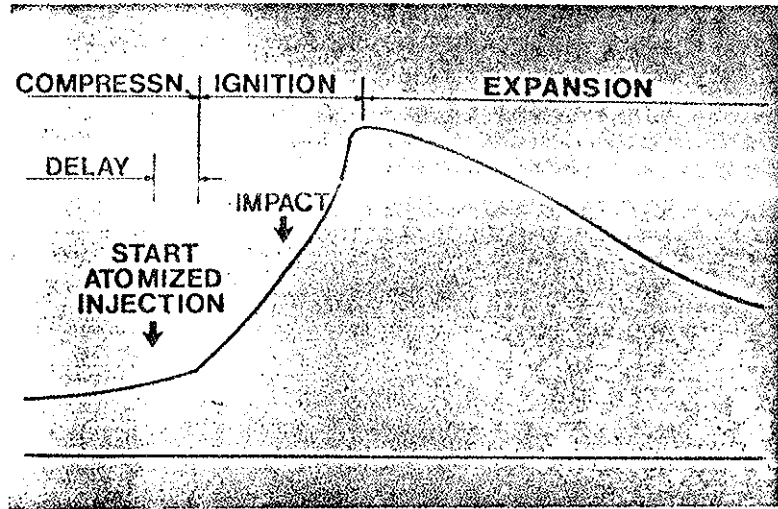
B36. In this slide a double acting diesel hammer is shown. Characteristic for this particular hammer is a pressure tank at the top of the unit which provides for a larger bounce chamber.



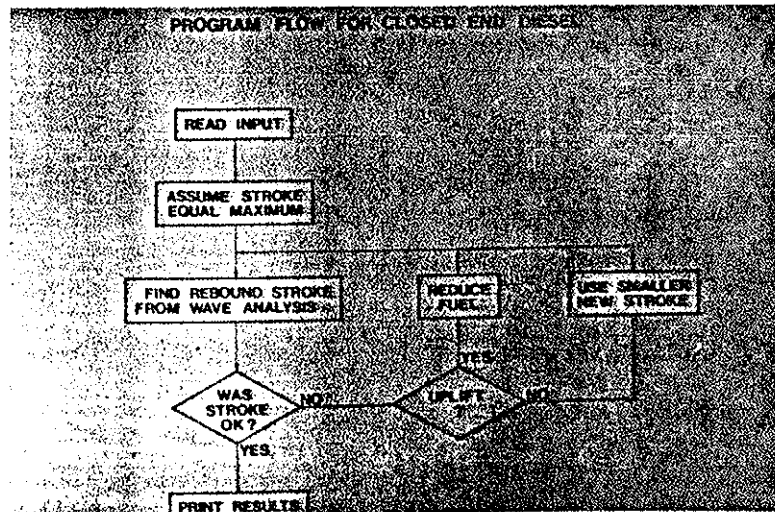
B37. The model used to represent the hammer is shown here. It is similar to that used for the open end diesel except for the bounce chamber pressure. The resulting force acting on top of the hammer must be calculated from the gas law and included in the computation of ram motion.



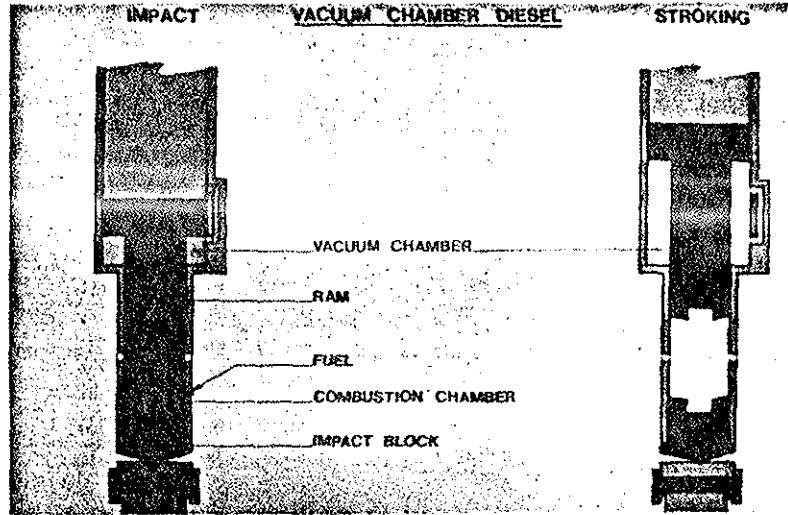
B38. Some hammers introduce the fuel in an atomized state. Thus, ignition occurs shortly after injection in a manner similar to the conventional diesel engine. Burning then takes place over a longer time depending on the rate of fuel injection.



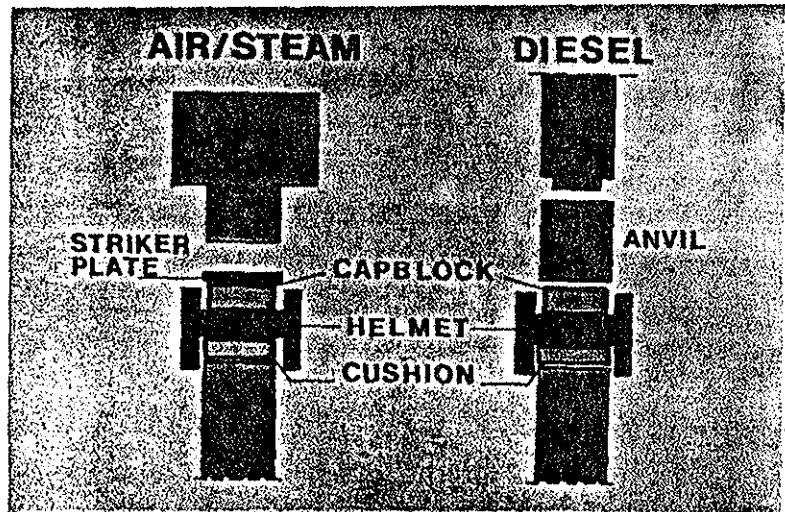
B39. The computer program for the closed end diesel hammer must operate somewhat differently than for open end hammers. If the open throttle stroke is such that lift off is induced then the combustion pressure, associated with a reduced throttle, must be found so that lift off is only incipient. On the other hand, if lift off does not occur then the computation is the same as for open end hammers where the stroke must converge.



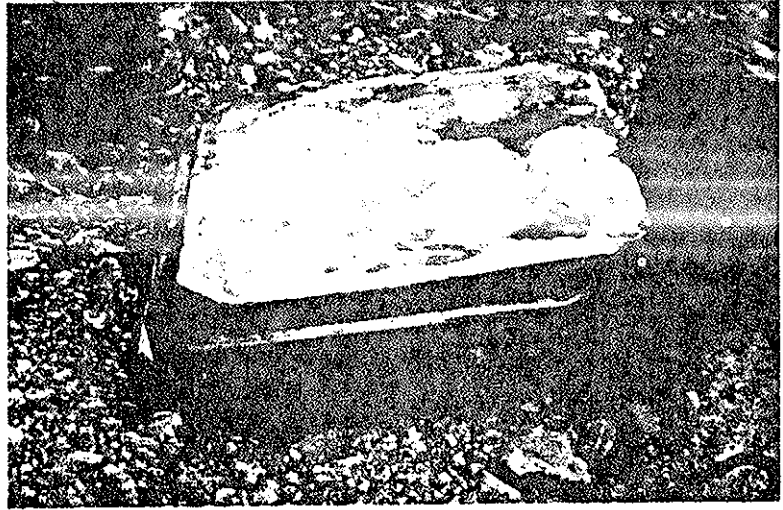
B40. An additional diesel hammer type known as the vacuum chamber diesel is also modeled in the program. The hammer is illustrated here schematically. This hammer is double acting in that on the up stroke a reduced pressure is induced in the vacuum chamber. This force increases rapidly to a near vacuum and then continues to increase only gradually. The stroke of the hammer is not limited by a hammer top so the computation convergence is only on stroke.



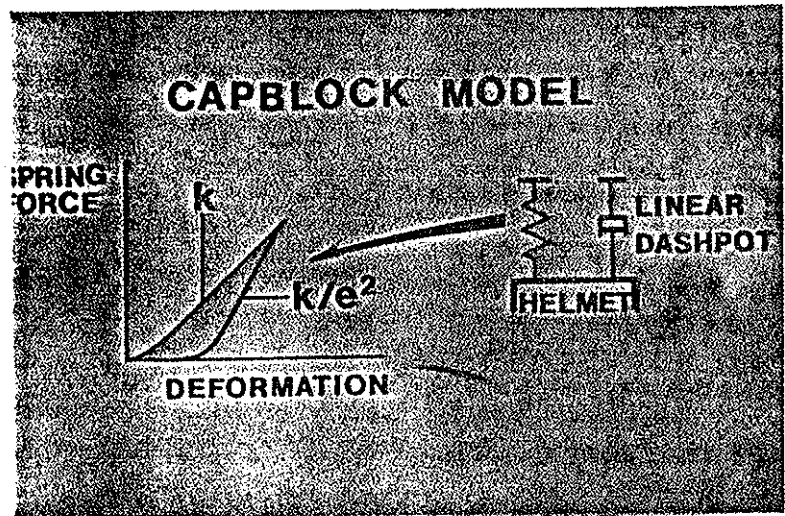
B41. So far only hammer models have been discussed. Now the next elements down in the driving system, the capblock, helmet, cushion model will be described. The capblock is that portion of the driving system which receives the impact force from the ram in the case of air/steam hammers or from the impact block from diesel hammers.



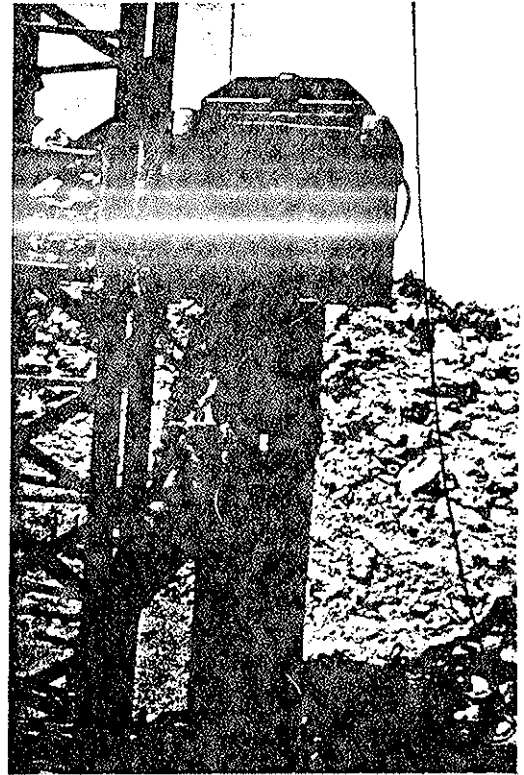
B42. Capblocks formerly consisted of hardwood or cable coils. Increasingly artificial composite materials are used. These newer materials have the advantage of offering more uniform and reproducible mechanical properties.



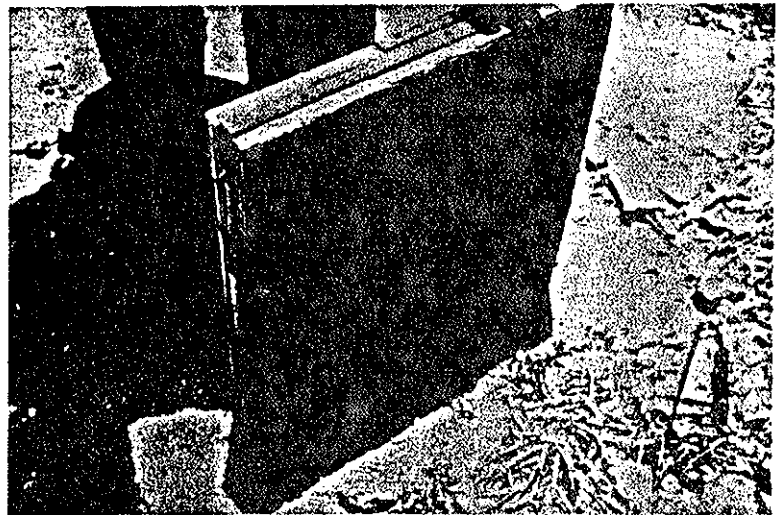
B43. The capblock is modeled as a spring with a steeper unloading portion to represent the coefficient of restitution. Also, a dashpot is used in its model. Comparison of measured with computed force and velocity curves proved this model to be valid.



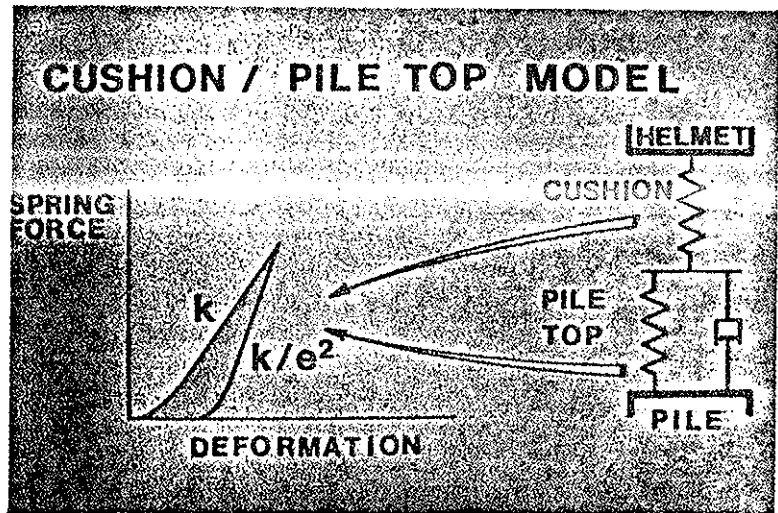
B44. The helmet is that steel element which contains the capblock on top and holds the pile in alignment with the appropriate recess in the bottom. It is modeled as a mass together with the mass of the capblock, strikerplates and pile adaptors, if present. If steel or timber piles are being driven the helmet fits directly on top of the pile.



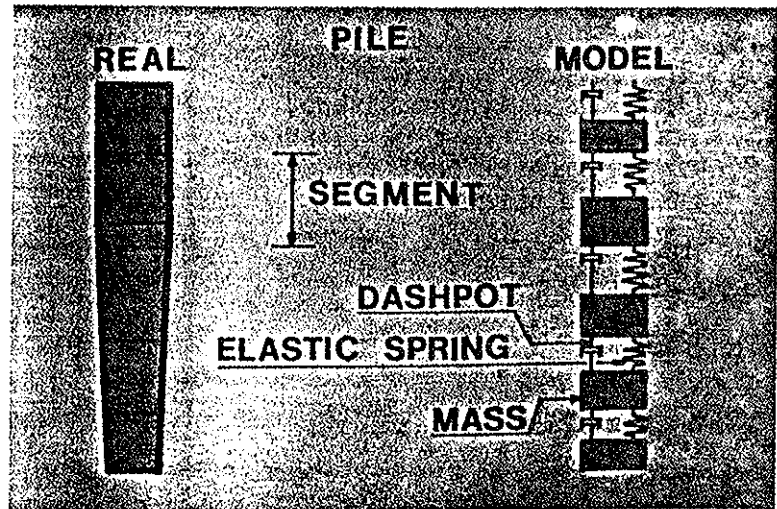
B45. For concrete piles a softwood cushion is placed between the helmet and the pile. The thickness of this cushion may vary widely.



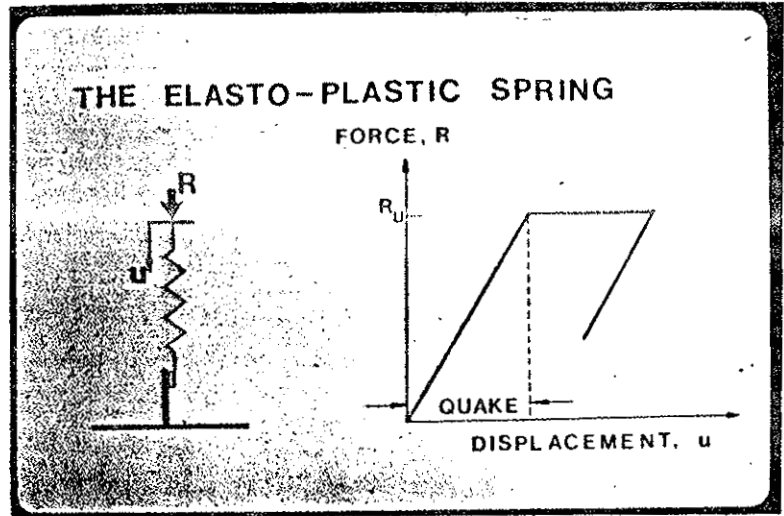
B46. It is modeled as a spring also with a coefficient of restitution. It is difficult to estimate the stiffness of the usual cushions since they are made of softwood struck across the grain and have substantially changing properties during driving.



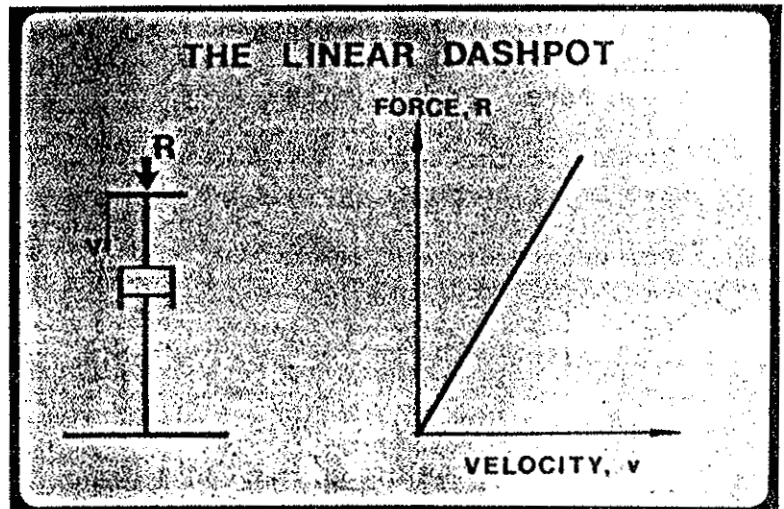
B47. The pile is modeled as a series of masses, springs, and dashpots. The complete model of the pile is shown here. Probably the pile is the simplest element in the total Wave Equation Model. Dashpots have been included to account for some structural damping especially for pile materials like timber.



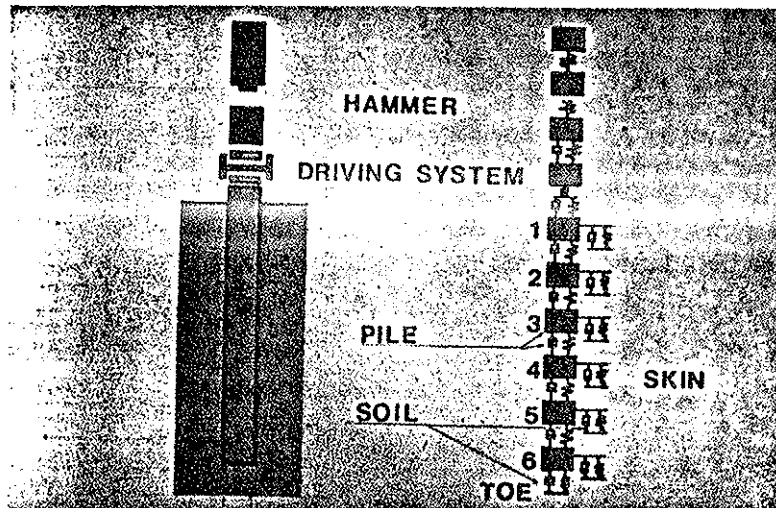
B48. The soil model was already introduced in the first section of this presentation. It consists of an elasto plastic spring which reaches an ultimate resistance force at a compression value called the quake.



B49. The soil model also includes a linear dashpot which produces a resistance force proportional to the pile velocity.

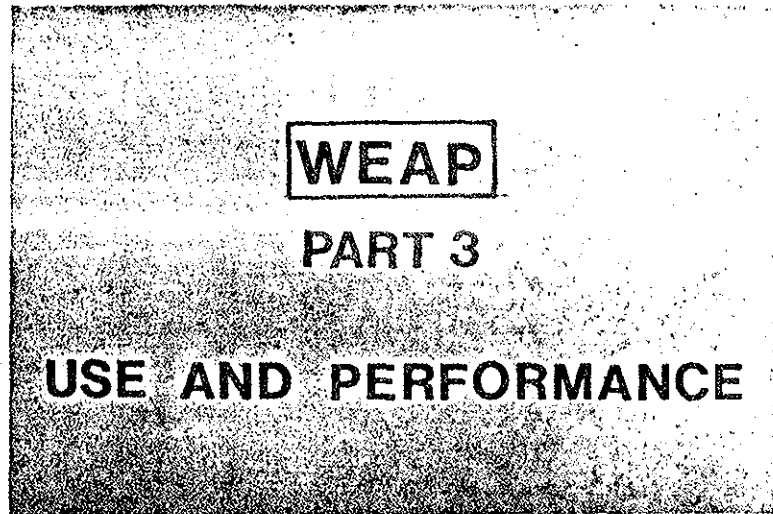


B50. The total wave equation model is shown here for the case of a diesel hammer driving a pile which is represented by six elements. A complete spring-dashpot soil model acts at each pile segment. An additional soil model unit acts at the toe to represent end bearing. The sum of the individual ultimate resistance forces acting at each pile segment, here six, is equal to the total ultimate skin resistance. Skin resistance plus end bearing equals the total pile resistance.



Part III
Program Use and Performance

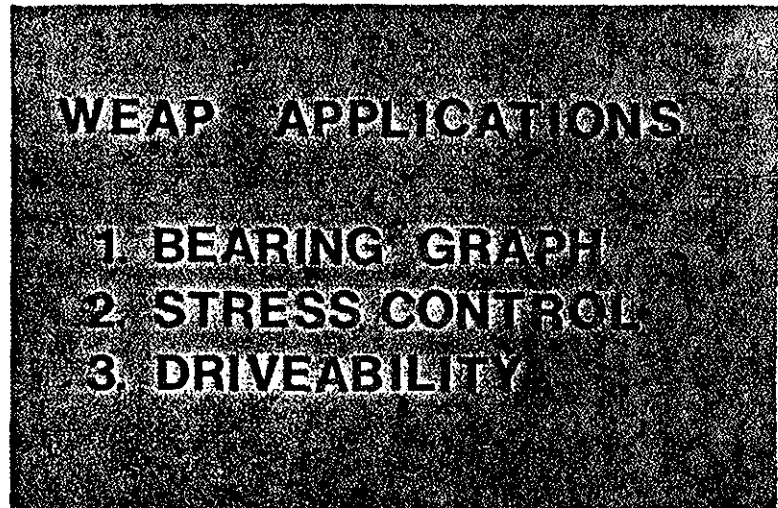
C1. In this, the third section of this presentation, further details of the WEAP program will be presented. An example of data input will be described, three sample problems will be presented and some results of the correlation between measured and calculated force and velocity curves will be given. The total data input structure with all options is too lengthy to describe in detail. Rather, an example which can be run with minimal input data will be described.



More detail is given in the User's Manual.

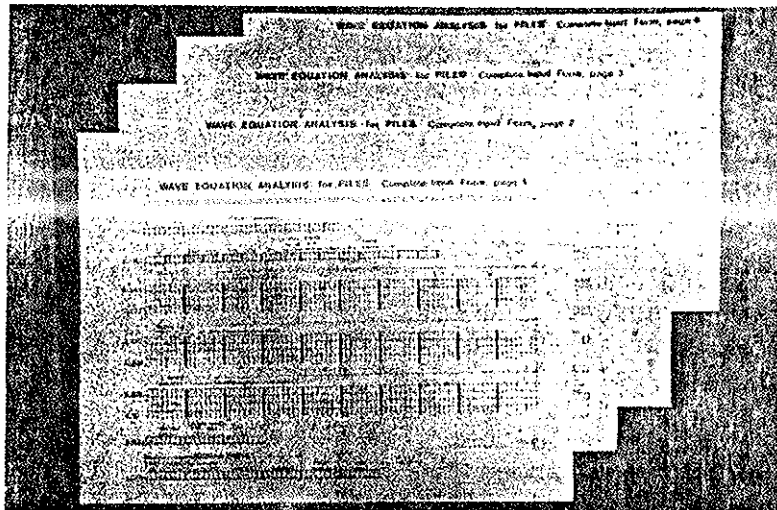
C2. There are three typical applications for a Wave Equation Program:

(a) The Construction of a Bearing Graph; (b) The Control of Stresses, in particular tension stresses in concrete and compression stresses in steel piles; and (c) The Analysis of driveability. By driveability we are referring to a check on a driving system, given a required pile bearing capacity.

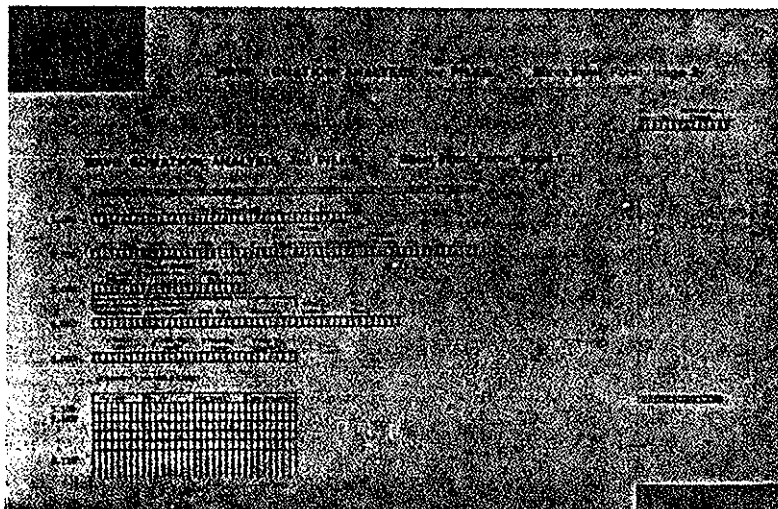


For each of these three groups an example will be given.

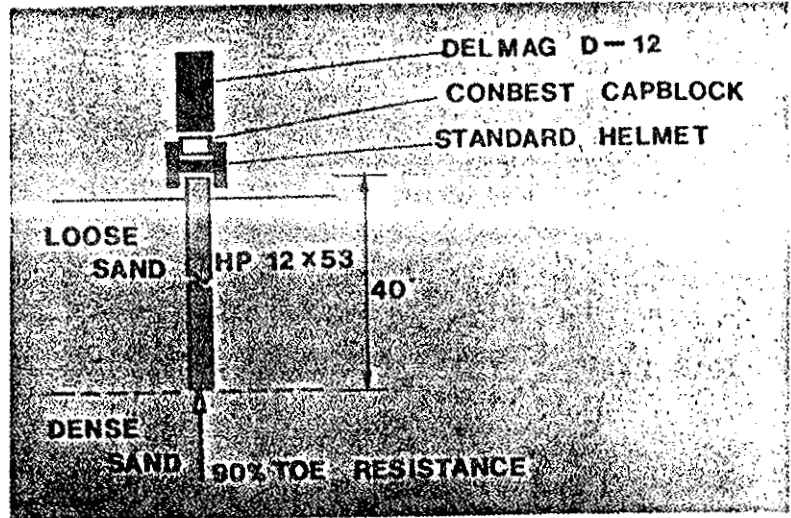
C3. The construction of a bearing graph will be discussed in more detail to demonstrate the coding of input data. Two different input coding forms were prepared, a SHORT and a COMPLETE form;



C4. only the short one is demonstrated here. The first problem can be stated as follows:

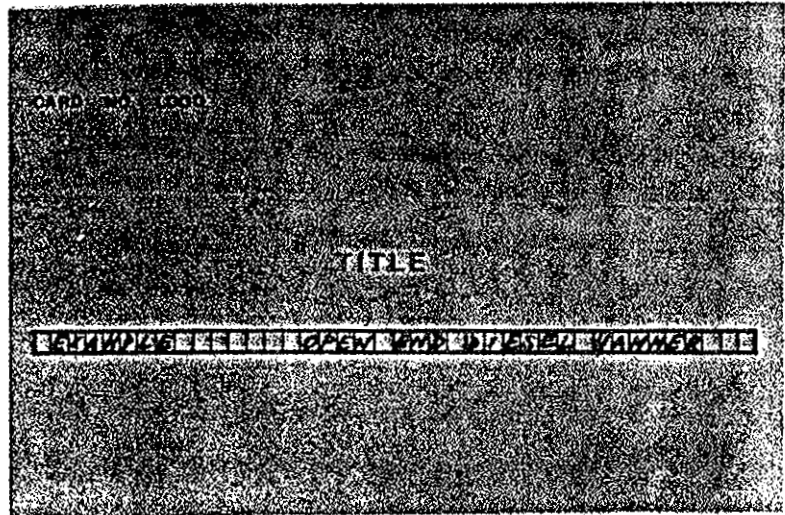


C5. An HP12 x 53 pile of 40 feet length is to be driven through loose into dense sand. A DELMAG D-12 which is an open end hammer is selected for pile driving. The contractor wants to use a standard conbest capblock and a twelve inch helmet. Because of the dense layer at 40 feet depth it is estimated that 90% of the total static capacity acts at the pile toe. The other 10% is assumed to be triangularly distributed along the side of the pile.

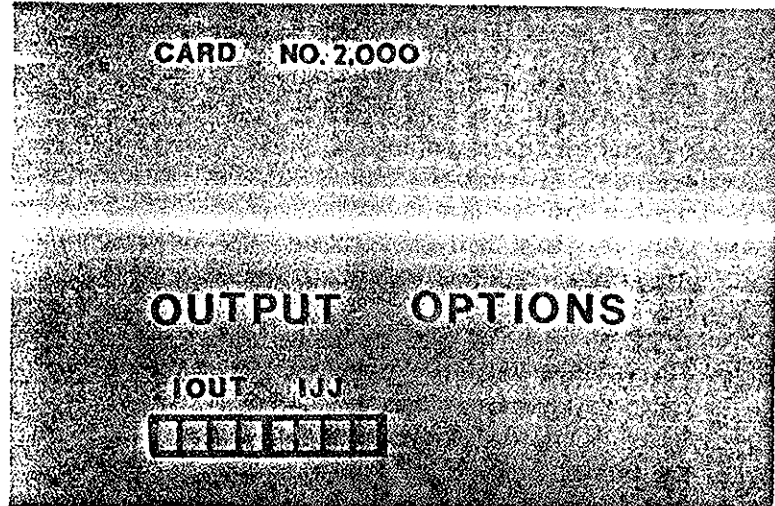


Input preparation is as follows:

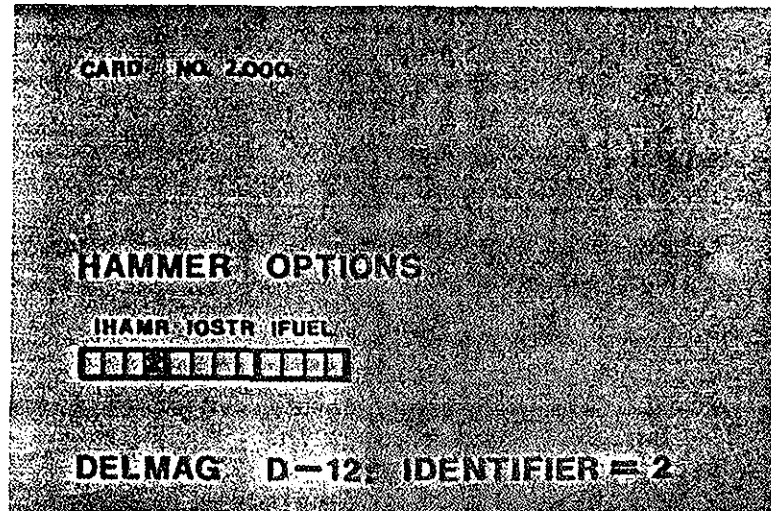
C6. On card 1 a title is entered that identifies the problem.



- C7. Card 2 allows the input of various options. The first two of these options deal with the type and the quantity of output. Leaving both data fields blank causes a small but sufficiently detailed output to be printed.



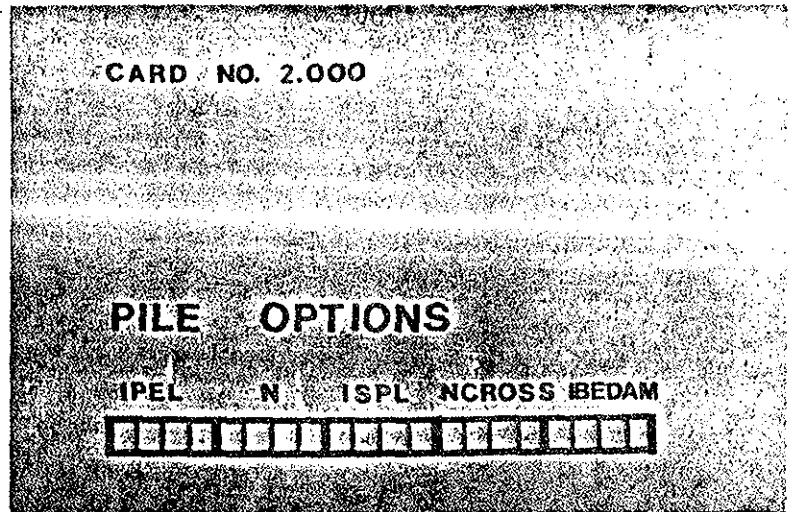
- C8. The next three fields of the option card are devoted to the hammer. IHAMR is the hammer identifier. A 2 is inserted here as given in the User's Manual for a D-12 hammer. Hammer information is stored on about 80 different hammers. Of course, all of the hammer information could be input, using data cards.



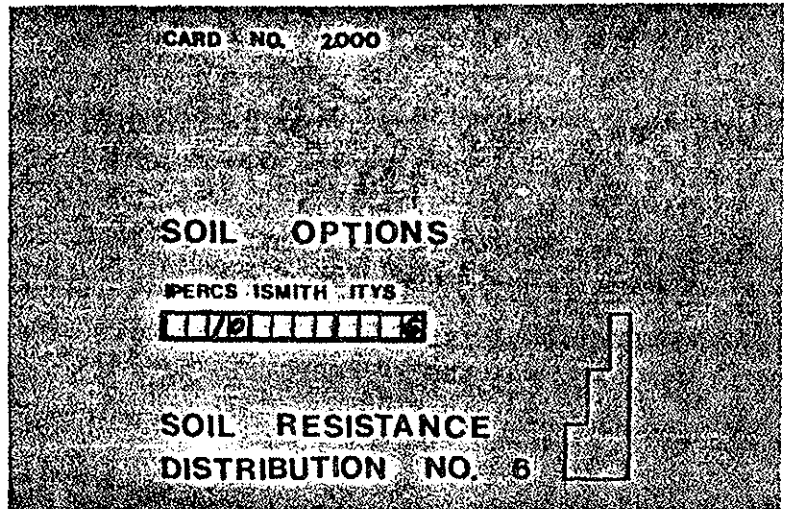
Leaving IOSTR blank causes the computer to find the proper hammer stroke. IOSTR set to -1 or 1 would produce an iteration with constant stroke or no iteration at all.

IFUEL determines where or not full combustion pressures are to be used. A blank field specifies normal operation.

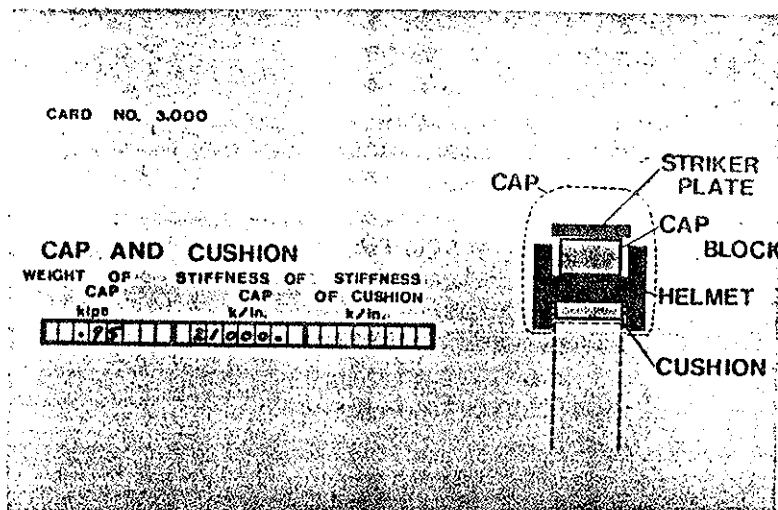
C9. The next five options deal with the pile input. Normally, segment stiffnesses and masses are determined by the program, and therefore, IPEL and N, the number of pile segments, can be left blank. There are no splices and the pile is uniform, thus ISPL and NCROSS are left blank. Finally, IBEDAM, the pile damping parameter, is left blank because a steel pile is being analyzed and its small damping is usually neglected.



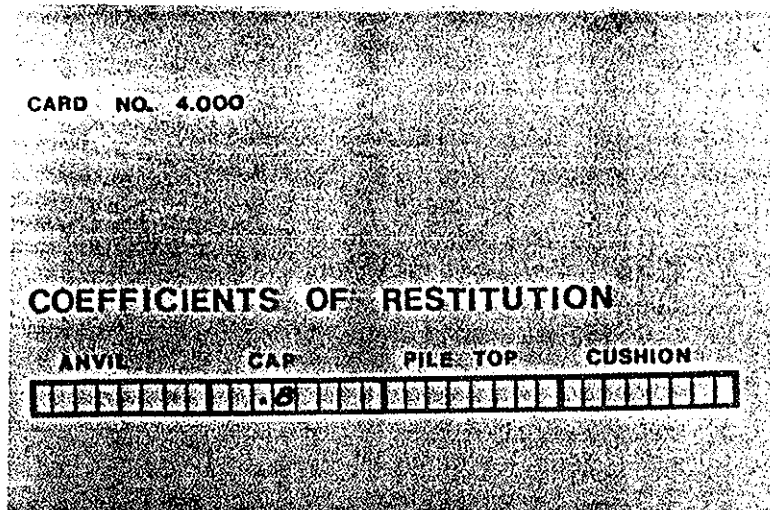
C10. Three more parameters dealing with the soil are given on card no. 2. IPERCS is the percentage of skin friction, here 10. ISMITH is left blank since viscous rather than Smith's soil damping approach is chosen. During the course of the piling research project at Case Western Reserve University a different approach to the treatment of soil damping was developed. This model, referred to here as viscous, uses the pile impedance to non-dimensionalize the damping constant. It has produced good results. Both methods are contained in the program. ITYS is set to 6 to distribute the resistance in a triangular manner. The resistance distributions which are available are given in the User's Manual.



C11. The driving system is identified on card no. 3. For this standard system one finds in the manual values of 0.95 kips for the helmet-capblock weight and 21,000 kips/inch for the conbest capblock stiffness. There is no cushion on top of the pile.



C12. Coefficients of restitution are left blank on card no. 4 except for the capblock where a 0.8 is inserted as appropriate for conbest. The computer will make a reasonable choice for all quantities that are left blank on this card.



C13. The pile has to be described on card no. 5. For the uniform case the following input is sufficient: the length equal to 40 feet, the cross sectional area equal to 15.5 square inches, the steel's elastic modulus of 30,000 ksi and the steel's specific weight of 492 lbs/cubic foot.

CARD NO. 5.000

PILE DESCRIPTION

LENGTH ft.	AREA in ²	MODULUS ksi	SPEC. WEIGHT lbs/ft ³
40.	15.5	30000.	492.

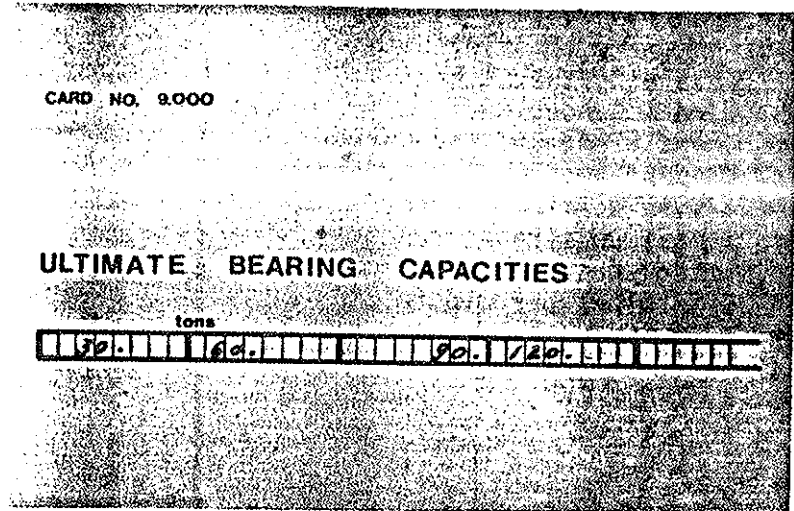
C14. Card no. 6 is used for soil parameter input. Quakes for both skin and toe bearing are set to 0.1 inches, soil damping was selected as 0.3 and 0.15 for the skin and toe, respectively. The ultimate soil bearing capacity was set to -1.0 since more than one value has to be analyzed. The soil's coefficient of restitution is left blank for normal operation.

CARD NO. 6.000

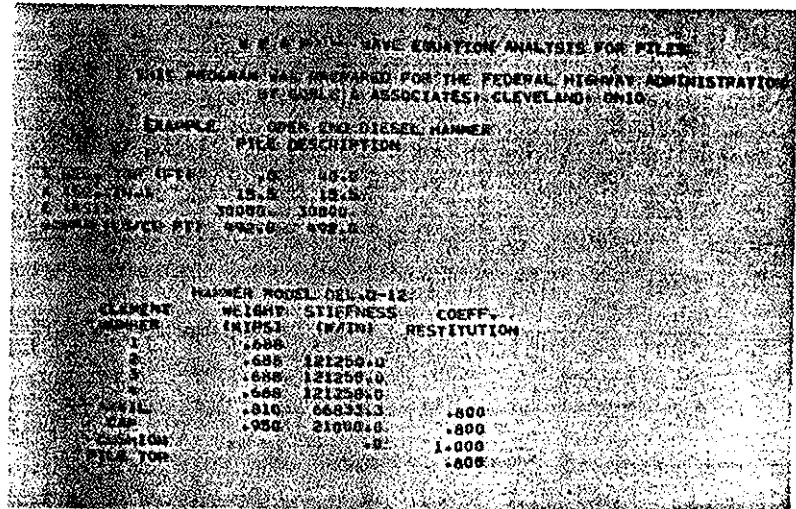
SOIL PARAMETERS

QUAKE SKIN	QUAKE TOE	DAMPING SKIN	DAMPING TOE	ULT.	COEFFICIENT OF RESTITUTION
0.1	0.1	0.3	0.15	-1.0	

C15. No other input except for card no. 9 is required. On this card four ultimate capacities; namely, 30, 60, 90, and 120 tons are specified. These four values are sufficient for the construction of a bearing graph.



C16. The output consists in this case of seven pages. There are two pages giving hammer, pile, soil, and option details. Four pages list the extrema of forces, stresses, velocities, and displacements for each element.

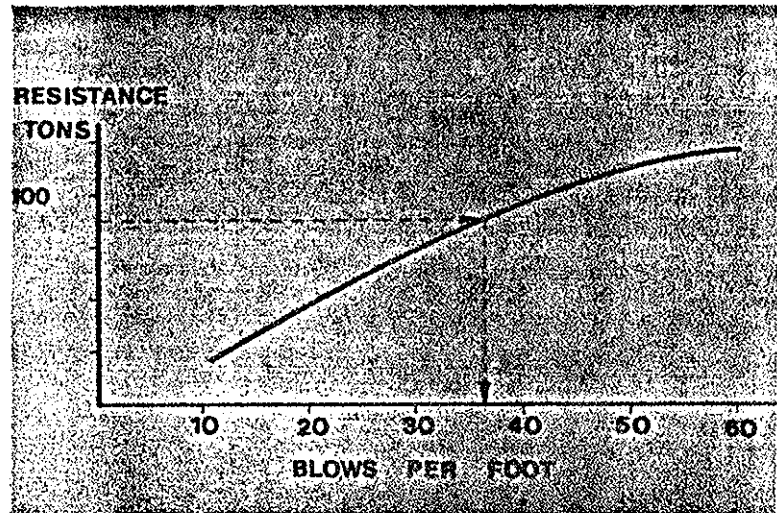


C17. On the last page a summary is printed containing the ultimate resistance, RULT, analyzed, and the corresponding blow count, stroke, minimum and maximum stresses and the speed of the hammer in blows per minute.

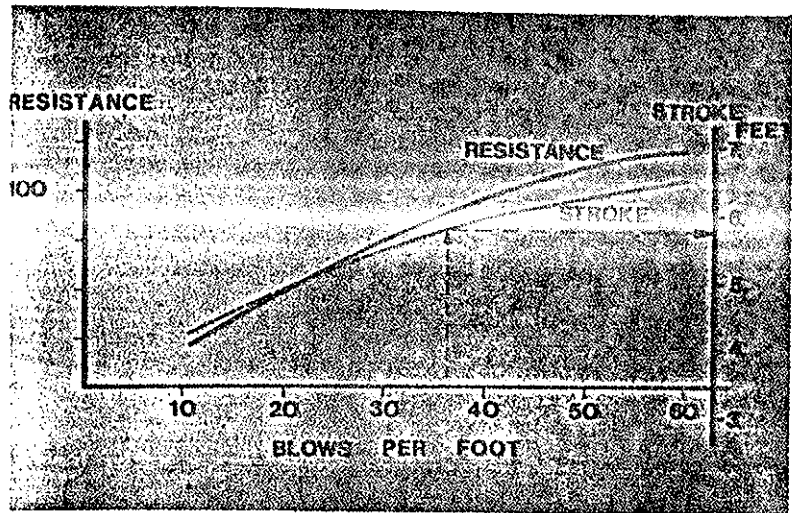
EXAMPLE OPEN END DIESEL HAMMER
SUMMARY

NO	R ULT TONS	BLOW CT 1/FT	STROKE FT	MIN STR KSI	MAX STR KSI	BLOWS/ MINUTE
1	30.0	14.	4.35	.00	18.11	56.3
2	60.0	24.	5.22	.00	21.72	51.4
3	90.0	40.	5.66	.00	25.43	49.5
4	120.0	54.	5.23	.00	25.29	47.2

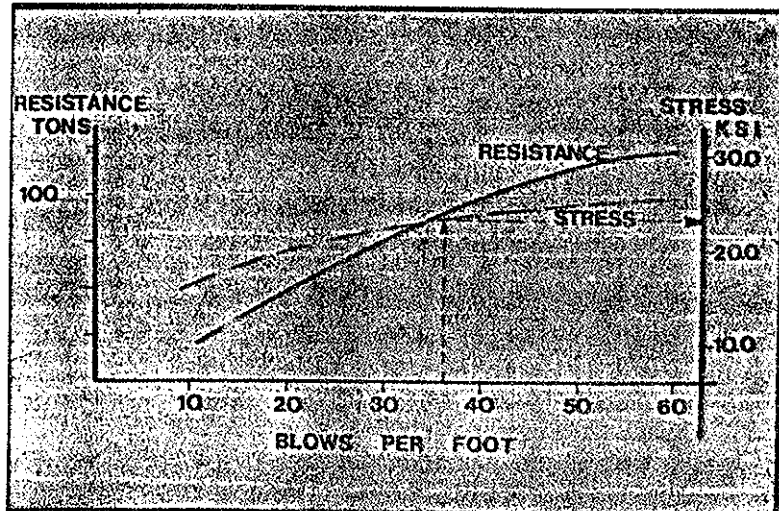
C18. The blow count results were used to construct a bearing graph as shown here. If an ultimate resistance of 90 tons was desired, then the blow count should be 36 blows per foot as indicated by the arrows.



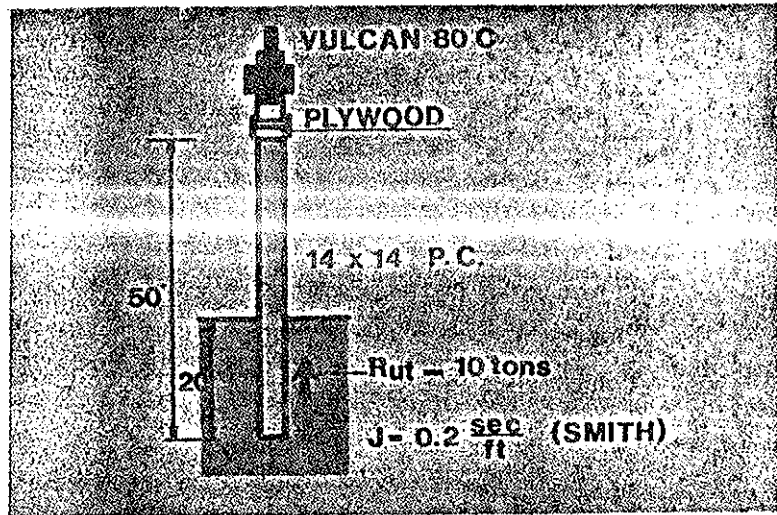
C19. Also plotted in this slide are the stroke results corresponding to the blow counts of the summary. The expected stroke for 90 ton resistance and 36 blows per foot is about 5.8 feet.



C20. In a similar manner the maximum compressive stresses are plotted and one finds a maximum stress of 24 ksi at 36 blows per foot driving resistance. It should be noted that the WEAP program produces the bearing graph automatically, if so desired.



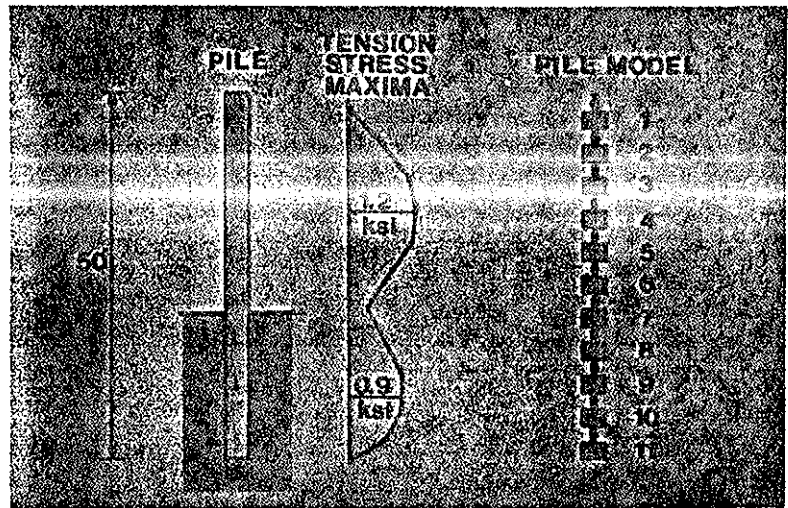
C21. A second type of wave equation application is for the control of tension stresses in concrete piles. As an example consider a Vulcan 80C differential acting air/steam hammer driving a 50 foot long, 14 x 14 inch square, prestressed concrete pile. The soil overlaying the bearing strata is very soft and a situation is to be analyzed with only 10 tons skin resistance and a Smith skin damping value of 0.2 at a penetration of 20 feet. The cushion thickness is to be determined such that the tension stresses are less than 1.0 ksi.



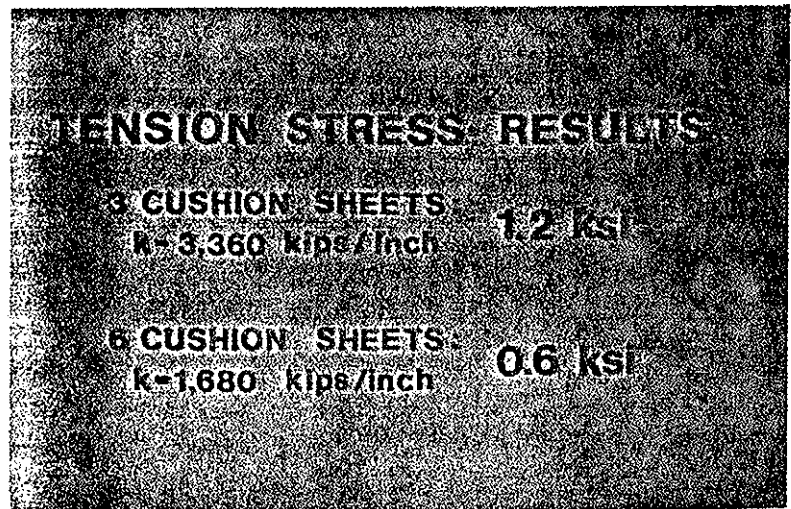
C22. In a first run the cushion is assumed to consist of 3 sheets of 3/4 inch plywood, 14 x 14 cross section. Its elastic modulus--after some time of driving--is assumed to be 30 ksi and the three sheets are probably compressed to a total thickness of 1.75 inches. The resulting cushion stiffness is, therefore, 3360 kips/inch. The coefficient of restitution is 0.5.



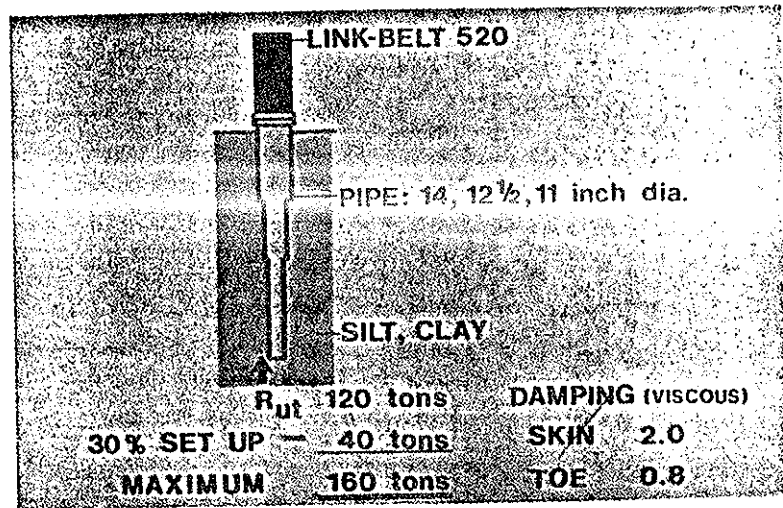
C23. In this slide the maximum tension stresses are plotted as determined by the computer for each element. The highest tension stress occurs at a depth of 18 feet and is 1.2 ksi.



C24. Thus the tension stresses are too high. A rerun of this problem with 6 cushion sheets and, therefore, a cushion stiffness which is only 1680 kips/inch, produces a tension stress of 0.6 ksi.



C25. The third problem to be demonstrated deals with driveability. A thin-walled pipe pile which is step tapered is to be driven by a Link-Belt 520 hammer. Estimates of bearing capacity in a cohesive silt and clay are 120 tons with 33% set up. This means that if driving is interrupted and the pile left at rest for an extended period of time, it would obtain a static capacity of 160 tons.



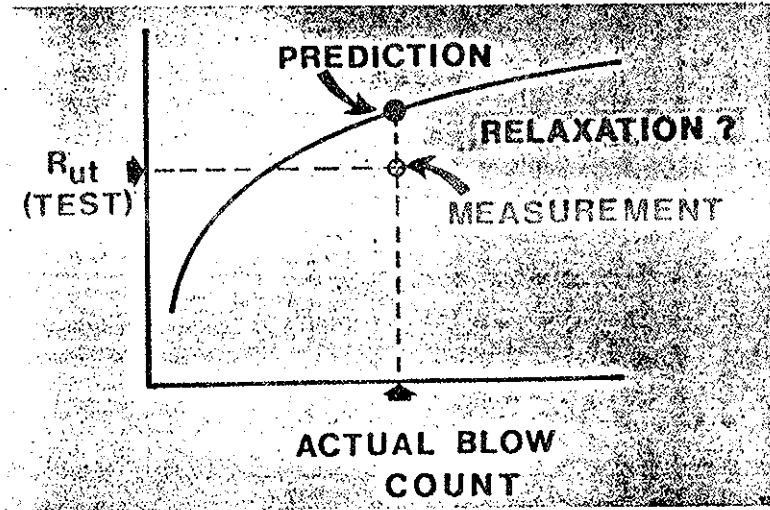
C26. A WEAP analysis with relatively high viscous damping factors of 2.0 and 0.8 for skin and toe, respectively, shows that the driving resistance would be 221 blows per foot at 120 tons resistance. It would be infinite, that is there would be no set, at the set up capacity of 160 tons. The conclusion is that the hammer/pile system would be insufficient if the pile had to be driven after a waiting period.

EXAMPLE DRIVEABILITY CHECK LB 520

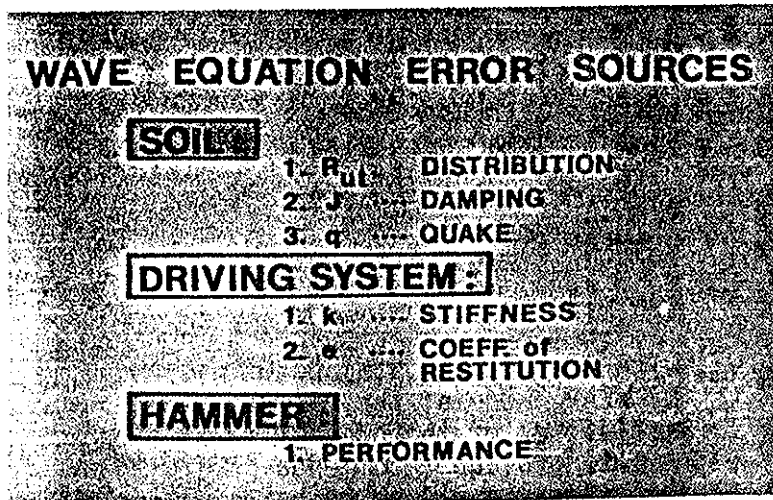
SUMMARY

NO	RESULT	BLOW CT	STROKE	MIN STR	MAX STR	MINUTE	B-C. PR.
	TONS	17FT	FT	KSI	KSI		PSI
1	120.0	221.	3.60(1)	80	30.54	73.9	24.7
2	160.0	999999.	3.60(2)	80	30.80		24.7

C27. Previously developed wave equation programs have been tested by comparing the predicted pile capacity with the value obtained from static load tests. This comparison could be meaningless for a number of reasons. The pile may gain or lose strength between driving and load testing.



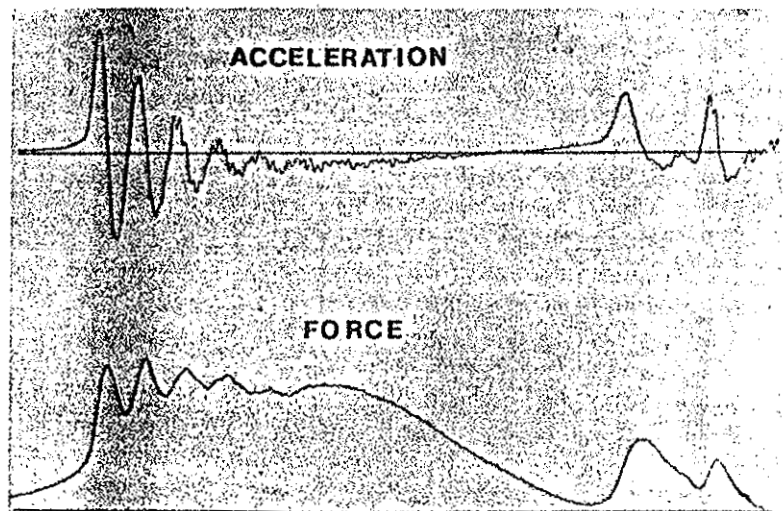
C28. The dynamic soil constants may be incorrectly estimated. The driving system constants may not be correctly known and the hammer performance may not be as expected. Thus, a very large number of constants can be adjusted to achieve a satisfactory correlation with capacity. The value of these constants are certainly not unique.



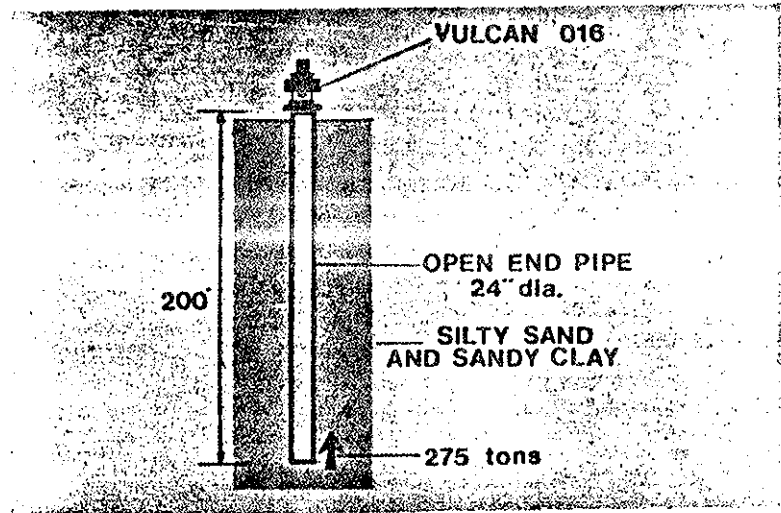
C29. During the Piling Research Program at Case Western Reserve University a large volume of force and acceleration measurements were made at the pile top during driving.



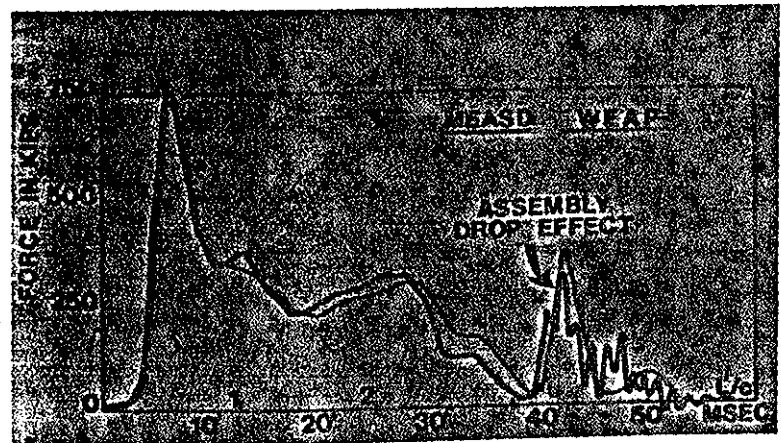
C30. This huge volume of data (several thousand hammer blows) was available to test the program. The available measurements consist of force and acceleration records at the pile top. These measurements were compared with the values obtained by WEAP. One of the major activities of the project was to provide recommendations on the values to be used for the hammer dynamic constants. Sixteen different jobs were so tested and the results are presented in the final report. Three of these examples will be shown here.



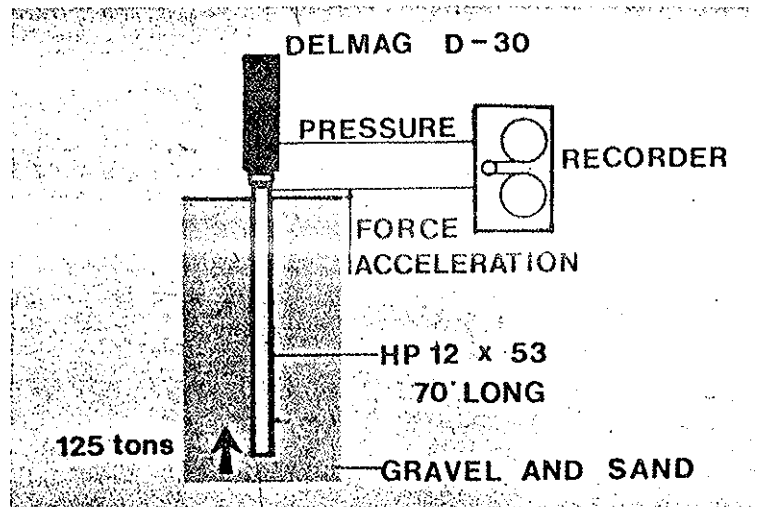
C31. The first case is an air/steam hammer driving a 200 foot long steel pipe pile. It was selected to test the assembly drop portion of the program.



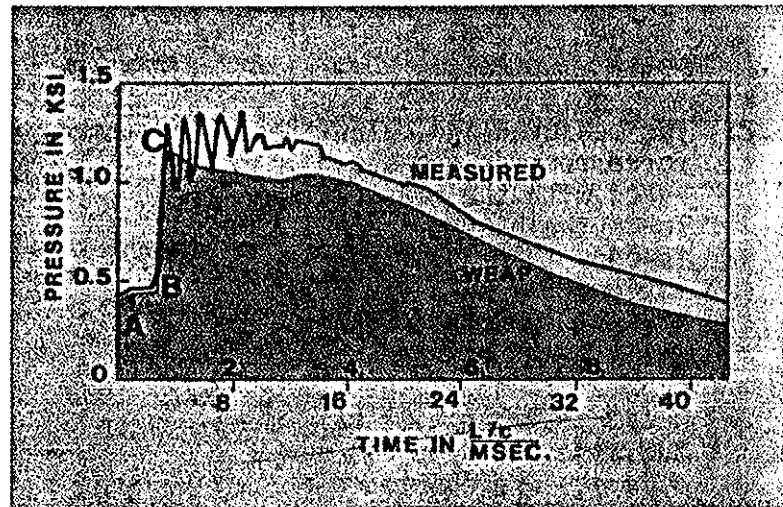
C32. The measured and calculated force records are shown here, with the measured force the heavy line. It is seen that the first maximum matches very well and the two curves agree well over their entire range. It should be noted that the time of the assembly drop was very accurately predicted.



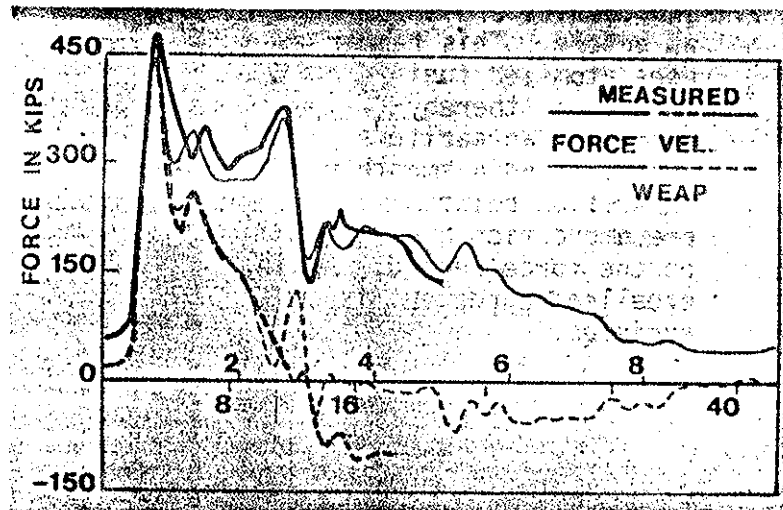
C33. The second test case was taken from a special test program on a DELMAG D-30 hammer. These tests were performed by Goble & Associates in 1971, for the Foundation Equipment Company of Newcomerstown, Ohio. In addition to force and velocity at the pile top the combustion chamber pressure was also measured.



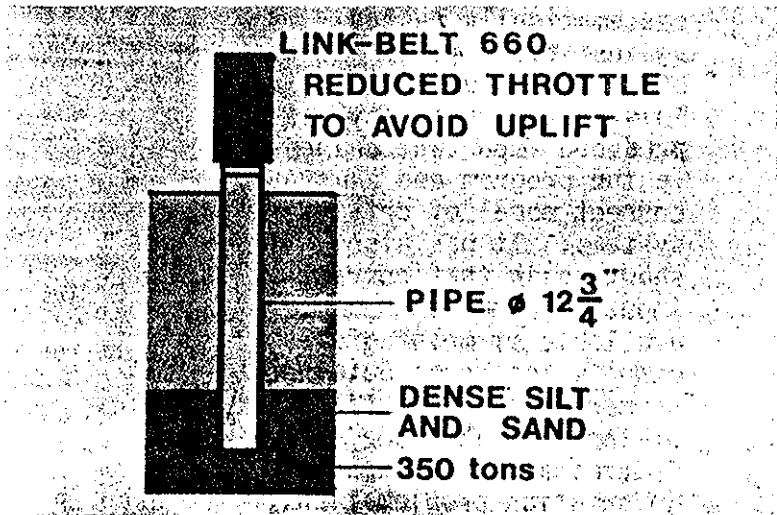
C34. The comparison of the pressure is shown here. Ram impact occurs at point A. Most of the precompression phase has not been included. The pressure remains constant for about 1 1/2 milliseconds until ignition begins at point B. Then it increases rapidly to point C. At ignition the measured record exhibits a high frequency oscillation characteristic of diesel engine ignition. Since this oscillation is unimportant for hammer performance no attempt is made to represent it. It can be seen that the pressure calculated by WEAP is similar but somewhat smaller than the measured value.



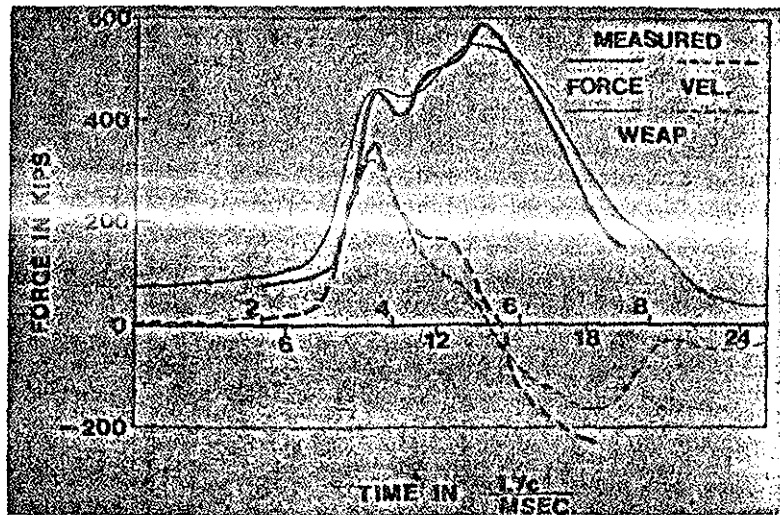
C35. A comparison of measured force and velocity for the top of this pile is shown here. The measured values are shown by the bold curves. The forces are solid lines and the velocity dashed. The agreement between these curves is quite good. In general, the velocity is a more sensitive quantity and, therefore, a better one to match. This sensitivity can be seen in this example at about 14 milliseconds where the measured and calculated velocities differ substantially.



C36. As a third example consider a pile driven by a closed end diesel hammer, the Link-Belt 660. The pile was a 43 foot long pipe of 12-3/4 inch outside diameter. It was driven into very dense silt and sand.



C37. The results are shown here as before. This hammer uses atomized fuel injection, thereby producing an earlier ignition and a smoother transition between precompression and impact on the force record. Again, excellent agreement is achieved.



C38. The WEAP program is the first wave equation program to be extensively and systematically tested against field measurements of force and velocity. This testing activity has produced important changes in the program and a more correct model of driving systems. It has also shown again that correct input information must be available if meaningful results are to be obtained. In many cases the state-of-the-art makes the determination of input information problematical. The user should not be surprised if such cases produce results which differ from the field experience. The wave equation concept is another engineering tool, the results of which should be evaluated by a knowledgeable engineer. Happy Pile Driving!

